

Soundra Prashanth
Sanjeet Kumar

PRINCIPLES OF IC ENGINE



ALEXIS PRESS
JERSEY CITY, USA



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First Published 2022

A catalogue record for this publication is available from the British Library

Library of Congress Cataloguing in Publication Data

Includes bibliographical references and index.

Principles of IC Engine by *Soundra Prashanth, Sanjeet Kumar*

ISBN 978-1-64532-848-3

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CHAPTER 1

CLASSIFICATION AND SOME BASIC DETAILS OF HEAT ENGINES

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ABSTRACT

A heat engine is a device that converts heat energy into mechanical work, providing a means to harness and utilize thermal energy for various applications. This abstract explores the fundamental principles of heat engines, their operation, and their significance in various industries. It discusses the key components of a heat engine, such as the heat source, working fluid, and the thermodynamic cycle involved. Additionally, different types of heat engines, including steam engines, internal combustion engines, and gas turbines, are highlighted, along with their respective advantages and limitations. The paper concludes by emphasizing the importance of heat engines in energy conversion and their role in meeting the growing demands for efficient and sustainable power generation. Additionally, internal combustion engines have a lower weight to power ratio than steam turbine power plants because they may achieve better thermal efficiency with a modest maximum operating pressure of the fluid in the cycle.

KEYWORDS

Heat Engine, Heat Source, Internal Combustion Engine, Steam Engine, Thermodynamic Cycle.

INTRODUCTION

The reciprocating internal combustion engine, gas turbine, and steam turbine are the three most often utilized forms of heat engines. Nowadays, the steam engine is being phased out gradually. Due to the lack of heat exchangers in the working fluid route (boilers and condensers in steam turbine plants), the reciprocating internal combustion engine has several benefits over the steam turbine. This leads to a significant mechanical simplicity and increased internal combustion engine power plant efficiency. Figure 1 provides a thorough categorization of heat engines.

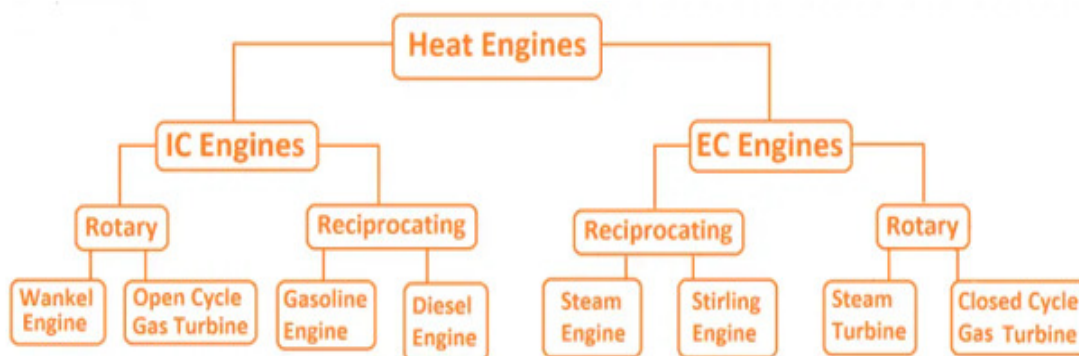


Figure 1: Classification of heat engines (ftp.idu.ac.id)

The reciprocating internal combustion engine has an additional benefit over the other two kinds in that all of its parts operate at an average temperature that is far lower than the highest

temperature of the working fluid in the cycle. This is due to the fact that only a very tiny portion of the cycle's duration is spent with the working fluid's high temperature. As a consequence, very high working fluid temperatures may be used, increasing thermal efficiency. Additionally, reciprocating internal combustion engines with negligible power output even a few hundred watts and good thermal efficiency have been developed. The primary drawback of this kind of engine is the vibration issue brought on by the reciprocating parts. Additionally, different fuels cannot be used in these engines. Only fuels that meet certain specifications, whether liquid or gaseous, may be utilized efficiently. These fuels cost more than average. Given the aforementioned considerations, it has been determined that reciprocating internal combustion engines are suited for use in cars, motorbikes, scooters, power boats, ships, slow-moving aircraft, locomotives, and power units with relatively modest output [1].

External Combustion and Internal Combustion Engines

Engines classified as external combustion engines are those in which combustion occurs outside the engine, as opposed to internal combustion engines, where combustion occurs within the engine. For instance, the heat produced by the burning of fuel is utilised in a steam engine or steam turbine to produce high pressure steam that serves as the working fluid in a reciprocating engine or a turbine. The working fluid of a petrol or diesel engine is made up of the combustion products that are produced when fuel and air are burned within the cylinder.

Basic Engine Components and Nomenclature

Internal combustion engines with reciprocating pistons seem to be relatively basic, but they are really quite sophisticated devices. To create output power, hundreds of components must each successfully carry out their respective tasks. There are two different kinds of engines: compression-ignition (CI) and spark-ignition (SI). Now let's go through some of the critical engine parts and the terminology used to describe engines.

Engine Components

A cross section of a single cylinder spark-ignition engine with overhead valves is shown in Figure 2. The major components of the engine and their functions are briefly described below.

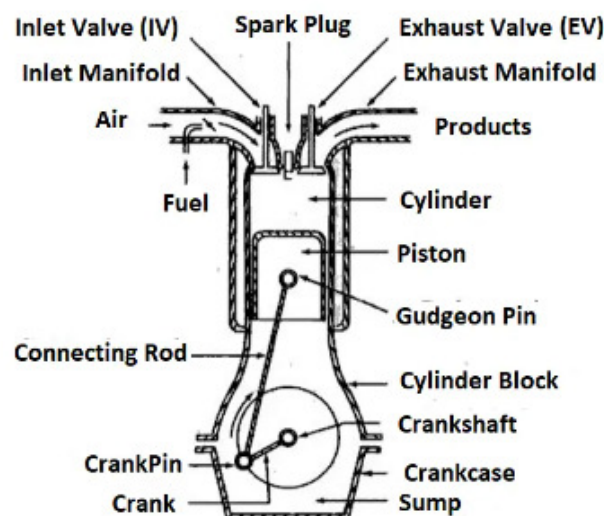


Figure 2: Cross-section of a spark-ignition engine (ftp.idu.ac.id)

Cylinder Block

The primary structural support for the numerous components is the cylinder block. A multi cylinder engine's cylinders are cast as a single unit known as the cylinder block. The cylinder block is attached to the cylinder head. When using water cooling or when using air cooling, the cylinder head and cylinder block are equipped with cooling fins or water jackets. Between the cylinder block and cylinder head is a cylinder head gasket. Numerous nuts or studs tightly secure the cylinder head to the cylinder block. The crankcase refers to the base of the cylinder block. The bottom of the crankcase has a cover called the crankcase that serves as a sump for lubricating oil. Bore or face refers to the inner surface of the cylinder block that has been precisely machined and polished to have a cylindrical form.

Cylinder

The term "cylinder" refers to the cylindrical vessel or chamber in which the piston reciprocates. The working fluid is fed into the variable volume that the engine's activity creates in the cylinder, which houses four IC engines that are each subjected to a separate thermodynamic process. The cylinder block supports the cylinder.

Piston

A piston is a cylindrical part that is inserted into the cylinder that serves as the combustion system's movable boundary. It creates a gas-tight space with the piston rings and lubrication by fitting exactly (and snugly) into the cylinder. It is the initial link in the chain that connects the output shaft to the gas forces [2].

Combustion Chamber

During combustion, the area in the upper half of the cylinder that is surrounded by the cylinder head and piston top is referred to as the combustion chamber. This area of the cylinder builds up pressure as a consequence of fuel combustion and the subsequent release of thermal energy.

Inlet Manifold

The inlet manifold is the conduit that links the intake system to the engine's inlet valve and is used to pull air or an air-fuel combination into the cylinder.

Exhaust Manifold

The exhaust manifold is the conduit that connects the exhaust system to the engine's exhaust valve and allows the combustion products to escape into the atmosphere.

Exhaust and inlet valves

Most valves are of the mushroom-shaped poppet kind. For controlling the charge entering the cylinder (inlet valve) and for releasing the combustion products from the cylinder (exhaust valve), they are either supplied on the cylinder head or on its side. The spark plug, which is often found on the cylinder head, is a part used in Spark Ignition (SI) engines to start the combustion process.

Connecting Rod

This component links the piston to the crankshaft and transfers the piston's gas forces to the latter. The little end and the large end of the connecting rod are its two ends. The large end is attached to the crankshaft by a crankpin, while the little end is connected to the piston by a dudgeon pin.

Crankshaft

The crankshaft transforms the piston's reciprocating action into usable rotational motion for the output shaft. A single cylinder engine's crankshaft has two crank arms, as well as balancing weights. The spinning system may be statically and dynamically balanced using the balance weights. A crankcase houses the crankshaft.

Piston Rings

Fitted into the slots all around the piston, piston rings provide a tight seal that stops combustion gases from escaping the cylinder.

Gudgeon Pin

This device connects the piston to the tiny end of the connecting rod.

Camshaft

The two valves' opening and shutting are controlled by the camshaft, which isn't shown in the diagram. Push rods, rocker arms, valve springs, and tappets are the related components. The ignition system is also driven by this shaft. Through timing gears, the crankshaft powers the camshaft. Integrated inside the camshaft, cams are constructed to open the valves at the right time and hold them open for the required amount of time (not indicated in the illustration).

Fly Wheel

Throughout a whole cycle of engine operation, the net torque applied to the crankshaft changes, changing the angular velocity of the shaft. The flywheel (not seen in the illustration) is a wheel-shaped inertia mass that is mounted to the output shaft in order to provide a consistent torque [3], [4].

DISCUSSION

If an engine is to work successfully then it has to follow a cycle of operations in a sequential manner. The sequence is quite rigid and cannot be changed. In the following sections the working principle of both SI and CI engines is described. Even though both engines have much in common there are certain fundamental differences. The credit of inventing the spark-ignition engine goes to Nicolaus A. Otto (1876) whereas compression-ignition engine was invented by Rudolf Diesel (1892). Therefore, they are often referred to as Otto engine and Diesel engine [5]–[7].

Four-Stroke Spark-Ignition Engine

In a four-stroke engine, the cycle of operations is completed in four strokes of the piston or two revolutions of the crankshaft. During the four strokes, there are five events to be completed, viz., suction, compression, combustion, expansion and exhaust. Each stroke consists of 180° of crankshaft rotation and hence a four-stroke cycle is completed through 720° of crank rotation. The cycle of operation for an ideal four-stroke SI engine consists of the following four strokes: (i) suction or intake stroke; (ii) compression stroke; (iii) expansion or power stroke, and (iv) exhaust stroke.

1. Intake or Suction Stroke

The piston begins its suction stroke when it is at top dead Centre and beginning to descend downward. The intake valve is anticipated to instantly open at this point, and the Figure 3 shows the exhaust valve in the closed position. The charge of fuel-air mixture is pulled into the cylinder by the suction produced by the piston's motion towards bottom dead Centre. The

suction stroke is over when the piston touches bottom dead Centre, and the intake valve immediately shuts.

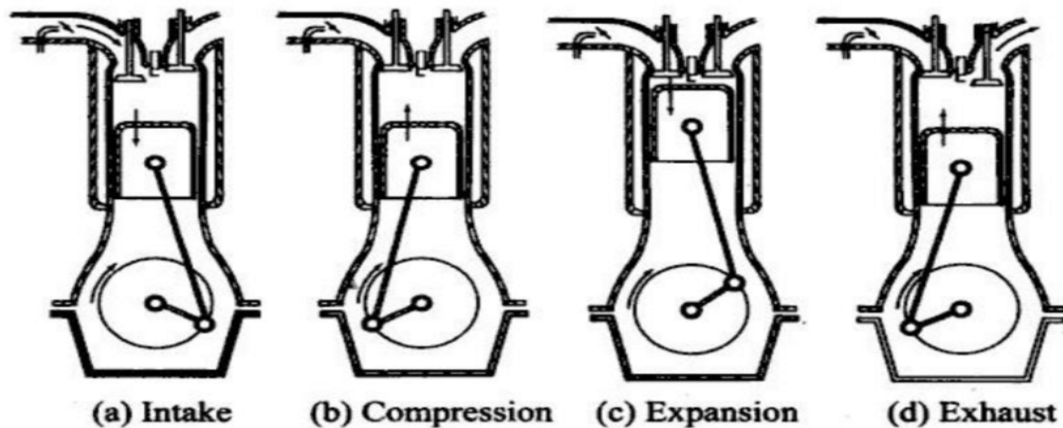


Figure 3: Working principle of a four-stroke SI engine (ftp.idu.ac.id)

2. Compression Stroke

During the piston's return stroke, the charge that was drawn into the cylinder during the suction stroke is compressed. Both the intake and exhaust valves are closed during this stroke, as shown. The whole cylinder's volume of mixture has now been crushed into the clearance volume. A spark plug on the cylinder head ignites the mixture at the conclusion of the compression stroke. The burning process may be roughly described as heat addition at constant volume since it is considered that in perfect engines burning occurs instantly when the piston is at top dead Centre. The chemical energy of the fuel is transformed into heat energy throughout the burning process, resulting in a temperature increase of around 2000 °C (process 2–3). The heat released from the fuel causes a significant rise in pressure towards the conclusion of combustion.

3. Expansion or Power Stroke

The piston is propelled towards the BDC by the high pressure of the burned gases. Both valves are in the closed position. Only during this stroke out of the four does power are created. During expansion, the temperature and pressure both falls.

4. Exhaust Stroke

Immediately after the expansion stroke, the exhaust valve opens while the intake valve stays shut. Part of the burning gases escape as the pressure drops to atmospheric pressure. The burned gases are swept out of the cylinder virtually at atmospheric pressure by the piston, which begins to move from the bottom dead Centre to the top dead Centre. When the piston reaches T DC at the conclusion of the exhaust stroke, the exhaust valve shuts, leaving some leftover gases trapped in the clearance volume in the cylinder. The subsequent cycle's new charge and these remaining gases combine to create the working fluid. In a four-stroke engine, each cylinder performs the aforementioned four tasks in two crankshaft revolutions: the first during the suction and compression strokes, and the second during the power and exhaust strokes. As a result, there is only one power stroke for a whole cycle, whereas the crankshaft completes two revolutions. The heat addition should be as high and the heat rejection as low as feasible in order to increase the engine's output. As a result, one should take care while creating the ideal p-V diagram, which should accurately portray the processes.

Four-Stroke Compression-Ignition Engine

Similar to the four-stroke SI engine, the four-stroke CI engine has a greater compression ratio than the latter. An SI engine's compression ratio ranges from 6 to 10, whereas a CI engine's ranges from 16 to 20. During the suction stroke of the CI engine, air is introduced rather than a fuel-air combination. The temperature at the conclusion of the compression stroke is high enough to self-ignite the gasoline delivered into the combustion chamber thanks to the use of greater compression ratios. A high-pressure fuel pump and an injector are used in CI engines to help with the fuel injection into the combustion chamber. The CI engine does not need the carburetor or ignition system that the SI engine needs [8], [9]. The ideal sequence of operations for the four-stroke CI engine as shown in Fig 4 is as follows,

1. Suction Stroke

During the suction stroke, only air is introduced. Inlet valve is open and exhaust valve is closed during this stroke, as shown in Figure 4 (a).

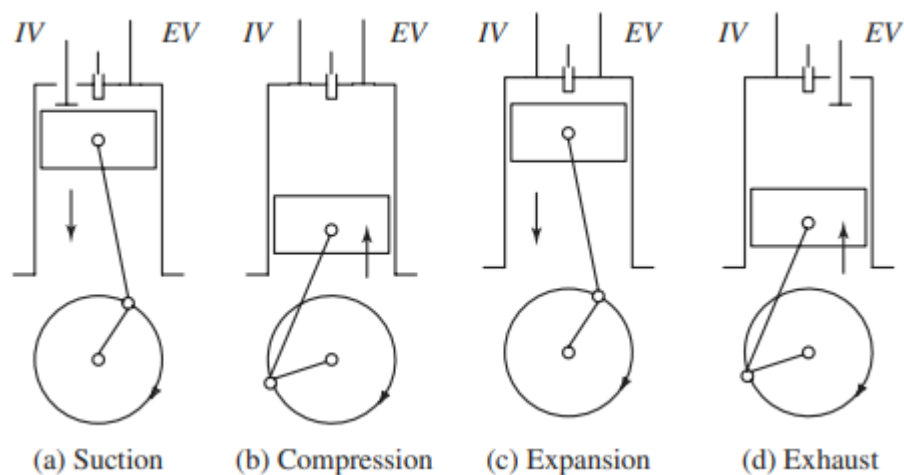


Figure 4: Cycle of operation of a CI engine (ftp.idu.ac.id)

2. Compression stroke

Inducted air is squeezed into the clearing volume during the suction stroke. During this stroke, both valves are closed, as seen in Figure 4 (b).

3. Expansion stroke

Nearly towards the conclusion of the compression stroke, fuel injection begins. Despite the piston moving throughout its expansion stroke, which increases volume, the rate of injection is such that combustion keeps the pressure constant. It is thought that heat was introduced under steady pressure. The combustion products expand after the fuel injection is finished (i.e., after cut-off). Figure 4 (c) shows that both valves are closed during the expansion stroke.

4. Exhaust stroke

The combustion byproducts are pushed out by the piston as it moves from BDC to TDC. During this stroke, Figure 4 (d) the intake valve is closed and the exhaust valve is open.

Four-stroke SI and CI Engines

In four-stroke SI and CI engines, there is one power stroke for every two crankshaft rotations. Two ineffective exhaust and suction strokes are required to flush the combustion products

from the cylinder and to load it with a new charge, respectively. If a different configuration could achieve this goal without using piston movement, it would be able to get a power stroke for each crankshaft rotation, boosting the engine's output. However, power can only be produced once per two crankshaft revolutions in SI and CI engines that run on a four-stroke cycle. Due to their similarities, SI and CI engines may be compared using a variety of key factors, including compression ratio, fuel induction, and the fundamental cycle of operation [10], [11].

CONCLUSION

In thermodynamics and engineering, a heat engine is a system that converts heat to usable energy, particularly mechanical energy, which can then be used to do mechanical work. This section has to be prepared very carefully as many readers go through this section and prepare a remark on the full paper. The future plan for the heat engine is to incorporate the cells into a grid-scale thermal battery that could absorb excess heat thermal energy from sources like the Sun and store that energy in heavily insulated banks of hot graphite. An apparatus that transforms heat energy into mechanical work is a heat engine. Internal combustion engines and external combustion engines are the two main categories of heat engines. A liquid is heated by external combustion engines, and as it expands, the heated liquid moves, producing useful work. The steam engine is one illustration of this. Contrarily, internal combustion engines produce heat by burning fuel inside the engine, which is subsequently turned into mechanical work. Examples of internal combustion engines include gasoline, diesel, and jet engines. Heat engines are widely employed in many different processes, such as industrial operations, power generation, and transportation.

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CHAPTER 2

AIR-STANDARD CYCLES AND THEIR ANALYSIS

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ABSTRACT

Air-standard cycles are thermodynamic models used to analyse the performance of internal combustion engines, gas turbines, and other power cycles. This abstract provides an overview of air-standard cycles and their analysis techniques. The fundamental assumption of air-standard cycles is that the working fluid behaves as an ideal gas and follows the ideal gas law. This simplification allows for the analysis of various thermodynamic processes within the cycle, such as compression, combustion, expansion, and exhaust. The analysis of air-standard cycles involves determining important parameters such as work output, thermal efficiency, power, and specific fuel consumption. Different cycles, including the Otto, Diesel, Brayton, and Rankine cycles, are discussed, highlighting their unique characteristics and applications. Additionally, advanced techniques such as intercooling, reheat, regeneration, and cogeneration are explored for improving cycle performance. The abstract concludes by emphasizing the importance of air-standard cycle analysis in designing and optimizing energy conversion systems.

KEYWORDS

Air-Standard Cycles, Combustion, Compression, Ideal Gas Law, Thermodynamics Analysis.

INTRODUCTION

Intake, compression, combustion, expansion, and exhaust are the individual steps that make up an internal combustion engine's operational cycle. Since the working fluid enters the system at one set of circumstances and exits at another, the internal combustion engine does not run on a thermodynamic cycle. However, by picturing one or more processes that would return the working fluid at the exit circumstances to the state of the beginning point, it is often feasible to analyse the open cycle as if it were a closed one [1]–[3]. Internal combustion engine operations are very difficult to accurately analyse. Analysing the performance of an idealised closed cycle that closely resembles the actual cycle is helpful in understanding them. The air-standard cycle is one strategy that is based on the following presumptions.

- i. The working medium is assumed to be a perfect gas and follows the relation $pV = mRT$ or $p = \rho RT$.
- ii. There is no change in the mass of the working medium.
- iii. All the processes that constitute the cycle are reversible.
- iv. Heat is assumed to be supplied from a constant high temperature source and not from chemical reactions during the cycle.
- v. Some heat is assumed to be rejected to a constant low temperature sink during the cycle.
- vi. It is assumed that there are no heat losses from the system to the surroundings.
- vii. The working medium has constant specific heats throughout the cycle.
- viii. The physical constants viz., C_p , C_v , γ and M of working medium are the same as those of air at standard atmospheric conditions. For example, in SI units.

These presumptions cause the analysis to be oversimplified, and the outcomes diverge from those of the real engine. The highest achievable values for work output, peak pressure, peak temperature, and thermal efficiency will be based on air-standard cycles and will be quite different from those of the real engine. It is often utilised, mostly because to how easy it is to get approximations of the complex processes in internal combustion engines. The numerous cycles will be discussed in this chapter, together with the formulae for work output, mean effective pressure, efficiency, etc. Additionally, a comparison between the Otto, Dual, and Diesel cycles will be done to determine which cycle is most effective given a particular set of operating circumstances.

The Carnot Cycle

In a reversible cycle, the working medium accepts heat at a higher temperature and rejects heat at a lower temperature, as postulated by French engineer Sadi Carnot in 1824. According to Figure 1, the cycle will include two isothermal and two reversible adiabatic processes. Engines may be compared to the Carnot cycle, which is regarded as the gold standard of perfection, to determine their level of perfection. It introduces the idea of maximising production while staying within two temperature thresholds.

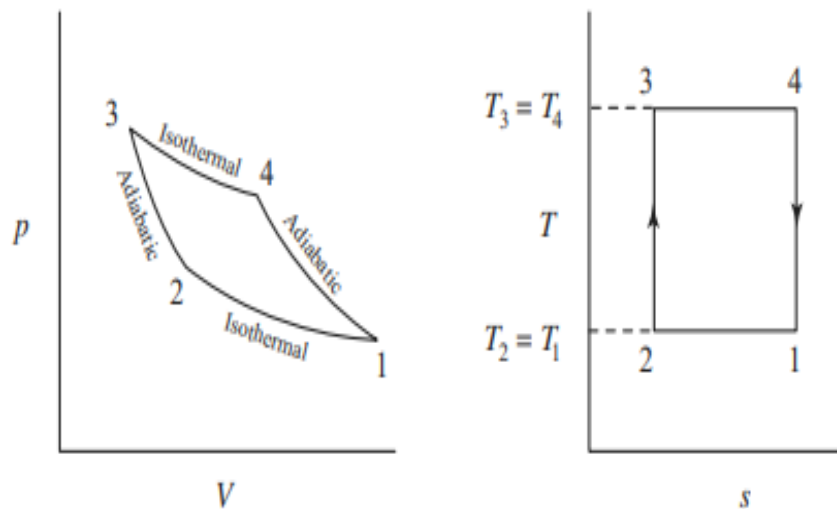


Figure 1: Carnot engine

Referring to fig 1 which depicts a cylinder and a piston arrangement operating without friction the operation of an engine based on the Carnot cycle may be described. It is believed that the cylinder's walls are ideal insulators. The cylinder head's design enables it to function perfectly as both a heat conductor and an insulator. A high temperature source (T_3) first transfer heat to the working media in the cylinder, which causes the working medium to expand. The isothermal process 3 to 4 in Fig 1 serves as a representation of this. Now the sealed cylinder head serves as an ideal insulator. It is now possible for the working medium in the cylinder to transition from state 4 to state 1 and is shown in the p-V and T-s graphs of Figure 1 by the reversible adiabatic process 4 to 1. As the cylinder head is now designed to function as an ideal heat conductor, the system is now brought into touch with a continuous low temperature sink, (T_1). The working medium is squeezed from state 1 to 2, which is symbolised by isothermal line 1 to 2, as a consequence of some heat being rejected to the sink without changing the temperature of the sink. Finally, the working medium is adiabatically compressed from state 2 to state 3, which is represented by process 2 to 3, and the cylinder head is once again made to function as a perfect insulator. The cycle is finished as a result.

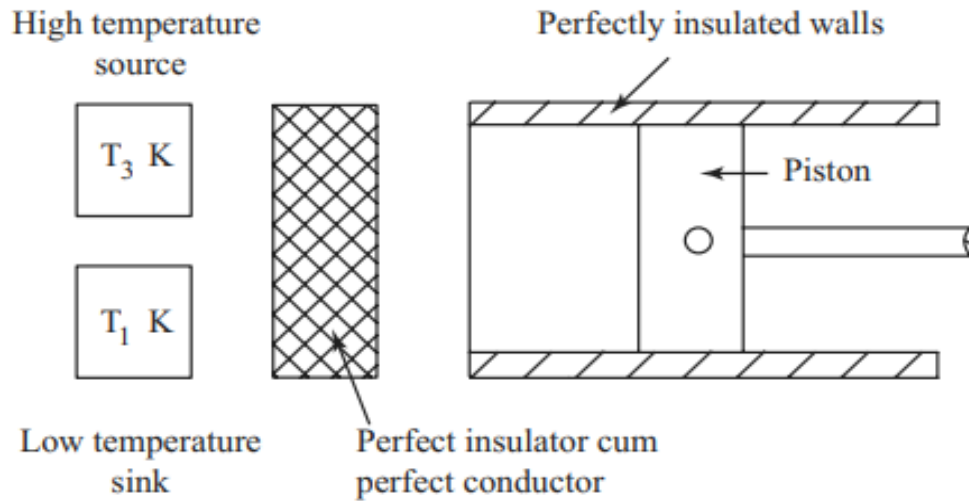


Figure 2: Working principle of a Carnot engine

The Stirling Cycle

Due to the very low work production of the Carnot cycle, it has a low mean effective pressure. Consequently, the Stirling cycle is one of the modified variants of the cycle to generate greater mean effective pressure while theoretically attaining complete Carnot cycle efficiency. It comprises of two isothermal processes and two processes with constant volumes. Heat is added while being rejected at a fixed temperature. The p-V and T-s diagrams for the Stirling cycle are shown in Figure 3(a) and 3(b) respectively. It is clear from Figure 3(b) that the amount of heat addition and rejection during constant volume processes is same.

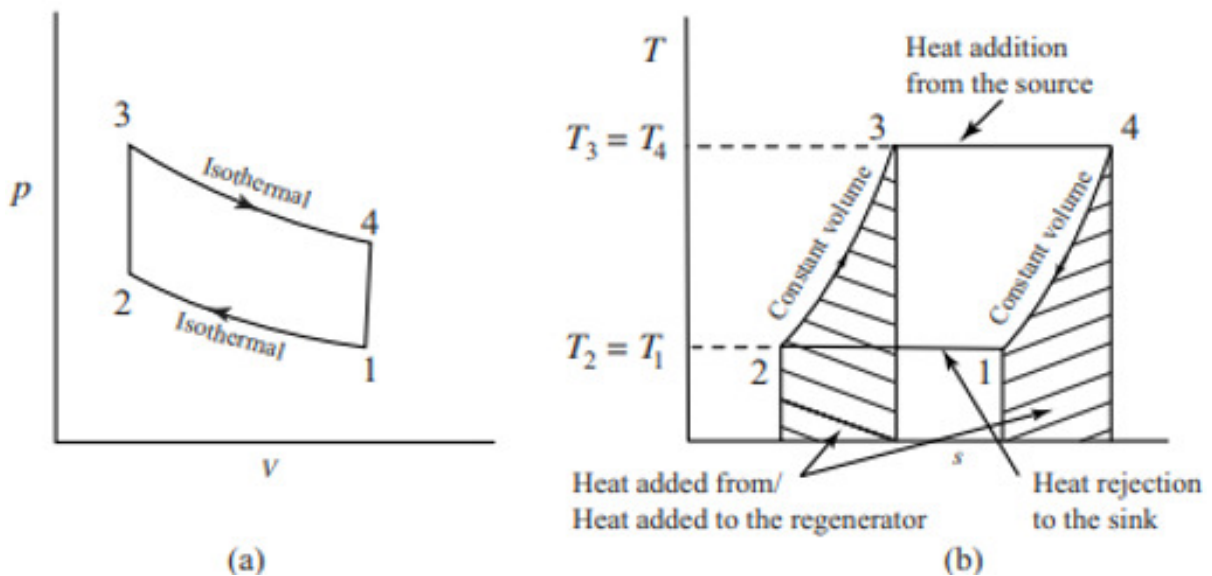


Figure 3: Stirling cycle

The Otto and Diesel cycles replaced the Stirling cycle, which was formerly utilised for hot air engines. The Stirling engine's design and construction of the heat exchanger, which must function constantly at very high temperatures, presents significant challenges. However, the Stirling engine has made a resurgence in practical use because to advances in metallurgy and much study into this kind of engine. The efficiency of the heat exchanger cannot be 100% in real life. Hence the Stirling cycle efficiency will be less than Carnot efficiency.

DISCUSSION

In the research and design of internal combustion engines and gas turbines, air-standard cycles are often employed. They serve as idealised models of these systems, which helps to clarify the intricate thermodynamic processes that take place inside of them. The fundamentals of air-standard cycles, their many varieties, and their applications may all be discussed in this topic [4]–[6]. The air-standard cycle operates in a closed loop within the engine or turbine and presumes that the operating fluid is air. It doesn't take into account the particulars of fluid characteristics, heat transmission, and combustion. Air-standard cycles, despite these simplifications, provide important insights into the performance characteristics of actual engines and turbines. The Otto cycle, Diesel cycle, Brayton cycle, and Rankine cycle are a few examples of air-standard cycles that are often used.

The Otto Cycle

The Otto cycle is used to model spark-ignition engines, such as gasoline engines. It consists of four processes: intake, compression, combustion, and exhaust. The combustion process occurs at a constant volume, representing the rapid combustion of the fuel-air mixture. Since the Carnot cycle uses high pressure and high-volume ratios with comparably low mean effective pressure, its fundamental disadvantage is that it is impractical. Modern spark-ignition engines are built on Nicolaus Otto's constant-volume heat addition cycle, which was first suggested in 1876. Figures 4 (a) and 4 (b) depict the cycle on p-V and T-s diagrams, respectively. The processes 0 to 1 and 1 to 0 on the p-V diagram represent the suction and exhaust processes, respectively, and their effects are negated when the engine is operating at full power. When the piston goes from the bottom dead centre to the top dead centre, the process 1/2 depicts the isentropic compression of the air. A consistent volume of 2 to 3 heat is provided throughout the procedure. In a real engine, combustion and spark-ignition are equivalent processes. Isentropic expansion and constant volume heat rejection, respectively, are represented by the processes 3 to 4 and 4 to 1.

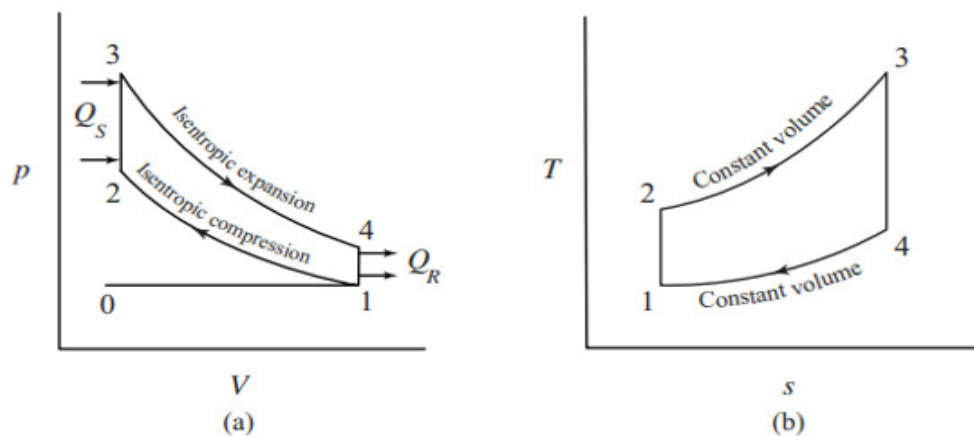


Figure 4: Otto cycle

The Diesel Cycle

Diesel engines, for example, are represented by the Diesel cycle. Intake, compression, combustion, and exhaust are among the four processes that it uses. The progressive burning of the fuel fed into the compressed air is simulated in the Diesel cycle by combustion occurring at constant pressure. The fuel's temperature at which it will spontaneously ignite in genuine spark-ignition engines sets a cap on the maximum compression ratio. By compressing the fuel and air separately and then combining them during combustion, it is

possible to get around the compression ratio's restriction. Such a configuration allows fuel to be fed into the cylinder containing compressed air at a temperature greater than the fuel's self-ignition temperature. As a result, the gasoline starts to burn on its own and doesn't need a particular mechanism like an ignition system in a spark-ignition engine. These motors run on heavy liquid fuels. These engines operate on a perfect cycle known as the Diesel cycle and are referred to be compression-ignition engines. Otto and Diesel cycles vary from one another in the manner in which heat is added. While it occurs at constant pressure in the Diesel cycle, the heat addition in the Otto cycle occurs at constant volume.

The Diesel cycle is often referred to as the constant-pressure cycle for this reason. This phrase should be avoided since it causes misunderstanding with the Joules cycle. On the p-V and T-s diagrams in Figures 5 (a) and (b), respectively, the Diesel cycle is shown. To analyse the diesel cycle the suction and exhaust strokes, represented by 0→1 and 1→0, are neglected as in the case of the Otto cycle. Here, the volume ratio V_1/V_2 is the compression ratio, r . The volume ratio V_3/V_2 is called the cut-off ratio.

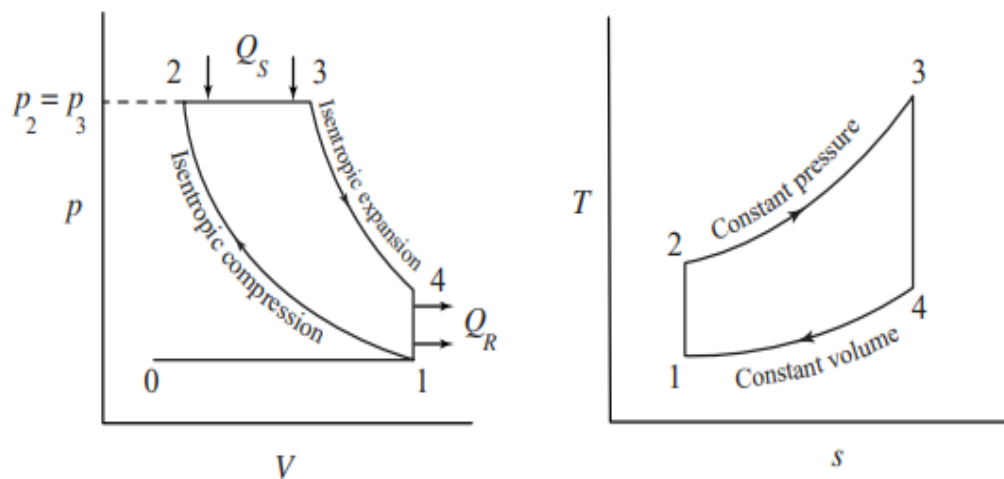


Figure 5: Diesel cycle

Brayton Cycle

Gas turbines and jet engines are often modelled using the Brayton cycle. Intake (compression), combustion, expansion (turbine), and exhaust are the four steps that make it up. The expansion process is modelled as an isentropic (reversible adiabatic) process, whereas the combustion phase occurs at constant pressure [7]–[9]. For gas turbines, the Brayton cycle is a hypothetical cycle. Two processes that are reversible, adiabatic or isentropic, and two processes that involve constant pressure make up this cycle. The Brayton cycle in p-V and T-s coordinates is shown in Figure. In terms of compression and heat addition, the cycle is comparable to the Diesel cycle. The Diesel cycle is expanded in an isentropic manner before being heated at constant pressure.

Engineers and scientists may analyse and improve the performance of engines and turbines using air-standard cycles. These idealized cycles' efficiency, power production, and heat transfer may all be calculated and compared by applying the principles of thermodynamics to them. The creation of more effective and ecologically friendly engines is aided by these assessments. It is crucial to keep in mind that air-standard cycles are oversimplified and do not account for all the intricacies of real-world systems. These idealized models do not take into account elements like mechanical losses, heat transfer losses, and combustion inefficiencies. As a result, whereas air-standard cycles provide an excellent place to start for study, actual performance may vary [10]–[12]. Figure 6 performance of engines and turbines.

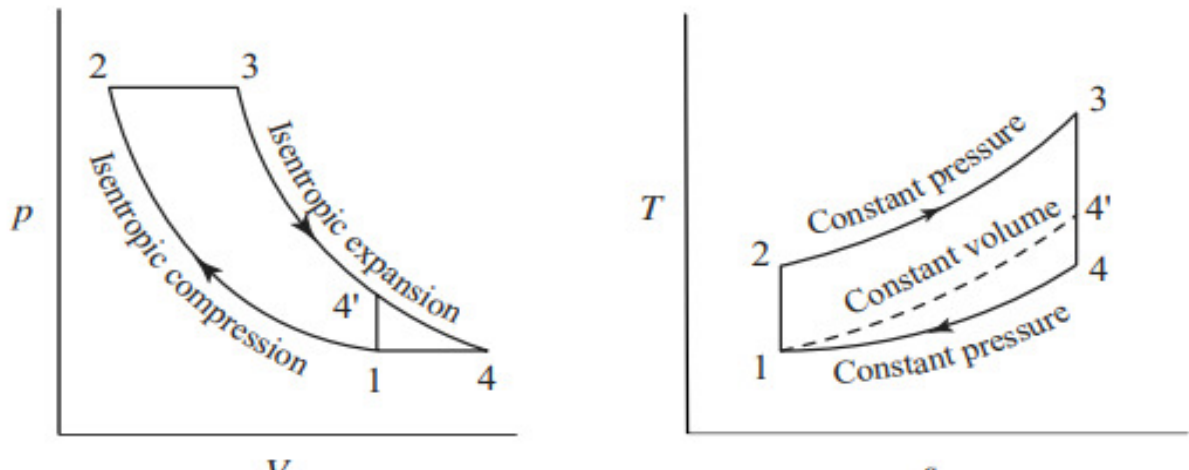


Figure 6: Performance of Engines and Turbines

CONCLUSION

Simplified models known as air-standard cycles are used to evaluate the thermodynamic efficiency of turbines and engines. They provide insightful data on power production, heat transport, and efficiency. The precision and scope of air-standard cycles will be improved in the future by developments in computer modelling and simulation methods, allowing more accurate optimization of engine and turbine designs. In addition, as the emphasis on renewable energy sources increases, air-standard cycles may be used to broaden the scope of cutting-edge technology like hybrid engines and innovative power generating systems. In conclusion, internal combustion engines and gas turbines both make extensive use of air-standard cycles for analysis and design. They provide streamlined illustrations of these systems, enabling engineers to calculate and contrast performance metrics. It's essential to comprehend these cycles in order to improve overall efficiency and construct engine and turbine designs

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CHAPTER 3

AN ANALYSIS OF FUEL AIR CYCLES

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ABSTRACT

Internal combustion engines' performance and efficiency are greatly influenced by fuel-air cycles. For engine design to be optimised and fuel efficiency to be increased, it is critical to comprehend the thermodynamic processes involved in these cycles. An overview of fuel-air cycles, including their kinds and characteristics, is given in this abstract. Additionally, it draws attention to their importance for the car industry as well as the possibilities for future developments in fuel-air cycle technology. Real engines have a working fluid that is a combination of air, fuel vapor, and leftover gases from the previous cycle. The working fluid's specific heats also change with temperature rather than being constant. Finally, at high temperatures, a specific dissociation of the combustion products occurs. It is possible to estimate pressures and temperatures that are pretty near to the real values found within the engine cylinder by taking into consideration the physical characteristics of the gases there before and after the combustion. The mean effective pressures and efficiencies determined by this study are just a few percentage higher than the actual values discovered during testing in the case of well-designed engines.

KEYWORDS

Engine Design, Fuel Air Cycle, Internal, Fuel Economy, Combustion Engine, Thermodynamics.

INTRODUCTION

Air-standard cycles were thoroughly discussed in the preceding chapter, with an emphasis on IC engines. High degree of simplification was used in the analysis. This results in a projected engine performance using air-standard cycle analysis that is greater than the real performance. For instance, a SI engine's real stated thermal efficiency, let's say with an 8:1 compression ratio, is in the range of 28%, but the efficiency of an engine operating at air-standard pressure is 56.5%. The gradual burning of the fuel, partial combustion, valve operation, etc., may to some degree be blamed for this significant variation. However, the primary causes of this may be attributable to the study's too simplistic assumptions[1]–[3]. The term "fuel-air cycle analysis" refers to the study that is based on the real qualities of the working medium, namely fuel and air, although even this analysis makes simplifying assumptions. In contrast to the settings utilized in the air-standard cycle analysis, they are more reasonable and nearer to the real-world situations.

The fuel-air cycle is greatly improved by improvements in engine design including direct fuel injection, variable valve timing, and modern engine management systems. These innovations enhance efficiency and performance by enabling precise control over the fuel-air mixture, timing of combustion, and other crucial factors. The fuel-air cycle is also changing as alternate fuels including electricity, hydrogen, and biofuels are researched. With the promise for cleaner and more sustainable energy sources in the future, these alternative fuels seek to lower emissions and dependency on fossil fuels.

Fuel–Air Cycles and their Significance

By using air-standard cycle analysis, it is clear how raising the compression ratio increases efficiency. However, since the working medium was believed to be air, research was unable to reveal the impact of the air-fuel ratio on the thermal efficiency. In this chapter, the existence of fuel in the cylinder is considered, and as a result, a combination of fuel and air will be used as the working medium. By analysing the fuel-air cycle, it will be feasible to show how the fuel-air ratio affects thermal efficiency and investigate how the peak pressures and temperatures throughout the cycle change depending on the fuel-air ratio. In general, by looking at the fuel-air cycles, it may be possible to better understand how numerous engine operating factors affect the pressures and temperatures within the engine cylinder. The following factors are included in the study of the fuel-air cycle:

- I. The exact make-up of the cylinder gases is as follows: Fuel, air, water vapour, and leftover gas are all present in the cylinder gases. During engine operation, the fuel-air ratio fluctuates, which affects the relative volumes of CO₂, water vapour, etc.
- II. Specific heats rise with temperature, with the exception of monoatomic gases, which show a specific heat fluctuation with temperature. As a result, temperature also affects the value of.
- III. The result of dissociation: under high temperatures (over 1600 K), the chemical combination of the fuel and the air is incomplete, and this results in the existence of CO, H₂, H, and O₂ under equilibrium circumstances.
- IV. A difference in the number of molecules: The ratio of fuel to air, as well as the subsequent pressure and temperature, all affect the number of molecules that remain after combustion.

Along with the above-mentioned elements, the following presumptions are often used:

- I. Prior to combustion, neither the fuel nor the air undergoes any chemical changes.
- II. After burning, the charge is always in a state of chemical equilibrium.
- III. Any process in which the gases are involved is adiabatic since there is no heat transfer between the gases and the cylinder walls. Frictionless methods are also used in compression and expansion.
- IV. It is believed that fluid motion within the cylinder may be disregarded in the case of reciprocating engines. It is also believed that, with special reference to the constant volume fuel-air cycle
- V. The air and fuel are properly blended and the fuel has entirely vaporised.
- VI. Top dead centre (at constant volume) is the exact location where the burning occurs instantly.

As previously indicated, the study of the air-standard cycle demonstrates the overall impact of merely the compression ratio on engine efficiency, but the analysis of the fuel-air cycle demonstrates the impact of variations in the fuel-air ratio, intake pressure, and temperature on the engine performance. You'll see that although intake conditions are still significant, compression ratio and fuel-to-air ratio are the two most crucial engine factors. About 85% of the projected fuel-air cycle efficiency corresponds to the actual efficiency of a decent engine. Fuel-air cycle analysis may be used to calculate the power that can be predicted from the actual engine. Furthermore, estimates of peak pressures and exhaust temperatures that have an impact on the construction and design of engines may be made that are quite similar to the real engine. As a result, fuel-air cycle analysis helps to clarify the impact of several factors on an engine's performance.

Composition of Cylinder Gases

Throughout the engine's functioning, the air-fuel ratio varies. The composition of the gases both before and after combustion is impacted by this shift in the air-fuel ratio, specifically the amount of carbon dioxide, carbon monoxide, water vapour, etc. in the exhaust gases. In four-stroke engines, the burned gases remaining in the clearance gap from the previous cycle come into touch with the new charge as it enters the engine cylinder. With according to speed and engine load, the quantity of exhaust gases in the clearance area fluctuates. This information is taken into consideration in the fuel-air cycle analysis, and the results are calculated to prepare the combustion charts. However, thanks to the advent of fast digital computers, it is now feasible to use the right numerical approaches to analyse how the composition of the cylinder gas affects the engine's performance. Results from the computer analysis may be produced quickly and precisely. As a result, computer analysis of the fuel-air cycle is easier to do than human computation [4]–[6].

DISCUSSION

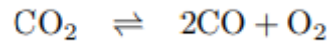
A concept related to the combustion process in internal combustion engines is the fuel-air cycle. It describes the series of activities that take place during the intake, compression, combustion, and exhaust of fuel and air within the engine. Maximising the engine's efficiency and power production is the purpose of the fuel-air cycle. An air and fuel mixture is brought into the combustion chamber during the intake stroke of the fuel-air cycle. Depending on the engine design, the fuel might be either petrol, diesel, or another kind. In order to obtain the optimum air-to-fuel ratio, which is normally approximately 14.7:1 for petrol engines, the quantity of fuel and air combination is carefully managed.

In the case of petrol engines, a spark is used to ignite the fuel and air combination during the power stroke. In the case of diesel engines, compression is used. The engine's power output is ultimately produced by the burning of the fuel, which releases energy that is then transformed into mechanical labour. Last but not least, during the exhaust stroke, the burnt combustion-process gases are forced out of the combustion chamber and out of the engine via the exhaust system. This opens the door for the start of the next cycle. In the fuel-air cycle, efficiency is a crucial factor. With higher efficiency, more of the energy produced during combustion is used to do meaningful work, such as driving a car or generating power. To increase the efficiency of engines, engineers are always working to improve the fuel-air cycle. Designing more sophisticated combustion systems, such direct injection, in which fuel is immediately fed into the combustion chamber at high pressure, is one method of increasing efficiency. Higher efficiency may result from being able to better manage the combustion process. Another strategy involves using sophisticated engine management systems to adjust the air-fuel ratio and timing of combustion. Real-time changes are made by these systems to optimise the fuel-air mixture and combustion process based on the monitoring of numerous factors, including engine load and temperature. As a way to increase the effectiveness and environmental impact of the fuel-air cycle, developments in alternative fuels like biofuels, hydrogen, or electric power are also being investigated.

Dissociation

It is possible to think of the dissociation process as the high-temperature disintegration of combustion products. You may think of dissociation as combustion going in the other direction. In contrast to combustion, which releases heat, dissociation absorbs heat. H₂O dissociation in IC engines is quite little. CO₂ is mostly split into CO and O₂ in these engines.

Around 1000 °C, the dissociation of CO₂ into CO and O₂ begins, and the reaction equation may be written as



Similarly, the dissociation of H₂O occurs at temperatures above 1300 °C and is written as



A rich fuel combination, which suppresses CO₂ dissociation by creating more CO, is an example of how the presence of CO and O₂ in the gases helps to inhibit CO₂ from dissociating. On the other hand, there is no dissociation in the burned gases of a low fuel-air combination. This is mostly because the temperature generated is insufficient for this phenomenon to occur. Therefore, the largest amount of dissociation occurs in the burned gases of the chemically right fuel-air combination when the temperatures are anticipated to be high but decreases with the leaner and richer mixes. Internal combustion engines experience a drop in maximum temperature and pressure due to heat transfer to the cooling medium. The separated elements rejoin when the temperature drops during the expansion stroke, releasing the heat that had been trapped during dissociation once again. However, it is too late in the stroke for the lost power to be fully recovered. The exhaust gases carry away some of this heat. A typical curve that depicts the drop in temperature of exhaust gas mixtures brought on by dissociation with relation to air-fuel ratio is shown in Figure 3.2. Maximum temperature is reached at the chemically ideal air-fuel ratio without dissociation. When the combination is only a little bit rich, dissociation occurs at its highest temperature. Even when the air-fuel ratio is chemically right, dissociation lowers the maximum temperature by around 300 C. In the Fig. 1, lean mixtures and rich mixtures are marked clearly [7]–[9].

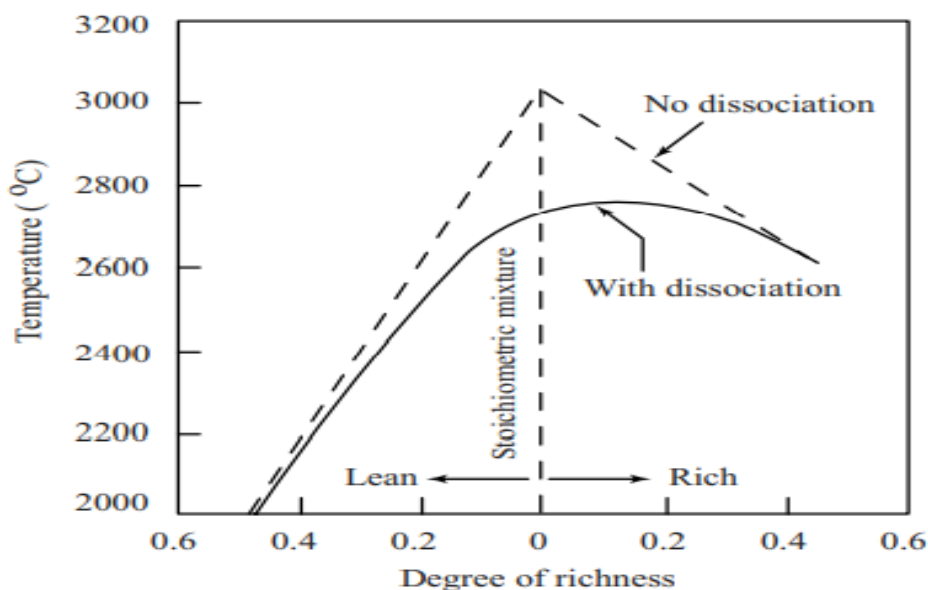


Figure 1: Effect of Dissociation on Temperature.

Figure 3.3 depicts the impact of dissociation on output power for a typical four-stroke, spark-ignition engine running at constant speed. When the mixture ratio is stoichiometric, the braking power output is greatest if there is no dissociation. The power loss resulting from dissociation may be seen in the dark region between the brake power graphs. There is no dissociation when the combination is very lean. The maximum temperature rises and dissociation starts as the air-fuel ratio falls, or as the mixture gets rich. When the combination strength is chemically right, the greatest dissociation takes place. Due to incomplete combustion, the dissociation effect tends to diminish as the mixture becomes richer.

Fuel–Air Ratio

Efficiency

The temperature increase caused by combustion will be minimised if the mixture is made leaner (with less fuel) since there is less energy input per unit mass of the mixture. The specific heat will be reduced as a consequence. Additionally, the losses brought on by dissociation and variations in specific heat will be reduced. As the fuel-air ratio is decreased, as seen in Figure 2, the efficiency is increased and actually approaches the air-cycle efficiency.

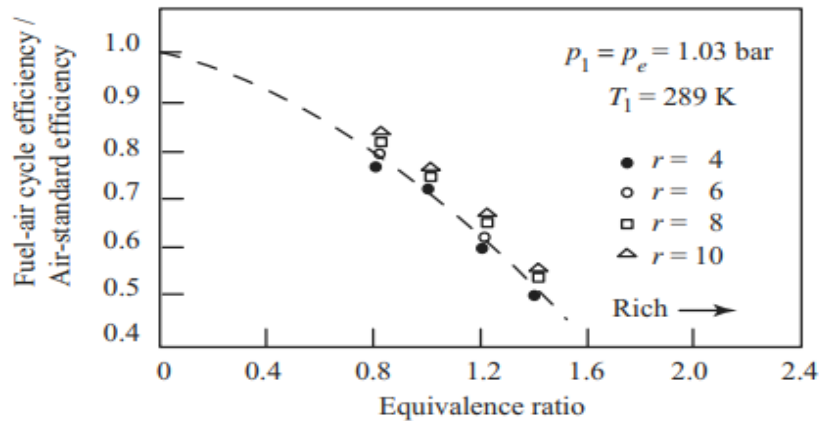


Figure 2: Variation of efficiency with mixture strength for a constant volume fuel-air cycle

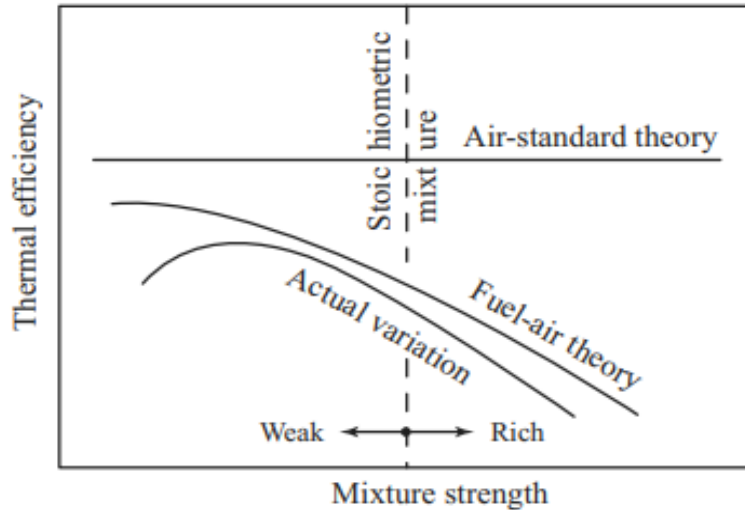


Figure 2: Effect of mixture strength on thermal efficiency

Greatest Power

The greatest power that an engine can produce is influenced by the fuel-air ratio. As seen in Fig 3, the variation. The experimental curve (Figs. 2 and 3), which shows how efficiency and power output change as the combination becomes richer, shows that efficiency and power output eventually decrease. Insufficient air will cause CO and H₂ to develop during burning, which constitutes a direct loss of fuel, in addition to greater specific temperatures and chemical equilibrium losses. Due to many simplifying assumptions, the fuel-air cycle analysis, however, cannot precisely mimic the experimental curve.

Maximum temperature

At a given compression ratio the temperature after combustion reaches a maximum when the mixture is slightly rich, i.e., around 6% or so ($F/A = 0.072$ or $A/F = 14:1$) as shown in Fig.3.12. At chemically correct ratio there is still some oxygen present at the point 3 (in the p-V diagram, refer Fig.3.1) because of chemical equilibrium effects a rich mixture will cause more fuel to combine with oxygen at that point thereby raising the temperature T_3 . However, at richer mixtures increased formation of CO counters this effect

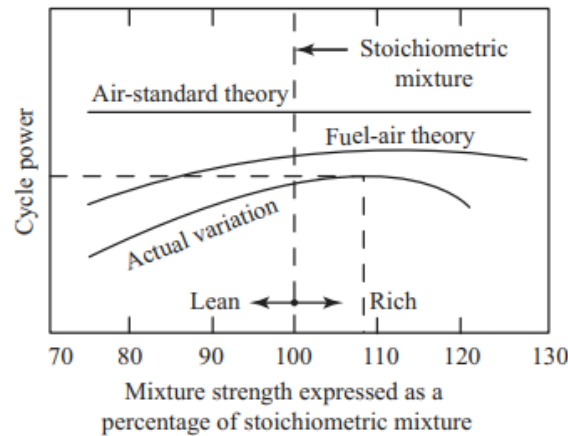


Figure 3: Effect of fuel-air ratio on power

Maximum Pressure

The pressure of a gas in a given space depends upon its temperature and the number of molecules. The curve of p_3 , therefore follows T_3 , but because of the increasing number of molecules p_3 does not start to decrease until the mixture is somewhat richer than that for maximum T_3 (at $F/A = 0.083$ or $A/F 12:1$), i.e., about 20 per cent rich (Fig 4).

Temperature of Exhaust Gas

According to Fig. 5, the chemically ideal combination results in the exhaust gas temperature, T_4 , being at its highest. Because chemical equilibria have little impact at this time, both fuel and oxygen have been totally consumed.

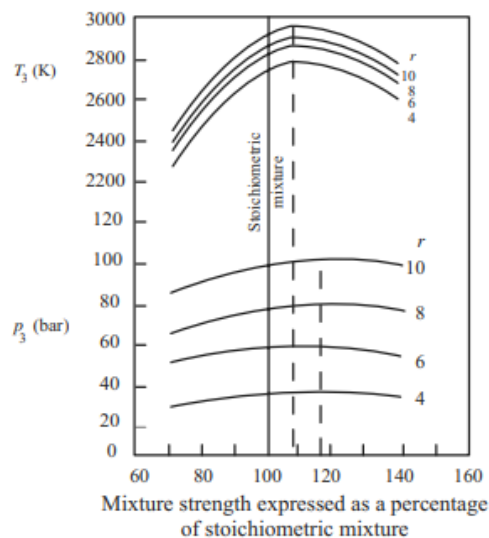


Figure 4: Effect of equivalence ratio on T_3 and p_3

Less fuel results in lower T_3 and, thus, lower T_4 at lean mixes. Less sensible energy develops in rich combinations, resulting in lower T_4 . That is, T_4 fluctuates with fuel-air ratio similarly to T_3 , with the exception that maximum T_4 occurs at the chemically right fuel-air ratio rather than the somewhat rich fuel-air ratio (6%), as in the case of T_3 .

The behaviour of T_4 with a compression ratio, however, differs from T_3 's as illustrated in Fig.5, which is a distinct characteristic. High compression ratios result in higher expansion, which makes the gas work harder on the piston and reduces the amount of heat that can be rejected at the conclusion of the stroke, lowering T_4 compared to T_3 (the exhaust gas temperature). Analysing the air-cycle also results in the same consequence[10]–[12].

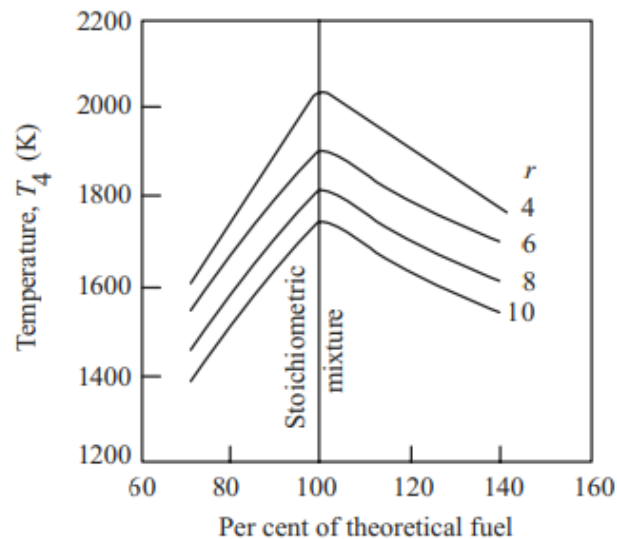


Figure 5: Effect of fuel-air ratio on the exhaust gas temperature

CONCLUSION

The process of air and fuel being drawn into an internal combustion engine, compressed, burned, and then expelled is known as the fuel-air cycle. Future applications of this technology include improving fuel economy, lowering pollutants, and developing engine technologies. The research of alternative fuels, advancements in direct fuel injection, variable valve timing, hybrid and electric powertrains, and other technologies are all included in this. In the transportation and energy industries, the fuel-air cycle's future goals include increased effectiveness, enhanced performance, and less environmental impact. The fuel-air cycle is a fundamental concept in the realm of internal combustion engines. It refers to the sequence of events that occur during the intake, compression, combustion, and exhaust of fuel and air within the engine. Understanding the fuel-air cycle is essential for optimizing engine performance, improving efficiency, and reducing environmental impact. A key factor in the fuel-air cycle is efficiency. The greatest energy possible from the fuel must be extracted and turned into productive activity in order to maximize efficiency. The fuel-air cycle is continuously being improved by engineers in an effort to increase engine performance in terms of efficiency, output, and overall quality.

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CHAPTER 4

ACTUAL CYCLES OF INTERNAL COMBUSTION (IC) ENGINES

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ABSTRACT

Actual cycles of internal combustion (IC) engines refer to the real-world thermodynamic cycles that occur during the operation of these engines. These cycles involve the processes of intake, compression, combustion, expansion, and exhaust, and they play a fundamental role in determining the engine's performance and efficiency. Analysing actual cycles of IC engines provides insights into the combustion process, heat transfer, and exhaust emissions, which are crucial for optimizing engine design and improving overall efficiency. This abstract explores the concept of actual cycles in IC engines and emphasizes their importance in understanding and enhancing engine performance. Actual cycles of internal combustion (IC) engines play a crucial role in understanding the performance and efficiency of these engines. An IC engine is a heat engine that converts the chemical energy of fuel into mechanical work through a series of thermodynamic cycles. These cycles involve a sequence of processes, including intake, compression, combustion, expansion, and exhaust. Analysing and studying the actual cycles that occur during the operation of IC engines is essential for optimizing engine performance, improving fuel efficiency, and reducing emissions.

KEYWORDS

Combustion Process, Engine Design, Engine Performance, Efficiency, Exhaust Emissions, Heat Transfer.

INTRODUCTION

The intake process involves drawing air and fuel mixture into the engine's combustion chamber, while the compression process compresses this mixture to increase its temperature and pressure. The actual combustion process occurs when the fuel-air mixture ignites, releasing energy in the form of heat, expanding the gases, and generating a high-pressure and high-temperature environment. This expansion process then converts the thermal energy into mechanical work, which is harnessed to produce power. Finally, the exhaust process expels the combustion by-products and prepares the engine for the next cycle [1], [2]. Studying the actual cycles of IC engines provides insights into various aspects of engine performance. It helps determine the efficiency of the combustion process, which influences the power output and fuel consumption. Understanding the heat transfer mechanisms within the engine and optimizing them based on the actual cycles can lead to improved thermal efficiency.

Additionally, analysing the exhaust gases and emissions generated during the cycle helps in developing strategies for emission control and complying with environmental regulations. Actual cycles can be influenced by several factors, including engine design, operating conditions, fuel properties, and combustion characteristics. These cycles may vary for different types of IC engines, such as spark-ignition (SI) engines and compression-ignition (CI) engines, as well as for different fuel types and engine configurations. In summary, the study of actual cycles of IC engines is vital for optimizing engine performance, improving fuel efficiency, and reducing emissions. By analysing these cycles, engineers can develop

innovative engine designs, enhance combustion efficiency, and implement effective emission control strategies, leading to more sustainable and environmentally friendly transportation systems [3]. In many ways, the actual cycles for IC engines are different from the fuel-air cycles and the air-standard cycles. Due to numerous losses that occur when the engine is really operating, the actual cycle efficiency is significantly lower than the air-standard efficiency. The largest losses result from

- I. Variation of specific heats with temperature.
- II. Dissociation of the combustion products
- III. Progressive combustion
- IV. Incomplete combustion of fuel
- V. Heat transfer into the walls of the combustion chamber
- VI. Blow down at the end of the exhaust process
- VII. Gas exchange process

From prior knowledge and a few straightforward engine tests, it is possible to estimate these losses, and these estimates can be applied to gauge an engine's performance.

Comparison of Air-Standard and Actual Cycles

Internal combustion engine actual cycles diverge significantly from air-standard cycles in a number of ways. These variations are primarily brought about by:

- I. The working substance is a mixture of air and fuel vapour, or liquid fuel that has been finely atomized in air with residual combustion products from the previous cycle.
- II. The change in chemical composition of the working substance.
- III. The variation of specific heats with temperature.
- IV. The change in the composition, temperature and actual amount of fresh charge because of the residual gases.
- V. The progressive combustion rather than the instantaneous combustion
- VI. The heat transfer to and from the working medium.
- VII. The substantial exhaust blow down loss, i.e., loss of work on the expansion stroke due to early opening of the exhaust valve.
- VIII. Gas leakage, fluid friction etc., in actual engines.

The details of Points (I) to (IV), which are connected to Fuel-Air Cycles, have already been covered in Chapter 4. In actuality, the remaining points, viz. (v) to (viii), are what account for the discrepancy between fuel-air cycles and real cycles. The majority of the above-mentioned elements have the tendency to reduce the thermal efficiency and power output of the actual engines.

On the other hand, analysis of the cycles with these parameters taken into account demonstrates clearly that the estimated thermal efficiencies are not significantly different from those of the actual cycles. Out of all of the aforementioned variables, a significant amount of impact is wielded by

- I. Time loss factor, or the loss brought on by the time needed for combustion as well as the mixing of fuel and air.
- II. Heat loss factor, which is the heat that is lost from gases to cylinder walls.
- III. Exhaust blow down, or loss of work during the expansion stroke as a result of an exhaust valve that opened too soon, is factor. In the sections that follow, we talk about these significant losses that weren't covered in the first two chapters.

DISCUSSION

Internal combustion (IC) engine actual cycles are discussed with reference to their characteristics, factors that influence them, and their relevance in terms of engine performance and efficiency [4]–[6].

Characteristics of Actual Cycles

Intake

During the intake phase, a fuel and air mixture is introduced into the engine's combustion chamber. Variations in intake manifold pressure, temperature, and air-fuel ratio are all taken into consideration during the actual cycle.

Compression

The air-fuel mixture is forced to a high pressure and temperature during compression. The actual cycle takes into account any variations from ideal gas behaviour as well as the impacts of compression ratio and heat transfer.

Combustion

An air-fuel mixture is ignited during the actual combustion process, which causes a sudden release of energy and a rise in pressure. The actual cycle is influenced by things like the length of the combustion, the spread of the flame, and the rate at which heat is released.

Expansion

When high-pressure gases expand, thermal energy is transformed into mechanical work. Variations in the expansion ratio, heat transfer to the cylinder walls, and frictional losses are all taken into account throughout the actual cycle.

Exhaust

The outflow of combustion by-products is a part of the exhaust process. The real cycle examines the changes in temperature and pressure that occur throughout the exhaust stroke, as well as how back pressure affects engine performance.

Factors Affecting Actual Cycles

Engine Design

The actual cycles' characteristics are influenced by engine factors like bore, stroke, compression ratio, valve timing, and geometry. The flow rates of the intake and exhaust gases, the efficiency of combustion, and the heat transfer all vary depending on the engine design.

Operating Conditions

The actual cycles are affected by variables like as engine speed, load, ambient temperature, and altitude. The air-fuel ratio, timing of combustion, and thermodynamic parameters of the working fluid can all change as a result of changes in operating conditions.

Fuel properties

Fuel Composition, Octane or Cetane Rating, and Fuel Injection Characteristics All Affect the Actual Cycles. Inflammation delay, combustion time, and emissions are all impacted differently by different fuels' different combustion properties.

Variability of Combustion

In actual use, turbulence, mixture inhomogeneity, and the quality of the fuel all contribute to variation in the combustion process in IC engines. The performance of the engine is impacted by the variability that causes variations from idealised cycles.

Significance in Engine Performance and Efficiency

Power Output

Examining real cycles allows for a better understanding of the variables affecting an IC engine's power output and torque characteristics. Engineers can develop engines with enhanced power delivery and performance by optimising the actual cycles.

Fuel Efficiency

Information about the engine's heat transfer and combustion efficiency is gleaned from the actual cycles. Engineers can increase fuel efficiency and minimise energy losses during combustion by researching and enhancing these cycles.

Control of Emissions

Researching and regulating emissions in IC engines requires the use of actual cycles. Engineers can create plans for minimising pollution and adhering to emission rules by studying the combustion process and its effects on emissions.

Engine Design and Optimisation

Analysing actual cycle aids in optimising engine design characteristics including compression ratio, valve timing, and fuel injection techniques. As a result, frictional losses are decreased, overall performance is better, and engine efficiency is improved.

Heat Loss Factor

Heat from the cylinder gases passes through the cylinder walls and cylinder head and into the water jacket or cooling fins during combustion and the ensuing expansion stroke. A small amount of heat enters the piston head, passes through the piston rings, and then escapes through the cylinder wall or is absorbed by the engine lubricating oil that splashes on the piston's underside. Figure 1 time loss, heat loss and exhaust loss in petrol engines

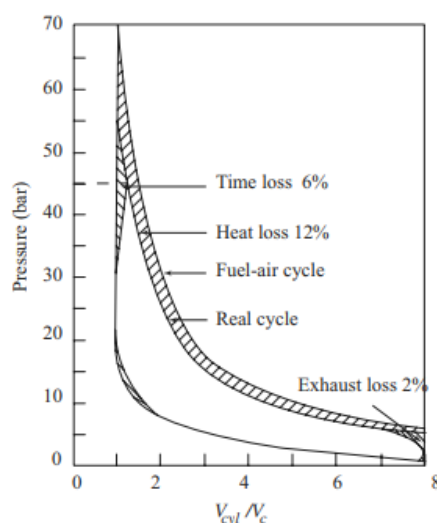


Figure 1: Time loss, heat loss and exhaust loss in petrol engines

The cycle efficiency will naturally be most affected by heat loss during combustion, whereas heat loss right before the conclusion of the expansion stroke can have very little impact because it contributes very little useful work. Even under the most ideal conditions anticipated for an air-standard cycle, only a portion of the heat lost during combustion could be turned into work, and the remainder would be rejected during the exhaust stroke. As a result, the heat lost during combustion does not represent a total loss.

In the process of combustion and expansion, about 15% of the total heat is lost. But a lot of it is lost too late in the cycle to have made a difference. Only approximately 20% of the heat loss could be perceived as useful work if it is completely recovered.

In a Cooperative Fuel Research (CFR) engine, time loss, heat loss, and exhaust loss are all depicted as percentages in Figure 4.8. Losses are expressed as a % of the effort in the fuel-air cycle. The consequence of heat loss during combustion is to lower the maximum temperature, leading to lower specific heats. The heat loss factor accounts for around 12% of the total losses.

Volumetric Efficiency

Volumetric efficiency, which is defined as the ratio of the volume of air actually admitted under ambient condition to swept volume, is a measure of the engine's ability to breathe, as it was already mentioned. However, it can also be defined in terms of mass as the ratio between the actual mass of air drawn into the engine over the course of a given period of time and the theoretical mass that should have been drawn in over the course of that same period of time, based on the total displacement of the engine's pistons as well as the temperature and atmospheric pressure. Only naturally aspirated engines fall under the scope of the aforementioned definition.

The theoretical mass of air should be computed for the supercharged engine, though, under the pressure and temperature that are present in the intake manifold. Numerous factors influence volumetric efficiency; among the crucial ones are: The density of a fresh charges. The density of the new charge: As the new charge is introduced into the hot cylinder, heat is transferred to it from the hot chamber walls and the hot residual exhaust gases, boosting its temperature.

As a result, the volumetric efficiency is decreased and the mass of fresh charge accepted decreases. Low temperatures and high pressure of the fresh charge boost volumetric efficiency because they increase density and allow for the induction of more mass of charge into a given volume (assuming there are no heat transfer effects).

The exhaust gas in the clearance volume

As the piston transitions from T to BDC on the intake stroke, these products have a tendency to expand and take up a portion of the piston displacement greater than the clearance volume, so lowering the space available for the incoming charge. Furthermore, these exhaust by-products have a propensity to increase the temperature of the fresh charge, lowering its density and further lowering volumetric efficiency.

The design of the intake and exhaust manifolds

The intake manifold should be made to allow for the maximum amount of fresh charge to be introduced, whilst the exhaust manifold should be made to allow for the easy escape of the exhaust products. This indicates that both the exhaust products being driven out of the cylinder and the fresh charge flowing into it are subjected to the least amount of restriction possible [7].

The intake and exhaust valves' timing

The regulation of the times of the cycle at which the valves are programmed to open and close is known as valve timing. A small "lead" time is required for proper opening and closing of the valves since they need a specific amount of time to open or close in order to function smoothly. The cam setting dictates the timing of the valve, while the design of the valve working cam ensures a seamless transition from one position to the next.

By affecting the amount of air injected per cylinder every cycle, or the volume of air sucked into one cylinder during one suction stroke, the intake valve timing's impact on the engine's air capacity may be seen. Intake valve timing examples for both a low speed and high-speed SI engine are shown in Figure 2. Referring to Fig. 2, the intake process should be followed in order to comprehend the impact of the intake valve timing on the charge injected per cylinder every cycle. Almost all SI engines use an intake valve opening that occurs a few degrees before TDC on the exhaust stroke, even though the intake valve should theoretically open at TDC. By doing this, it will be ensured that the valve will be fully open and that the fresh charge will begin to flow into the cylinder as soon as the piston reaches TDC. The intake valve begins to open 10° before TDC in Figure 2. The intake valve closes 10° after BDC for low-speed engines and 60° after BDC for high-speed engines, as shown in Figure 2.

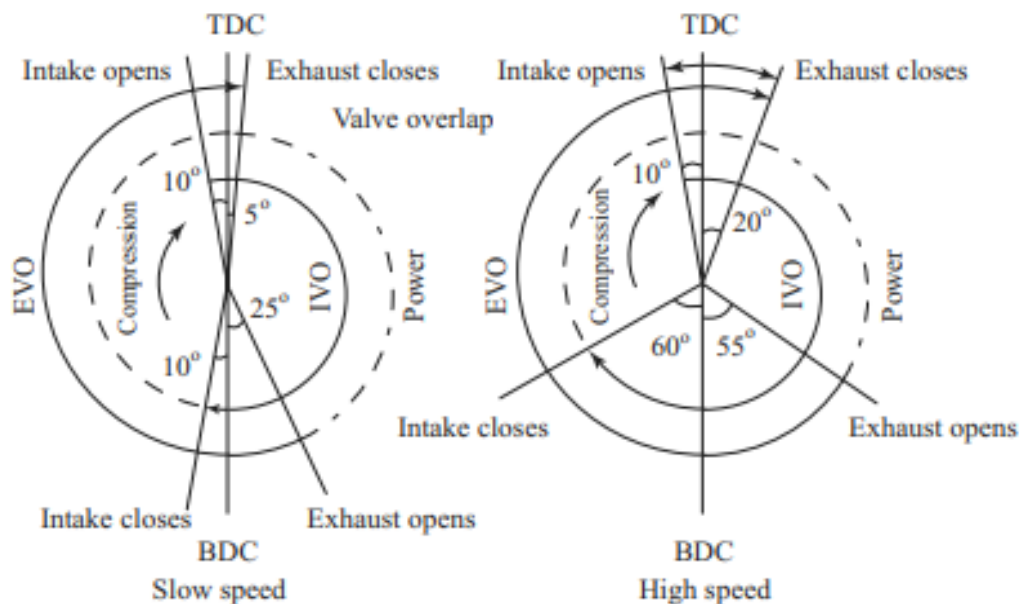


Figure 2: Valve timing diagram of four-stroke engines

The fresh charge is brought in through the intake port and valve as the piston depresses on the intake stroke. The piston tends to continue moving inside the cylinder until it reaches BDC and begins to rise during the compression stroke due to the inertia of the incoming new charge. The charge enters the cylinder very slowly at low engine speeds because of its low inertia. The upward motion of the piston during the compression stroke would tend to force some of the charge already in the cylinder back into the intake manifold, which would reduce volumetric efficiency if the intake valve were to remain open for a long time after BDC. As a result, for an engine running at a low RPM, the intake valve closes rather soon after BDC. However, high speed engines inject the charge via the intake manifold at higher rates, and the charge has more inertia. The entering mixture creates a "ram" effect when the piston rises during the compression stroke, which tends to fill the cylinder with more charge. To take advantage of this "ram" and induct the greatest amount of charge, the intake valve shutting is consequently delayed for a longer time after BDC in the high-speed engine.

There is a specific speed at which the charge per cylinder per cycle reaches its maximum for a given valve setting, whether the engine is working at low or high speeds within its range of speeds. Beyond this point, the low-speed engine's revolutions will rise at the expense of a lower charge per cylinder every cycle due to an intake valve that closes prematurely. If the high-speed engine's rotations are raised over this limit, fluid friction may cause the flow to become impeded. The charge per cylinder every cycle can drop as a result of these losses growing larger than the ram's advantages. A compromise between the best setting for the low-speed end of the range and the best setting for the high speed end must always be made when choosing an intake valve setting for an engine operating over a range of speeds. The volumetric efficiency is influenced by the timing of the exhaust valve as well. When the piston is on the expansion stroke, the exhaust valve typically opens before BDC. As a result, there is a reduction in the work required to expel the burned products during the exhaust stroke, which increases overall output while also reducing the work required to expand gases during the power stroke.

The piston expels the burned gases at a high rate of speed during the exhaust stroke. Increased volumetric efficiency occurs as a result of the exhaust valve's inertia, which tends to scavenge the cylinder more effectively by carrying out a greater mass of the gas still in the clearance volume if the exhaust valve is left open for longer than T DC. The exhaust valve is often designed to close a few degrees after T DC on the exhaust stroke. It should be noted that it is very conceivable for the exhaust and intake valves to both be open or partially open at the same time. The valve overlap refers to this. Naturally, there must be some overlap in order to prevent the fresh charge from escaping through the exhaust valve or the consumed gases from being pulled into the intake manifold.

In the previous section, only the dynamic impacts of gas flow were taken into account to illustrate the reasons why valve overlap and timings other than at T DC or BDC are necessary. But it's important to understand that a mechanical issue with the valves' actuation can affect how soon they open. The problem of acceleration involved prevents the valve from being raised to the correct height instantly; instead, it must be opened gradually [8], [9]. Whenever designing a cam, a quick change in acceleration from positive to negative values may occur. A blow on the cam may arise from the cam follower losing contact with the cam and then being pressed by the valve spring to re-establish close contact. The cam contours are made in such a way as to provide moderate and smooth changes in directional acceleration since this kind of activity must be avoided [10].

CONCLUSION

The term "actual cycles of IC engines" refers to the thermodynamic cycles that actually take place while these engines are running, taking into account differences in the processes of intake, compression, combustion, expansion, and exhaust. Future applications of actual cycles will focus on analysing and improving engine performance using cutting-edge computational methods like multidimensional modelling and simulation. The comprehension and optimisation of real cycles will be further improved by the use of alternative fuels, hybrid technologies, and modern control systems, resulting in more effective and environmentally friendly IC engines. Consequently, the valve must start opening before it is fully opened. The closure hour is justified using the same logic. It is clear from this that mechanical and dynamic factors play a role in the timing of valves. The settings are chosen using the prototype of the actual engine, and both the intake and exhaust valves are typically timed to provide the most favourable outcomes for the average operating conditions of the specific engine.

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CHAPTER 5

AN ANALYSIS OF DIFFERENT CONVENTIONAL FUELS

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ABSTRACT

Conventional fuels, derived from fossil resources such as crude oil, natural gas, and coal, have long served as the primary energy sources for transportation, electricity generation, and industrial processes. These fuels, including gasoline, diesel, and coal, offer high energy density and a well-established infrastructure. However, their extensive use contributes to environmental issues, such as air pollution, greenhouse gas emissions, and climate change. As the world seeks cleaner and sustainable energy alternatives, the future of conventional fuels faces challenges, prompting the exploration and adoption of alternative and renewable energy sources. This abstract provides an overview of conventional fuels, their environmental impact, and the need for transition to more sustainable energy options. Conventional fuels are widely used energy sources derived from fossil resources, including crude oil, natural gas, and coal. They have been the primary energy carriers for transportation, electricity generation, and industrial processes for many years.

KEYWORDS

Coal, Conventional Fuels, Crude Oil, Fossil Fuels, Natural Gas.

INTRODUCTION

Conventional fuels, such as gasoline, diesel, and coal, offer high energy density, ease of transportation, and a well-established infrastructure for extraction, refining, and distribution. However, their use has significant environmental consequences, including air pollution, greenhouse gas emissions, and contribution to climate change. As the world focuses on sustainable and clean energy solutions, the future of conventional fuels is being challenged, leading to increased efforts to develop and adopt alternative and renewable energy sources. Since these engines were first developed, fuels for IC engines have been the subject of research. With the help of the engine, heat energy can be converted into mechanical energy. Heat energy is produced chemically when fuel and oxygen are combined. A basic understanding of the many fuel kinds and their properties is necessary to comprehend the combustion phenomenon since the fuel is where the heat energy originates. The design, performance, output, and especially the dependability and durability of the engine are all greatly influenced by the properties of the gasoline that is utilised. Additionally, the fuel's properties are very important in determining how much pollution is produced by car engines.

Fuels

Different fuel types, such as liquid, gaseous, and even solid fuels, can be used to power internal combustion engines. The engine needs to be built in accordance with the type of fuel that will be utilised.

Solid Fuels

As a result of difficulties handling the fuel and getting rid of the solid by-products of combustion, or ash, solid fuels currently have limited practical use. But solid fuels, such as finely ground coal, were used in the early stages of engine development. Solid fuels are much

more difficult to work with than gaseous or liquid fuels, and they take up a lot more space to store and feed. These fuels are no longer appropriate in their solid state due to difficulties in the design of the fuel feed systems. In order to employ charcoal in IC engines, efforts are being made to turn it into gaseous or liquid fuels.

Gaseous Fuels

Internal combustion engines can run on gaseous fuels with no difficulty. They do not have the distribution or starting issues associated with liquid fuels since they are gaseous, which mixes more uniformly with air. The usage of gaseous fuels in automobiles is limited despite the fact that they are the best fuels for internal combustion engines. This is because of storage and handling issues. As a result, stationary power plants close to the fuel's source are frequently equipped with them. In order to decrease the storage volume, some gaseous fuels can be liquefied under pressure, however this method is both exceedingly expensive and dangerous. A lot of research is being done to enhance the design and performance of petrol engines, which became outdated when liquid fuels were first used because of the recent energy crisis.

Liquid Fuels

Liquid fuels, which are derived from liquid petroleum, are used in the majority of modern internal combustion engines. Benzyl, alcohol, and petroleum products are the three main commercial types of liquid fuels. But as of right now, petroleum products are the primary fuels for internal combustion engines [1], [2].

Chemical Structure of Petroleum

Petroleum as obtained from the oil wells, is predominantly a mixture of many hydrocarbons with differing molecular structure. It also contains small amounts of sulphur, oxygen, nitrogen and impurities such as water and sand. The carbon and hydrogen atoms may be linked in different ways in a hydrocarbon molecule and these linking influences the chemical and physical properties of different hydrocarbon groups. Most petroleum fuels tend to exhibit the characteristics of that type of hydrocarbon which forms a major constituent of the fuel. The carbon and hydrogen combine in different proportions and molecular structures to form a variety of hydrocarbons. The carbon to hydrogen ratio which is one of the important parameters and their nature of bonding determine the energy characteristics of the hydrocarbon fuels. Depending upon the number of carbon and hydrogen atoms the petroleum products are classified into different groups. The differences in physical and chemical properties between the different types of hydrocarbons depend on their chemical composition and affect mainly the combustion processes and hence, the proportion of fuel and air required in the engine. The basic families of hydrocarbons, their general formulae and their molecular arrangement are shown in Table 1.

Table 1: Basic families of hydrocarbons

Family of hydrocarbons	General formulae	Molecular structure	Saturated/Unsaturated	stability
Paraffin	C_nH_{2n+2}	Chain	Saturated	Stable
Olefin	C_nH_{2n}	Chain	Unsaturated	Unstable
Naphthenic	C_nH_{2n}	Ring	Saturated	Stable
Aromatic	C_nH_{2n-6}	Ring	Highly unsaturated	Most unstable

Paraffin Series

The normal paraffin hydrocarbons are of straight chain molecular structure. They are represented by a general chemical formula, C_nH_{2n+2} . The molecular structures of the first few members of the paraffin family of hydrocarbons are shown below. Figure 1 butane and isobutane.

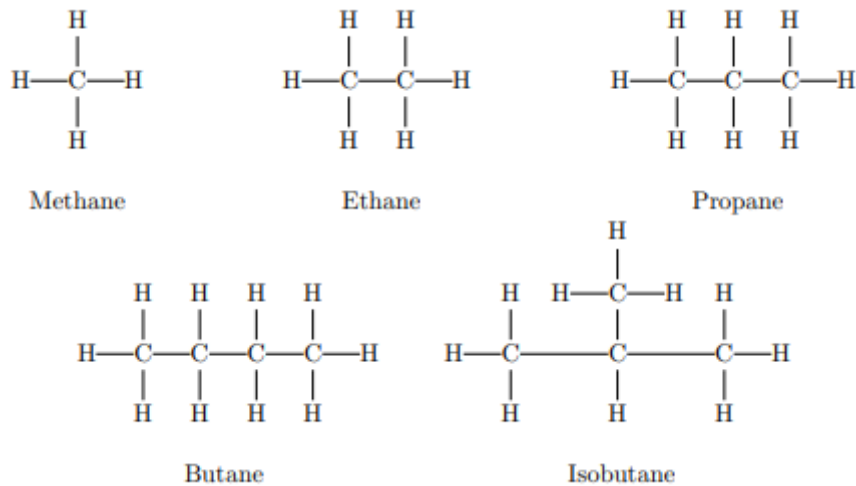


Figure 1: Butane and Isobutane.

All of the carbon atoms in these hydrocarbons have single bonds with hydrogen atoms that completely utilise their valency. As a result, the paraffin hydrocarbons are saturated substances and have a reputation for being extremely stable. A branched chain paraffin is a variant of the paraffin family that has an open chain structure with an attached branch. Isomers are hydrocarbons that share the same chemical formula but differ in their structural formula. The isobutane seen above has a different molecular structure and set of physical properties despite sharing the same overall chemical formula and molecular weight as butane. It is referred to as isobutane and is an isomer of butane. Is paraffins are likewise stable substances [3].

Olefin Series

Similarly, to paraffins, olefins are straight-chain compounds, but they are unsaturated because they have one or more double bonds between the carbon atoms. Their molecular structure is C_nH_{2n} . While diolefins have two double bonds, mono-olefins only have one. Figure 2 hexene and butadiene.

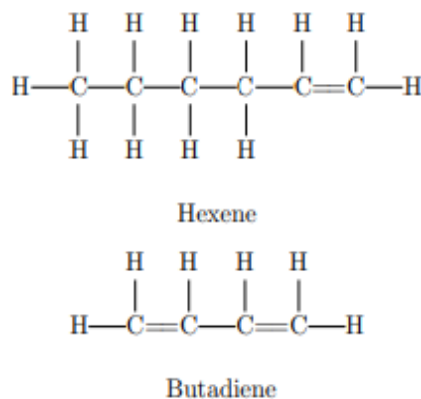


Figure 2: Hexene and butadiene.

Due to the double bonds in their structure, olefins are less stable than paraffins with a single bond. As a result, compounds easily undergo oxidation during storage and develop gummy deposits. Olefin content in several petroleum products is therefore restricted by specification.

Naphthene Series

Despite having a ring structure and the same chemical formula as the olefin class of hydrocarbons, naphthenes are frequently referred to as cyclo-paraffins. They are stabilised and saturated. Olefins are unsaturated chemicals, whereas naphthenes are saturated. One of the substances in the naphthene series (C_nH_{2n}) is cyclopentane. Figure 3 shows the cyclopentane.

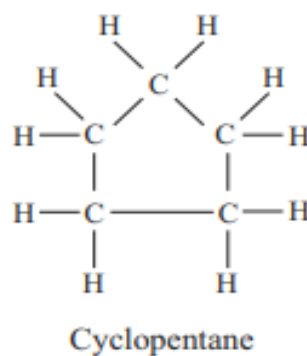


Figure 3: Shows the cyclopentane.

DISCUSSION

Petroleum Refining Process

Methane and ethane make up the majority of the gases in crude petroleum when it comes from the oil wells, along with other gases, solids, and other contaminants. By using a fractional distillation technique, crude oil is divided into different types of liquids like petrol, kerosene, and fuel oil. The theory behind this procedure is that as molecular weight increases, different hydrocarbons' boiling temperatures rise [4].

An easy-to-understand picture of the refining of crude oil can be shown; however, this model omits the locations of the facilities used for process utilisation, desulphurization, etc. The petroleum is sent through a separator in the first phase, where the fumes are taken out and a substance known as natural petrol is produced. After being heated to 600 C in a still, the liquid petroleum is then evaporated, and the resulting vapour is introduced at the base of the fractionating tower. The liquid fuel is maintained in trays at various temperatures, and the vapour is pushed to move through them as they are arranged like a labyrinth on a set of plates. Lower boiling point substances travel up to higher levels where they are condensed in trays at the proper temperature, while higher boiling point substances condense out at lower levels. The top fraction, commonly referred to as straight-run petrol, is obtained at an increasing range of boiling temperatures, whereas the other fractions, such as kerosene, diesel oil, and fuel oils, are obtained under other conditions[5]–[7]. The conversion of some of these fractions into compounds with higher demand can be accomplished by a variety of procedures. The following list includes a few of the major refinery procedures. Figure 4 refining process of petroleum.

In order to crack, huge, complex hydrocarbon molecules must be broken down into smaller, simpler ones. Large hydrocarbon molecules that are subjected to thermal cracking are broken up into smaller molecules with lower boiling points by being exposed to high pressure and

temperature. The use of catalysts in catalytic cracking allows for thermal cracking to be done at temperatures and pressures that are generally lower. By catalysis, naphthenes are converted to olefins, paraffins, and olefins, which are subsequently converted into isoparaffins, which are needed to make petrol.

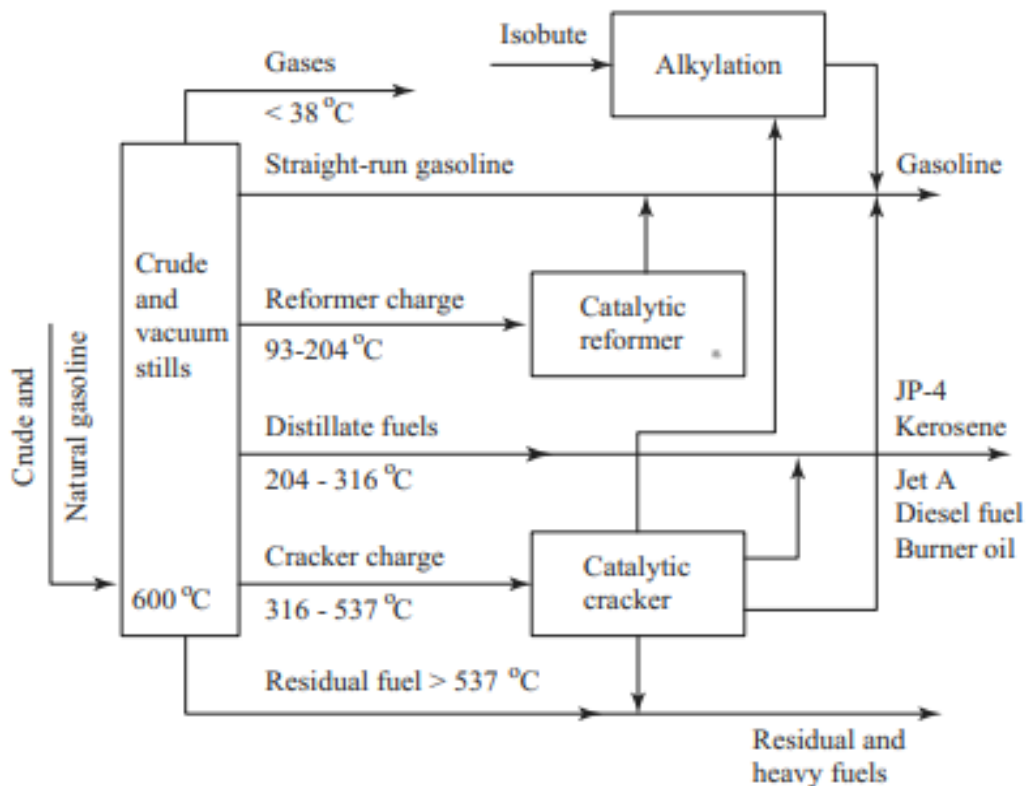
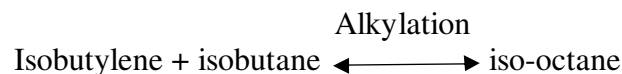


Figure 4: Refining Process of Petroleum

Compared to thermal cracking, catalytic cracking gives petrol superior anti-knock properties. To create more aesthetically pleasing molecules, certain hydrocarbons are subjected to the process of hydrogenation, which involves the addition of hydrogen atoms at high pressure and temperature. It is frequently used to change unstable molecules into stable ones. The process of polymerization involves turning olefins, the unsaturated by-products of cracking, into heavier and more stable molecules. Alkylation, which occurs in the presence of a catalyst, joins an olefin and an isoparaffin to create a branched chain isoparaffin. Example,



Isomerization modifies the relative positions of the atoms within a hydrocarbon molecule without altering the molecule's molecular formula. For instance, isomerization is used to change n-butane into isobutane so that it can be alkylated. Another example is the conversion of n-pentane and n-hexane into isoparaffins to raise the knock rating of highly volatile petrol.

- I. A ring compound of the naphthene family is created via cyclization, which links the ends of a straight chain molecule together.
- II. Aromatization is a process that is comparable to cyclization, with the difference being that the end result is an aromatic molecule.

- III. Reformation, a sort of cracking procedure, is used to turn low antiknock quality stocks into petrol with a higher-octane rating. The overall volume of petrol is unchanged.
- IV. Blending is the process of combining various products in an appropriate ratio to produce a product of the desired quality.

Table 2: Products of petroleum refining process.

S.no.	Fraction	App. boiling range, °C	Remarks
1.	Fuel gas	-160 to -44	Methane, ethane and some propane used as refinery fuel
2.	Propane	-40	LPG
3.	Butane	-12 to 30	Blended with motor gasoline to increase its volatility
4.	Light Naphtha	0 to 150	Motor gasoline for catalytic reforming
5.	Heavy Naphtha	150 to 200	Catalytic reforming fuel, blended with light gas oil to form jet fuels
6.	Kerosene Middle distillate	200 to 300	Domestic, aviation fuels
7.	Light Middle distillate	200 to 315	Furnace fuel oil, diesel fuels
8.	Heavy gas oil	315 to 425	Feed for catalytic cracking
9.	Vacuum gas oil	425 to 600	Feed for catalytic cracking
10.	Pitch	>600	Heavy fuel oil, asphalts

Rating of Fuels

The antiknock properties of fuels are often rated. The octane number and cetane number, which are used to rate different types of fuels for petrol and diesel oil, respectively, are two metrics. In this part, we talk about fuel rating for both SI and CI engines.

Rating of SI Engine Fuels

An essential quality of gasoline for spark-ignition engines is resistance to knocking. Based on their chemical make-up, these fuels' capacity to resist knock varies greatly. For comparing the antiknock properties of the various fuels, a reliable rating system has been developed. Other operating parameters, such as fuel-air ratio, ignition time, dilution, engine speed, combustion chamber shape, ambient conditions, compression ratio, etc., affect the likelihood to knock in the engine cylinder in addition to the chemical properties of the hydrocarbons in the fuel. Therefore, the engine and its operating variables must be set at standard values in order to calculate the knock resistance characteristic of the fuel. The antiknock value of a SI engine fuel is often calculated by contrasting it with a blend of iso-octane (C₈H₁₈) and regular heptane (C₇H₁₆), two reference fuels. Chemically, iso-octane is a very good antiknock

gasoline, hence it is arbitrarily given an octane value of 100. On the other hand, normal heptane (C_7H_{16}) receives a rating of 0 octane number due to its extremely low antiknock properties. The proportion of iso-octane by volume in a blend of iso-octane and regular heptane that precisely matches the fuel's knocking intensity in a typical engine under a set of standard operating conditions is known as the Octane number fuel. Iso-octane can be combined with some substances, such as tetraethyl lead, to provide fuels with higher antiknock qualities (octane numbers exceeding 100). Tetraethyl lead's antiknock effectiveness declines with increasing lead content in the fuel for the same amount of lead added. Additionally, compared to the identical unit at the lower end of the scale, each octane number in the higher region of the octane scale will generate a stronger antiknock impact. For instance, an increase in octane from 92 to 93 produces a stronger antiknock effect than an equivalent rise from 32 to 33.

Laboratory Procedure

The engine is operated under predetermined circumstances with a specific compression ratio and a specific mixture of reference fuels. Standard knock is the level of knock in these usual circumstances. The knock metre is set up to deliver a specific reading in these circumstances. The engine is currently running on the test fuel, and the air-fuel ratio has been tweaked to produce the most knock. When the knock metre reading is the same as it was during the prior run (standard knock), the engine's compression ratio is gradually increased. Now that the compression ratio is determined, the engine runs on well-known mixtures of reference fuels. The reference fuel mixture that produces a knock metre reading equal to the benchmark value will match the test fuel's knocking characteristics. The blend's specific octane number is calculated as a percentage of iso-octane by volume.

Rating of CI Engine Fuels

The knock resistance of compression-ignition engines is influenced by chemical properties as well as by the engine's operating and design parameters. As a result, the knock rating of a diesel fuel is determined by contrasting it with primary reference fuels while it is operated in a specific engine under predetermined conditions. The reference fuels are alpha methyl naphthalene ($C_{11}H_{10}$) and normal cetane ($C_{16}H_{34}$), both of which have arbitrary cetane numbers of 0 and 100, respectively. When combustion is carried out in a typical engine under predetermined operating conditions, the cetane number of a fuel is defined as the volume percentage of normal cetane in a mixture of normal cetane and α -methyl naphthalene that has the same ignition characteristics (ignition delay) as the test fuel. It is logical to assume that knock should be closely connected to the ignition delay of the fuel as ignition delay is the Conventional Fuels 159 key component affecting the initial autoignition in the CI engine. Diesel oil's ability to withstand knocking can be increased by sparingly including substances like ether, ethyl nitrate, or amyl nitrate [8]–[10].

Laboratory Procedure

The test is conducted under the conditions listed in Table 3 in a typical single-cylinder engine, such as a CFR diesel engine or a Ricardo single-cylinder variable compression engine. The test fuel is initially used in the engine running under the predetermined circumstances. An ideal fuel-to-air ratio is achieved by adjusting the fuel pump delivery. A 13-degree injection advance is achieved by adjusting the injection timing as well. The ignition delay can be raised or lessened by adjusting the compression ratio until combustion starts at TDC. The test fuel experiences an igniting delay of 13 degrees after this point is discovered. Table 3 conditions for ignition quality test on diesel fuels

By noting the compression ratio for a 13-degree delay and then consulting a prepared chart that illustrates the relationship between cetane number and compression ratio, it is possible to estimate the cetane number of an unknown fuel. To ensure accuracy, two reference gasoline blends chosen to bracket the unknown sample differ from one another by no more than 5 cetane digits. To achieve the standard ignition delay (13 degrees), the compression ratio is changed for each reference blend. The cetane rating of the unknown fuel is then calculated by interpolating the compression ratios [5]–[7].

Table 3: Conditions for ignition quality test on diesel fuels

Engine speed	900 rpm
Jacket water temperature	100 °C
Inlet air temperature	65.5 °C
Injection advance	Constant at 13°bT DC
Ignition delay	13°

CONCLUSION

Conventional fuels, such as gasoline and diesel, are commonly used in internal combustion (IC) engines for transportation and power generation. These fuels are ignited in the engine's combustion chamber to produce the energy needed for mechanical work. However, the future scope of conventional fuels in IC engines is shifting due to environmental concerns and the need for sustainable energy sources. The future lies in exploring alternative fuels, such as biofuels, hydrogen, and synthetic fuels, as well as hybrid and electric propulsion systems, to reduce greenhouse gas emissions, improve fuel efficiency, and promote a greener and more sustainable transportation sector.

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CHAPTER 6

EXPLORING THE IMPORTANCE OF ALTERNATE FUELS

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ABSTRACT

Alternate fuels in internal combustion (IC) engines refer to non-conventional fuel sources used as substitutes for conventional fossil fuels like gasoline and diesel. These fuels include biofuels, hydrogen, natural gas, and synthetic fuels. The adoption of alternate fuels in IC engines aims to reduce greenhouse gas emissions, enhance fuel efficiency, and promote sustainable energy solutions. This abstract provides an overview of alternate fuels in IC engines, their properties, impact on engine performance and emissions, and their potential for a greener and more sustainable transportation sector. Crude oil and petroleum products are anticipated to become extremely expensive and in short supply in this century. Engine fuel efficiency is increasing daily and will keep increasing. The demand for fuel is now being controlled by the massive rise in the number of automobiles. In the not-too-distant future, petrol and diesel will be expensive and scarce. Alternative fuel technology will spread over the next few decades as a result of rising demand and fossil fuel depletion.

KEYWORDS

Alternate Fuels, Biofuels, Hydrogen, Natural Gas, Synthetic Fuels.

INTRODUCTION

There have always been a few IC engines that run on diesel or other non-gasoline fuels. They have, however, been few, in terms of numbers. Some developing nations are attempting to adopt different fuels for their automobiles due to the high cost of petroleum goods. Concern about the emission issues with petrol and diesel engines is another factor driving the development of alternative fuels for IC engines. The high number of cars in the world, when combined with other air-polluting technologies, is a significant factor in the global air quality issue. In order to reduce emissions from car engines, a lot of changes have been implemented. It is important to keep in mind that if a 35% improvement is made over a number of years, the increase in the global automobile population of 40% will cancel out the improvement. A great deal of work has been put into cleaning up automotive exhaust, with positive results. To reduce the ever-rising air pollution caused by the population of automobiles, new advancements are necessary.

The requirement for a significant portion of crude oil to be imported from nations that control the major oil reserves is a third justification for the development of alternative fuels. Currently, a variety of alternative fuels are employed in autos in small amounts. Taxis, delivery vans, and utility company trucks, among others, have frequently been utilized for testing. As a result, it will be easier to compare these automobiles to ones that run on equivalent amounts of petrol. Engines that are used to burn other fuels are repurposed versions of engines that were built to run on petrol. Since they are not the best design for the other fuels, they are not recommended. The greatest performance and efficiency of these engines can only be attained after years of intensive research and development. While waiting for the fuels to be approved as suitable for a significant number of engines, it is challenging to justify the research and development.

Market availability of some diesel engines has begun. They use a small amount of diesel fuel injected at the right time to ignite both fuels, together with methanol or natural gas and another fuel source. Due to their current low usage, the majority of alternative fuels are relatively expensive. If their utilization reaches a level that is on par with that of petrol, several of these fuels will become far less expensive. There would be a reduction in the price of production, shipping, and marketing. Lack of distribution locations (service stations) where the gasoline is made available to the general population is another issue with alternative fuels. Unless there is a substantial network of service stations where fuel for that automobile can be purchased, the people will be hesitant to buy an automobile. On the other hand, until there are enough vehicles to make them lucrative, it is challenging to justify constructing a network of these service stations. A few distribution terminals for some of these fuels, including propane, natural gas, LPG, and methanol, have been established in a few cities. The process of switching from one main fuel type to another will be drawn-out, expensive, and even traumatic. The numerous alternative fuels will be covered in the sections that follow [1].

Internal combustion (IC) engines using alternative fuels have drawn a lot of interest as a solution to the environmental problems brought on by using traditional fossil fuels. It is crucial to investigate and use alternative fuel sources as worries about climate change, air pollution, and energy security continue to rise. The alternatives covered by these alternative fuels are many and include biofuels, hydrogen, natural gas, and synthetic fuels. It is conceivable to lower greenhouse gas emissions, increase fuel economy, and pave the road for a more sustainable transportation sector by using these fuels in IC engines. The necessity to reduce greenhouse gas emissions is one of the main reasons to investigate alternative fuels. When burnt, conventional fossil fuels like petrol and diesel generate carbon dioxide (CO₂) and other pollutants that contribute to air pollution and global warming. A potential option is biofuels, which are produced from renewable resources like biomass, agricultural products, or algae. Because they are made from recently collected atmospheric carbon during plant development, these fuels, like ethanol and biodiesel, may be combined with or even used in lieu of conventional fuels, greatly lowering net CO₂ emissions.

Another alternative fuel with enormous promise is hydrogen. Hydrogen combustion in IC engines results in just water vapor as a byproduct, completely eliminating CO₂ emissions. However, there are still issues with infrastructure development, storage, and manufacturing of hydrogen. The hydrogen fuel cell technology, which promises even better efficiency and zero emissions, is now being advanced. Another alternative fuel for IC engines is natural gas, which mostly consists of methane. Natural gas burning produces less CO₂ and air pollutants than combustion of petrol and diesel. Compressed natural gas (CNG) and liquefied natural gas (LNG) are becoming more widely available, which has increased their usage in automobiles, notably in business fleets and public transit.

Renewable energy sources, including solar or wind power, are transformed into liquid or gaseous fuels to create synthetic fuels, sometimes referred to as e-fuels or power-to-liquids (PtL) fuels. These fuels, like synthetic petrol or synthetic diesel, are made from sustainable sources yet have attributes that are comparable to those of their traditional equivalents. Synthetic fuels are a realistic solution for decreasing emissions and reaching carbon neutrality since they may be utilized directly in current IC engines without requiring major changes. Future applications for alternative fuels in IC engines go beyond those related to the environment. It is feasible to improve energy security and lessen reliance on limited fossil fuel supplies by diversifying fuel sources. In addition, alternative fuels provide a chance to establish new businesses and add workers to the renewable energy industry, promoting economic expansion and technical innovation. However, issues with scalability, cost-effectiveness, and infrastructure development still exist. In order for alternative fuels to be

widely used, it is necessary for governments, businesses, and academic institutions to work together, invest in research and development, and implement supporting legislation [2], [3].

In conclusion, using alternative fuels in IC engines opens the door to a transportation industry that is greener and more sustainable. These fuels, such as biofuels, hydrogen, natural gas, and synthetic fuels, show promise for reducing environmental effects and fostering a switch to a more sustainable energy future since they have the ability to increase fuel economy, lessen greenhouse gas emissions, and increase energy security. To fully realize the promise of alternative fuels and develop a cleaner, more sustainable transportation system, it is imperative to invest in ongoing research, innovation, and supporting regulations.

DISCUSSION

Solid fuels

For IC engines, solid fuels are no longer useful. Some of the previous initiatives are mentioned in this section to provide historical context. Before petroleum-based fuels were mastered in the second part of the 1800s, a variety of alternative fuels were tried and employed in IC engines. One of the fuels Rudolf Diesel utilized when constructing his engine was coal dust diluted with water. Early diesel engines used water to disperse fine coal (carbon) particles, which were then injected and burnt. Many experimental engines have been created during the last century that use this fuel, even though it was never widely used as fuel.

On this fuel technology, considerable work is still being done today. The decrease in average coal particle size has been this fuel type's most significant advancement. The typical particle size in 1894 was in the range of 100 ($1 = 1 \text{ micron} = 10^6 \text{ m}$). Between 1940 and 1970, this was lowered to around 70, and by the early 1980s, it had been further decreased to about 10. A typical slurry has a bulk composition of around 50% coal and 50% water. The abrasiveness of the solid particles in this fuel is a significant issue since it causes piston rings and injector wear. Given its widespread availability, coal is a desirable fuel. Other forms of utilization, however, seem more practical when used as motor fuel. These include the coal being liquefied or gasified. Due to World War II, petroleum products were very difficult to come by in the late 1930s and early 1940s, particularly in Europe. The German military utilized almost all petrol products, leaving no fuel for private automotive usage [4], [5]. Even though this caused the general population difficulty, they continued to use their cherished cars. Solid fuels like coal, wood, or charcoal have been successfully used to power vehicles in a number of nations, primarily Sweden and Germany [6].

Liquid Fuels

Because they are convenient to store and have a respectable calorific value, liquid fuels are favored for IC engines. The primary substitute for liquid fuel in this category is alcohol.

Alcohol

Alcohols may be derived from both natural and artificial sources, making them a desirable alternative fuel. The two alcohol types that look most promising are methanol (methyl alcohol) and ethanol (ethyl alcohol). The benefits of using alcohol as a fuel include:

1. It may be acquired from a variety of natural and artificial sources.
2. It has an anti-knock index number (octane number) of above 100, making it a high-octane gasoline. By adopting greater compression ratios, high-octane fuel-consuming engines may operate more effectively. Alcohols burn more quickly.
3. In comparison to petrol, it generates less total emissions.

4. Burning alcohols produces more exhaust gas molecules, increasing the pressure and power of the expansion stroke.
5. It has a high latent heat of vaporisation, which makes the intake process colder. This improves the engine's volumetric efficiency and lowers the amount of effort that must be put into the compression stroke.
6. The fuel's sulphur concentration is low for alcohols.

Disadvantages Of Alcohol

1. Alcohol has a low energy content, or in other words, it only has around half the calorific value of petrol. This implies that to provide the same amount of energy input to the engine, roughly twice as much alcohol than petrol must be burnt. In order to go the same distance with the same thermal efficiency and engine power, two times as much gasoline would need to be bought, and the range would be halved. The amount of storage needed for both automobiles and distribution centres would need to be doubled, as would the number of storage facilities, the volume of storage at the service station, the number of tank trucks and pipes, etc. Engine power would be about the same for a given displacement even with alcohol's decreased energy content. Alcohol requires a lower air-fuel ratio, which accounts for this. Since alcohol includes oxygen, stoichiometric combustion needs less air. The same quantity of air may burn more fuel.
2. Alcohol combustion increases the amount of aldehydes in the exhaust. Aldehyde emissions would be a significant exhaust pollution issue if alcohol fuel was used on a par with petrol.
3. Alcohol corrodes copper, brass, aluminium, rubber, and many polymers significantly more than gasoline does. This imposes certain limitations on the development and production of engines using this fuel. Long-term alcohol usage may cause gasoline tanks, gaskets, and even metal engine components to degrade (leading to fractured fuel lines, the need for a special fuel tank, etc.). Metals are severely corroded by methanol.
4. Due to low vapour pressure and evaporation, it has poor cold weather starting characteristics. Engines running on alcohol often have trouble starting at low temperatures (below 10 °C). Alcohol fuel is often mixed with a tiny quantity of petrol, which significantly enhances cold-weather starting. But doing so significantly lessens the appeal of any alternative fuel.
5. Alcohols often have weak ignition properties.
6. Alcohols have very imperceptible flames, which is harmful when working with fuel. This risk can be eliminated with a modest quantity of petrol.
7. Because of the low vapour pressure in storage tanks, there is a risk of fire. Storage tanks may have air leaks that result in flammable mixtures.
8. Due to the low flame temperatures, there will be less NO_x emissions.
9. Though it takes longer to heat the catalytic converter to an effective working temperature as a consequence of the lower exhaust temperatures.
10. A lot of individuals find the strong alcohol smell to be highly repulsive. When refuelling a car, people have reported experiencing headaches and light-headedness.
11. In fuel distribution systems, vapour lock is a potential problem.

Methanol

Methanol has undergone extensive research and development and is one of the most promising fuels being evaluated as a substitute for petrol. For a number of years, pure methanol and blends of methanol and petrol in varying ratios have been thoroughly tested in engines and vehicles. M85, which contains 85% methanol and 15% petrol, and M10, which

contains 10% methanol and 90% petrol, are the most popular blends. The performance and emission statistics from these experiments are compared with those of pure petrol (M0) and pure methanol (M100). Some intelligent flexible fuel (or variable fuel) engines may run on any arbitrary blend of methanol and petrol, from pure methanol to pure petrol. Utilising two fuel tanks, different flow rates of the two fuels may be supplied to the engine while travelling via a mixing chamber. The electronic monitoring system (EMS) changes the appropriate air fuel ratio, ignition timing, injection timing, and valve timing for the fuel mixture being utilised using data from sensors in the intake and exhaust (where feasible) [7].

Alcohol's propensity to interact with any water in gasoline-alcohol combinations as fuel is one issue. This results in a non-homogeneous mixture as the alcohol locally separates from the petrol. Due to the significant variances in the air-fuel ratios of the two fuels, this results in the engine running erratically. There are several sources of methanol, both renewable and fossil. Coal, oil, natural gas, biomass, timber, landfills, and even the ocean are examples of these. However, the cost of gasoline increases with every source that needs considerable manufacture or processing. An engine running on M10 fuel emits emissions that are similar to those of a gasoline-powered engine. The biggest benefit (and drawback) of utilising this fuel is the 10% drop in petrol consumption. There is a discernible reduction in HC and CO exhaust emissions while using M85 gasoline. However, formaldehyde emissions have increased significantly (by around 500%) and NO_x emissions have increased. Some dual-fuel CI engines utilise methanol. Due to its high-octane number, methanol is not an acceptable CI engine fuel on its own, but it may be used successfully if a tiny quantity of diesel oil is used for ignition. This is extremely appealing to underdeveloped nations since methanol is often available for a much less money than diesel fuel.

Ethanol

For many years, ethanol has been utilised as motor fuel in a number of nations across the globe. Early in the 1990s, Brazil was undoubtedly the top user. 5 million cars and trucks used gasoline that was 93% ethanol. Gasohol, a mixture of petrol and alcohol, has been sold at petrol stations in the US for a while. Petrol contains 10% ethanol and 90% petrol. The development of systems that use petrol and ethanol combinations is ongoing, much as with methanol. The mixtures E85 (85% ethanol) and E10 (gasohol) are noteworthy examples of mixture combinations. To solve some of the issues with pure alcohol (such as cold starting and tank flammability), E85 is essentially alcohol fuel with 15% petrol added. E10 decreases petrol consumption without requiring any changes to car engines. We are testing flexible-fuel engines that can run on any ethanol-to-gasoline ratio. Either ethylene or the fermentation of cereals and sugar may be used to make ethanol. Corn, sugar beets, sugar cane, and even cellulose (wood and paper) are used to make a large portion of it. Due to the extensive production and processing needed, ethanol is currently expensive. If more of this fuel were utilised, this would be decreased. High output would, however, lead to a food-fuel price war and increased prices for both. According to some research, the energy needed to cultivate, plant, harvest, ferment, and transport ethanol-producing crops in the United States today is more than the energy contained in the finished product. This negates a key benefit of adopting alternative fuels. While it emits more HC than methanol, ethanol emits less than petrol[8]–[10].

SI Engines

Alcohols have a stronger anti-knock property than petrol. Engine compression ratios between 11:1 and 13:1 are thus typical when using an alcohol fuel. The compression ratio of modern petrol engines is typically 7:1 or 9:1, which is much too low for pure alcohol. Alcohol emits less damaging exhaust emissions when used in an engine and fuel system that have been

appropriately constructed. Per litre, alcohol has roughly half the thermal energy of petrol. Alcohol has a lower stoichiometric air fuel ratio than petrol. A carburetor's or fuel injector's fuel passageways should be doubled in area to allow for increased fuel flow in order to deliver the correct fuel-air combination.

Unlike petrol, alcohol does not quickly vaporize. It has a substantially higher latent heat of vaporisation. Starting cold weather is impacted by this. Alcohol won't burn correctly if it liquefies in the engine. As a result, in severely cold climates, the engine may be difficult or even impossible to start. In order to get around this, petrol is added to the engine up until it starts and heats up. Alcohol injected after the engine has warmed up will swiftly and totally vaporize and burn properly. To totally vaporize alcohol, more heat may need to be provided even during regular operation. Alcohol burns at a rate that is about half that of petrol. As a result, the ignition timing has to be adjusted in order to produce additional spark advance. As a result, the alcohol that burns slowly will have more time to build up the pressure and power within the cylinder. Furthermore, as alcohols are naturally corrosive, corrosion resistant materials are needed for the fuel system.

Petrol for SI Engine Reformulated

Reformulated petrol is regular petrol with a minimally altered formulation and additives to aid in the reduction of engine emissions. The gasoline contains oxidation inhibitors, corrosion inhibitors, metal deactivators, detergents, and additives for deposit management. Alcohols and oxygenates are combined to include 1-3% oxygen by weight, such as methyl tertiary butyl ether (MTBE). This will aid in lowering the amount of CO in the exhaust. The concentrations of benzene, aromatic, and high-boiling substances are decreased, along with the vapour pressure. Cleaning chemicals are used since it is known that engine deposits contribute to emissions. Some additives clean carburetors, some clean fuel injectors, and some clean intake valves, but often none of these additions clean other parts of the vehicle.

Positively, this fuel may be used in any gasoline-powered engine, regardless of age, without the need for modification. The usage of petroleum products is not drastically decreased, costs go up, and only a minor reduction in emissions are achieved.

For SI engines, a water-gasoline mixture

Petrol with water added to it burns more slowly and cools the gas in the cylinder, which likely prevents explosion. When water was introduced to the intake charge, there were less engine combustion chamber deposits noted. There have been significant decreases in nitric oxide emissions, according to reports. Contrarily, adding water almost certainly causes a rise in hydrocarbon emissions. Last but not least, the addition of water seems to have no impact on carbon monoxide emissions. Only a very little amount of work has been done using water addition through an emulsion with the fuel rather than separately. Emulsion may make a separate tank unnecessary, provide better atomization, and enhance fuel safety. The separation of the water and fuel might be a concern, however [11].

CONCLUSION

Alternate fuels in IC engines are non-conventional fuel sources used as substitutes for conventional fossil fuels like gasoline and diesel. These fuels include biofuels, hydrogen, natural gas, and synthetic fuels. The future scope of alternate fuels lies in their potential to reduce greenhouse gas emissions, improve fuel efficiency, and promote a more sustainable transportation sector. By exploring and adopting alternate fuels, IC engines can contribute to environmental sustainability and energy security while paving the way for a cleaner and greener future. The urge to enhance the compression ratio for better efficiency and fuel

economy has been present throughout the development of the spark-ignition engine. The octane rating of the available petrol has sometimes proved a barrier to this economic improvement. Water was suggested as an antiknock additive to get around this restriction.

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CHAPTER 7

INVESTIGATING THE CARBURATION'S BENEFITS AND LIMITATIONS

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ABSTRACT

In an internal combustion engine, the process of carburetion involves using a carburetor to combine the air and fuel in the right amounts for burning. The liquid fuel is turned into a fine mist by the carburetor, where it is then combined with entering air to ignite. An overview of carburetion is given in this abstract, along with information on its historical relevance, the fundamental concepts governing how carburetors work, and its function in the internal combustion engine combustion process. The abstract also discusses the applicability of carburetion to contemporary engine technology, as well as its benefits and drawbacks. Typically, liquid fuels that are volatile are used in spark-ignition engines. A homogenous mixture is often not formed at the intake manifold since the preparation of the fuel-air mixture is done outside the engine cylinder. Even throughout the suction and compression operations, fuel droplets that are in suspension continue to evaporate and mix with air. Spark-ignition engines place a lot of importance on the preparation of the mixture. Carburetion's main goal is to provide a combustible combination of gasoline and air in the precise amount and quality needed for the engine to run effectively in all circumstances.

KEYWORDS

Air-Fuel Mixture, Atomization, Carburetion, Carburetor, Combustion.

INTRODUCTION

Internal combustion engines depend on the basic mechanism of carburetion to combine the fuel and air precisely and in the right amounts for burning. It was important to the development of automotive engineering at the time, and it is now a key idea in the area of engine technology. By atomizing liquid fuel into a tiny mist and combining it with incoming air to generate a combustible combination, a device known as a carburetor is used in the carburetion process. Before the invention of fuel injection systems, carburetion was the main fuel delivery technique used in internal combustion engines. In order for the engine to produce power, it offered a simple yet efficient method of combining air and fuel. In a system that uses a carburetor, the carburetor pulls gasoline from the fuel tank and measures it according to the throttle position, engine load, and other factors. Bernoulli's law and the Venturi effect are the foundational ideas of carburetion. The air's velocity rises as it travels through the venturi of the carburetor, which narrows the passageway and creates a low-pressure area. Fuel from the carburetor's float chamber is drawn into this area of low pressure by way of the primary fuel jet. A combustible mixture is created by atomizing the fuel into tiny droplets and combining it with the incoming air before being fed into the engine cylinders.

Simpleness, cost-effectiveness, and ease of maintenance are only a few benefits of carburetion. It contributed greatly to the advancement of internal combustion engines and supplied proper fuel-air mixing for a long time. Carburetion, nevertheless, is not without its drawbacks. It may be difficult to get accurate air-fuel ratios under different engine circumstances, which might lead to inefficiencies and pollution problems. Due to the lack of exact control over the fuel-air mixture, carbureted engines may also have trouble starting in chilly conditions or at high elevations. The carburetion process in contemporary engines has been mostly superseded by fuel injection systems because to technological improvements. Greater control over the air-fuel ratio is made possible by fuel injection, which also improves engine performance and fuel economy while lowering pollutants. Carburetion, however, is still useful in a few situations, including those involving tiny engines, antique cars, and recreational vehicles. In conclusion, the evolution of internal combustion engines has been greatly aided by carburetion, which has long been the main way to distribute fuel. Despite the fact that fuel injection has supplanted carburetion as the primary method of delivering gasoline, it still has use in certain situations. Understanding the physics and principles of carburetion is crucial for recognizing the advances that have produced more efficient and environmentally friendly engines and for understanding the development of engine technology [1], [2].

Carburation's benefits and limitations

Simpleness, cost-effectiveness, and ease of maintenance are only a few benefits of carburetion. The benefits of carburetion systems are highlighted in this section along with their drawbacks. A fair view of the technology is presented by examining difficulties in attaining accurate air-fuel ratios under various engine situations, possible inefficiencies, and pollution problems.

Carburetion in Contemporary Culture

Carburetion still has use in certain applications even though fuel injection technologies have mostly superseded it in contemporary engines. The circumstances in which carburetion is still important, such as tiny engines, classic cars, and recreational vehicles, are examined in this section. It goes through the justifications for its continuous usage as well as the factors to take into account while maintaining and maximizing carbureted engines.

Development and upcoming trends

Technology developments have made fuel injection the preeminent method of delivering gasoline to contemporary engines. This section examines the development of fuel delivery systems, emphasizing the switch from carburetion to fuel injection and the advantages it provides to engine performance, fuel economy, and emissions reduction. It also sheds light on potential engine technology changes and the declining significance of carburetion.

Definition of Carburetion

The act of creating a combustible fuel-air combination by combining the right quantity of gasoline with air before allowing it into the engine cylinder is known as carburetion, and the tool that does this is known as a carburetor.

Factors Affecting Carburetion

Of the various factors, the process of carburetion is influenced by

- (i) the engine speed
- (ii) the vaporization characteristics of the fuel

- (iii) the temperature of the incoming air and
- (iv) the design of the carburetor

The time available for mixture creation is quite constrained since current engines are of the high-speed kind. The amount of time an engine has for mixture induction during the intake stroke, for instance, is only approximately 10 milliseconds (ms) for an engine operating at 3000 rpm.

Only 5 milliseconds are left when the speed reaches 6000 revolutions per minute. As a result, the air stream velocity at the point where the fuel is injected has to be raised in order to achieve high quality carburetion (i.e., a mixture with a high vapor content). This is accomplished by directing the air via a venturi portion. The primary metering jet's principal release of fuel occurs near the venturi's throat, which is its narrowest cross section.

The presence of highly volatile hydrocarbons in the fuel is another element that guarantees excellent quality carburetion in a short amount of time. As a result, effective carburetion is required, particularly at high engine speeds, and is determined by the fuel's evaporation properties as evidenced by its distillation curve [3]. Carburetor efficiency is significantly influenced by the temperature and pressure of the surrounding air.

A more homogenous mixture is produced when the temperature of the mixture is higher because it enhances the vaporization of fuel (the proportion of fuel vapor rises with temperature). However, while the air-fuel ratio remains fixed, the engine's power output decreases with a rise in ambient temperature because of a reduction in mass flow into the cylinder, or, in other words, a reduction in volumetric efficiency.

On how evenly the mixture is distributed to the engine's numerous cylinders, the design of the carburetor, the intake system, and the combustion chamber all have a significant role. The provision of the required mixture composition under various engine running situations is only made possible by properly designed carburetor components. You can prepare this section depending upon the objective of your paper. This part adds a lot of value to a research paper [4], [5].

DISCUSSION

Principle of Carburetion

The suction produced by the piston's descent draws both fuel and air into the engine cylinders and through the carburetor. This suction results from a rise in the cylinder's volume and a corresponding fall in the gas pressure inside of this chamber. The air flows into the chamber as a result of the pressure differential between the atmosphere and the cylinder. Air entering the combustion chamber of the carburetor takes up fuel that is released via a tube. A tiny hole called a carburetor jet is present in this tube and is open to the air flow.

The pressure head or difference in pressure between the float chamber and the venturi's throat, as well as the size of the tube's exit, determine how quickly fuel is released into the atmosphere. The suction action must be strong and the nozzle output relatively tiny in order for the fuel extracted from the nozzle to be completely atomized. The pipe in the carburetor that carries air to the engine is engineered to have a restriction in order to provide a powerful suction. Due to the increase in flow velocity at this limitation known as the throat, a suction action is produced. To reduce throttling losses, the limitation is built in the shape of a venturi, as illustrated in Fig. 1. The venturi or throat of the carburetor is where the fuel jet's terminus is situated.

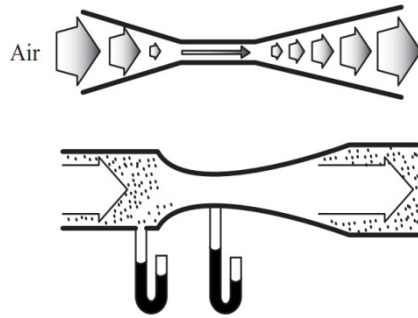


Figure 1: Operation of the Venturi Tube

The venturi tube's geometry is seen. The air must travel through a much smaller flow region since the channel is narrower in the core. The air will move at its fastest speed via the tube's narrowest point because the equal quantity of air must flow through all of the tube's points. The suction will rise proportionally as the area decreases due to the higher air velocity. As was already established, where the suction is strongest is typically where the fuel discharge jet's entrance is situated. In most cases, this is directly below the venturi tube's narrowest point. In this area, the air coming in from the venturi tube and the petrol spray from the nozzle combine to create a combustible combination that enters the cylinders through the intake manifold. A little amount of fuel vaporizes at the same time that the majority of it is atomized. The pace at which fuel vaporises is aided by increased air velocity near the venturi's throat. The increased air velocity at the venturi throat alone cannot completely address the challenge of obtaining a mixture of 196 IC Engines sufficiently high fuel vapour-air ratio for effective starting of the engine and for uniform fuel-air ratio in different cylinders (in case of multicylinder engine) [6].

The Simple Carburetor

Carburetors are extremely complicated devices. Let's first comprehend the operation of a basic or simple carburetor, which produces an air-fuel combination for cruising or standard range at a single speed. Later, more mechanisms will be added to support the many unique requirements, such as beginning, idling, variable load and speed operation, and acceleration. A straightforward carburetor is depicted in detail in Figure 2. The basic components of a carburetor are a float chamber, a fuel discharge nozzle, a metering orifice, a venturi, a throttle valve, and a choke.

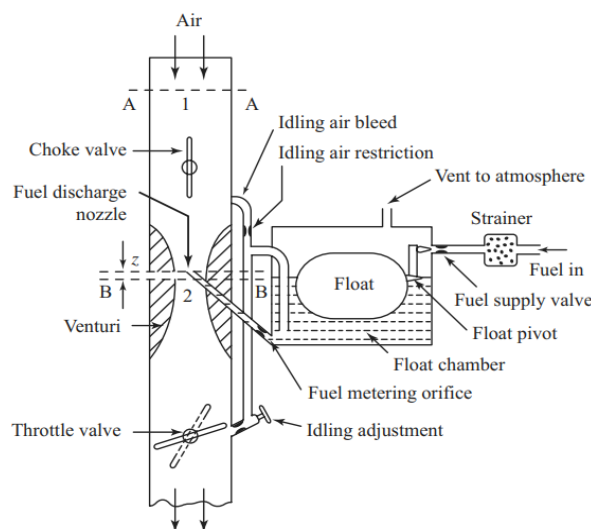


Figure 2: Simple Carburettor

An even amount of petrol is kept in the float chamber by a float and needle valve arrangement. The fuel supply valve opens and fuel is admitted if there is less fuel than is intended in the float chamber, which causes the float to drop. The float closes the gasoline supply valve after the predetermined level has been reached, preventing the flow of more fuel from the supply system. Either the atmosphere or the side of the venturi upstream receives venting from the float chamber. The venturi is filled with air during the suction stroke. A tube with a diminishing cross-section and a small space at the throat is what a venturi is, as was previously mentioned. When the throttle is fully open, the pressure at the throat is typically between 4 and 5 centimetres of mercury (cm Hg) below air pressure and seldom more than 8 cm Hg [7].

The liquid level in the float chamber is kept at a level just slightly below the discharge jet's tip to prevent fuel from overflowing through the jet. The nozzle tip is what is described above. The area in Fig. 2 with the letter h represents the height difference between the nozzle's top and the level of the float chamber. The quantity-governed nature of the petrol engine allows for precise control over how much charge is delivered to the cylinder at each speed in order to adjust power output. The throttle valve, which is often of the butterfly type and is placed behind the venturi tube, is used to do this. When the throttle is closed, less air passes through the venturi tube and less fuel-air mixture is delivered to the cylinder, which results in less power being produced. The amount of mixture fed to the engine increases as the throttle is opened because more air passes through the choke tube. Engine power output is increased as a result.

Because it only offers the necessary A/F ratio at one throttle position, a basic flaw in a simple carburetor of the sort mentioned above exists. Depending on whether the throttle is opened more or less at the other throttle locations, the mixture is either leaner or richer. The pressure difference between the float chamber and the venturi throat changes as the throttle opening changes, which also affects the air flow. Fuel flows through the nozzle at a set pressure differential. As a result, there is a corresponding variation in air and fuel flow velocity. Increasing air flow also causes the pressure at the venturi throat to decrease while the density of the fuel stays constant. With increasing throttle opening, a straightforward carburetor produces a mixture that is progressively richer. The next section provides a mathematical breakdown of how well a basic carburetor works.

Types of Carburetors

Depending on how the air flows, there are generally three different types of carburetors. The first is the up draught kind, which is depicted in Fig. 3(a), in which the air enters at the bottom and exits at the top, causing it to flow upward. The updraft carburetor's drawback is that air friction is used to elevate the fuel droplet that has been sprayed. The mixing tube and throat must be relatively small in order for the air velocity to be sufficient to lift and convey the fuel particles even at low engine rpm. If this doesn't happen, the fuel droplets will likely split, giving the engine a lean mixture instead. However, because it is limited in size and unable to provide mixture to the engine quickly enough at high speeds, the mixing tube must be used. The downdraught carburetor [Fig.3 (b)] is utilised to get around this problem. It is positioned higher than the input manifold and has a downward-moving air and mixture flow. In this case, even though the air velocity is low, the fuel enters the cylinders by gravity instead of having to be lifted by air friction as in up draught carburetors [8].

As a result, it is possible to increase the size of the mixing tube and throat, enabling high engine speeds and high specific outputs. A horizontal mixing tube with a float chamber on one side makes up a cross-draught carburetor [Fig.3(c)]. In engines, one right-angled turn in

the inlet tube is removed and the flow resistance is decreased by the use of a cross-draught carburetor.

Constant Choke Carburetor

The air and fuel flow zones are continuously maintained to remain consistent in a constant throttle carburetor. But depending on how much work the engine needs to do, the pressure difference or depression that causes the flow of fuel and air might change. This class of carburetors includes Solex and Zenith.

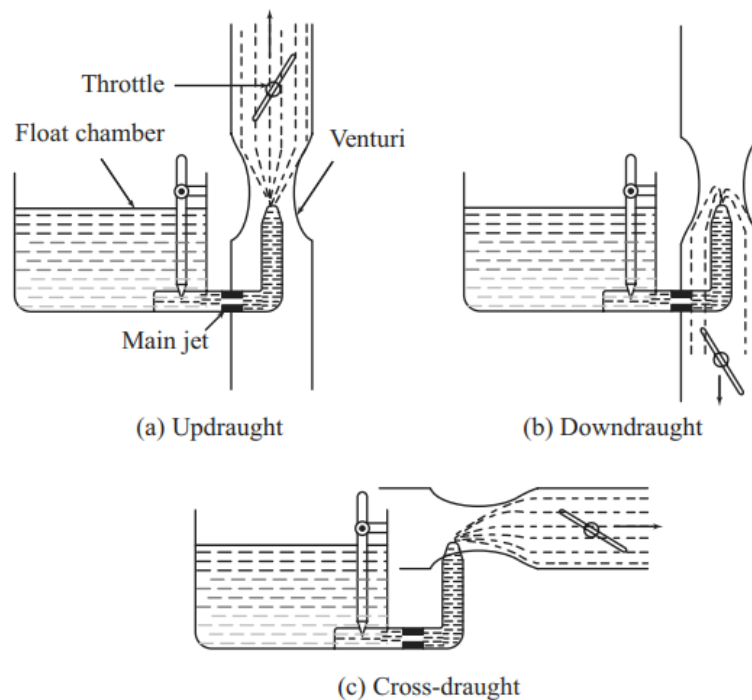


Figure 3: Types of carburetors

Constant Vacuum Carburetor

The constant vacuum carburetor, also known as a variable choke carburetor, adjusts the air and fuel flow regions to meet the needs of the engine while maintaining a constant vacuum. These types of carburetors include the S.U. and Carter models.

Multiple Venturi Carburetor

A double or triple venturi is used in a multiple venturi system (Fig. 4 (a) or (b)). Within the main venturi, the boost venturi is centred. The main venturi's throat is where the discharge edge of the boost venturi is situated. Upstream of the throat of the bigger main venturi is where the boost venturi is located. The amount of air passing via the boost venturi is quite little [9], [10]. The pressure at the main venturi throat and the boost venturi exit are now equal. The fuel nozzle is situated near the boost venturi's throat. The following outcomes from this arrangement:

1. The area around the gasoline nozzle has a high depression. Consequently, it is feasible to optimise atomization and have greater control over fuel flow. Air may move at speeds of up to 200 m/s at the boost venturi throat.
2. An annular air blanket forms. The induction tract's walls are protected from the fuel (or fuel droplets) by this blanket.
3. Excellent full throttle functioning at low speeds is feasible.

4. A more effective mixing of the air and fuel is achieved without a tolerable loss in volumetric efficiency. Due to the fact that the higher pressure drop only affects a fraction of the entering air, volumetric efficiency is only marginally reduced.

In certain carburetors, three venturiers are employed in sequence instead of two. A triple venturi carburetor is seen in Figure 4(b). The primary venturi, secondary venturi, and major venturi are the three venturi. The secondary venturi's throat contains the main venturi's exit. The main venturi's throat is where the secondary venturi's exit is located.

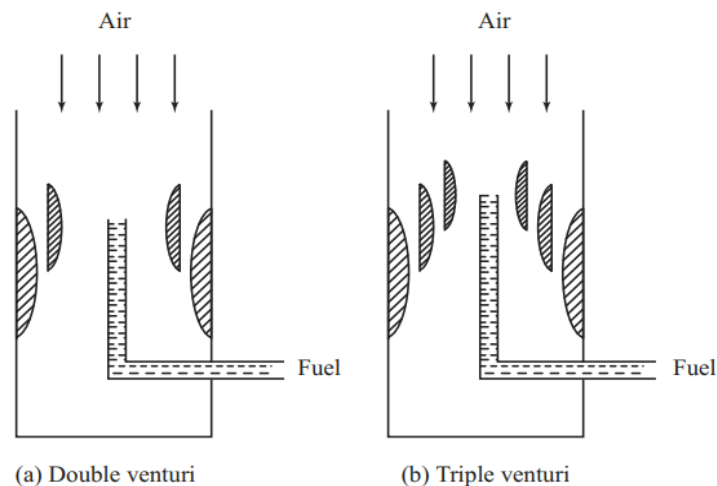


Figure 4: Double and Triple Venturi Carburetors

CONCLUSION

Carburetion is the process of mixing fuel and air in an internal combustion engine using a carburetor. It was widely used in older vehicles before the advent of fuel injection systems. However, with the increasing focus on fuel efficiency and emissions reduction, carburetion has been largely replaced by electronic fuel injection. The future scope of carburetion is limited as modern vehicles require more precise control over fuel delivery, which is achieved more effectively through electronic fuel injection systems. Venturi tubes, also known as choke tubes, are formed in such a way as to provide the least amount of resistance to air flow. A maximum velocity is reached at the venturi throat as the air moves through the venturi. A minimal pressure is reached as a result of the pressure decreasing. The fuel is delivered to a discharge jet from the float chamber, and the tip of the discharge jet is situated in the venturi's throat. Fuel is released into the air stream as a result of the carburetor depression, or pressure difference between the float chamber and venturi throat. The size of the discharge jet, which is selected to provide the requisite air-fuel ratio, has an impact on the fuel discharge.

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CHAPTER 8

AN ANALYSIS OF MECHANICAL INJECTION SYSTEMS

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ABSTRACT

Internal combustion engines employ fuel delivery methods called mechanical injection systems. Mechanical injection systems use mechanical components to accurately measure and distribute gasoline to the engine, as opposed to carburetors or electronic fuel injection. This abstract gives a general overview of mechanical injection systems, stressing their significance in certain applications while outlining their operating principles, benefits, and limits. As the compression stroke comes to a conclusion, the fuel is fed into the combustion chamber and atomized into extremely small droplets. In order to create a fuel-air combination, these droplets vaporize as a result of heat transfer from the compressed air. The temperature increases beyond its self-ignition point as a result of continuing heat transfer from hot air to the fuel. By causing the fuel to spontaneously ignite as a result, the combustion process is started.

KEYWORDS

Fuel Delivery, Fuel Metering, Internal Combustion Engines, Mechanical Components, Mechanical Injection Systems.

INTRODUCTION

The most crucial element of CI engines' operation is the fuel-injection system. The efficacy of the fuel-injection system has a significant impact on engine performance, including power output, economy, etc. The critical task of starting and managing the combustion process must be carried out by the injection system. The preparation of the combustible charge is essentially the same for both fuel injection and carburetion. However, in the case of carburetion, the procedures used to atomize the fuel depend on air speeds larger than the fuel speeds at the fuel nozzle, but in the case of fuel injection, the fuel speeds at the point of delivery are higher than the air speeds to atomize the fuel. When discussed in Chapter 7, in carburetors, gasoline is taken in by a nozzle positioned in a venturi when air passes past it. Depending on the air velocity in the venturi, the engine's fuel demand is determined. A pump that applies pressure to the fuel in a fuel-injection system regulates how much fuel is injected into the air stream that is directed towards the engine [1]–[3]. These systems have benefits in terms of ease of use, dependability, and capacity to adapt to various engine configurations. They gave the engine a regulated and effective way to receive fuel, guaranteeing perfect combustion and the production of power.

The fuel pump, which pulls gasoline from the tank and presses it to the necessary pressure, is the main part of a mechanical injection system. In order to achieve synchronization with the engine's rotation, the fuel pump is often operated by the engine's camshaft. The fuel metering system, which accurately regulates the quantity of gasoline given to the engine cylinders, is likewise controlled by this mechanical connection. Mechanical injectors or fuel nozzles are used in mechanical injection systems to measure the fuel. These parts are designed to spray gasoline into the combustion chamber or intake manifold in a precisely atomized pattern. The

mechanical parts interact with the engine's camshaft and other driving systems to control the timing and duration of fuel injection [1], [2].

The ability of mechanical injection systems to distribute fuel consistently under a variety of engine operating circumstances is one of its main advantages. Compared to carburetors, these systems are less sensitive to fluctuations in intake airflow, ambient temperature, and altitude. Mechanical injection systems are well-liked options in high-performance applications like racing engines because they can be modified and altered to match certain performance needs. Mechanical injection techniques do, however, have several drawbacks. To attain best performance, they need careful calibration and fine-tuning, which may be labour- and time-intensive. In comparison to contemporary electronic fuel injection systems, the mechanical nature of these systems restricts their ability to respond to changing engine characteristics, such as load and speed. Additionally, mechanical injection systems may not be able to adjust to pollution restrictions or properly manage fuel-air mixture ratios.

As a result of its ability to distribute fuel precisely and reliably, mechanical injection systems have been crucial in the development of internal combustion engines. Although mechanical fuel injection systems have mostly been superseded by electronic fuel injection in current cars, these systems are still important in certain situations, especially with older or more powerful engines where their simplicity and tunability are prized. The development of fuel delivery technology has been influenced by the history and evolution of mechanical injection systems, which has shaped the growth of the automobile industry [1]–[6].

Functional Requirements of an Injection System

The injection system's performance must satisfy the following criteria for the engine to function correctly and perform well:

1. Accurate fuel injection metering every cycle. Due to the tiny amounts of gasoline being handled, this is particularly important. Metering mistakes might significantly alter the output from what is expected. As the engine's speed and load needs change, so should the amount of gasoline being metered.
2. Proper fuel injection timing throughout the cycle to provide optimum power while guaranteeing fuel efficiency and clean burning.
3. Properly controlling the injection rate to ensure combustion results in the optimum heat-release pattern.
4. Proper fuel atomization, which produces extremely small droplets.
5. A suitable spray pattern to facilitate quick fuel and air mixing.
6. The combustion chamber's fuel droplets are distributed evenly.
7. To provide multi-cylinder engines with identical amounts of metered fuel to each cylinder.
8. No lag at the start or conclusion of injection, i.e., to prevent gasoline dribbling into the cylinder.

DISCUSSION

Mechanical injection systems have been widely discussed and debated in the automotive industry due to their historical significance and continued relevance in certain applications [3].

Injection Pump Governor

In a CI engine, the fuel delivery is unaffected by the characteristics of the injection pump and the air intake. While air intake decreases with speed, fuel delivery from a pump rises with

speed. As a consequence, the engine tends to over fuel at higher speeds and stall out at lower speeds when the engine is just idling.

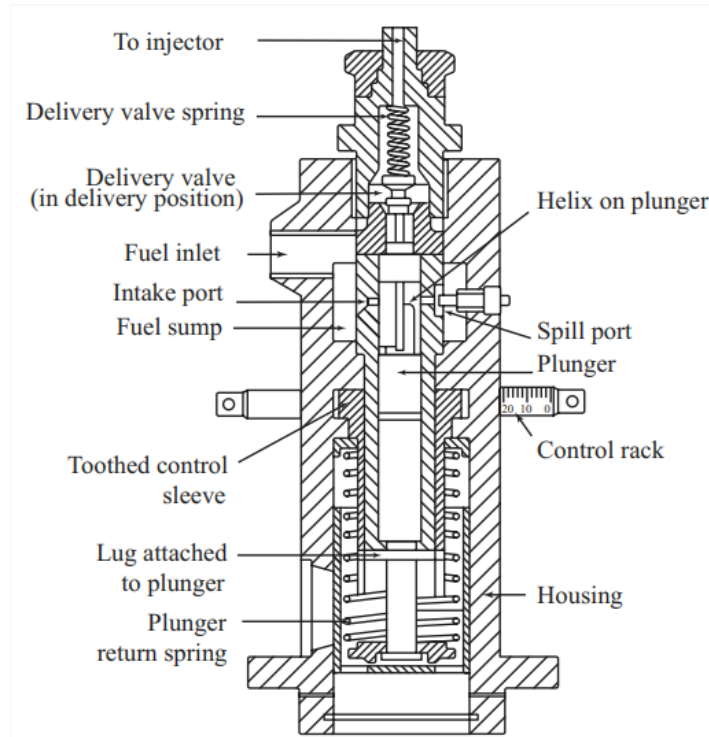


Figure 1: Single Cylinder Jerk Pump Type Fuel-Injection System

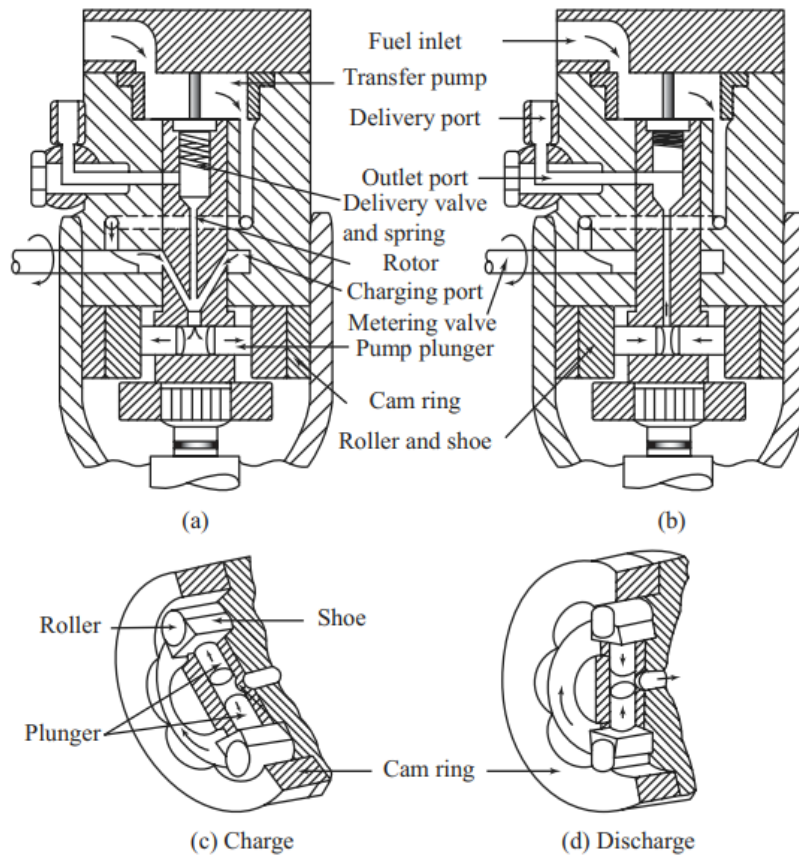


Figure 3: Schematic of Roosa Master Distributor Pump

With increasing load, more fuel is given, resulting in excessive carbon build up and high exhaust temperature. A sharp decrease in load will increase over speeding to dangerous levels. An injection pump governor is responsible for maintaining the aforementioned restrictions. There are typically two sorts of governors.

1. Mechanical Governor, and
2. Pneumatic Governor.

Mechanical Governor

Fig. 3 presents a schematic representation of how a mechanical governor functions. Weights fly apart when an engine tends to run over its maximum speed. As a result, the control lever is moved lower and the sleeve is raised by the bell crank levers. By doing this, the fuel-injection pump's control rack is actuated in a way that decreases the quantity of fuel provided. The engine's speed drops as fuel consumption drops. The opposite occurs when engine speed tends to drop.

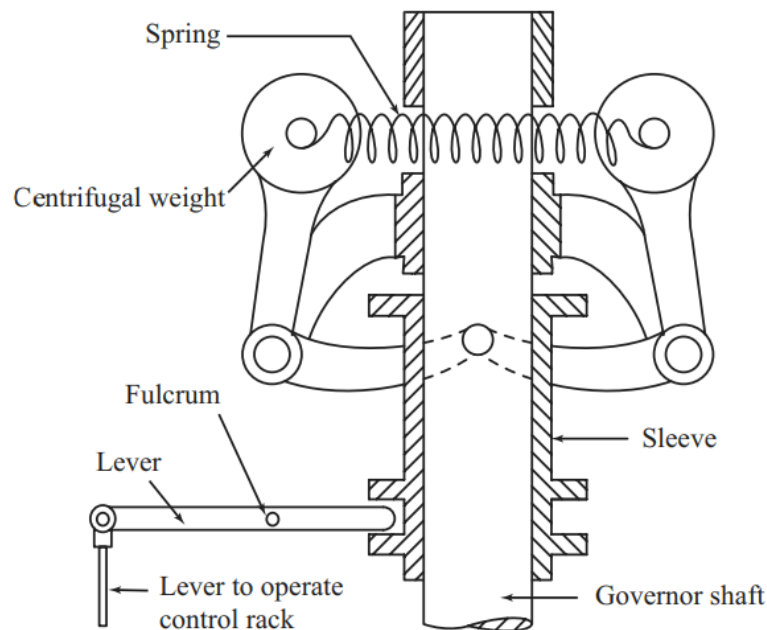


Figure 3: Principle of Mechanical Governor

Pneumatic Governor

Figure 4 depicts a pneumatic governor in detail. By moving the butterfly valve in the venturi unit, the accelerator pedal may alter how much vacuum is supplied to the diaphragm. The control rack for the fuel pump is joined via a diaphragm. The quantity of gasoline injected is therefore determined by the position of the accelerator pedal as well as the position of the pump control rack.

Fuel Injector

A fuel injector with good design ensures quick and thorough combustion. It improves mixing and subsequent combustion by atomizing the fuel into extremely small droplets, which increases their surface area. The process of atomization involves applying intense pressure to a tiny aperture of fuel. The elements of the injector assembly include,

1. A needle valves
2. A compression springs

3. A nozzle
4. An injector bodies

Fig. 8.11 shows a cross-sectional view of a typical Bosch fuel injector. When fuel is provided by the injection pump, it exerts enough force against the spring to raise the nozzle valve, causing fuel to spray into the combustion chamber in the form of tiny, highly atomized particles.

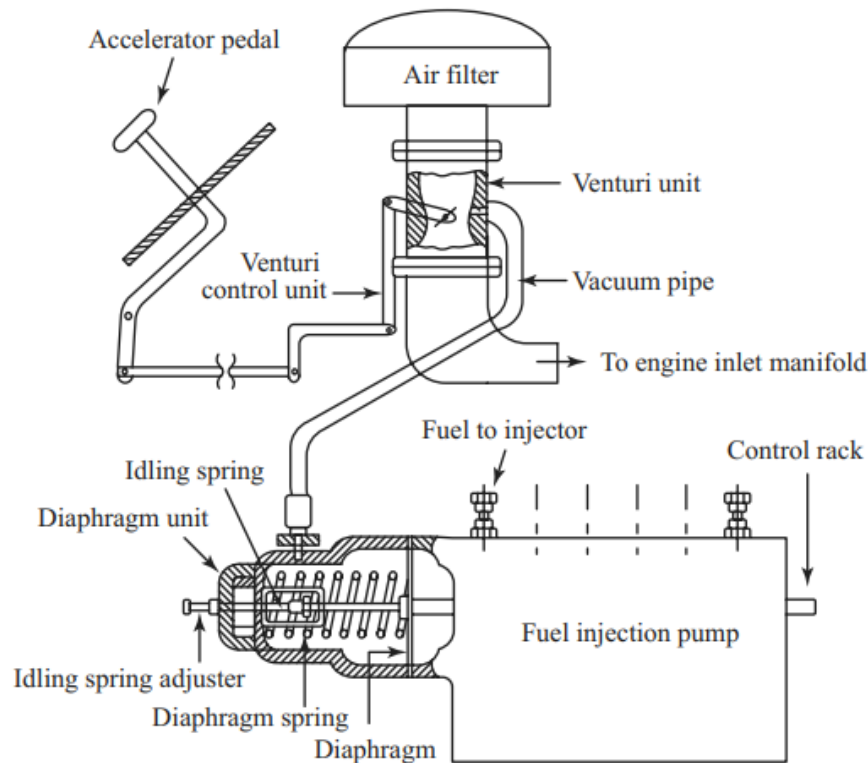


Figure 4: Principle of Pneumatic Governor

The nozzle valve is forced back onto its seat by spring pressure when the gasoline from the delivery pump runs out. A limited amount of gasoline is allowed to seep through the space between them for adequate lubrication between the nozzle valve and its guide before being emptied back into the fuel tank via a leak off connection. By changing the screw at the top, the spring tension and subsequently the valve opening pressure are regulated [7], [8].

Injection In SI Engine

In ci engines, fuel-injection systems are often used. Due to the following disadvantages of carburetion, petrol injection systems are now popular in SI engines.

1. In multicylinder engines, the mixture is not distributed evenly.
2. Volumetric efficiency loss as a result of mixture flow constraints and potential fire back.

All these issues are resolved with a petrol injection system. Any of the following techniques, which are shown, may be used to inject gasoline into a SI engine.

1. direct fuel injection into the cylinder
2. fuel injection near the intake valve
3. fuel injection into the inlet manifold

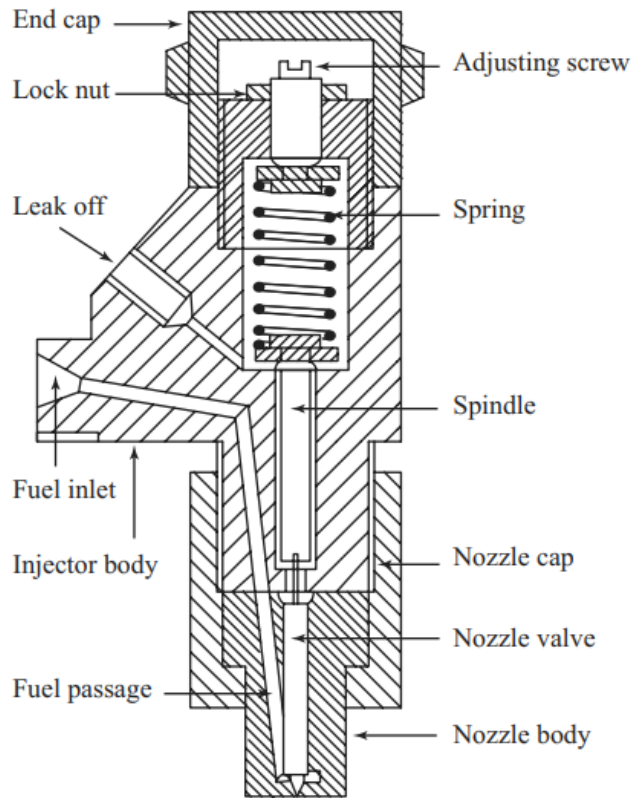
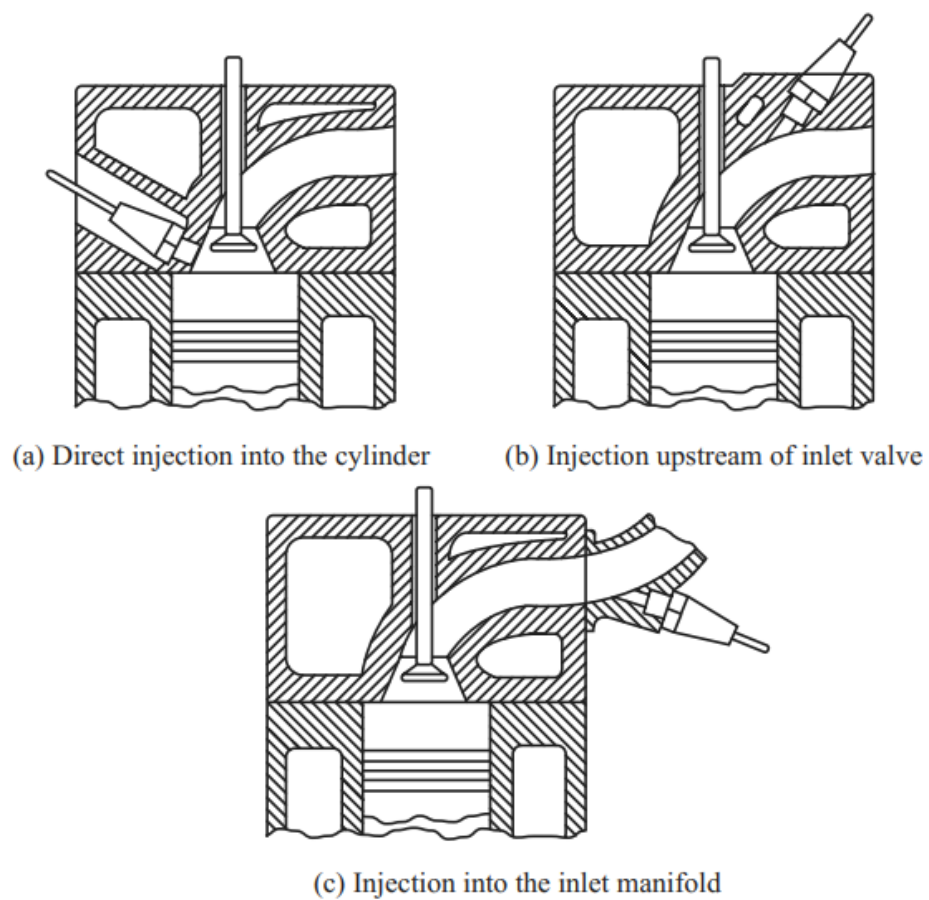


Figure 4: Fuel Injector (Bosch)



(a) Direct injection into the cylinder (b) Injection upstream of inlet valve

(c) Injection into the inlet manifold

Figure 5: Location of Injection Nozzle

There are two different kinds of petrol injection systems:

Continuous Injection

Fuel is injected constantly. When manifold injection is considered, it is used.

Timed Injection

Only during the induction stroke for a brief time, fuel is injected. In SI engines, the timing of the injection is not crucial. The following are some of the main benefits of fuel injection in a SI engine:

1. higher volumetric efficiency
2. better thermal efficiency
3. reduced exhaust emissions
4. superior fuel distribution

Petrol injection's application is limited by its high initial cost, intricate design, and higher maintenance needs. In comparison to carburetion, petrol injection is seen to have a brighter future and might eventually take the place of the carburetor [9], [10].

CONCLUSION

Mechanical injection systems are a type of fuel injection system that uses mechanical components to meter and deliver fuel to an engine's combustion chamber. They are known for their simplicity, reliability, and adaptability, making them attractive for specific niche applications such as vintage cars and high-performance racing engines. While electronic fuel injection has become the norm, mechanical injection systems still have a future in specialized applications and research and development projects. Internal combustion engines have evolved significantly thanks to mechanical injection systems, which administer fuel precisely for optimum performance.

Mechanical injection systems use mechanical components to metre and distribute gasoline to the engine cylinders, as opposed to carburetors or electronic fuel injection. These systems have a long history and have been used in many different applications, from industrial machines to high-performance racing engines. In the automobile industry, mechanical injection systems have been frequently employed, particularly in the middle of the 20th century when electronic fuel injection was still uncommon.

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CHAPTER 9

A REVIEW STUDY ON ELECTRONIC INJECTION SYSTEMS

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ABSTRACT

Internal combustion engine fuel delivery has been transformed by electronic injection systems, which provide precise control and mixture optimization. The main features, benefits, and potential applications of electronic injection systems are highlighted in this abstract. The function of sensors, computers, and actuators in real-time monitoring and modification of fuel distribution, which results in higher performance, fuel economy, and lower emissions in contemporary automobiles, is covered in the debate. To guarantee a smooth running engine in the case of automobile engines, a continual measured amount of the gasoline air combination must be provided. A fuel injector in a petrol injection system injects the fuel into the intake manifold or close to the intake port. From the pump, the injector receives the petrol, which is then finely atomized and sprayed into the air stream. The air stream and petrol are better mixed in this scenario compared to carburetion.

KEYWORDS

Electronic Injection Systems, Fuel-Air Mixture, Fuel Delivery, Fuel Efficiency, Internal Combustion Engines.

INTRODUCTION

In the history of the creation of IC engines, the year 1903 is noteworthy. The first flight powered by a SI engine was accomplished by the Wright Brothers in that year, and the engine's intake ports were fueled by a gear pump. Santos Dumont, a Brazilian, made his maiden flight with a plunger pump in Europe shortly after, in 1906. The well-known Grade Enddecker, a two-stroke engine in which the crankcase pressure of each cylinder was used to act as the injection pressure for the fuel, accomplished the legendary 13-kilometer flight in 1909 using injection. Injection technology for petrol engines is thus not a novel idea. You'll notice that the throttle valve's location affects how much mixture flows into the intake manifold in both the carburetor and the fuel injection system. In today's autos, petrol injection is used in several models. Petrol injection systems have been introduced by well-known manufacturers including Ford, Daewoo, Fiat, Mitsubishi, Honda, and others in their latest vehicles.

Internal combustion engines now get fuel with unprecedented accuracy and control thanks to electronic injection systems. Electronic injection systems use advanced sensors, computers, and actuators to monitor and modify the fuel-air mixture in real-time, enhancing engine performance and lowering emissions. These systems don't depend on mechanical components. Instead, they use them. Electronic injection system development has been an ongoing process of invention and refinement, with smaller, quicker, and more dependable components being made possible by technological and material advancements. Modern automobiles now use electronic injection systems more often, which has boosted power production, reduced emissions of harmful pollutants, and fuel economy. Fuel injectors, a fuel pump, an electronic control module (ECM), and a number of sensors, such as mass airflow sensors, oxygen sensors, and throttle position sensors, are the essential parts of an electronic injection system. Taking into consideration variables including engine load, temperature,

altitude, and driver inputs, these parts work together to monitor and modify the fuel-air mixture [1], [2].

The capacity to accurately adjust the fuel-air mixture, enabling ideal combustion and maximum power production, is one of the main benefits of electronic injection systems. The fuel flow may be adjusted by electronic injection systems to account for changing circumstances, which improves fuel efficiency and lowers pollutants. Electronic injection systems also provide diagnostic capabilities, which is a big plus. Onboard diagnostic (OBD) systems are included in the majority of contemporary systems. These systems can identify problems and notify drivers of them, giving technicians important information and lowering the possibility of catastrophic engine failures.

Electronic injection systems have a promising future because of continuous research and development activities aimed at enhancing performance, cutting emissions, and boosting dependability. Electronic injection systems are expected to become much more advanced and effective in the years to come because to developments in artificial intelligence, machine learning, and materials science. In conclusion, electronic injection systems have revolutionized the automobile sector by enabling unheard-of fuel delivery control, enhancing engine efficiency, and lowering pollutants. Electronic injection systems have a bright future ahead of them with potential uses beyond conventional internal combustion engines because to continued technological and innovative advancements [3].

Why gasoline injection?

It is challenging to achieve homogeneity of mixture strength in each cylinder of a multicylinder engine using a carburetor. An intake manifold of a multicylinder engine with a typical pattern of mixture distribution is shown in Figure 9.1. The intake valve of cylinder 2 is open, as can be seen. The gasoline travels to the manifold's end and builds up there, as can also be seen. The mixture entering into the end cylinders is enriched as a result. The leanest mixture, however, is sent to the middle cylinders, which are in close proximity to the carburetor. As a result, the air-gasoline combination is delivered to the individual cylinders in varied amounts and richness. The port injection system may address this issue, known as maldistribution, by injecting the same volume of fuel into each intake manifold. Therefore, the creation of injection systems for petrol engines is critical. By using petrol injection, maldistribution may be greatly reduced and each cylinder can get the same richness of the air-gasoline combination.

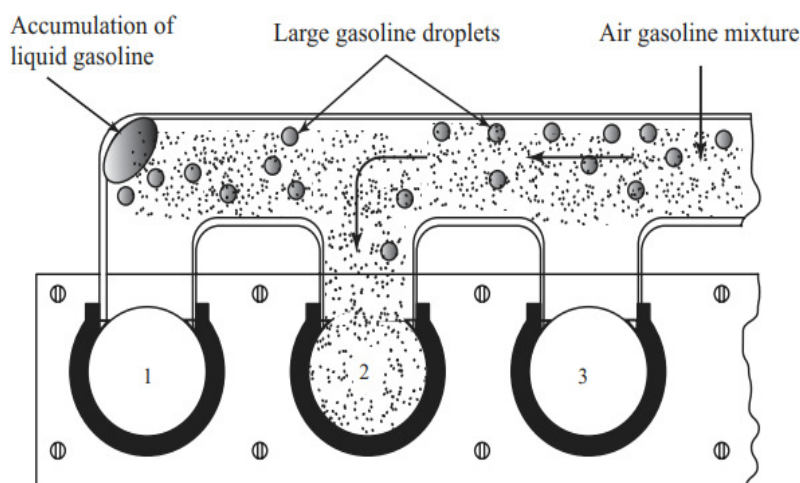


Figure 1: Typical pattern of mixture distribution in a multi-cylinder engine (Quora).

For one or more of the reasons listed below, certain modern vehicle engines use petrol injection systems rather than carburetors, as was previously mentioned:

- (i) To ensure that gasoline is distributed evenly throughout a multicylinder engine.
- (ii) To increase volumetric efficiency, or the capacity for breathing.
- (iii) To lessen or do away with detonation.
- (iv) In the case of two-stroke engines, to avoid fuel loss during scavenging.

DISCUSSION

Types of Injection Systems

The fuel injection system can be classified as:

- (i) Gasoline direct injection into the cylinder (GDI)
- (ii) Port injection
 - (a) Timed, and (b) Continuous
- (iii) Manifold injection

The aforementioned fuel injection systems may be divided into two categories: single-point injection and multi-point injection. In the throttle body assembly of a single point injection system, one or two injectors are installed. Sprays of fuel are aimed towards the intake manifold's centre or at a single location. The throttle body injection method is a different term for single point injection. One injector per engine cylinder is used in multipoint injection. At this mechanism, gasoline is injected at several places. This is more prevalent and often known as a port injection system [4], [5]. The gasoline fuel injection system used in a spark-ignition engine may either be a continuous injection system or a timed injection system, as was before explained.

Systems with continuous injection

Typically, these systems use a rotary pump. A pressure of around 0.75 to 1.5 bar is maintained in the fuel line gauge by the pump. An injector nozzle in the manifold just behind the throttle plate is used by the system to inject fuel. The entry of the supercharger in a supercharged engine receives fuel injection. Electronic Control Unit (ECU) takes into account the load and speed while determining the timing and duration of the fuel injection.

Timed fuel injection system

When the engine is operating at its highest speed, this system's fuel supply pump delivers fuel at a low pressure of just around 2 bars. The system's other components are a fuel metering or injection pump and a nozzle. With pressures ranging from 16 to 35 bar, the nozzle injects gasoline into the combustion chamber or the manifold at about 6.5 bar or into the cylinder head port at pressures of about 6.5 bar. In most cases, the early portion of the suction stroke is when fuel is injected via a timed injection system. As soon as the exhaust valve closes during maximum power operation, injection starts and often finishes after BDC. Manifold injection is inferior to direct in-cylinder injection, which is always preferable and better. Higher volumetric efficiency may be attained by using both low and high volatile fuels in this situation. If the vehicle is utilised for everyday transportation, it was shown that direct injection results in oil dilution during the repeated warm-up stages [6].

Electronic Fuel Injection System

To precisely measure and inject the proper quantity of fuel into the engine cylinders, modern petrol injection systems include engine sensors, a computer, and solenoid-operated fuel injectors. Electrical and electronic equipment are used by these systems, also known as

electronic fuel injection (EFI), to monitor and manage engine functioning. Electrical signals in the form of current or voltage are received by an electronic control unit (ECU) or the computer from numerous sensors. The injectors, ignition system, and other components connected to the engine are then operated using the recorded data.

As a consequence, the car gets better mileage and emits less unburned gasoline as pollutants. The following are examples of typical sensors for an electronic fuel injection system.

- (i) Exhaust gas or oxygen sensor - determines the air-fuel ratio by measuring the quantity of oxygen in the engine exhaust. The air-fuel ratio has an impact on sensor output voltage.
- (ii) The engine temperature sensor measures the coolant temperature, and using this information, the computer changes the mixture's strength to the rich side for cold starting.
- (iii) Air flow sensor - regulates the amount of fuel by monitoring the mass or volume of air moving into the intake manifold.
- (iv) Air intake temperature sensor: used to fine-tune the mixture strength, this sensor measures the temperature of ambient air entering the engine.
- (v) In order to optimize the mixture flow for engine speed and acceleration, the throttle position sensor (optional) detects the movement of the throttle plate.
- (vi) The engine intake manifold vacuum is monitored by the manifold pressure sensor, which allows the mixture strength to be changed in response to variations in engine load.
- (vii) The camshaft position sensor measures the engine's camshaft and crankshaft rotation to determine the time and pace of injection.
- (viii) A knock sensor is a microphone-style sensor that may be used to detect ping or pre-ignition noise and delay the ignition.

In an EFI, the fuel injector is only a fuel valve. The injector remains closed when it is not powered up by spring pressure, which prevents gasoline from entering the engine. The injector armature is drawn to the magnetic field created when the computer delivers the signal via the injector coil. The intake manifold is then sprayed with fuel. The length of time that each injector is powered on and held open is indicated by the injector pulse width. Based on the information from the different sensors, the computer chooses and regulates the injector pulse width [7], [8].

The computer will detect a wide open throttle, high intake manifold pressure, and high inlet air flow when the engine is fully loaded. Then, in order to enrich the mixture and allow the engine to create more power, the ECU will widen the injector pulse width. The ECU will reduce the pulse width while the engine is idle or under low load to keep the injectors closed for a longer length of time. As a consequence, the air-fuel combination will be leaner, improving fuel efficiency.

A cold start injector is also part of the electronic fuel injection system. When the engine is cold, this additional injector pours gasoline into the centre of the engine's intake manifold. It

does the same task as a carburetor choke. In very cold temperatures, the cold start injector provides simple engine starts.

Merits of EFI System

The spark ignition engine with an EFI system has the following advantages over a carburetor unit:

- (i) An increase in volumetric efficiency brought on by the intake manifolds' relative low resistance, which will result in lower pressure losses. It almost eliminates the need for manifold heating and removes the bulk of carburetor pressure losses.
- (ii) Because the gasoline is injected into or near to the cylinder and does not need to flow via the manifold, manifold wetness is removed.
- (iii) Because fuel atomization is independent of cranking speed, starting will be simpler.
- (iv) The engine will be less likely to knock with improved atomization and vaporization.
- (v) The throttle plate no longer develops ice.
- (vi) Less volatile fuel may be utilized since distribution is not reliant on vaporization. Even when the vehicle is in various locations, such as turning, travelling on grades, uneven roads, etc., the variation in the air-fuel ratio is essentially minor. Because the injection unit's position is less important, the engine (and hood) may be built lower.

Demerits of EFI System

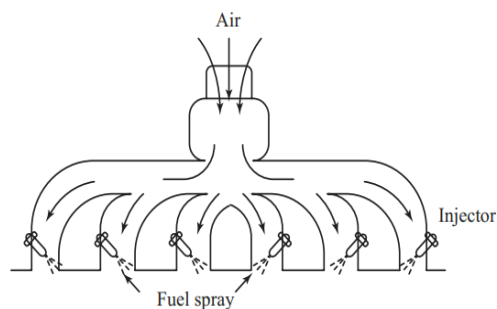
Some of the disadvantages of EFI system are:

- (i) High maintenance cost,
- (ii) Difficulty in servicing, and
- (iii) Possibility of malfunction of some sensors.

Multi-Point Fuel Injection (MPFI) System

The Multi-Point Fuel Injection (MPFI) system's primary goal is to deliver the right proportion of petrol and air to the cylinders. These systems operate using two fundamental configurations, namely

- (i) Port injection
- (ii) Throttle body injection



**Figure 1: Multi-Point Fuel Injection (MPFI) Near Port
(PDFCOFFEE.COM)**

Port Injection

In the port injection configuration, the injector is positioned close to the intake port on the side of the intake manifold (Fig. 2). This injector sprays petrol into the intake manifold. It takes some time for the petrol and air to blend evenly. After passing through the intake valve and into the cylinder, this petrol and air mixture.

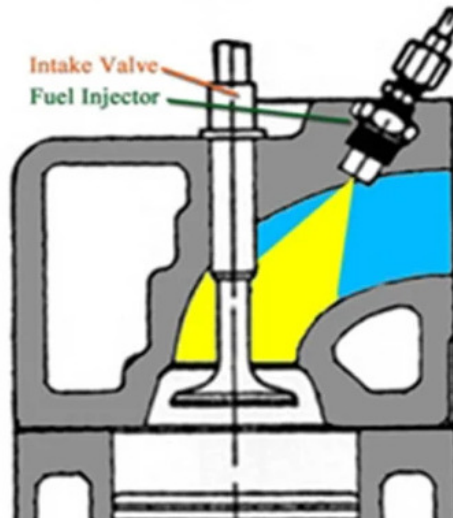


Figure 2: Port Injection (Skill-Lync)

Every cylinder is provided with an injector in its intake manifold. If there are six cylinders, there will be six injectors. Figure. 1 shows a simplified view of a port or multi point fuel injection (MPFI) system.

Throttle Body Injection System

The simplified drawing of the throttle body injection system (also known as single point injection) is shown in Figure 9.5. The volume of air entering the intake manifold is controlled by the throttle valve in this throttle body, which is identical to the carburetor's throttle body.

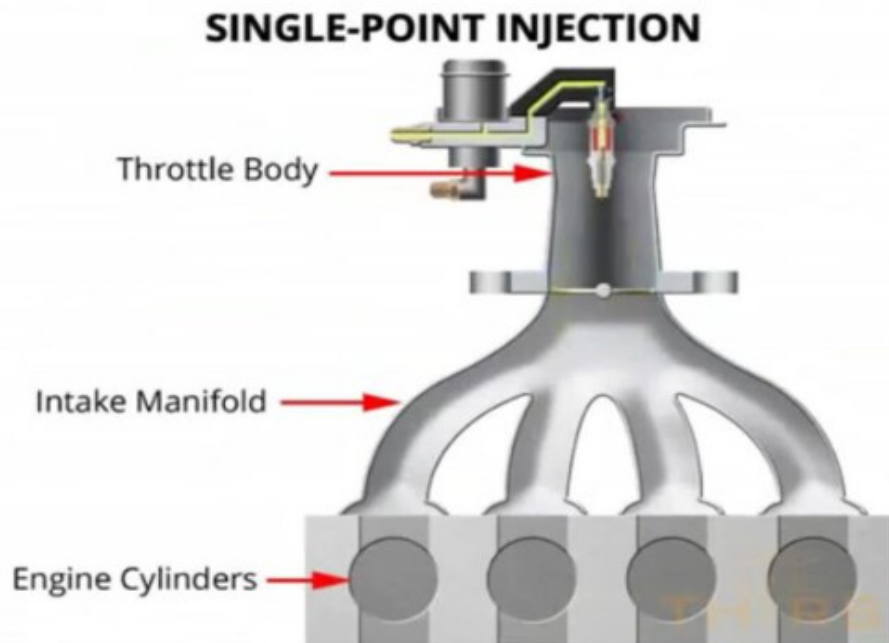


Figure 3: Throttle Body Injection (Single Point) [Spinny]

A little amount of space is left above the throttle body's throat for an injector. The intake manifold is where petrol and air are mixed after being sprayed into the air by the injector. This mixture then reaches the intake manifold after passing via the throttle valve. Fuel-injection systems may be timed or continuous, as was before indicated. Petrol is pulsed out of the injectors in the timed injection system. Petrol is continually sprayed from the injectors in a continuous injection system. Both the throttle-body injection system and the port injection system may be continuous or pulsed systems. The quantity of petrol injected in each method is based on the engine's speed and power requirements. DMPFI and L-MPFI are two categories under which MPFI systems are categorized in certain literature [9], [10].

D-MPFI System

The manifold fuel injection system is called the D-MPFI system. In this style, the intake manifold vacuum is initially detected. Additionally, it detects the air volume based on its density. Figure 9.6 shows a block diagram of how the D-MPFI system operates. The manifold pressure sensor measures the intake manifold vacuum as air enters the intake manifold and transmits the information to the ECU. The speed sensor also communicates to the ECU data on engine rpm. To control the quantity of petrol supplied for injection, the ECU in turn sends orders to the injector. The gasoline and air combine when the injector sprays fuel into the intake manifold, and the combination then enters the cylinder.

L-MPFI System

A port fuel-injection system is the L-MPFI system. The engine speed and the quantity of air that actually enters the engine control the fuel metering in this kind. This process is known as air-mass or air-flow metering. Figure 9.7 depicts the block diagram of an L-MPFI system. The air flow sensor detects the volume of air entering the intake manifold and transmits data to the ECU. In a similar manner, the speed sensor informs the ECU of the engine speed. To control the quantity of petrol supplied for injection, the ECU analyses the data it has received and provides the relevant orders to the injector. When injection occurs, the petrol and air combine, and the resulting mixture enters the cylinder.

CONCLUSION

Modern internal combustion engines employ electronic injection systems, often known as electronic fuel injection (EFI) systems, as sophisticated fuel delivery systems. They are made up of parts like fuel injectors, an engine control unit (ECU), and different sensors, and they replace carburetors. Electronic injection systems' future potential includes further improving engine performance, cutting emissions, integrating with cutting-edge technologies like direct injection and hybrid/electric powertrains, enhancing control algorithms and connectivity for data analysis, and converting to alternative fuels.

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CHAPTER 10

EXPLORING THE IGNITION SYSTEM IN IC ENGINE

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ABSTRACT

A spark or other method is used to start the combustion of the air-fuel combination during the ignition process, which is a crucial step in internal combustion engines. It is essential to engine efficiency, output, and emissions. An overview of ignition systems, including compression ignition and spark ignition, their operating concepts, and their effects on engine performance are given in this abstract. Additionally, it talks about the possible advantages of direct ignition, multi-spark systems, and laser ignition for next engine development.

KEYWORDS

Ignition, Internal Combustion Engines, Laser Ignition, Multi-Spark Systems, Spark Ignition.

INTRODUCTION

Internal combustion engines are built around the basic mechanism of ignition. It acts as the ignition source for the air-fuel combination, starting the combustion process and producing the power required to move objects down the road, run equipment, and provide energy for other uses. The effectiveness and management of the ignition process have a big impact on an engine's performance, emissions, and efficiency. The numerous features of ignition, including their importance in internal combustion engines, the various ignition system types, the combustion process, and the effect of ignition on engine performance, efficiency, and emissions, will all be covered in this thorough introduction. We will also explore recent developments in ignition technology and talk about potential future developments that might further enhance this critical component of engine functioning[1]–[3].

The process of starting the burning of the air-fuel combination within the engine cylinders is known as ignition. It includes igniting the combustible mixture by releasing a high-energy source, often a spark or a controlled compression. An engine's pistons are eventually driven by this ignition event's quick gas expansion, which transforms chemical energy into mechanical work and produces power. Because it controls the timing and effectiveness of combustion, the ignition process is essential for engine functioning. By ensuring optimum combustion, proper ignition timing maximises the engine's output of power, fuel efficiency, and overall performance. A poor or faulty ignition, on the other hand, might result in decreased efficiency, greater fuel consumption, and higher levels of toxic pollutants.

Internal combustion engines typically employ one of two ignition system types: compression ignition or spark ignition. In petrol engines, spark ignition is common. A spark plug generates an electrical discharge that ignites the air-fuel combination. On the other hand, diesel engines use compression ignition, which relies on the strong compression of the air-fuel combination to provide the needed heat for self-ignition.

Engine efficiency and performance have been dramatically impacted by developments in ignition technology. For example, direct ignition systems have taken the place of conventional distributor-based systems, allowing for more accurate ignition timing control

and improving combustion efficiency. Multiple sparks may be delivered during each ignition event by multi-spark systems, which further optimize combustion and increase power production. Furthermore, cutting-edge innovations like laser and plasma ignition hold promise for creating even more precise and efficient igniting procedures.

There are many stages to the combustion process itself, such as burn rate, flame propagation, and ignition delay. These combustion stages are directly impacted by ignition timing, which is the time at which the spark or compression happens. It also has an effect on variables like peak cylinder pressure, combustion stability, and the occurrence of unwanted phenomena like banging or pre-ignition. Achieving efficient and clean combustion depends on the right air-fuel mixture quality, ignition timing, and combustion chamber design.

It is impossible to overstate the effect of ignition on engine output, effectiveness, and emissions. The engine's overall performance and manoeuvrability are directly impacted by the quality of the ignition, which also influences the engine's capacity to produce power, torque, and acceleration.

Additionally, ignition efficiency is important for fuel economy since inefficient ignition may lead to incomplete combustion and lost fuel. The decrease of nitrogen oxides (NO_x), particulate matter (PM), unburned hydrocarbons (HC), and carbon monoxide (CO) is another way that ignition regulates emissions by influencing the combustion process.

Future ignition technology advancements promise to significantly improve engine performance, efficiency, and emissions management. Advanced timing algorithms and adaptive ignition techniques are examples of ignition control system improvements that offer increased accuracy and optimization. Combining the best features of spark and compression ignition, hybrid ignition systems have the potential to increase efficiency across a wider variety of operating situations. When ignition systems are integrated with electric drivetrains, opportunities for synergistic optimization and improved system efficiency open up. Innovative technologies like plasma and laser ignition provide promising opportunities for greater.

Due to the lower compression ratio and higher self-ignition temperature of gasoline in spark-ignition engines, an ignition system is essential for igniting the mixture and starting combustion.

In a spark-ignition engine, the ignition system creates an electrical discharge between the two electrodes of a spark plug to initiate the combustion process. This occurs towards the conclusion of the compression stroke. The spark's high temperature plasma kernel matures into a self-sustaining, spreading flame front. Several exothermic chemical reactions take place in this thin reaction sheet. The ignition system's job is to start the flame propagation process. It should be emphasized that the spark must be generated in a repeatable way, i.e., cycle after cycle, across the whole range of the engine's load and speed, at the proper point in the cycle.

Inferentially, ignition just serves as a condition for combustion. To comprehend the phenomena of combustion, it is necessary to examine ignition in order to develop a standard for determining if ignition has taken place. Although the ignition process is closely related to the beginning of combustion, it is not tied to the unsavory characteristics of combustion. Rather, it is a localized, localized event that only affects a limited area of the combustion chamber[4]–[6]. Ignition does not have a degree, either intensely or broadly, according to the basic definition. The medium's combustion is either started or it isn't. Therefore, it makes sense to think about ignition from the perspective of the start of the combustion process that it triggers.

Energy Requirements for Ignition

The sum of the spherical flame's surface area and the enthalpy per unit area yields the total enthalpy needed to maintain the flame and encourage ignite. It is acceptable to infer that the ignition system's fundamental requirement is that it must provide this energy inside a limited volume. Additionally, ignition should take place during a period of time that is brief enough to prevent energy loss other than that required to start the flame. Given the final criteria, it is clear that the pace of energy delivery is just as crucial as the overall amount of energy delivered.

Most of the conditions for ignition would seem to be met by a modest, brief electric spark. Applying a high enough voltage between two electrodes with a gap may produce a spark, and there is a threshold voltage below which sparking is not possible. The size of the distance between the electrodes, the fuel-to-air ratio, and the gas pressure all affect the critical voltage. Regarding the amount of energy needed, it is also crucial to consider how the voltage is increased to the critical value as well as the arrangement and state of the electrodes.

The law of energy conservation is observed throughout an igniting process. Therefore, it may be seen of as a balance between the energy that is

- (i) Supplied by an external source,
- (ii) Produced during chemical reactions, and
- (iii) Lost to the environment via thermal conduction, convection, and radiation.

The Energy and Duration of the Spark

A spark with energy of around 1 mJ and duration of a few microseconds would be sufficient to start the combustion process in a cylinder with a homogenous mixture. In actuality, though, the situation is less than perfect. The voltage necessary to ignite a spark is greatly influenced by the pressure, temperature, and density of the mixture between the spark plug electrodes. Therefore, during the whole range of engine operation, the spark energy and duration must be of sufficient order to start combustion under the most adverse circumstances anticipated in the area of the spark plug. Reliable ignition is often achieved when the spark energy surpasses 40 mJ and the duration is more than 0.5 ms. The loss of electrical energy via these deposits may stop the spark discharge if the resistance of the deposits on the spark plug electrodes is high enough.

DISCUSSION

An essential step in internal combustion engines is ignition, which starts the burning of the fuel and air combination that produces power. Peak cylinder pressure and burn rate are two elements that are impacted by proper ignition timing, which is crucial for effective and ideal combustion. In contrast to compression ignition systems, which depend on strong compression to accomplish self-ignition, spark ignition systems employ spark plugs to produce an electrical discharge. Engine efficiency and performance have increased because to developments in ignition technology including direct ignition and multi-spark systems. In order to reduce emissions and achieve cleaner combustion, effective ignition control is crucial.

Ignition System

Spark discharge is influenced by the voltage applied across the spark plug electrodes. Additionally, under all working circumstances, it should provide the spark with the necessary energy to ignite the combustible mixture close to the plug electrodes. It should be noted that the ideal spark timing for a certain engine design depends on the engine speed, intake

manifold pressure, and mixture composition. These elements should be considered in the design of a conventional ignition system to provide the spark of the right energy and duration at the right moment[7]–[9].

An air gap in an electric circuit operates as a high resistance since air is a poor conductor of electricity. However, a spark crosses the gap when a high voltage is delivered between a spark plug's electrodes. The spark-ignition system refers to the process by which such a spark is generated to ignite a homogenous air-fuel mixture in an engine's combustion chamber. Depending on how the main energy required to operate the circuit is made accessible, there are many types of ignition systems.

- (i) Battery ignition systems
- (ii) Magneto ignition systems

Ignition System Requirements

For an engine to perform properly, the ignition system must run smoothly and consistently. These are the specifications for such an ignition system:

- (i) At the appropriate time, it should produce a strong spark between the electrodes of the plugs.
- (ii) It ought to work well over the whole engine speed range.
- (iii) It must be portable, efficient, and dependable in use.
- (iv) It need to be portable and simple to maintain.
- (v) It should be affordable and simple to use.
- (vi) The radio and television receivers inside a car shouldn't be affected by the high voltage source's interference.

Battery Ignition System

A battery ignition system is used by the majority of contemporary spark-ignition engines. In this method, a 6 or 12 volt battery is used to provide the spark-producing energy. A battery ignition system may be built in a huge variety of ways. It is based on the kind of ignition energy storage and the ignition performance needed for the specific engine. This is due to the fact that an ignition system is just one component of the internal combustion engine, which is the engine's heart and does not function entirely independently. Therefore, it is crucial that the ignition system be well suited to the engine.

Battery ignition systems are seen in passenger vehicles, light trucks, certain motorbikes, and big stationary engines. Fig.1 depicts the specifics of a six-cylinder engine's battery ignition system.

The following are the system's main parts:

- (i) battery
- (ii) ignition switch
- (iii) ballast resistor
- (iv) ignition coil
- (v) contact breaker
- (vi) capacitor
- (vii) distributor
- (viii) spark plug

The ignition coil's secondary side is where the last four components are located, whereas the primary side is where the first three components on the list above are located. The following sections provide a short description of each component's specifics.

The main winding of the coil is linked to the positive terminal post of the storage battery when the ignition switch is engaged. A current known as primary current flows when the primary circuit is closed using the breaker connections.

A magnetic field is created in the core when this current passes through the main coil, which is coiled over a soft iron core. Every time an ignition discharge is needed, the breaker points are set up to be opened by a cam that is powered by the engine shaft. The current that had been flowing through the points now flows into the condenser, which is linked across the points, when the breaker points open. The main current decreases and the magnetic field collapses as the condenser charges up. The main winding experiences a voltage due to the field collapsing, which charges the condenser to a voltage far greater than battery voltage. The magnetic field and primary current are then reversed when the condenser discharges into the battery. The secondary winding of the ignition coil experiences an extremely high voltage due to the magnetic field's quick collapse and reversal in the core. The secondary winding is made up of several rounds of very tiny wire coiled around the same core as the main winding. The distributor, a spinning switch found in the secondary or high tension circuit of the ignition system, directs the high secondary voltage to the appropriate spark plug.

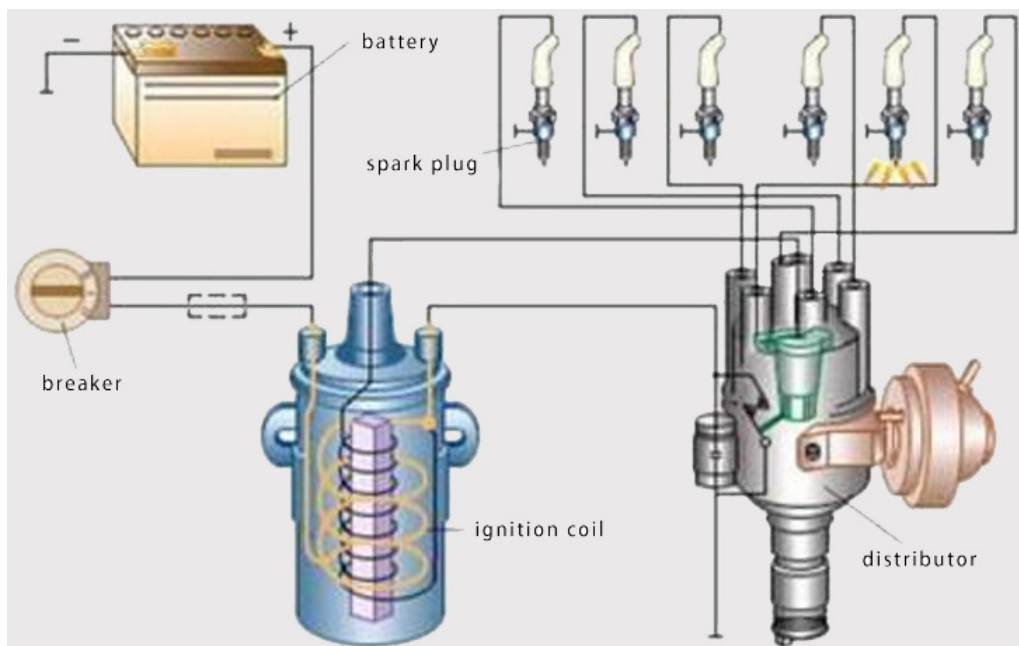


Figure 1: Battery Ignition System for a Six-Cylinder Engine (Alibaba.Com)

Operation of a Battery Ignition System

The ignition coil in the battery ignition system is the source of the ignition energy. This coil stores the energy in its magnetic field and transfers it to the proper spark plug at the time of ignition (the firing point) in the form of a burst of high voltage current (the ignition pulse). We also refer to the ignition coil as an inductive storage device since energy storage in the magnetic field is based on an inductive process. Fig.2 is the schematic diagram of a typical battery ignition system for a four-cylinder engine.

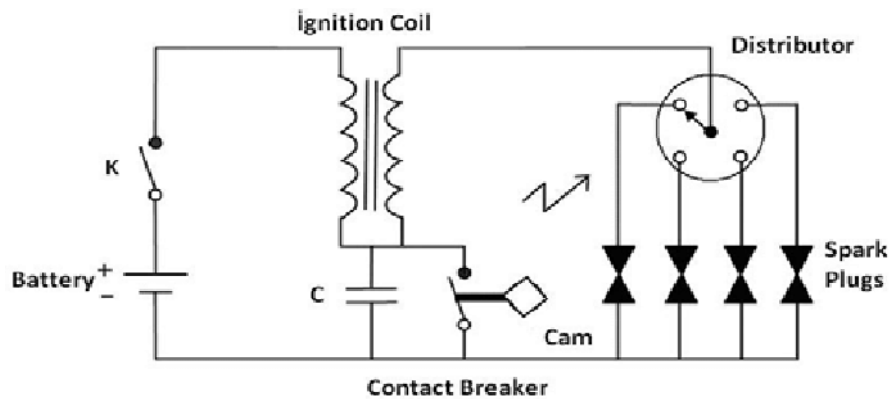


Figure 2: Diagram of a Four-Cylinder Engine's Conventional Coil Ignition System (Researchgate.Net)

The ignition coil, as was already noted, consists of two wire coils that are separated from one another: the primary winding, L1, which has a few twists of heavy copper wire, and the secondary winding, L2, which has several twists of fine copper wire. A laminated iron core is encircled by the main and auxiliary windings, enhancing the magnetic field's strength and, thus, the amount of energy retained.

The positive terminal post of the storage battery is linked to one end of the main winding through the ignition switch, while the other end is grounded by the contact breaker. The contact breaker and ignition capacitor are linked in parallel. The secondary winding is linked to the contact breaker on one end and to the center electrode of the spark plug on the other end through the distributor and high-tension ignition wires. Note that the distributor only decides the order in which the spark plugs fire; the timing of the spark is controlled by the crank angle at which the breaker points open. Rotating the plate that holds the breaker points in relation to the cam may change the ignition timing. As a result, if the plate is moved in the direction of the camshaft's rotation, ignition will be delayed.

Limitations

- (i) Because the breaker system's inability to switch enough current, the main voltage drops as engine speed rises.
- (ii) As engine speed rises, the dwell duration shortens, reducing the amount of time available for the primary coil's current to build up and the stored energy.
- (iii) Due to the system's sensitivity to side-tracking over the spark plug insulator, the system has a high source impedance of roughly 500 k.
- (iv) The continual electrical and mechanical wear that the breaker points experience causes short maintenance windows. Increased currents quickly reduce system dependability and breaker point life. With a main current restriction of no more than 4 amperes, these systems may function for an acceptable amount of time.

Advantage of A 12 V Ignition System

All automobile engines featured 6-volt ignition systems up until roughly 1950. The main benefit of a 6-volt system is the use of a three-cell storage battery, which is less expensive, lighter, and bulkier than a six-cell battery with a comparable watt-hour capacity. The 6-volt system performed well with the then-common low compression ratios. The voltage needed to destroy the spark gap grew as compression ratios and engine speeds climbed. Because 12-volt systems can achieve much greater secondary voltages, they have become more popular [10]–[12].

The following are some additional benefits:

- (i) The cables in a 6-volt system need theoretically be twice as thick as 12-volt cables in order to convey equivalent power without experiencing an excessive voltage drop.
- (ii) The 12-volt system, in particular, greatly enhances starting since the ignition coil has access to approximately twice as much power during the beginning surge.
- (iii) The 12-volt system is able to deliver enough electricity to the growing number of utilized electrical accessories.

CONCLUSION

In internal combustion engines, ignition is the process of starting combustion by the use of a spark or compression. It is necessary for controlling emissions and improving engine efficiency and performance. Future developments in ignition technology will likely include better ignition control systems, hybrid ignition systems, integration with electric propulsion systems, and the investigation of cutting-edge ignition techniques like laser igniting. These innovations seek to improve combustion efficiency, lower emissions, and help the shift to cleaner, more environmentally friendly transportation. The high primary voltage brought on by the collapse of the magnetic field around the primary winding would result in an arc across the breaker points if a condenser were not utilized in the primary circuit. The main current and magnetic field would not decrease quickly enough to produce the high secondary voltage without the arc burning and destroying the points.

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CHAPTER 11

CLASSIFICATION OF COMBUSTION AND COMBUSTION CHAMBERS

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ABSTRACT

The fast oxidation of a fuel in the presence of oxygen causes combustion, a chemical process that releases energy in the form of heat and light. It is a basic procedure in internal combustion engines, where the power required for vehicle propulsion is produced by the burning of the air-fuel combination. An overview of combustion concepts is given in this abstract, along with information on the combustion process, variables affecting combustion efficiency, and combustion chamber design concerns. The significance of combustion optimization for engine performance, efficiency, and emissions reduction is also covered. A key component of many applications, such as internal combustion engines, power production, industrial operations, and heating systems, combustion is a basic chemical reaction. It entails the quick oxidation of a fuel in the presence of oxygen, releasing energy in the form of heat and light as a consequence. For many technological advancements, combustion must be understood and optimized for effectiveness, performance, and emissions reduction.

KEYWORDS

Combustion, Combustion Chambers, Combustion Efficiency, Combustion Process, Internal Combustion Engines.

INTRODUCTION

A chemical process known as combustion occurs when specific fuel components, such as hydrogen and carbon, mix with oxygen to produce heat energy and raise the temperature of the gases. The existence of a combustible mixture and a way to start the process are prerequisites for combustion. The theory of combustion is a tremendously complicated subject that has long been the focus of much study. Despite this, there is little information accessible about the phenomena of combustion. Depending on the kind of engine, the process of combustion typically occurs in either a homogeneous or a heterogeneous fuel vapor-air combination [1]–[3]. In this thorough introduction, we will discuss the fundamentals of combustion, the combustion process itself, and the importance of combustion chambers in ensuring efficient and regulated combustion. We'll examine the variables that affect combustion efficiency and go through the design parameters to take into account while building efficient combustion chambers.

When a fuel and oxygen, often from the air, mix in an exothermic process, combustion is the chemical reaction that results. Heat, which is released during this process and may be used for a number of things, is a kind of energy. Fuel, oxygen, and an ignition source are the three primary components required for combustion. Particularly for the purpose of producing power, combustion is a major factor in internal combustion engines. A limited area known as the combustion chamber is where the air-fuel combination is burned in these engines. For regulated and effective combustion, this chamber has been thoughtfully built. There are a number of clearly defined phases in the combustion process. To make a combustible combination, the fuel and oxygen must first be completely combined. The combustion process is then started by lighting this mixture on fire using an ignition source, such a spark

or compression. After being lit, the fuel quickly oxidizes, releasing hot gases that expand and create pressure. The movement of engine parts like pistons or turbine blades is driven by this pressure, which transforms the chemical energy of the fuel into meaningful mechanical work.

For engines to operate effectively and efficiently as well as to reduce emissions, maximum combustion efficiency must be attained. The air-fuel ratio, combustion time, combustion length, and ignition timing are some of the variables that affect combustion efficiency. To achieve complete combustion without using too much fuel or oxygen, the air-fuel ratio has to be properly managed. In the combustion chamber, turbulence promotes effective mixing and more complete burning. Engine performance and emissions are influenced by the length of the combustion phase and the timing of the ignition. Achieving an effective and regulated combustion depends heavily on the combustion chamber's architecture. The airflow, fuel distribution, and combustion properties inside the chamber are influenced by a number of variables, including chamber size, shape, and fuel injector location. The objective is to design a combustion chamber that encourages effective mixing, reduces heat losses, maximizes fuel burning, and reduces the production of undesirable byproducts.

Significant gains in combustion efficiency and performance have been made because to advancements in combustion chamber design, including better forms, more sophisticated fuel injection systems, and optimization via computer modelling. Engineers may design combustion chambers that maximize power production, fuel efficiency, and emissions management by carefully taking into account elements like air swirl, fuel atomization, and combustion stability. In summary, combustion is a basic chemical reaction that drives internal combustion engines and a variety of other uses. The key to maximizing engine performance, efficiency, and emissions management is efficient and regulated combustion. An atmosphere that encourages effective mixing, complete combustion, and the limitation of toxic byproducts is crucially influenced by the design and optimization of combustion chambers. For further improving the effectiveness and sustainability of combustion-based systems, ongoing research and improvements in combustion technology are essential.

The Homogeneous Mixture

A virtually homogenous combination of fuel and air is created in the carburetor of spark-ignition engines. Thus, a homogeneous mixture is created outside the engine cylinder, and at a certain moment towards the conclusion of the compression stroke, combustion begins within the cylinder. With a certain velocity, the flame front spreads throughout a flammable mixture. Fuel and oxygen molecules are more or less evenly distributed in a homogenous gas mixture. When the combination of fuel vapor and air ignites, a flame front develops and quickly spreads across it. Heat transmission and the diffusion of burning fuel molecules from the combustion zone to the surrounding layers of unburned mixture are what cause the flame to spread. The new mixture and the combustion byproducts are separated by a small area known as the flame front. The normal flame velocity is the speed at which the flame front advances towards the unburned mixture in a direction perpendicular to its surface.

The flame speed is typically in the range of 40 cm/s in a homogenous mixture with an equivalency ratio, (the ratio of the actual fuel-air ratio to the stoichiometric fuel-air ratio) close to 1.0. The maximum flame speed in a spark-ignition engine is achieved when is between 1.1 and 1.2, or when the mixture is only a little richer than stoichiometric. The flame speed quickly decreases to a low value if the equivalency ratio is outside of this range. The flame extinguishes when the flame speed reaches a very low value, at which point the heat loss from the combustion zone equals the heat generated during burning. For optimal combustion, it is thus much better to run the engine at an equivalency ratio of 1.1 to 1.2. In

mixes beyond the aforementioned range, the flame speed may be enhanced by adding turbulence and appropriate air movement.

The Heterogeneous Mixture

The mutual diffusion speed of air and fuel vapours in a heterogeneous gas mixture, rather than the rate of chemical reaction, determines the rate of combustion in this system. The fundamental factor in defining the parameters of combustion is self-ignition or spontaneous ignition of the fuel-air combination at the high temperature created by greater compression ratios. Since there are always local zones where fluctuates between 1.0 and 1.2, corresponding to the maximal rate of chemical reaction, combustion may occur in a heterogeneous mixture even when the mixture is overall lean. This zone is where ignition begins, and the flame that results helps to burn the fuel in the nearby zones where the mixture is thinner. Similar to this, combustion happens in the zones where the mixture is rich owing to the high temperature created by combustion that was begun in the zones where is 1.0 to 1.2. The next sections provide a thorough analysis of both compression- and spark-ignited engine combustion.

Combustion in Spark-Ignition Engines

In a normal spark-ignition engine, as was previously said, the fuel and air are uniformly mixed in the intake system before being introduced via the intake valve into the cylinder, where they combine with residual gases and are then compressed. Under typical operating circumstances, an electric discharge at the spark plug starts combustion near the end of the compression stroke. Following ignition, a turbulent flame forms and spreads through this premixed charge of fuel and air as well as the leftover gas in the clearance volume until it reaches the walls of the combustion chamber. The two primary forms of combustion in the SI engine are normal combustion and aberrant combustion.

DISCUSSION

Stages of Combustion in SI Engines

Figure 1 depicts a typical theoretical pressure-crank angle diagram for the compression (a), combustion (b), and expansion (c) phases of an ideal four-stroke spark-ignition engine. The graphic shows that in an ideal engine, the whole pressure increase during combustion occurs at constant volume, or at TDC. This does not occur in a real engine, however. Below is a thorough description of the combustion process of a real SI engine [4]–[6]. The combustion process in a SI engine, according to Sir Ricardo, who is regarded as the founder of engine development, has three stages:

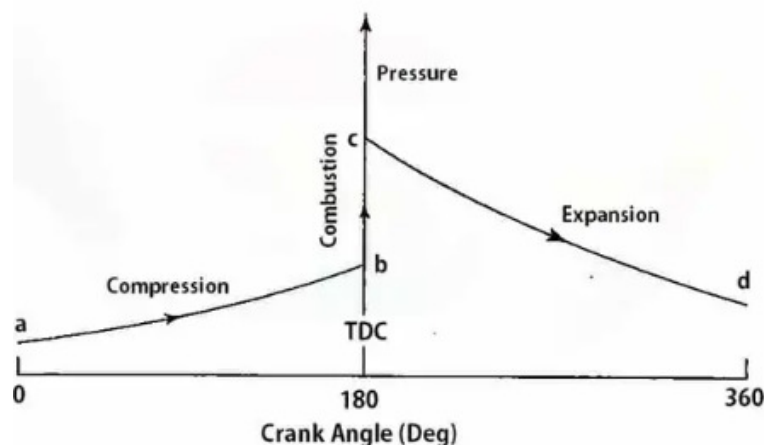


Figure 1: Theoretical p- θ diagram (extrudesign.com)

Fig. 2 depicts the pressure change brought on by combustion in a real engine. In this diagram, A represents the spark passing point (let's say 20 bT DC), B represents the starting point of the pressure increase (let's say 8 bT DC), and C represents the achievement of peak pressure. As a result, AB stands for the first stage, BC for the second stage, and CD for the third stage.

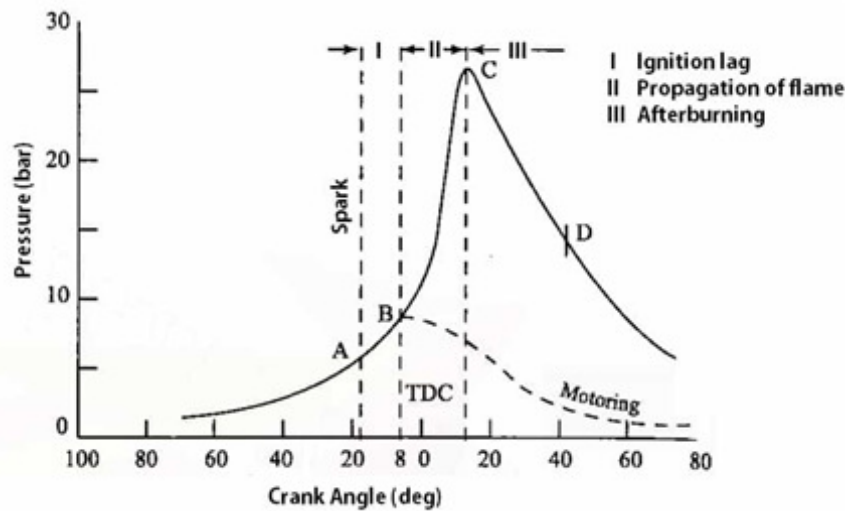


Figure 2: Stages of Combustion in a SI Engine (Jhotpotinfo.Com)

The first stage (A → B) is known as the ignition lag or preparation phase, during which a self-propagating flame nucleus grows and develops. This is a chemical process that is influenced by temperature, pressure, fuel type, and the amount of exhaust residual gas. Additionally, it depends on how the temperature and response rate are related. The physical second stage (B-C) is concerned with how the flame spreads across the combustion chamber. The indicator diagram's first detectable increase in pressure, or the point where the line of combustion diverges from the compression line (point B), marks the beginning of the second stage. The departure from the driving curve demonstrates this. The flame spreads nearly at a steady speed throughout the second stage. Because just a tiny portion of the burning mixture makes contact with the cylinder wall during this time, there is little heat transmission to the wall.

The amount of turbulence present and the reaction rate, which is influenced by the makeup of the mixture, both have a significant impact on the rate of heat release. Since the piston is close to the top dead center at this point and the combustion chamber capacity is essentially constant, the rate of pressure rise and the rate of heat release are proportionate. The moment the maximum pressure on the indicator diagram is achieved (point C), is often used as the third stage's beginning point. In this phase, the flame velocity reduces. Low flame velocity and decreased flame front surface cause the rate of combustion to decrease. There can be no pressure increase at this stage of combustion since the expansion stroke begins before it, with the piston moving away from the top dead center.

Flame Front Propagation

The flame front's pace of propagation within the cylinder is very important for effective combustion. The response rate and the transposition rate are the two key variables that affect how quickly the flame front moves across the combustion chamber. The process of pure chemical combination in which the flame consumes the unburned charge yields the reaction rate. The pressure difference between the burning gases and the unburned gases in the combustion chamber and the physical movement of the flame front in relation to the cylinder wall both contribute to the transposition rate.

The rate of flame propagation is seen in Figure 3. Area I, (A–B), has a low transposition rate and little turbulence, which causes the flame front to advance rather slowly. Due to the very tiny initial charge burnt mass, there is virtually little flame front transposition. The sluggish progress of the flame is mostly caused by the poor response rate. The absence of turbulence also slows down the response rate, which in turn slows down the flame speed since the spark plug must be placed in a gas layer that is quiescent and near to the cylinder wall. The flame front advances faster and at a constant rate (BC) as it moves out of the quiescent zone and into more turbulent regions (area II), where it consumes a larger amount of mixture, as illustrated in Fig.3. The amount of unburned charge is much smaller near the conclusion of flame travel, and as a result, the transposition rate again declines, slowing the flame speed. Since the flame is approaching a region (area III) with comparatively moderate turbulence (C–D) in Figure.3, the response rate is likewise decreased once again.

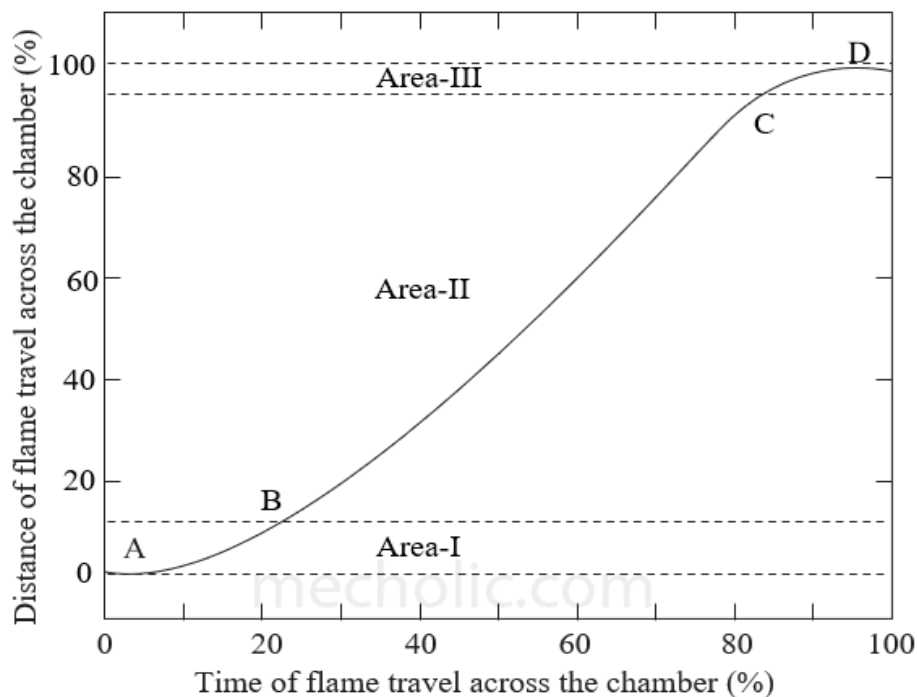


Figure 3: Details of Flame Travel

Abnormal Combustion

When combustion occurs normally, the spark-generated flame spreads rather uniformly across the combustion chamber. Under certain operational circumstances, combustion deviates from its usual path, resulting in performance loss and potential engine damage. A banging combustion or aberrant combustion are two terms that might be used to describe this sort of combustion. Loss of power, recurrent pre-ignition, and engine damage are all effects of this aberrant combustion process.

The Phenomenon of Knock in SI Engines

In a spark-ignition engine, combustion spreads over the combustible mixture after starting between the spark plug electrodes. From the spark plug to the opposite end of the combustion chamber, a distinct flame front separates the new mixture from the combustion by-products. The temperature and pressure of the burnt portion of the mixture are higher than those of the unburned portion owing to heat release caused by combustion. The burnt portion of the mixture will adiabatically expand and compress the unburned portion of the mixture in order to equalise the pressure, raising both its pressure and temperature. As the flame front moves

through the mixture, this process keeps going as the temperature and pressure of the unburned mixture rise.

A variety of pin-point sites experience spontaneous ignition or autoignition if the temperature of the unburned mixture is higher than the fuel's self-ignition temperature and stays there or higher during the pre-flame reaction time (ignition lag). Knocking is the name for this occurrence. Knocking of the engine results from autoignition [6]–[8].

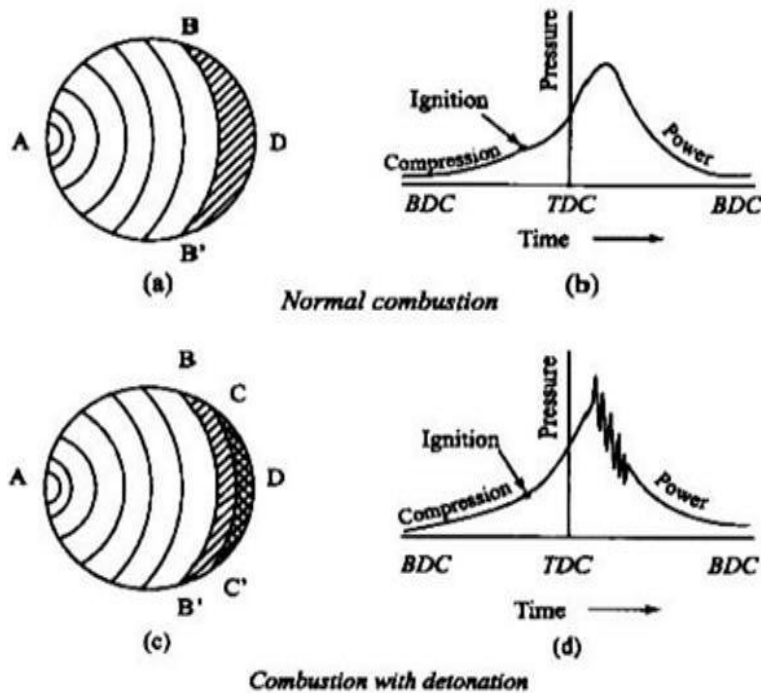


Figure 4: Normal and Abnormal Combustion (researchgate.net)

Referring to Fig. 4(a), which depicts the cross-section of the combustion chamber with flame advancing from spark plug point A without knock while Fig. 4(c) depicts the combustion process with knock, can help illustrate the phenomena of knock. In a typical combustion, the flame moves from A to D across the combustion chamber. The end charge BB D that is farthest from the spark plug is compressed by the moving flame front, increasing its temperature. Heat transmission from the hot, moving flame-front raises the temperature in addition. Additionally, some pre-flame oxidation may occur in the final charge, further raising the temperature. Despite these conditions, the charge would not self-ignite if the temperature of the end charge had not reached its self-ignition temperature. Instead, the flame would advance and devour the charge BB D. This is the typical combustion process, which is shown in Fig. 4(b)'s pressure-time curve. The charge will auto ignite, resulting in knocking combustion, if the end charge BB D achieves its autoignition temperature and stays there during the pre-flame processes. In Fig. 4(c), it is assumed that the charge ahead of the flame has attained critical autoignition temperature when it reaches point BB. If the flame front could only go from BB to CC during the pre-flame response phase, the charge in front of CC would auto-ignite.

The autoignition causes a second flame front to begin moving in the opposite direction of the first flame front. During the collision of the two flame fronts, a strong pressure pulse is produced. Along the pressure pulse, the gas in the chamber is compressed and rarefied until pressure equilibrium is restored. The walls of the combustion chambers may be forced to vibrate at the same frequency as the gas as a result of this disturbance. The frequency of gas

vibration in vehicle engines is in the range of 5000 cps. In such a case, the pressure-time trace is shown in Fig. 4(d).

It should be remembered that the characteristics of the fuel have a significant impact on when knocking first appears. The explanation above makes it obvious that there won't be any banging if the unburned charge doesn't reach its autoignition temperature.

Additionally, there won't be any banging if the first phase, or ignition lag period, lasts longer than the time needed for the flame front to burn through the unburned charge. However, the end charge will explode if the critical temperature is attained and sustained and the ignition lag is less than the time needed for the flame front to burn through the unburned charge. Therefore, a high autoignition temperature and a lengthy ignition lag are desired characteristics for SI engine fuels in order to prevent or suppress detonation.

In conclusion, two distinct kinds of vibration may be generated during autoignition. One scenario involves a significant quantity of mixture auto-igniting, causing a sharp spike in pressure throughout the combustion chamber and a direct hit on the engine's structure. The ensuing thudding sound and subsequent noise from free vibrations of the engine components are audible to the human ear. In the alternative scenario, there may be significant pressure fluctuations in the combustion chamber, which might cause the gas to vibrate at a frequency that matches the chamber walls.

It's possible to hear a sound. Engine failure may result from the knock's effects on the structure and parts of the engine, and the noise produced by the vibration of the engine is always unpleasant. Due to the pressure variations in the combustion chamber, the gas vibrates and scrubs the chamber walls, increasing the heat transfer to the coolant. A characteristic sound is often used to determine whether or not there is knocking combustion in engines. Utilising a pressure transducer is a scientific way to find knocking. Typically, a cathode ray oscilloscope is linked to this transducer's output. For a normal combustion and a knocking combustion, respectively, typical pressure-time traces that may be acquired from a pressure transducer are shown in Fig. 4(b) and Fig. 4(d), respectively.

Combustion in Compression-Ignition Engines

The combustion process in the SI and CI engines differs in several fundamental ways. In the SI engine, a homogeneous carburetted mixture of petrol vapour and air is compressed in a certain ratio (compression ratio 6:1 to 10:1), and the mixture is ignited by an electric spark at a single location just before the compression stroke is completed. After ignition, a single flame front moves through the air-fuel combination.

Only air is compressed in the CI engine at high compression ratios (16:1 to 20:1), which results in elevated air temperature and pressure. In the combustion chamber, this highly compressed air is injected with fuel using one or more jets. Here, the fuel jet detaches into a fuel core that is encircled by an air and fuel spray envelope. The atomization and vaporisation of the fuel both contribute to the formation of this spray envelope. The fuel particles are torn away from the core by the turbulence of air moving through the jet in the combustion chamber. At some point in the spray envelope, fuel and air combine, and oxidation begins.

The latent heat of vaporisation from the air around the liquid fuel droplets causes them to evaporate. This lowers the temperature of the thin layer of air around the droplet, and it takes some time before this temperature can be increased again by absorbing heat from the majority of the air. When the temperature of the air and this vapour reaches the autoignition level and the local A/F ratio is within the combustible range, ignition occurs. Thus, it follows that there is initially a considerable wait before ignition occurs.

The fuel-air combination is fundamentally heterogeneous because the fuel droplets cannot be injected and dispersed evenly across the combustion area. Under these circumstances, if the air inside the cylinder remained still, there wouldn't be enough oxygen in the burning zone, causing the fuel to burn slowly or not at all because it would be encircled by its own combustion products. Therefore, it is necessary to provide the air and fuel an organised and regulated motion so that fresh air is continuously given to each burning droplet and the by-products of combustion are whisked away. The impact of this air swirl, as it is sometimes known.

Turbulence in a SI engine is a disorganised air motion with no obvious flow direction. The swirl, which is necessary in CI engines, helps to break up the fuel jet by being an ordered movement of the whole air mass in a specific direction of flow. This swirl also causes the burnt and unburned mixture components to mix together. A single point of ignition and a gradual increase in pressure characterise a SI engine, while many sites of ignition and a quick rise in pressure characterise a CI engine. There is no distinct flame front in CI engines, in contrast to the combustion process in SI engines.

From no load to maximum load, the air-fuel ratio in a SI engine stays quite near to the stoichiometric value. But regardless of load or speed, an almost constant flow of air enters the cylinder in a CI engine. As the load changes, the amount of fuel injected changes, changing the air-fuel ratio. Thus, the total air-fuel ratio fluctuates from around 18:1 at maximum load to approximately 80:1 at idle. Since the mean effective pressure and power output are at their highest while the CI engine is running at full load, the fundamental objective of the engine's designer is that the A/F ratio should be as near to stoichiometric as possible. The operation of an engine with a leaner air-fuel ratio always offers a greater thermal efficiency, but the mean effective pressure and the power output decrease, according to a thermodynamic study of the engine cycles. Because of this, if an engine is operated close to stoichiometric conditions, the engine size increases for a given output. This is because the A/F ratio in some areas of the combustion chamber is likely to be so high that some fuel molecules won't be able to find the oxygen they need to burn, leading to the production of visibly black smoke. Thus, depending on the application, the CI engine is always built to run with an excess of air ranging from 15% to 40%. The shaded area denotes the general area of A/F ratios where visible black smoke develops.

The Phenomenon of Knock in CI Engines

As a result, more droplets are being injected into the chamber as the first few droplets pass through the ignition delay interval. A very little quantity of fuel will have collected in the chamber when real burning starts if the ignition delay of the fuel being injected is short and the initial few droplets start the actual burning phase quickly after injection. The mass rate of mixture combustion will be such as to provide a rate of pressure increase that will impose a smooth force on the piston. On the other hand, if the ignition delay is longer, a higher amount of fuel droplets build up in the chamber and the actual burning of the initial few droplets is delayed.

The extra fuel may result in an excessively quick rate of pressure increase when the real burning starts, which can jam the piston and produce rough engine running. If the ignition delay is sufficiently lengthy, there may be enough fuel accumulation for the rate of pressure increase to be almost instantaneous. The presence of audible knock indicates the presence of the severe pressure differentials and intense gas vibrations known as knocking. Similar to what happens in the SI engine, the phenomena. However, knocking happens towards the onset of combustion in the CI engine as opposed to the end of combustion in the SI engine [9]–[11].

Table 1: Effect of Variables on the Delay Period

Increases in variable	Effect on Delay Period	Reasons
Cetane number of fuel	Reduces	Reduces the self-ignition temperature
Injection pressure	Reduces	Reduces physical delay due to greater surface-volume ratio
Injection timing advance	Reduces	Reduced pressures and temperatures when the injection begins
Compression ratio	Reduces	Increases air temperature and pressure and reduces autoignition temperature
Intake temperature	Reduces	Increases air temperature
Jacket water temperature	Reduces	Increases wall and hence air temperature
Fuel temperature	Reduces	Increases chemical reaction due to better vaporization
Intake pressure (supercharging)	Reduces	Increases density and also reduces autoignition temperature
Speed	Increases in terms of crank angle. Reduces in terms of millisecond	Reduces loss of heat
Load (fuel-air ratio)	Decreases	Increases the operating temperature
Engine size	Decreases in terms of crank angle. Little effect in terms of milliseconds	Larger engines operate normally at low speeds
Type of combustion chamber	Lower for engines with pre-combustion chamber	Due to compactness of the chamber

Starting the real burning as soon as feasible after the injection starts is crucial to reduce the likelihood of knock. In other words, the quantity of fuel present when the initial few droplets actually begin to burn must be reduced in order to shorten the ignition delay.

Comparison of Knock in SI and CI Engines

It may be interesting to note that the primary cause of knocking in both compression- and spark-ignition engines is the fuel-air mixture's spontaneous ignition. Both times, the fuel-air mixture's autoignition latency is what causes the knocking. However, a close comparison of the knocking behaviour in compression-ignition and spark-ignition engines shows the following distinctions. On the pressure-time graphs, the knocking process in SI and CI engines is compared.

- (i) In spark-ignition engines, knocking is caused by the autoignition of the end gas far from the spark plug, presumably close to the end of the combustion. However, the autoignition of the charge that causes banging in compression-ignition engines occurs at the commencement of combustion. In compression-ignition engines, knocking is caused by the initial charge auto igniting. It provides an illustration of this. It makes it very evident that explosive auto-ignition is essentially finished before the peak pressure for the compression-ignition engines. However, following the peak pressure for spark-ignition engines, the conditions for explosive autoignition of the terminal charge are more favourable. In spark-ignition engines, autoignition of the end gas must be completely avoided in order to prevent banging. The earliest autoignition is required in a compression-ignition engine in order to prevent knocking.
- (ii) The intensity of knocking or the rate of pressure rise at explosive autoignition is likely to be higher in spark-ignition engines because the charge that auto ignites there is homogeneous, as opposed to compression-ignition engines where the fuel and air are not homogeneously mixed even when explosive autoignition of the charge occurs. As a result, detonation is a common name for it in SI engines.

Only air is compressed in compression-ignition engines during the compression stroke, and the ignition can only happen once fuel is introduced immediately before top dead centre. Therefore, unlike spark-ignition engines, compression-ignition engines cannot pre-ignite. It has previously been mentioned that autoignition is the typical method of combustion in compression-ignition engines.

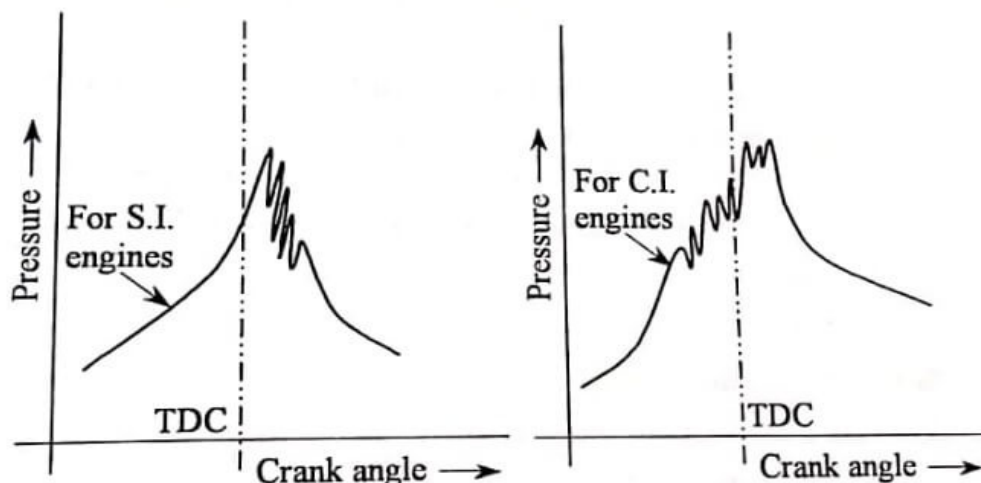


Figure 4: Diagrams Illustrating Knocking Combustion in SI and Ci Engines
(theteche.com)

CONCLUSION

A chemical process called combustion releases energy in the form of heat and light when a fuel is rapidly oxidized in the presence of oxygen. Internal combustion engines and several other applications use it as a basic process. Designated areas called combustion chambers are where the combustion process takes place. These chambers enable efficient and regulated burning. In order to improve combustion efficiency, decrease emissions, and promote the transition to cleaner and more sustainable energy systems, improved design approaches, optimization methodologies, and the integration of new technologies will shape the future of combustion and combustion chambers. In terms of crank rotation per degree, the usual rate of pressure increase for the initial portion of the charge for a compression-ignition engine is greater than for a spark-ignition engine. And in a compression-ignition engine, loud knock is often present at all times. The engine is considered to be knocking when the audible noise intensifies and the engine vibrates heavily. As a result, it also involves using discretion. It is exceedingly difficult to distinguish clearly between knocking combustion and regular combustion. In compression-ignition engines, the rate of pressure increase may reach 10 bar per degree of crank rotation. Knock is encouraged in CI engines by the same variables that lengthen the autoignition response time and lessen knock in SI engines. Additionally, a good fuel for a compression-ignition engine is a bad fuel for a spark-ignition engine. Diesel fuels have high cetane ratings of approximately 45 to 65 and low octane ratings of about 30, while spark-ignition fuels have high octane ratings of 80 to 100 and low cetane ratings of around 20.

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CHAPTER 12

IMPACT OF ENGINE FRICTION AND LUBRICATION

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ABSTRACT

Internal combustion engines' performance, efficiency, and longevity are significantly influenced by engine friction and lubrication. Power production, fuel consumption, and component wear are all directly impacted by friction, which is the resistance to motion between two surfaces. On the other side, lubrication decreases friction by creating a barrier between moving components, assuring smooth operation and limiting wear. This abstract gives a general introduction of engine lubrication and friction concepts, emphasizing the need of efficient lubrication procedures and friction reduction methods. Additionally, it looks at how friction and lubrication affect engine output, fuel efficiency, and emissions, highlighting how crucial it is to optimize these elements for better engine performance.

KEYWORDS:

Engine Friction, Engine Performance, Fuel Economy, Friction Reduction, Lubrication, Lubrication Strategies.

INTRODUCTION

Internal combustion engines' longevity, performance, and efficiency are all greatly influenced by engine friction and lubrication. Power production, fuel efficiency, and component wear are all directly impacted by friction, which is the barrier to motion between two surfaces in contact. Conversely, lubrication ensures smooth operation and reduces wear by forming a protective layer between moving elements, which helps to reduce friction. This thorough introduction will cover the fundamentals of engine friction and lubrication, the role they play in engine performance, and the methods used to lower friction and improve lubrication effectiveness. We will go into detail on how friction and lubrication affect engine performance, fuel efficiency, and emissions, emphasizing how crucial it is to optimize these elements for better engine performance and sustainability.

Engine friction is the resistance to motion that results from the interaction of two or more surfaces. In internal combustion engines, friction may develop from a number of places, such as piston rings, bearings, valve train parts, and contact between the piston and cylinder walls. The engine's output energy is significantly decreased due to frictional resistance, which results in power losses and a general decrease in efficiency. Lubrication is the process of inserting a lubricant, usually a fluid, between moving surfaces in order to reduce friction. In order to prevent direct metal-to-metal contact, lubricants provide a thin film or boundary layer that separates the contacting surfaces. This lowers friction and wear. Smooth operation, less energy loss, and increased component lifespan are all benefits of good lubrication.

To minimize friction and improve engine efficiency, a variety of lubrication methods are used. Engine oils play a crucial function since they are designed precisely to provide lubrication and protection. They include additives that increase lubricity, reduce friction, and provide defense against corrosion and wear. The viscosity, additives, and composition of engine oil are carefully chosen to meet the engine's working environment and

needs. Techniques for reducing friction are also used to reduce power losses brought on by friction. Improvements in engine design, such as better bearing surfaces, piston ring construction, and smaller internal clearances, aid in lowering frictional losses. To further minimize friction and increase longevity, crucial engine components get surface coatings and treatments like diamond-like carbon (DLC) coatings and low-friction coatings.

Beyond power production and efficiency, engine friction and lubrication have a significant influence. Fuel economy is directly impacted by friction since more friction causes more energy losses and higher fuel use. Improved dependability and lower maintenance costs are a result of effective lubrication, which lowers wear and increases the life of engine components. Additionally, friction and lubrication are very important in the regulation of emissions.

High operating temperatures and higher emissions of pollutants like nitrogen oxides (NO_x) and particulate matter (PM) may result from excessive friction. Low operating temperatures are maintained and friction-related pollutants are decreased with optimal lubrication [1], [2].

In conclusion, internal combustion engine performance, efficiency, and durability are significantly influenced by engine friction and lubrication. Engine designers and lubrication experts may increase power output, improve fuel efficiency, lengthen component life, and minimize emissions by minimizing friction and optimizing lubrication procedures. For engines to become more efficient, dependable, and environmentally friendly, ongoing research and development in friction reduction methods and cutting-edge lubricating technology is essential. In general, friction refers to forces that occur between surfaces that are moving relative to one another.

Frictional losses in engines are mostly caused by spinning and sliding components. In its broadest meaning, engine friction is often defined as the difference between the braking power (bp) and the indicated power (ip). Engine friction is often measured in terms of frictional power, abbreviated as f_p . The following mechanical losses are mostly responsible for frictional loss.

- (i) Direct frictional losses,
- (ii) Pumping losses,
- (iii) Power loss for the components that charge and scavenge, and
- (iv) Power loss for various auxiliary components.

A good engine design should prevent the overall frictional losses in reciprocating engines from exceeding 30% of the energy input. A competent designer's goal ought to be to reduce wear and friction on the components involved in relative motion. Proper lubrication helps to accomplish this. This section lists the several friction-related losses that may occur.

Direct Frictional Losses

It is the energy lost as a result of the relative motion of several bearing surfaces, including cam shaft bearings, main bearings, and piston rings. Because there are so many moving components, reciprocating engines have relatively larger frictional losses [3], [4].

Pumping Loss

For four-stroke engines, the intake and exhaust processes need a significant amount of energy. The net power applied to the working medium (gases) by the engine (piston) during the intake and exhaust strokes is known as the pumping loss. As the incoming new mixture is utilised to scavenge the exhaust gases in two-stroke engines, this is minimal.

Power loss for the components that charge and scavenge

The intake charge is given at a greater pressure in certain four-stroke engines than in normally aspirated ones. A mechanically driven compressor or a compressor powered by a turbine are both utilized for this. The engine is referred to as a supercharged or turbocharged engine as a result. While the turbine in a turbocharged engine is propelled by the exhaust gases of the engine, in the case of a supercharged engine, the compressor is powered by the engine itself. These gadgets reduce the engine's power in some manner. This loss is seen as a frictional loss that is negative. When two-stroke engines include a scavenging pump, the engine provides the power to operate the pump.

Power loss for various auxiliary components

A significant portion of the power output created is used to operate auxiliary devices such the water pump, cooling fan, fuel pump, lubricating oil pump, generator, etc. This is seen as a loss since the existence of each of these elements lowers the engine's net output [5].

DISCUSSION

Mechanical Friction

As was previously indicated, because of their relative motion, friction loss is a factor in the bearing surfaces of engine components. The six kinds of mechanical friction in an engine are addressed in the following sections.

(i) Fluid-film or Hydrodynamic Friction

The phenomenon known as hydrodynamic friction occurs when there is a full layer of lubricant between the two bearing surfaces. In this instance, the friction force is entirely dependent on the viscosity of the lubricant. The primary mechanical friction loss in the engine is caused by this kind of friction.

(ii) Partial-film Friction

There is contact between the rubbing surfaces in certain areas when rubbing (metal) surfaces are insufficiently lubricated. The only metallic contact that occurs during normal engine operation is between the compression (upper) piston ring and the cylinder walls. The piston velocity is almost zero towards the conclusion of each stroke, which is mostly when this occurs. The journal bearings function in partial-film friction while the engine is beginning. Due to its little contribution to overall engine friction, partial-film friction may be disregarded.

(iii) Rolling Friction

Because the two surfaces are rolling against one another, there is rolling friction. Rolling friction is experienced by tappet rollers, ball and roller bearings, and others. This kind of bearings have a coefficient of friction that is essentially unaffected by load and speed. Localized rubbing from distortion under load and roller climbing are also contributing factors to this friction. When an engine is starting and operating for the first time, rolling friction is lower than journal bearing friction. The oil's high viscosity and partial friction in journal bearings during engine starting, which employs plain journal bearings on the crankshaft, are the causes. Comparing rolling friction to overall friction, it is little.

(iv) Dry Friction

Even when an engine is not operated for a long time there is little possibility for direct metal to metal contact. Always some lubricant exists between the rubbing surfaces

even after long periods of disuse. One can take the dry friction to be non-existent and hence, this can be safely neglected while considering engine friction.

(v) Journal Bearing Friction

A journal, or circular cylindrical shaft, revolves against a bearing, or cylindrical surface. When the bearing surface is smaller than the whole circle, journal bearings are referred to be partial. The rotary motion might be oscillatory or continuous. To determine the performance of journal bearings under different operating situations, a great deal of theoretical and experimental study has been done. Engine journal bearings are subject to loads that change over time in terms of intensity and direction. Engine journal bearings, however, are subject to the same fundamental relationships found for regular journal bearings, even if the coefficients of friction are often different [6], [7].

(vi) Friction due to Piston Motion

There are many types of friction caused by a moving piston:

1. viscous friction due to piston
2. non-viscous friction due to piston ring

The non-viscous piston ring friction can be further subdivided into

1. friction due to ring tension
2. friction due to gas pressure behind the ring

Test results reveal that there is often not enough lubricant present to completely cover the area between the piston and the cylinder wall. The piston side-thrust and the accompanying vibrations have an impact on the oil film thickness between the piston and the cylinder as well. With load and speed, the average oil film thickness between the piston and cylinder wall fluctuates. The viscosity of the oil and the temperature at different places on the piston are other factors that affect piston friction. Compression rings and oil rings are the two types of piston rings. To seal against gas pressure, compression rings are located on the upper section of the piston. The compression ring's pressure on the cylinder wall is caused in part by the ring's elasticity and a little amount by gas pressure that seeps into the area between the ring and the piston. The gas pressure behind the top ring (compression ring) is less than the cylinder pressure in the second ring groove and much less in the third ring groove. It is about equal to the cylinder pressure [8].

Oil rings are made with radial passages that enable oil to scrape off portion of the cylinder wall and return to the oil sump. Because holes were bored into the inside of the piston to serve as vents, there is no gas pressure below the grooves for the rings. In this instance, the elasticity (tension) of the rings is the only factor contributing to the pressure of the ring surface on the cylinder wall. Due to their spring action, piston rings constantly push against the cylinder walls. As a result, there is constantly friction caused by piston action.

Pumping Loss

Pumping loss refers to the effort required to fill the cylinder with a new mixture during the suction stroke and to expel the combustion byproducts during the exhaust stroke. By expanding the valve regions, this pumping loss may be decreased. Due to the practical restriction on the amount of space in the cylinder head, this area cannot be significantly extended. Additionally, engine speed affects the pumping loss. According to the indication diagram in Fig.1, the pumping loss for four-stroke engines may be separated into three components.

Exhaust Blowdown Loss

The exhaust valve is designed to open before the piston reaches BDC on its expansion stroke in order to lessen the effort required of the piston to force out the exhaust gases. Due to the pressure differential, combustion gases rush out of the cylinder at this time. As a result, there is a certain loss of power, which is represented by the region in Fig.1. The blowdown defeat is the name of this loss. The timing and size of the exhaust valve play a major role in this. The blowdown loss will be larger with bigger valve areas and early exhaust valve opening, but it tends to be lower with increased speed.

Exhaust Stroke Loss

Exhaust loss is the effort needed to drive the combustion products out of the cylinder after blowdown. The gas pressure within the cylinder is more than three to four times greater than the gas pressure inside the exhaust pipe at the beginning of the blowdown process, which causes the gases to exit at a rapid rate. The high velocity of exhaust gases often continues even during the exhaust stroke as a result of inertia, and the cylinder pressure may briefly fall below the gas pressure in the exhaust pipe. Gases are forced into the exhaust pipe as the piston travels up once again due to increased pressure. As a result, the power needed to push the exhaust gases out is known as the exhaust loss. A schematic of an indicator displays this loss. This loss is influenced by the valve's size, timing, and flow coefficient. A larger valve, an earlier valve opening, and a greater valve flow coefficient could all work together to lessen exhaust stroke loss. The exhaust stroke loss rises as speed rises. Therefore, features that tend to increase exhaust stroke loss, such as high speed, late exhaust valve opening, and smaller exhaust valve size, may also tend to lower blowdown loss. For the engine to operate better, the sum of the two losses should be as low as possible.

Intake Stroke Loss

A pressure differential across the intake valve is created by the piston receiving energy, allowing new charge to be pulled into the cylinder. The power used by the piston to do this is known as the intake stroke loss. This is done to overcome the friction and inertia of the gas in the intake system. The pumping loss is the result of the combined intake and exhaust stroke losses.

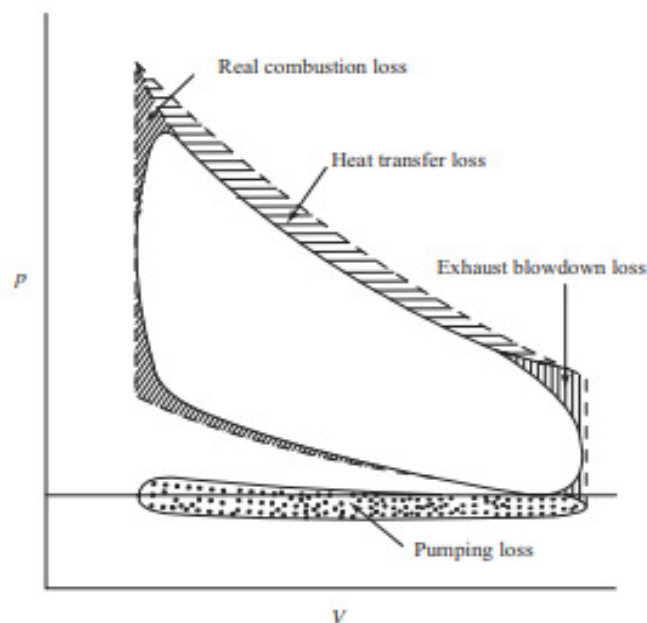


Figure 1: Pumping and Blowdown Losses (Quora)

Factors Affecting Mechanical Friction

Engine friction is influenced by several things. This section discusses how some of these variables affect mechanical friction.

Engine Design

The following design variables affect friction losses:

Stroke bore ratio

Lower stroke-bore ratios have the potential to somewhat reduce the fmep. It is primarily caused by smaller frictional area when the stroke to bore ratio is lower.

Effect of Engine Size

More frictional surfaces are present in larger engines. As a result, their engines need additional lubrication.

Piston rings

Friction may be decreased by lowering the number of piston rings and the ring's contact area with the cylinder wall. Additionally, low ring pressure lessens friction.

Compression ratio

As the compression ratio rises, the friction mean effective pressure also rises. But due to the rise in imep, the mechanical efficiency is either same or may even significantly improve.

Journal bearings

The fmep is decreased in journal bearings by reducing the journal diameter/diametrical clearance ratio. Inertia loads will be kept to a minimum by short pistons with less mass at the gudgeon pin axis. This will thus lower the friction loss.

Engine Speed

With increased speed, friction rises sharply. Mechanical efficiency begins to decline significantly at greater speeds. This is one of the justifications for limiting engine speeds.

Engine Load

As the load grows, the cylinder's maximum pressure rises as well, slightly increasing the friction values. Additionally, when the load rises, the temperature within the cylinder and the temperature of the lubricating fluid both rise. Higher temperature causes the oil viscosity to drop, which significantly lessens friction. As the throttle is opened wider to deliver more fuel, the throttling losses in petrol engines decrease. The aforementioned effects may contribute to reduce frictional losses in such engines (SI engines). Diesel engines don't experience throttling, hence the frictional losses caused by engine load are more or less constant.

Cooling Water Temperature

By lowering oil viscosity, an increase in cooling water temperature marginally lowers engine friction. Due to the low temperature and high viscosity of the water and oil at beginning, friction losses are considerable.

Oil Viscosity

Friction loss and viscosity are (directly) proportional to one another. The viscosity may be decreased by raising the oil's temperature. However, if the oil temperature rises over a

specific point, the local oil film may collapse, causing partial fluid film friction or even metal to metal contact, which is very detrimental to the engine.

Lubrication

It is clear from the explanation above that lubrication is necessary to reduce friction and wear between the engine's components. The specifics of engine lubrication are explained in the sections that follow.

Function of Lubrication

A lubricant (oil, grease, etc.) is admitted between two surfaces that are in touch and moving relative to one another via the practise of lubrication. One or more of the following tasks are what lubrication in an engine is intended to do.

- (i) To cut down on wear and friction between moving components, which will cut down on energy waste and lengthen engine life.
- (ii) To operate as a sealing agent, for instance, the lubricating oil aids the piston rings in maintaining a strong seal to prevent the high pressure gases in the cylinder from seeping out into the crankcase.
- (iii) To reduce surface temperatures by removing heat from engine parts.
- (iv) To remove carbon and metal wear-related particles from the surfaces.

The first function is regarded as being the most significant of all these functions. The high temperatures experienced during the combustion process and the wide range of temperatures encountered during the cycle make lubrication issues in internal combustion engines more challenging. As previously mentioned, adequate lubrication can reduce the amount of energy lost due to friction between different engine parts.

Additionally, the gas force on the piston and the inertia force of the moving parts are susceptible to change, which makes it extremely challenging to lubricate effectively under all operating situations. To reduce starting issues when the flow of lubricating oil is challenging, the temperature extremes are highlighted. As a result, there could be metal-to-metal contact due to a lack of lubricating oil supply for the moving parts. In this chapter, the fundamental issues related to the appropriate lubrication of the various types of bearings found in internal combustion engines will be generally covered. Additionally, the characteristics of lubricating oils, how engine operation affects these characteristics, and various lubricating system types will be covered [9], [10].

Lubrication of Engine Components

Numerous surfaces in a reciprocating engine come into touch with one another; as a result, lubrication is necessary to lessen friction.

Piston

Due to the high temperatures that the piston and piston rings are exposed to, the oil that is supplied to the cylinder walls must provide enough lubrication under difficult circumstances. In order to reduce the amount of combustion gases that pass the piston rings and enter the crankcase, the lubricant on the cylinder walls must also serve as a seal.

Automobile high-speed engine pistons often don't require any extra lubrication. The crank webs and connecting rods receive splashes of oil from the main bearings and crankpin, and the pistons receive enough lubrication from the oil mist that the fast-moving components

create inside the enclosed crankcase. These conditions remain regardless of whether the oil level in the crankcase is high enough for the connecting rods to pick it up, as they do even when it is low due to the continual ignition that occurs in the sump of many vehicles.

Positive feed mechanical oilers are used to lube the pistons of low- and medium-speed engines. A tiny plunger on a positive feed mechanical oiler releases oil onto the cylinder surface as it advances. A check valve stops the oil from being sucked back in during the return stroke, and oil is instead brought in through a different route.

According to tests, the piston rings and cylinders spend a significant amount of time in the thin film lubrication region. An excessive amount of oil would be consumed with thick-film lubrication. Oiliness and surface quality will be key factors in decreasing wear and scuffing in the thin film zone. A smooth surface has flat areas with numerous tiny indentations positioned in between them. These imperfections serve as both individual oil reservoirs and aid in keeping the oil in a film on the surface. By including sulphur and chlorine chemicals in the lubricant, surface degradation may be further diminished. These substances transform the material into sulphides and chlorides while preventing stressed regions from becoming weldable. With the aid of an oil gallery, which is present in high speed engines with higher specific power output, lubricant helps to dissipate heat more effectively.

Crankshaft Bearing

The crankshaft's primary bearings are manufactured with ring oilers in tiny stationary engines. The only component of a ring oiler is a ring that is slid over the shaft and runs over the journal. Because the ring's diameter is greater than that of the journal, it can slide across the shaft while it is revolving. The ring will incline to revolve due to the shaft's rotation. The oil is drawn from the reservoir by the spinning bottom portion of the ring, which then transports it to the journal's top where it can flow into the bearing oil grooves and bearing clearance gap and cover the whole bearing surface.

Crankpin Bearings

Small engines have minimal bearing pressure, and the crankpins may occasionally be lubricated by oil splashing. For this reason, the connecting rod's bottom is attached with a dipper that, as it descends, dips into the oil.

The centrifugal banjo oiler is used with various two-stroke vertical engines and small horizontal engines. Before the dead center, the oil hole going to the crankpin's surface is frequently drilled at an angle of around 30 degrees, allowing oil to enter the upper shell at a relatively low pressure point before to ignition. A bored hole in a heavy duty engine allows oil to be delivered from the main bearing to the crankpin.

Wristpin Bearing

A sight feed oiler lubricates the wristpin in tiny horizontal engines. The configuration and a unique scraper scoop is offered to scrape the oil flowing from the cylinder walls. In some vertical engines, the wristpin is lubricated by extra oil from the crankpin bearing, which is sent under pressure to the main and crankpin bearings. The connecting rod will have a hole punched through it for this reason.

Properties of Lubricants

In an engine, lubricant has a wide range of responsibilities. The lubricant is required to restrict and regulate the following

- (i) Friction between the components and metal to metal contact

- (ii) Overheating of the components
- (iii) Wear and corrosion of the components

The lubricant needs the following qualities to carry out the aforementioned tasks:

- (i) suitable viscosity
- (ii) When lubrication is applied at the boundary zone, oiliness helps to reduce friction and wear and serves as a covering against corrosion.
- (iii) High strength to avoid metal-to-metal contact and seizure while under excessive load.
- (iv) Must not interact with the lubricating surfaces.
- (v) A low pour point to enable lubricant flow to the oil pump at low temperatures
- (vi) No tendency to deposit when reacting with fuel, water, air, or combustion products
- (vii) Cleaning prowess, nontoxic, non-flammable, and non-foaming qualities
- (viii) Low cost.

CONCLUSION

Engine friction is the resistance that is experienced between moving components, which leads to energy loss and poor efficiency. As opposed to friction and wear, lubrication entails the use of oils or lubricants to prolong the lifespan and ensure smooth operation of engine components. In order to further minimize frictional losses, improve fuel efficiency, and lower emissions in engines, improved lubricants and friction-reducing technologies, like nanotechnology-based coatings and additives, are on the horizon. These developments may lead to more effective and environmentally friendly transportation systems. Positive feed lubricators are used to lube the main bearings of big engines that have an open or at least not overly constrained crankcase, as is the case with most horizontal engines. Engines with enclosed crankcases, such as those that are the most vertical, are typically constructed with pressure lubrication. A gear pump draws the oil, delivering it to the oil gallery in the crankcase, where it is distributed to the bearings. A scavenging pump collects the oil that leaks from the bearing and deposits it in an oil sump. It is then pumped through a filter to an oil cooler and a supply tank, where it flows back to the pressure pump after cooling. Large volumes of oil are typically circulated since they not only operate as a lubricant but also as a cooling medium for the bearings.

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CHAPTER 13

HEAT REJECTION AND COOLING SYSTEM

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ABSTRACT

There are many different sectors and applications where heat rejection and cooling are crucial operations. Cooling is the reduction of temperature within a system or item, whereas heat rejection is the transfer of excess heat from a system to the surrounding environment. For maintaining ideal operating conditions, minimizing overheating, and assuring the durability and performance of equipment, efficient heat rejection and cooling mechanisms are essential. This abstract examines the ideas behind heat rejection and cooling and emphasizes their usefulness in a variety of industries. The temperature of the cylinder gas increases significantly during combustion. The walls of the combustion chamber receive a lot of heat from the fire. Therefore, it is essential to provide adequate cooling, particularly to the combustion chamber's walls. The prevailing high temperatures may also cause chemical and physical changes in the lubricating oil. This results in cylinder wall scoring, piston seizing, and wear and sticking of the piston rings. Therefore, a rise in the operating temperature of the piston head will be brought on by excessive cylinder-wall temperatures. This in turn will have a significant impact on the piston's strength.

KEYWORDS

Cooling, Equipment Performance, Heat Rejection, Temperature Reduction, Thermal Management.

INTRODUCTION

The percentage of chemical energy in fuel that can be converted into mechanical energy by internal combustion engines is optimal between 25 and 35 percent. The cooling medium loses about 35% of the heat produced, with the remaining heat being released through exhaust and lubricating oil. Additionally, an overheated cylinder head may cause pre-ignition due to an overheated spark plug electrode. The exhaust valve could structurally fail or get too hot to prevent pre-ignition. Pre-ignition can also raise cylinder head temperatures to the point where the engine fails or loses all power. In SI engines, auto-ignition will occur due to a high cylinder wall or cylinder head temperature since the final portion of the charge to burn will come into touch with the walls of the combustion area during the burning phase.

In light of the above, it is recommended that the inner surface temperature of the cylinder walls be maintained within a range that will guarantee proper clearances between components, encourage fuel vaporization, maintain the oil's optimal viscosity, and avoid the condensation of hazardous vapors. In order to continually remove the heat that is transported into the combustion chamber's walls, a cooling system is used. The cooling medium removes between 30 and 35 percent of the total heat produced by the fuel. 5% of the total heat provided is lost via radiation and heat transported away by lubricating oil. Engine seizure will occur if the engine is not properly cooled. The specifics of engine heat transport, heat rejection, and cooling are taken into account in this chapter.

Cooling and heat rejection are essential processes used in many different sectors, technical systems, and daily life. These procedures are essential for preserving ideal working conditions, avoiding overheating, and guaranteeing the durability and effectiveness of equipment. Cooling is the deliberate lowering of temperature inside a system, while heat rejection is the transfer of extra heat from a system or item to the environment.

In many different applications, such as power generation, industrial procedures, HVAC (Heating, Ventilation, and Air Conditioning) systems, electronic gadgets, cars, and many others, efficient heat rejection and cooling mechanisms are crucial. The entire performance, dependability, and safety of these systems are significantly influenced by the systems' capacity to manage and regulate temperature.

The fact that too much heat may harm machinery and processes is one of the main justifications for the significance of heat rejection. Inadequate heat removal may cause overheating, component breakdowns, decreased efficiency, and even system shutdowns. In order to maintain ideal temperatures and guarantee the efficient functioning of equipment, good heat rejection is crucial [1], [2].

The purposeful practice of lowering temperature inside a system or item to create desired conditions is known as cooling. Different techniques, including conduction, convection, radiation, and phase change, may be used to cool an object. Various cooling media, such as air, water, refrigerants, or specialized cooling fluids, are used in these techniques. Since many systems and processes produce heat as a byproduct, cooling is necessary. For instance, while they are operating, power plants, industrial equipment, and electronic gadgets all produce significant quantities of heat. To get rid of this extra heat and keep temperatures within ideal and safe ranges, cooling is required.

Heat rejection and cooling processes depend heavily on thermal management. It entails the planning, execution, and management of systems that regulate heat transport and maintain target temperature ranges. To improve heat dissipation and efficiently regulate temperature, efficient thermal management methods make use of heat sinks, heat exchangers, fans, pumps, insulation, and control systems. Different systems now function and operate more efficiently because to improvements in heat rejection and cooling technology. Innovative cooling methods and materials, including enhanced heat transfer fluids, phase-change materials, microfluidics, and thermal management algorithms, are still being investigated by researchers and engineers. With these advancements, system performance will be improved overall while energy usage will be decreased.

In conclusion, heat rejection and cooling are crucial procedures in a variety of applications, including manufacturing, electronics, and power production. Effective heat removal from systems prevents overheating and maintains ideal operating conditions by ensuring that excess heat is adequately removed from them. On the other hand, cooling is the purposeful lowering of temperature to bring about desirable circumstances. Improved equipment performance, energy efficiency, and system dependability are results of efficient thermal management strategies and continuous development in cooling technology.

Parameters Affecting Engine Heat Transfer

It should be clear from the explanation above that a variety of factors affect how much heat the engine transfers. The design of an effective cooling system would be challenging without knowledge of the impact of these elements. This section discusses briefly how different factors affect engine heat transmission.

Fuel-Air Ratio

The temperature of the cylinder gases and the flame speed will alter in response to a change in the fuel-to-air ratio. At an equivalency ratio of around 1.12, or at a fuel-to-air ratio of approximately 0.075, the maximum gas temperature will occur. T will be at its highest at this fuel-to-air ratio. However, based on experimental findings, it is discovered that a combination that is a little leaner than this ratio would exhibit the highest heat rejection.

Compression Ratio

The temperature of the gas will only slightly rise at the top dead centre as the compression ratio rises, while towards the bottom dead centre, where a big cylinder wall is exposed, the temperature of the gas will significantly fall due to increased gas expansion. Due to increased expansion, the exhaust gas temperature will also be substantially lower, which will result in less heat being rejected during blowdown. In general, heat rejection tends to slightly decrease as compression ratio rises.

Spark Advance

Increased heat rejection to the cooling system will be the outcome of a spark advance that is both more than the ideal value and less than the ideal value. This is primarily because spark timing that is different from the MBT value (Minimum Spark Advance for Best Torque) would result in less power production and higher heat rejection.

Pre-ignition and Knocking

Pre-ignition has the same result as advancing the time of the ignition. Large spark advance might result in erratically knocking and running. Although knocking significantly alters the local heat transfer conditions, it seems to have little overall impact on heat transmission. Regarding the impact of pre-ignition and knocking on engine heat transmission, no quantitative data is provided.

Engine Output

Less heat will be rejected by engines built for high mean effective pressures or fast piston speeds. In bigger engines, less heat will be wasted for a given stated power.

Cylinder Wall Temperature

In contrast to the cylinder wall temperature, the average temperature of the cylinder gas is much greater. As a result, any little variation in the temperature of the cylinder gas will have very little impact on the temperature differential and, thus, on heat rejection.

DISCUSSION

Power Required to Cool the Engine

If there is no cooling system, theoretically the engine's thermal efficiency will increase, but in practise the engine won't run. The major cause of engine seizure is excessive temperature, which causes metals to lose their properties and pistons to expand significantly. Cooling the engine is essential to preventing seizure. However, the engine should provide the necessary power to operate the cooling system. This section briefly discusses the amount of power needed to operate an air-cooled engine's cooling system [1]–[3]. Some presumptions must be made in order to streamline the calculations and arrive at an expression for the power needed to cool an engine. The crucial ones are:

- (i) Engine operation is predicated on a steady fuel-to-air ratio.

- (ii) The temperature differential between the gases in a cylinder and the cylinder wall, ΔT , is believed to remain constant since changes in fin or coolant temperature are thought to be small enough.
- (iii) It is thought that the density and temperature of cooling medium are unaffected by how it flows through the fins of an engine or radiator.

Need For Cooling System

It should be clear from the discussion of heat rejection in the preceding sections that high temperatures are generated in the engine's cylinders as a consequence of combustion during the process of converting thermal energy to mechanical energy. The cylinder head, walls, piston, and valves all get a significant amount of heat from the combustion gases. The engine will sustain harm if this extra heat is not removed and these components are not sufficiently cooled. In addition to keeping the temperature of these components within predetermined ranges to ensure optimal engine performance, an engine must include a cooling system to avoid damage to its critical components. So, an essential necessity of reciprocating internal combustion engines is adequate cooling. A cooling system is thus required to prevent the engine from overheating and yet allow it to operate hot enough to maintain the engine's optimal efficiency. In other words, the cooling system's job is to prevent the engine from becoming too hot while also preventing it from getting too cold [3].

Characteristics of an Efficient Cooling System

The following are the two main characteristics desired of an efficient cooling system:

Under all engine operating circumstances, it has to be able to remove roughly 30% of the heat produced in the combustion chamber while keeping the engine's ideal temperature. If the engine is heated, it ought to dissipate heat more quickly. However, cooling should be kept to a minimum when the engine is starting, allowing the functioning components to achieve their operational temperatures quickly.

Types of Cooling Systems

A cooling medium is needed to cool the engine. This might either be liquid or air. Accordingly, two different kinds of cooling systems are often used for IC engines. Those are

- (i) Liquid Or Indirect Cooling System
- (ii) Air Or Direct Cooling System

Liquid Cooled Systems

In this method, the majority of the heat is extracted by water that is forced to flow via jackets that are placed around the cylinder, cylinder head, valve ports, and seats. Figure 1 is a diagrammatic illustration of a water jacket, which is a water circulating channel. It consists of a long, flat tube with a thin wall and an entrance towards the exit of the water pump, as well as several smaller apertures spaced along its length to send water towards the exhaust valves. The tube may be taken out of the block's front end and fits into the water jacket [4].

Convection and conduction are used to transfer heat from the cylinder walls and other components. The liquid is heated as it moves through the jackets and is then cooled by a radiator system that uses air cooling. In turn, the heat from the liquid is transmitted to the air. Thus, the indirect cooling system is what it is called.

Any one of the following five techniques may be used for water cooling:

- (i) Direct or non-return system

- (ii) Thermo syphon system
- (iii) Forced circulation cooling system
- (iv) Evaporative cooling system
- (v) Pressure cooling system

Direct or Non-return System

Large installations with ample of water access may benefit from this technology. Through an entrance valve, water from a storage tank is immediately delivered to the engine cooling water jacket. The hot water is just discarded rather than being cooled for later use.

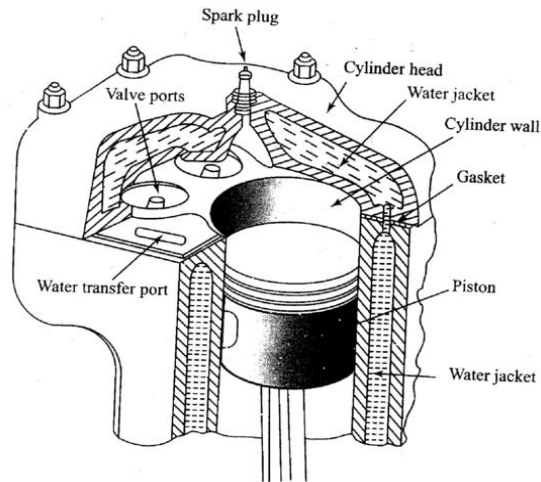


Figure 1: Liquid Cooled System (slideplayer.com)

Thermo syphon System

With reference to Fig. 2, the thermosyphon's fundamental concept may be described. The fluid in tank A receives heating. The heated fluid ascends due to the relatively lower density, and is replaced by the relatively cold fluid from tank B through pipe p2 as it passes.

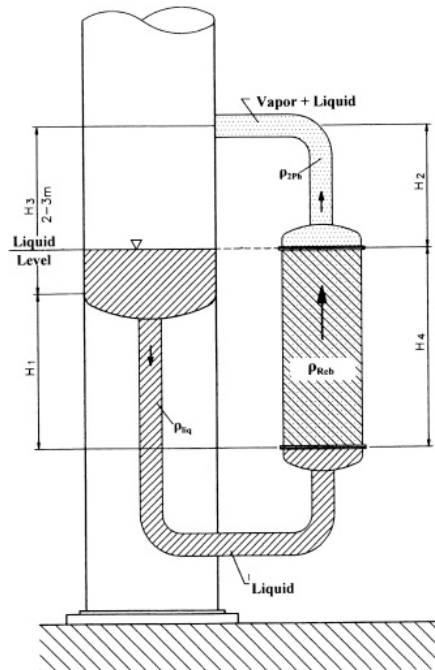


Figure 2: Principle of Thermosyphon System (sciencedirect.com)

Through the pipe p1, the heated liquid travels to tank B, where it cools. As a result, convection currents carry the fluid through the system. In an engine application, tank A stands in for the cylinder jackets, tank B for the radiator, and the circulating fluid is water. The water jackets are situated below the radiator to guarantee that the cylinder jackets always have access to the coldest water. The system's simplicity and automated cooling water circulation are its key benefits. The system's biggest drawback is its inability to provide the high water flow rates needed, especially for powerful engines.

Forced Circulation Cooling System

Numerous vehicles, including light trucks, buses, and even cars, employ this technology. Here, convection with a pump's help moves water from radiators to water jackets. An explanation of this system's fundamental workings is provided by the block diagram in Fig. 3. The engine drives a centrifugal pump that circulates water or coolant via jackets around the engine sections that need to be cooled. The air is pulled through the radiator by a fan and by the air draught created by the vehicle's forward motion, which cools the water as it passes through. To regulate the water temperature needed for cooling, a thermostat is utilised. The radiator, fan, water pump, and thermostat are the four primary parts of this system. Fig. 4 depicts these components' specifics in depth [5].

Radiator

A radiator's main function is to provide a sizable quantity of cooling surface area in order to effectively cool the water that is sent downward through it in thin streams. There are a variety of different configurations to do this.

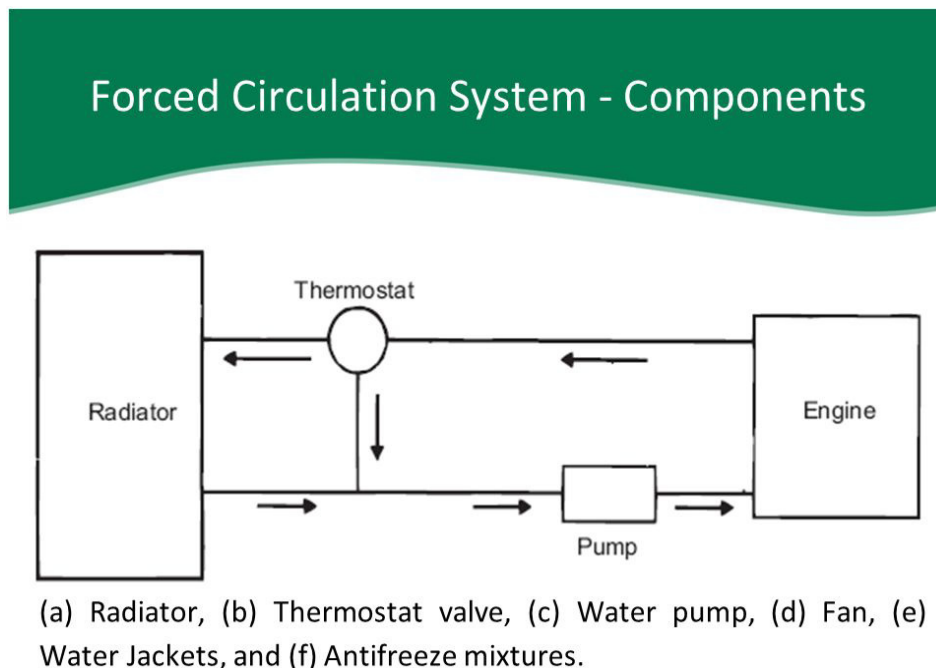


Fig. 3 Principle of Forced Circulation cooling system using the thermostat (slideplayer.com)

A lower tank and an upper tank (header tank) make up the majority of the radiator. To prevent dust from entering the radiator when filling it with water, certain designs may include a detachable filter mesh in the top tank. The radiating core is located in the space between the two tanks. Rubber hoses link the higher tank to the engine jacket's water outputs, while another rubber hose connects the lower tank to the engine jacket's input at the pump. Tubular

or cellular radiator cores are the two different types. A tubular radiator is made of several brass elliptical or circular tubes that have been squeezed into numerous appropriate brass fins that have been punched out. The tubes are staggered and have fins to prevent corrosion. The primary drawback is how difficult it is to repair any broken tubes. However, the system's starting cost is rather low. Soft water should be utilised for cooling. If hard water is utilised, silt buildup on water jackets and tubes serves as an insulator and impairs cooling. For every 10 litres of water, 30 g of sodium bichromate should be added if soft water is not available.

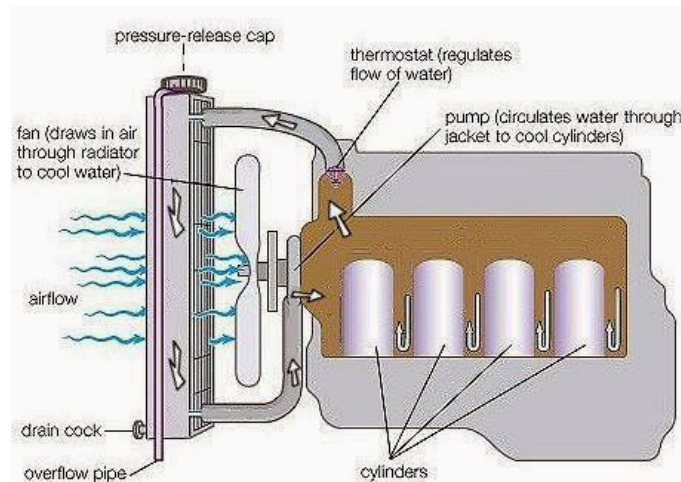


Fig. 4 cooling of an automobile (mevision.blogspot.com)

Fan

Air is drawn into the intervals between the radiator tubes by a fan installed on the impeller spindle, as illustrated in Fig. 4. This significantly lowers the water's temperature. The fan is powered by a suitable belt and pulley system [6], [7].

Pump

The pump keeps the water flowing through the system at a constant rate. The suction side of the pump is linked to the bottom of the radiator. A pulley positioned on the end of the camshaft or crankshaft transmits the power to the pump spindle. A centrifugal pump installed in this system ensures a constant flow of water under all circumstances (Fig. 4). This guarantees a good rate of water circulation. Therefore, using less water and a smaller radiator would be appropriate. Conveniently located on the engine, a pump is powered by the crankshaft through a fan belt. On the drive shaft, movable packing glands are available to stop water leaks. Using grease with a high melting point, bearings are lubricated. In certain circumstances, special bushes are utilised that don't need lubrication. For multi-cylinder engines, a header is often used to distribute water evenly throughout all of the cylinders. The exhaust valve seats and other important engine components, such as the header, are surrounded by tubes or ducts that provide a high rate of flow. The majority of diesel and automobile spark-ignition engines use this method. Typically, 3 to 4 litres per minute per kilowatt are circulated [6], [8].

Some engines include a pump that pushes cold water from the radiator into the engine jacket. This pump is situated between the radiator's output and the engine block. However, in vehicles, this configuration would result in the pump being located so low that the fan could not be properly mounted on the pump shaft. This arrangement seems to have the drawback that, in the event of a water loss, circulation ceases as soon as the level reaches the bottom of the cylinder head jacket, as opposed to a pump in the supply line, which continues to operate for as long as there is still water in the system.

Thermostat

In order to reduce warm-up time when the engine is started cold, the coolant temperature must be raised to the required level. This may be accomplished by installing a thermostat in the system, which at first stops the passage of water through the radiator below a certain temperature so that the water is heated up rapidly. The thermostat opens the radiator's water valve when the predetermined temperature is attained. Typically, a Bellows type thermostat is used; further information may be found in Fig. 5. A wax-element type thermostat is often used in contemporary engines.

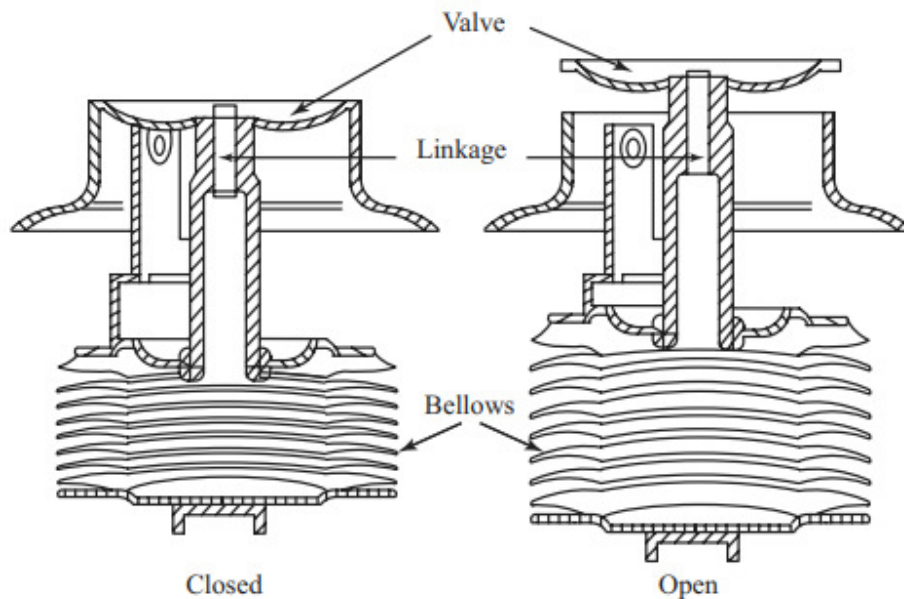


Figure 5: Bellows Type Thermostat (semanticscholar.org)

The device consists of a closed bellows holding a flammable liquid at a low pressure. The liquid vaporises when the bellows are heated, producing enough pressure to cause the bellows to expand. Bellows movement activates a connection, which opens the valve. The bellows contracts to seal the valve when the unit cools, causing the gas to condense, the pressure to drop, and the gas to condense.

Evaporative Cooling System

Moderate pressures, say up to 2 bar, are often utilised in pressure cooling systems. A cap is equipped with two valves, a safety valve that is loaded by a compression spring and a vacuum valve, as shown. Both valves are closed when the coolant is cold, but as the engine warms up, the coolant temperature rises until it reaches a preset value that corresponds to the desired pressure, at which point the safety valve opens; however, if the coolant temperature drops while the engine is running, the valve will close again until the temperature rises once more to the equivalent pressure value [6], [7]. When the engine is turned off and the coolant cools, a vacuum form in the cooling system. However, when the internal pressure goes below atmospheric, the greater outside pressure opens the vacuum valve, bringing the cooling system's pressure up to atmospheric.

Evaporative cooling using a water-cooled condenser is shown in Figure 7. In this instance, the heat exchanger C, which is cooled by a secondary water circuit, condenses the vapour that developed in the above tank B, and the water flows back to B via gravity. Pump A maintains circulation by delivering hot water from the engine to tank B while also circulating cooling water to the engine [9].

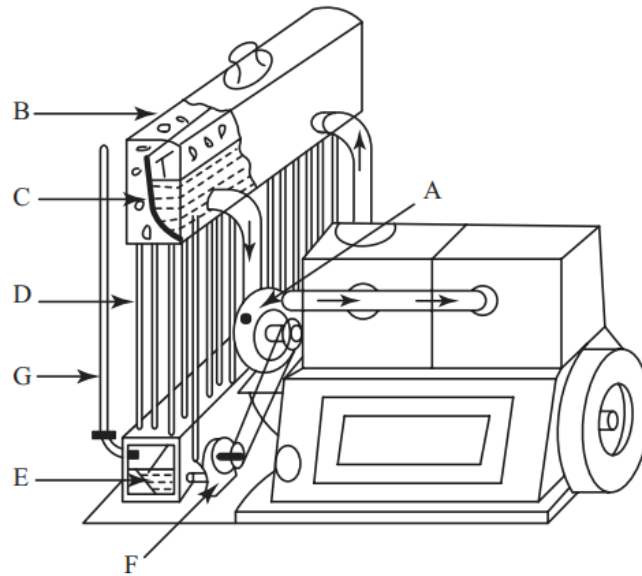


Figure 6: Evaporative Cooling with Air-Cooled Condenser (Slideshare.Net)

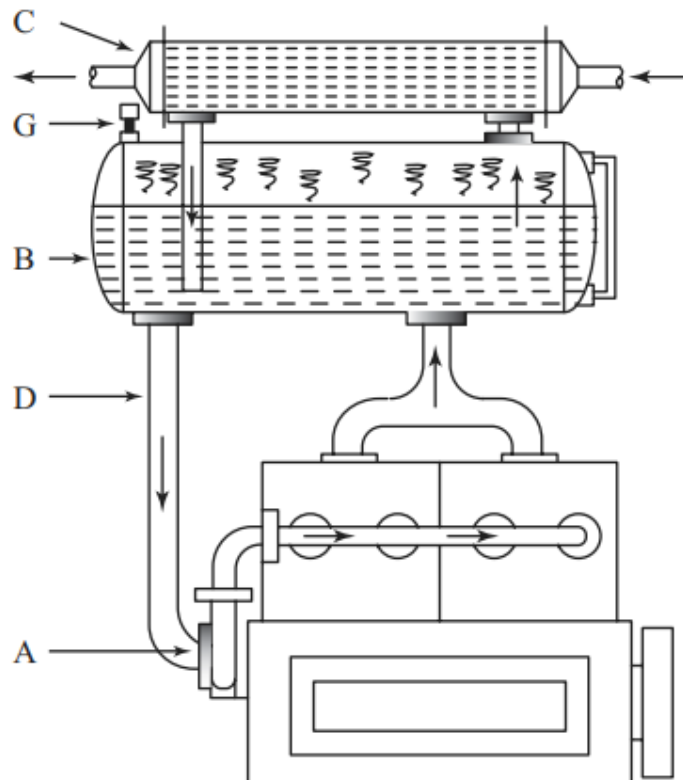


Figure 7: Evaporative cooling with water-cooled condenser (Slideshare.Net)

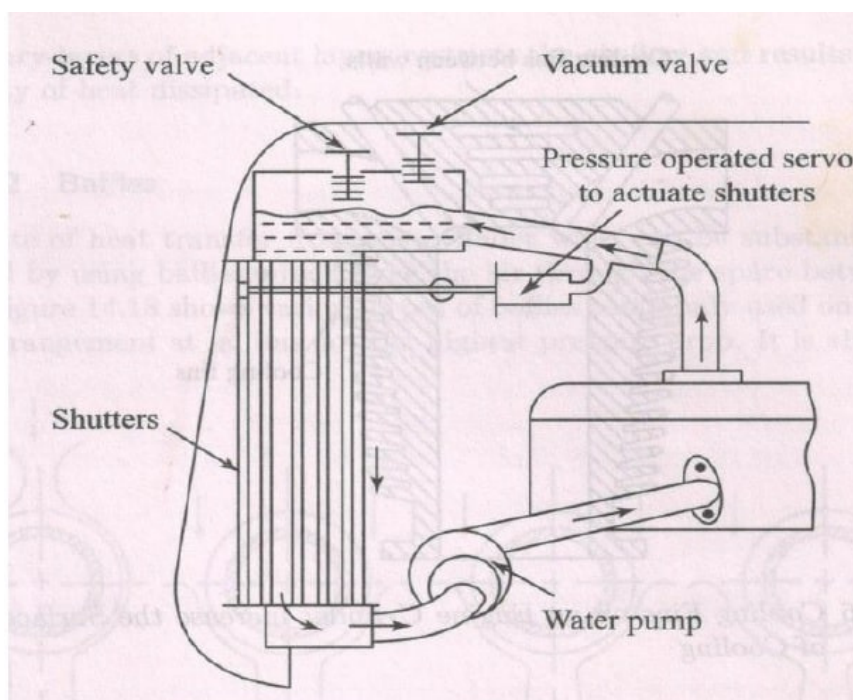
Pressure Cooling System

As was previously noted, the rate of heat transfer is influenced by the difference in temperatures between the two mediums, the surface area that is exposed, and the conductivity of the materials. In the case of radiators, it is suggested to seal the cooling system from the atmosphere and to permit a certain amount of pressure to build up in the system in order to take advantage of the fact that the temperature of the boiling point of water rises as the pressure increases. This will allow the radiator to be smaller. Table 1. Displays the boiling point of water at different pressures [8], [10].

Table 1: Boiling Point of Water at Various Pressures

Pressure (bar)	1.0	2.0	5.0	10.0
Temperature (°C)	100	121	153	180

The filler cap has a safety feature built in so that if it is attempted to be unscrewed while the system is under pressure, the first movement of the cap immediately releases the pressure, preventing the release of scalding steam or the cap from blowing off due to increased internal pressure [9].

**Figure 8: Pressure cooling (learnmech.com)**

CONCLUSION

This section has to be prepared very carefully as many readers go through this section and prepare a remark on the full paper. Typically, stationary engines employ this method. Because the water in the cylinder jackets will have evaporated into steam, the engine will be cooled. Here, the water is allowed to evaporate in the cylinder jackets in order to benefit from the high latent heat of vaporization of water. The temperature will be higher than the typical allowable temperature if the steam is produced at a pressure higher than atmospheric. Evaporative cooling using an air-cooled condenser is shown in Figure 6. In this instance, the pump A circulates water, and some of it boils off as it reaches the above tank B. There is partition C in the tank. The condensate flows into the lower tank E once the vapour rises over the partition C due to the condensing action of the radiator tubes D. The little pump F then picks it up and returns it to the tank B. To keep the tanks B and E from collapsing as the pressure within them decreases due to condensation, the vertical pipe G is connected to the atmosphere outside

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CHAPTER 14

EXPLORING THE ENGINE EMISSIONS AND THEIR CONTROL

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ABSTRACT

Regarding environmental pollution and climate change, engine emissions and their regulation have become crucial issues. Engine emissions are the pollutants that are released into the environment as a consequence of combustion in different kinds of engines. In order to reduce or completely eradicate dangerous pollutants including carbon dioxide (CO₂), nitrogen oxides (NO_x), and particulate matter (PM), and hydrocarbons (HC), engine emissions must be controlled. This abstract examines the idea of engine emissions, emphasises its environmental impact, and talks about how crucial emission control methods are in reducing the negative consequences on climate change and air quality. Fossil fuels, such as petrol, diesel, and coal, produce pollutants during burning that endanger human health and the environment. Carbon dioxide (CO₂), nitrogen oxides (NO_x), particulate matter (PM), hydrocarbons (HC), and other hazardous chemicals are the main pollutants released by engines. These emissions have been connected to a number of negative consequences, such as respiratory conditions, the development of smog, acid rain, and the escalation of climate change.

KEYWORDS

Combustion Processes, Emission Reduction Measures, Engine Emissions, Environmental Impact, Pollutant Control.

INTRODUCTION

Due to their considerable effects on air quality, human health, and climate change, engine emissions and their regulation have elevated to the top of the list of urgent worldwide problems. Engine emissions are the pollutants that are released into the environment as a consequence of combustion processes in a variety of engine types, such as those used in automobiles, power plants, and industrial machines. These emissions are made up of an intricate mix of gases, particulate matter, and other hazardous materials. Significant efforts have been undertaken to regulate and lower engine emissions in response to these concerns. Implementing different technologies and tactics to reduce or completely stop the discharge of dangerous pollutants into the atmosphere is part of the process of controlling engine emissions. These actions are intended to safeguard public health, enhance air quality, and lessen their negative effects on the environment.

The improvement of combustion processes is one of the primary strategies for controlling emissions. To cut emissions, engine makers have been creating cleaner and more effective combustion processes. Improved combustion chamber designs, lean-burn engines, and stratified charge engines are a few of them. Other methods include enhancing fuel-air mixes, using direct injection systems, and using better combustion chamber designs. These developments increase combustion efficiency, which lowers pollutant emissions. The creation and use of after-treatment systems is another essential component of emission control.

Devices including catalytic converters, diesel particulate filters, and selective catalytic reduction (SCR) systems are used in these systems. Prior to being discharged into the environment, these technologies treat exhaust gases to convert or trap pollutants and lessen their negative effects. Nitrogen oxide, particulate matter, and other dangerous gas emissions have all been successfully decreased using after-treatment systems.

Measures to reduce emissions concentrate on both technical developments and better fuel quality. Low-sulfur fuels and alternative fuels, such as electric power and biofuels, have the potential to cut emissions. While alternative fuels provide better burning qualities and lower greenhouse gas emissions, low-sulfur fuels assist reduce the release of sulphur dioxide (SO₂) emissions [1], [2]. Governments and international organizations have had a significant impact on the adoption of pollution control technology by enacting strict emission standards and regulations. These regulations create testing protocols to verify compliance and set limits on the permitted emissions from engines. Engine makers now priorities adhering to these regulations, which has sparked ongoing innovation and study in pollution control systems.

In conclusion, engine emissions and their regulation have drawn a lot of attention because of their negative effects on the environment, human health, and air quality. To lessen the discharge of dangerous pollutants, it is essential to design and execute emission control techniques, such as changes to combustion processes, after-treatment systems, and fuel quality. The adoption of greener technology is also greatly influenced by strict emission limits and regulations. We can improve air quality, slow down climate change, and protect people's health now and in the future by efficiently managing and lowering engine emissions. During the combustion process, internal combustion engines produce harmful pollutants. In this, both the SI and CI engines share equal accountability. The emissions that are released into the environment contaminate the atmosphere, which results in the following issues.

- (i) Global warming
- (ii) Acid rain
- (iii) Smog
- (iv) Odours
- (v) Espiratory and other health hazards

Non-stoichiometric combustion, nitrogen dissociation, and contaminants in the fuel and air are the main drivers of these emissions. Unburned hydrocarbons (HC), carbon oxides (CO_x), nitrogen oxides (NO_x), sulphur oxides (SO_x), and solid carbon particles are the pollutants that are of concern. Engineers and scientists strive to create engines and fuels that produce a negligible amount of hazardous emissions that may be released into the environment without causing significant damage to the ecosystem. With current technology, this is not feasible, thus it is crucial to treat exhaust gases after they exit the engine and reduce emissions within the cylinder. The employment of thermal or catalytic converters and particle traps constitute after-treatment in this scenario. Exhaust gas recirculation (EGR) and various fuel additives are being tested for in-cylinder reduction. Non-exhaust pollutants in addition to exhaust emissions also contribute. We shall examine the specifics of these emissions and their regulation in this chapter.

Air Pollution Due To IC Engines

Because there were so few IC engines in operation up until the middle of the 20th century, the pollution they produced was bearable. The sunshine helped to keep the surroundings reasonably clean at that time. Power plants, industries, and an ever-increasing number of cars started to pollute the air to an unacceptable level as the world's population increased. In the

Californian Los Angeles basin, air pollution as a concern was first identified in the late 1940s. The high density of people in the region and the local weather patterns were two factors that contributed to this. Smog was created when smoke and other pollutants from several industries and cars met with the frequent fog in this maritime region. Along with rising population density and vehicle density in the 1950s, the pollution issue became worse. At this point, it was understood that one of the main causes of the issue was the vehicle. In California, emission restrictions started to be implemented in the 1960s [3]–[5].

The remainder of the United States, as well as Europe and Japan, enacted emission regulations within the next ten years. During the 1970s and 1980s, emissions of HC, CO, and NO_x per vehicle were decreased by roughly 95% via the improvement of engines' fuel efficiency and the use of exhaust after-treatment. During the 1980s, lead, one of the main air pollutants, was phased out as a gasoline additive. The development of more fuel-efficient engines led to the typical automobile's 1990s fuel consumption being less than half that of its 1970 use. However, a significant rise in the number of cars during this period prevented a general drop in gasoline use. More carbon reductions will be far more difficult and expensive. Due to need, emission limits have been stricter as the world's population has increased. California often pioneers the strongest regulations, with the rest of the US and the globe following. Despite the fact that air pollution is a worldwide issue, certain areas of the globe still lack emission regulations or restrictions. However, a lot of nations, including India, have begun adhering to emission standards. The next section will provide a quick overview of the standards followed in India.

DISCUSSION

Emission Norms

The quantity of pollutants that may be emitted into the environment is limited by laws known as "emission norms." Norms concentrate on controlling the pollution that come from powered vehicles like cars. In addition, they can control pollution from factories, power plants, and diesel generators. The emissions of nitrogen oxides (NO_x), sulphur oxides, particulate matter (PM) or soot, carbon monoxide (CO), or volatile hydrocarbons are the pollutants that are often controlled. The Environmental Protection Agency (EPA) oversees emissions rules in the US. California has a particular exception that allows it to enact stricter automobile emissions regulations. The national or Californian norms may be adopted by other states. All new automobiles must adhere to the emissions regulations established by the European Union. All road vehicles, railroads, barges, and "nonroad mobile machinery" (including tractors) are now subject to standards. Seagoing vessels and aircraft are not subject to any norms.

Euro 4 will be implemented by the European Union on January 1, 2008, followed by Euro 5 on January 1, 2010, and Euro 6 on January 1, 2014. Due to the need for oil refineries to upgrade their facilities, these dates have been postponed for two years. The first emission restrictions in India were the idle emission limits, which went into force in 1989. These mass emission standards, which were increasingly increased during the 1990s, quickly took the place of these idle emission regulations for both petrol (1991) and diesel (1992) cars. India began implementing European fuel and pollution standards for four-wheeled light- and heavy-duty vehicles in the year 2000. Two- and three-wheeled vehicles are still subject to Indian pollution standards [6], [7].

Hydrocarbon Emission

Emission levels vary for a SI engine depending on the equivalence ratio. It is obvious that the air-fuel ratio has a significant role. Because there is insufficient oxygen in a fuel-rich combination for all the carbon to react, there are significant amounts of HC and CO in the

exhaust products. This is especially true when the air-fuel combination is intentionally made highly rich, which occurs during beginning. It is also somewhat true while accelerating quickly when under load. If the air-fuel ratio is too lean, worse combustion takes place, producing HC emissions once again. Total misfire is the extreme of poor combustion for a cycle. As the air-fuel ratio is reduced, this happens more often. Exhaust emissions are 1 gm/kg of gasoline consumed when one misfire occurs out of 1000 cycles.

The causes for hydrocarbon emissions from SI engine are:

- (i) Incomplete combustion
- (ii) Crevice volumes and flow in crevices
- (iii) Leakage past the exhaust valve
- (iv) Valve overlap
- (v) Deposits on walls
- (vi) Oil on combustion chamber walls

Hydrocarbon Emission from CI Engines

CI engines have only around one-fifth the HC emissions of a SI engine since they run with an overall fuel-lean equivalency ratio. Diesel fuel has greater boiling and condensing temperatures than petrol because its constituent parts typically have heavier molecular weights. Therefore, CI engines produce more soot. Some of the HC particles that are produced during combustion condense onto the surface of the solid carbon soot. As mixing and combustion continue, the majority of this gets burnt. Only a tiny portion of the initially generated carbon soot exits the cylinder. Along with the solid carbon particles themselves, the HC components that have collected on the surface of the carbon particles also contribute to the engine's HC emissions.

A CI engine typically has a combustion efficiency of about 98%. This indicates that just 2% of the HC fuel is being released. Some small areas in the combustion chamber will be too lean to adequately ignite due to the no homogeneity of the fuel-air combination. There could not be enough oxygen in other areas because they are too rich in fuel. There are several flame fronts present at once, with local locations ranging from very rich to extremely lean. Due to a shortage of oxygen, under mixing causes certain fuel particles in fuel-rich zones to never react. Fuel combustion is constrained in fuel lean zones, and some fuel is not consumed. Over mixing prevents complete combustion because some fuel particles will combine with already-burned petrol.

It's crucial that injectors be made such that there is no dribble when injection ceases. On the nozzle tip, however, a little quantity of liquid fuel will remain trapped. The size of this very tiny amount of fuel, known as the sac volume, depends on the nozzle design. Because it is surrounded by a fuel-rich zone and there is no pressure driving the liquid fuel into the cylinder after the injector nozzle shuts, this small amount of gasoline evaporates relatively slowly. Because some of this fuel does not completely evaporate until after combustion has ended, HC emissions are increased. For some of the same reasons why SI engines emit HC (such as wall deposit absorption, oil film absorption, crevice volume, etc.), CI engines also emit HC [8].

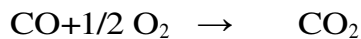
Emission Control Methods

It should be mentioned that, depending of the operating circumstances, the four-stroke cycle's combustion phase lasts only for roughly 25 to 50 ms. The partly burnt components of the cylinder gas mixture's exhaust gas continue to react after combustion is complete throughout the expansion stroke, exhaust blowdown, and the exhaust process. During this period, over 90

to 95% of the unburned HC reacts, either in the cylinder, close to the exhaust port, or in the upstream section of the exhaust manifold. Unwanted emissions are decreased when CO and tiny component hydrocarbons combine with oxygen to produce CO₂ and H₂O. These secondary reactions take place more often and reduce engine emissions as exhaust temperature rises. Stoichiometric air-fuel combustion, high engine speed, a delayed spark, and/or a low expansion ratio may all lead to an increase in exhaust temperature. As a result, various after-treatments with add-on devices are required to decrease emissions. The next parts go into the specifics.

Thermal Converters

If the temperature is high, secondary reactions take place considerably more quickly and fully. As a result, some engines use thermal converters as a way to reduce emissions. Exhaust gas passes through high-temperature chambers called thermal converters. They aid in the oxidation of the CO and HC that are still present in exhaust.



The temperature has to be maintained above 700 °C for this reaction to proceed at a meaningful pace. Now, think about the response.



To significantly diminish HC, the aforementioned reaction requires a temperature over 600 C for at least 50 milliseconds. A thermal converter must consequently operate at a high temperature in order to be effective. It should also be big enough to allow exhaust gases to stay there for a long enough period to encourage these secondary reactions. In essence, the majority of thermal converters are just an expanded exhaust manifold that is linked to the engine just outside the exhaust ports. This is required to reduce heat losses and prevent the exhaust gases from chilling to irreversible levels.

However, this poses two very critical issues for the engine compartment in cars. Space in the engine compartment of contemporary, low-profile, aerodynamic cars is very constrained, making it almost difficult to fit in a large, often insulated thermal converter chamber. Second, even with insulation, heat losses produce a major temperature issue in the engine compartment since the converter must run over 700 °C to be effective.

Some thermal converter systems include an air inlet that adds more oxygen for the CO and HC to react with. The system becomes more expensive, sophisticated, and large as a result. It is particularly important to provide air during rich operating circumstances like startup. It is vital to maintain the high temperatures via internal system reactions since engine exhaust is often at a lower temperature than required for a thermal converter to operate well. Maintaining the required operating temperature becomes more difficult when cooler outside air is added. Although oxidation may decrease HC and CO emissions, a thermal converter cannot lower NO_x emissions.

Catalytic Converters

The catalytic converter found on the majority of cars and other contemporary medium- or large-sized engines is the most efficient aftertreatment for lowering engine emissions. If the temperature is maintained between 600 and 700 °C, CO and HC may be oxidised to CO₂ and H₂O in thermal converters and exhaust systems. The temperature required to support these oxidation processes is decreased to 250–300 C in the presence of certain catalysts, making for a far more desirable system.

A catalyst is a material that reduces the energy required for a chemical reaction to occur, speeding up the process. As long as it is not damaged by heat, ageing, pollutants, or other reasons, the catalyst may continue to work indefinitely. The exhaust gases travel via catalytic converters, which are placed in the flow system. The catalytic material in these chambers encourages the oxidation of the pollutants present in the exhaust flow. Because they are employed to lower the levels of CO, HC, and NO_x in the exhaust, they are often referred to as three-way converters. Typically, it is a stainless steel canister installed anywhere along the engine's exhaust pipe. The exhaust gas flows through a porous ceramic structure that is within the container. The ceramic is often a single honeycomb structure with several flow openings in converters. Some converters transmit the gas between packed spheres of loosely granular ceramic. A converter's ceramic structure typically has a volume that is roughly half that of the engine's displacement. As a consequence, the converter receives a volumetric flow rate of 5 to 30 gas changes per second or more of exhaust gas. The solid soot in the exhaust streams necessitates bigger flow paths in catalytic converters for CI engines. Small embedded pieces of catalytic material on the ceramic passageways' surface support oxidation processes in the exhaust gas as it flows through. The most common kind of ceramic material used in catalytic converters is aluminium oxide (alumina). Alumina is chemically inert and can resist high temperatures [9]. It does not experience thermal degradation with time and has a very low thermal expansion. Platinum, palladium, and rhodium are the catalyst materials that are most often utilised.

Particulate Traps

Particulate traps are installed in the exhaust flow of compression ignition engine systems to minimise the number of particles emitted into the atmosphere. Traps are filter-like devices that are often built of metal wire mesh or ceramic in the shape of a monolith or mat. 60 to 90% of the particles in the exhaust flow are normally removed via traps. Traps steadily fill up with the soot particles as they are caught in them. This limits the flow of exhaust gases and increases the engine's back pressure. The engine runs hotter, the exhaust temperature rises, and the fuel consumption increases as a result of increased back pressure. Particulate traps are renewed as they start to saturate in order to lessen this flow limitation. The process of regeneration involves burning the surplus oxygen and particles in the exhaust of the lean-running CI engine. At typical operating temperatures, CI engine exhaust is between 150 and 350 °C, whereas carbon soot ignites at about 550 to 650 °C. The exhaust temperature increases as soot builds up in the particle trap and inhibits flow, but it is still insufficient to ignite the soot and renew the trap. Automatic flame igniters are used in certain systems, and they begin burning in the carbon when the pressure drop over the trap reaches a specific level. These igniters might be diesel-fueled flame nozzles or electric heaters. The temperature required to ignite carbon soot is lowered to a range between 350 and 450 °C if catalyst material is put in the traps. When the exhaust temperature increases due to increased back pressure, certain of these traps may replenish themselves spontaneously by self-igniting.

In other catalytic systems, flame igniters are used. Utilising catalyst additives in the diesel fuel is another method of reducing the carbon soot's ignition temperature and encouraging self-regeneration in traps. These additions often include copper compounds or iron compounds, and it's typical to use 6 to 8 grammes of additive per 1000 litres of gasoline. In a catalytic system, traps may be installed as near to the engine as feasible, even before the turbocharger, to maintain temperatures high enough for the system to self-regenerate.

The particle trap is changed as it approaches the filled position on certain bigger stationary engines, some construction equipment, and large vehicles. The removed trap is then

externally regenerated, and the carbon is removed using a furnace. The trap may then be utilised once again. When soot accumulation becomes excessive and regeneration is required, several techniques are applied. The most typical technique is to measure the pressure decrease that occurs when the exhaust flow passes through the trap. Regeneration begins when a certain pressure decrease, p , is attained. The regeneration controls must be set to account for the fact that pressure drop is also a function of exhaust flow rate. Transmitting radio frequency waves through the trap and measuring the percentage of absorption is another way to detect soot formation. Radio waves are absorbed by carbon soot but not by ceramic materials. The percentage drop in radio signal may consequently be used to estimate the quantity of soot accumulation. The soluble organic fraction (SOF) cannot be easily detected using this approach. Modern particle traps fall short of expectations, particularly for automobiles. When fitted for regeneration, they are expensive and complicated, and they lack long-term durability. An ideal catalytic trap would need the least amount of additional fuel, be simple, affordable, and dependable. It would also be self-regenerating.

Modern Diesel Engines

Modern CI engines generate much less carbon soot particle pollution because to better fuel injector and combustion chamber geometry design. Large areas of mixes rich in fuel may be avoided when combustion begins thanks to significantly higher mixing efficiency and speeds. These are the areas where carbon soot accumulates, and by decreasing their volume, soot accumulation is greatly reduced. Indirect injection, improved combustion chamber geometry, improved injector design and higher pressures, heated spray targets, and air-assisted injectors work together to increase mixing rates.

CONCLUSION

Pollutants including CO₂, NO_x, PM, HC, and other dangerous compounds are emitted during the combustion process in engines, or engine emissions. To reduce or completely eliminate these emissions and enhance air quality while reducing climate change, these emissions must be controlled. The future potential rests in generating cleaner fuels, improving combustion processes, and incorporating cutting-edge technology like AI and IoT. Additionally, the use of renewable energy and electric vehicles (EVs) may dramatically lower engine emissions and advance sustainability. The air-fuel mixing process is substantially accelerated by indirect injection into a secondary chamber that fosters strong turbulence and swirl. Higher injection pressures and improved nozzle design result in finer fuel droplets that mix and evaporate more quickly. Evaporation is accelerated by injection against a heated surface and by air-assisted injectors. Some cutting-edge, current CI automotive engines have decreased particle production to the point that they no longer need particulate traps to fulfil strict criteria.

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CHAPTER 15

MEASUREMENTS AND TESTING OF IC ENGINE

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ABSTRACT

In many sectors, including research, engineering, manufacturing, and quality assurance, measurements and testing are essential. While testing refers to the process of assessing the performance, qualities, or compliance of a product or system, measurements deal with the quantification of physical quantities. For the purpose of guaranteeing product quality, safety, and regulatory compliance, accurate and trustworthy measurements and testing are crucial. This abstract examines the idea of measuring and testing, demonstrates its relevance to many businesses, and emphasizes the value of using standardized procedures and tools. Measuring several metrics and physical quantities linked to engine performance, combustion characteristics, emissions, and fuel economy in the context of IC engines. Engineers may enhance design, identify problems, and boost performance by using these measures to better understand how the engine behaves under various operating situations. Engine torque, cylinder pressure, temperature, exhaust gas composition, and fuel consumption rates are important parameters.

KEYWORDS

Measurements, Physical Quantities, Product Quality, Regulatory Compliance, Testing.

INTRODUCTION

Internal combustion engines (IC engines), where precision and accuracy are essential for optimizing performance, guaranteeing compliance with pollution standards, and attaining economical fuel usage, play a critical role in measurements and testing. The development, use, and maintenance of IC engines depend on measurements and testing since the power a variety of vehicles, pieces of equipment, and power production systems. In order to assess engine performance, emissions, durability, and dependability, testing in IC engines entails a thorough set of methods and tests. In order to mimic real-world operating circumstances, engine testing is often carried out in controlled settings like dynamometer test cells or chassis dynamometers. To evaluate the engine's capabilities and adherence to legal requirements, several testing are carried out, such as power output tests, pollution tests, durability tests, and thermal efficiency measures.

In IC engines, measurements and testing have many uses. They first provide helpful information for engine design and optimization. Engineers may refine combustion processes, improve air-fuel mixture management, and optimize engine characteristics via measurements and testing in order to increase power output, increase efficiency, and lower emissions. Second, testing and measurements are essential for adhering to emissions laws. To lessen the negative effects on the environment and safeguard air quality, governments and regulatory organizations set strict limitations on the amount of pollution that IC engines may emit. In order for producers to certify their goods and adhere to legal regulations, engine measurements and emission testing assist verify that engines satisfy these regulatory criteria.

Additionally, measurements and testing help with problem diagnosis, problem solving, and engine health monitoring. Engineers may spot anomalies in combustion, differences in valve timing, problems with the fuel system, or component failures by examining measures including cylinder pressure, exhaust gas composition, and temperature profiles. Through prompt maintenance and repair, which is made possible by this diagnostic capacity, dependability is improved and downtime is decreased [1], [2]. The development of monitoring and testing technology will continue to fuel innovation in IC engines in the future. New insights into combustion processes and emissions characteristics are provided by non-intrusive monitoring methods including optical sensors and laser-based diagnostics. More thorough and accurate measurements are made possible by cutting-edge data collecting systems and real-time monitoring, which improve engine management and optimization.

Additionally, virtual testing is now possible because to developments in simulation and modelling methods, which eliminate the need for physical prototypes and quicken the engine development process. In order to create engines that are more effective and burn cleaner, computational fluid dynamics (CFD) simulations are used in conjunction with observed data to offer a greater knowledge of fluid flow, combustion dynamics, and heat transport inside the engine. In conclusion, measuring and testing are essential instruments in the field of IC engines, allowing engineers to increase performance, assure compliance with regulations, identify problems, and optimize operation. Future generations of internal combustion engines will have more honed designs, better fuel efficiency, lower emissions, and more dependability thanks to advancements in measurements and testing for IC engines.

The primary goal of engine design and development is to lower manufacturing costs while increasing production efficiency and power output. The development engineer must evaluate the output and efficiency of the produced engine in comparison to that of other engines in order to complete the aforementioned mission. He must test the engine and take measurements of important variables that represent the engine's performance in order to do this. He must generally carry out a broad range of engine testing. Numerous variables affect the kind and type of tests that will be performed. The discussion of all the elements is beyond the purview of this work. This chapter examines a few fundamental as well as significant measures and tests.

- i. Friction power
- ii. Indicated power
- iii. Brake power
- iv. Fuel consumption
- v. Air flow
- vi. Speed
- vii. Exhaust and coolant temperature
- viii. Emissions
- ix. Noise
- x. Combustion phenomenon

Friction power

Friction power is the term used to describe the difference between an engine's indicated and braking power. Pumping losses and friction losses are the two main categories of internal losses in an engine. The gaseous pressure on the piston is greater on its forward side during the inlet and exhaust strokes (on the underside during the inlet and on the upper side during the exhaust stroke), so the piston must be moved against a gaseous pressure during both strokes, which results in the so-called pumping loss. The friction loss is made up of the friction between the piston and the cylinder walls, the friction between the piston rings and

the cylinder walls, the friction between the crankshaft and camshaft and their bearings, the loss experienced when operating the necessary accessories, such as the water pump, ignition unit, etc. The goal of the designer should be to minimise power loss due to friction. For the assessment of stated power and mechanical efficiency, friction power is employed. The friction power is determined using the following techniques in order to gauge the engine's performance [3].

DISCUSSION

Consumption of Fuel

Fuel consumption can be expressed in one of two ways: by weight or by volume over a given period of time. It is stated in terms of km per liter for autos. In engine testing, precise fuel consumption measurement is crucial. Although it may appear like a straightforward affair, it is far from it, as the occurrence of the following phenomena makes clear:

- (i) In the fuel line, vapor bubbles develop as a result of engine heat. The fuel volume grows as the bubble expands, and fuel backflow occurs. This backflow is measured as forward flow by some fuel flow meters. While a forward flow that is counted occurs when the bubble bursts, certain meters do not count backward flow.
- (ii) The flow measured can be significantly higher than the real flow if bubbles occur inside or before the flow meter.
- (iii) The fuel flow may be recorded as having a higher flow rate if there is any swirl in the fuel flow, especially when using a turbine type flow meter.
- (iv) The temperature, which can range from -10 to 70 degrees Celsius, affects the fuel's density and might cause measurement mistake.
- (v) The measurement of some flow meters that use a laser beam may be impacted by the color of the fuel.
- (vi) Periodically, the carburetor's needle valve in the float bowl opens and closes, letting fuel rush into the bowl. This could result in a water hammer effect that keeps the turbine type flow meter spinning even when the fuel flow has stopped, leading to inaccurate flow measurements.

As previously indicated, there are two primary types of fuel measurement methods:

- (i) Volumetric type
- (ii) Gravimetric type

Volumetric Type Flow meter

The simplest way to measure volumetric fuel use is with glass bulbs that have been marked on both sides and have a known volume. A stop watch is used to time how long it takes the engine to consume this volume. The volumetric flow rate is calculated as volume divided by time.

Burette Method

This method uses two spherical glass bulbs with corresponding capacities of 100 and 200 cc. Three-way cocks join them so that one can feed the engine while the other is filling. While the spherical form combines strength with a slight change of fuel head, which is most significant in the case of carburetor engines in particular, the glass bulbs are of varying capacities in order to make the duration of the tests generally constant regardless of the engine load. Burette photocells are used to prevent errors when comparing the fuel level to the mark on the burette. Such a setup where the measurement is automatic is shown in Figure 1.

Automatic Burette Flow meter

Commercially available automatic volumetric fuel flow metering equipment is shown in Figure 2. It is made up of a measuring volume (A) with tubular housings for a photocell (B) and a light source (C). As illustrated in Fig. 2, these housings are placed one on each of the lower and upper portions of the measuring cylinder, opposed to one another at an angle that creates a point of light on the axis of the measuring volume. In order to provide an air cushion at supply line pressure and to store fuel during measurement, an equalization chamber (D) is linked to the measuring tube via the air tube (E), magnetic valve (F) and equalization pipe (G) [4], [5].

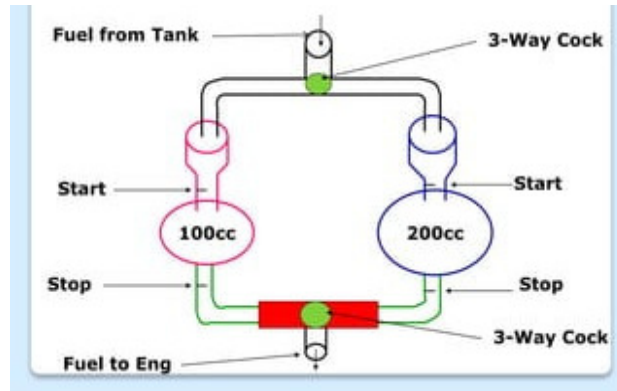


Figure 1: Burette method of measuring fuel consumption (slideshare.net)

When the start button is pressed, the lamps in the two photoelectric systems turn on, and the magnetic valve shuts off the instrument's flow of fluid. Depending on how much fuel the engine is using, the fuel level in the measuring volume begins to decline. A similar amount of flows through the equalization tube and into the equalization chamber at the same time. The focused light beam from the lamp is reflected onto the opposing photocell and transformed into an electrical signal to start a timer counter when the fuel level reaches the top measuring level (H). The new signal created stops the timer counter when the fuel level has decreased enough to reach the lower measuring level. The lamps are turned off automatically, the valve is opened, and the usual flow is resumed. Thus, the duration for consuming the selected fuel volume is noted.

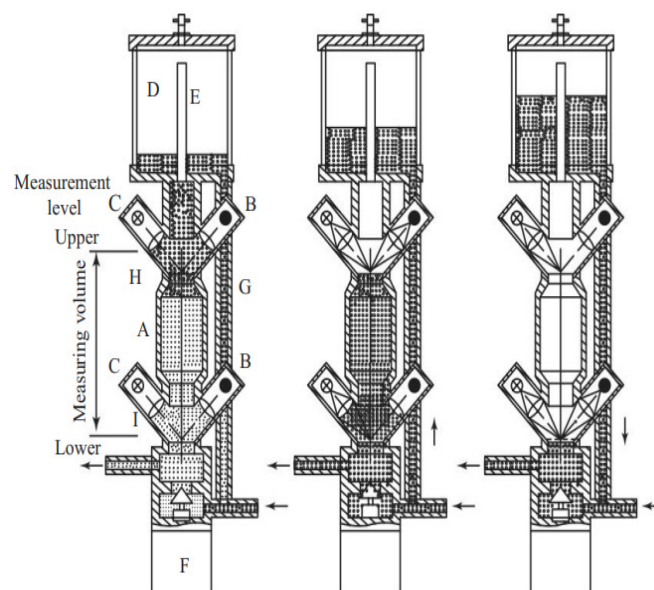


Figure 2: Automatic volumetric fuel flow meter (ftp.idu.ac.id)

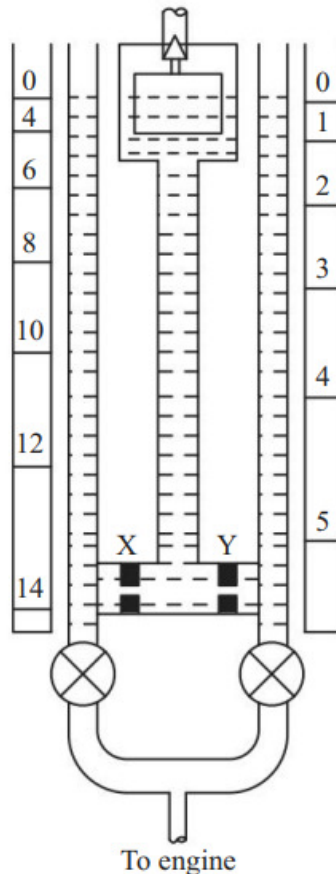


Figure 3: Orifice flow meter (ftp.idu.ac.id)

Orifice Flow meters: Flow meters are occasionally employed for this purpose as well. The pressure difference across an aperture is what drives flow meters. In Fig. 3, two orifices, X and Y, are depicted. Direct observations of the volume of gasoline provided each hour can be made using these orifices, which are pre-calibrated for the fuel being used. It is possible to use one or two orifices at once. Orifices can also be adjusted for various flow rates. Scales used for calibration must also be modified in that circumstance.

Gravimetric Fuel Flow Measurement

The amount of gasoline burned per kilogram, rather than per liter, determines an engine's efficiency in most cases. The process of measuring volume flow and then adjusting it for differences in specific gravity is highly cumbersome and has a finite degree of accuracy. Instead, there can be a significant improvement in cost and accuracy if the weight of the gasoline consumed is directly measured [6], [7].

The procedure entails weighing the fuel supplied to the engine via the setup seen in Fig. 4. This method involves closing valve B to allow fuel from the tank to flow straight to the engine and opening valve a whenever the engine is to be operated without first measuring the rate of fuel supply. When measuring the fuel, valves A and B are opened to allow fuel to flow from the tank into the flask. The weight of the fuel is measured on the balance. The valve A is closed while keeping the valve B open to allow the gasoline from the flask to be syphoned off to the engine. The specific gravity of the fuel does not need to be determined separately using this method. A stop watch is used to record the amount of time needed to completely syphon off the weighed fuel. This results in the measurement of fuel consumption in gravimetric units.

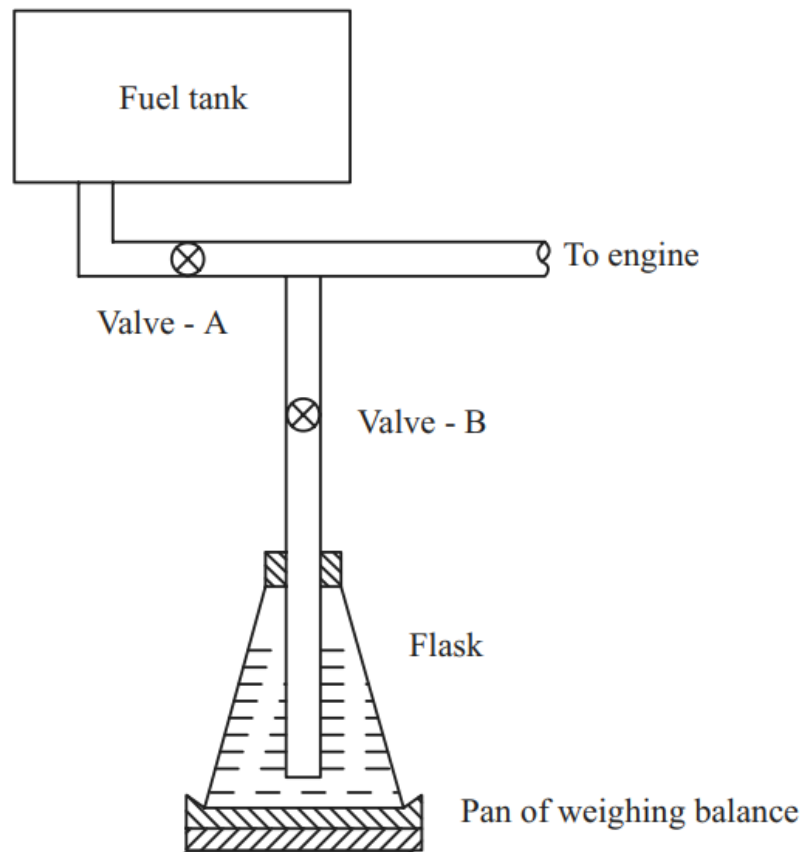


Figure 4: Gravimetric measurement of fuel flow (ftp.idu.ac.id)

Fuel Consumption Measurement in Vehicles

Measuring fuel consumption in km per liter is the third way to express fuel consumption. The apparatus seen in Fig. 5 is typically used to precisely measure the fuel consumption of any vehicle being tested. To accurately measure km/liter on a real road test, a glass burette with a 1 liter capacity is connected by tubes and control valves. The instrument displays how fuel consumption is affected by factors like speed, traffic, load, driving style, and engine conditions. The tester is attached using special plastic tubes and the included adapters to the top edge of the right front door glass or another suitable spot. When the needle valve is opened, the excess fuel capacity of the fuel pump fills the glass burette, and when a test run is performed, the control tap sends the gasoline to the intake side of the fuel pump. As a result, the carburetor receives its usual working pressure supply. Calculating actual usage is easy and precise thanks to speedometer readings in kilometers and tenths for each 0.5 liters utilized.

Air Consumption

An engine's diet consists of fuel and air. Accurate measurement of both values is crucial to determining the engine performance. Due to the cyclic nature of the engine and the fact that air is a compressible fluid, measuring air consumption in IC engines accurately is rather challenging. Because the reading will be pulsing and unpredictable, the straightforward method of employing an opening in the induction pipe is inadequate. The relationship between flow rate and differential pressure is a square law for all kinetic flow inferring systems, including nozzles, orifices, and ventures, leading to severe errors on unstable flow. For a specific set of flow parameters, the pressure across the orifice is roughly inversely proportional to the errors caused by pulsation.

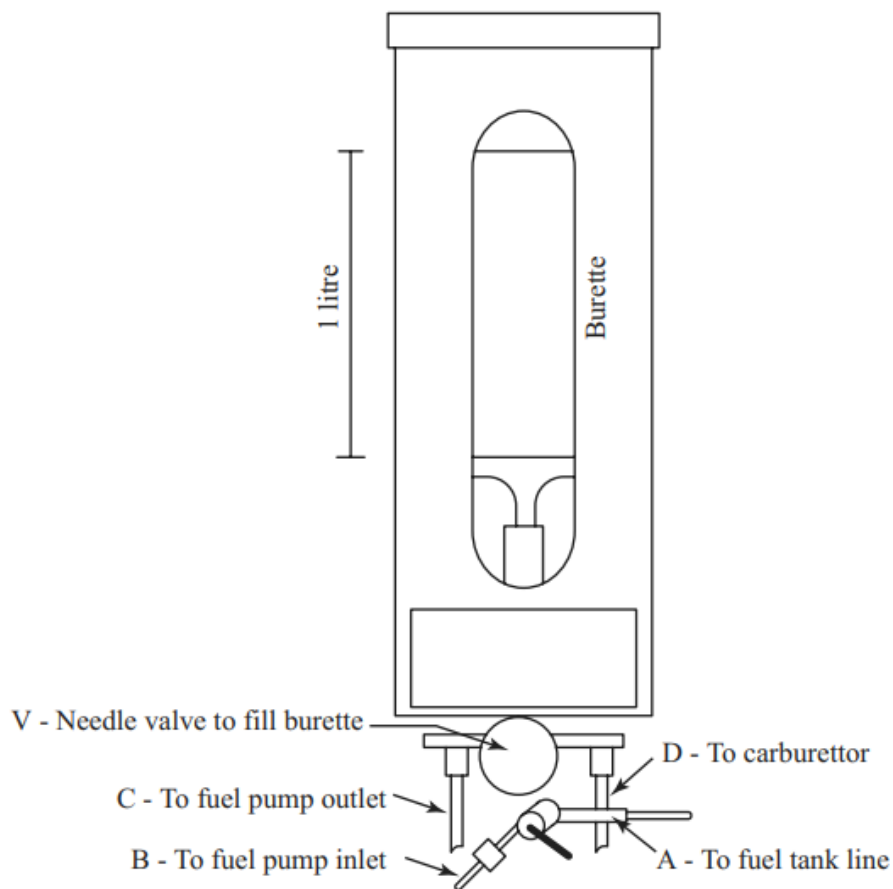


Figure 5: Fuel consumption measurements in vehicles (ftp.idu.ac.id)

Noise

Noise is a collection of different noises that irritates the listener. A thing vibrating produces sound. Pressure waves are used to communicate the vibrations to the surrounding air. When the pressure wave's frequency and intensity fall within a certain range (15 to 15000 Hz and 0 to 120 dB intensity), the feeling of sound is produced. As shown in Fig. 15.30, a sound level metre comprises of a microphone, a calibrated attenuator, an electrical amplifier, and an indicator metre that displays decibels (dB). The frequency distribution of light bands in the range of 20 to 10,000 Hz can be obtained using an octave band frequency analyzer. Since it is the complete noise emitted by a motor vehicle, including the gear box and gearbox, the measurement of the noise produced by motor vehicles is based on a moving vehicle [8], [9].

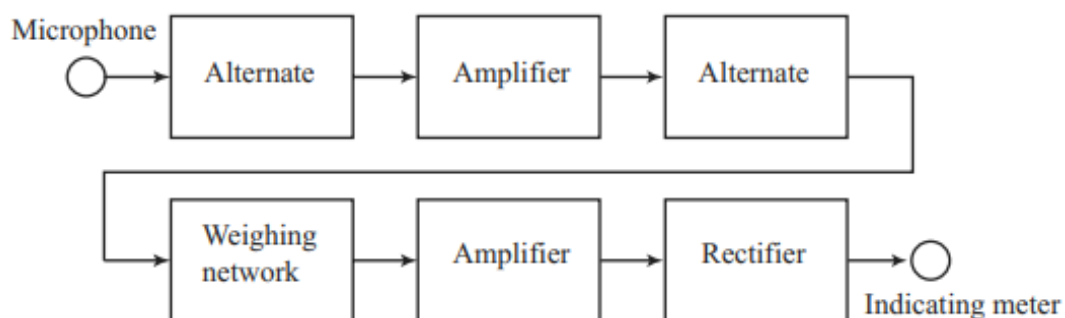


Figure 6: Sound level meter

CONCLUSION

Internal combustion engine measurements and testing entail the evaluation of numerous performance factors and parameters to guarantee optimal performance and adherence to legal requirements. This includes gauging factors like fuel economy, emissions, torque, and power output as well as combustion traits and engine longevity. The scope of measurements and testing in IC engines will expand as technologies become more precise and effective, testing procedures become more automated, sensors and data analytics are integrated for real-time monitoring, and testing procedures for alternative fuels and electric powertrains are developed.

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CHAPTER 16

PERFORMANCE PARAMETERS AND CHARACTERISTICS

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ABSTRACT

this abstract provides an overview of the performance parameters and characteristics of internal combustion engines (IC engines). It discusses the key factors that affect engine performance, such as power output, fuel efficiency, emissions, combustion efficiency, and durability. Various measurement techniques and testing methodologies are utilized to evaluate these parameters and ensure optimal engine operation. Understanding and optimizing these performance parameters are crucial for enhancing engine efficiency, reducing emissions, and improving overall vehicle performance. Power output is one of an IC engine's key performance metrics. Power output, which is typically expressed in terms of horsepower or kilowatts, refers to an engine's capacity to generate mechanical work. It has a direct impact on a vehicle's acceleration, towing capability, and general performance. An important goal in the design and optimization of IC engines is to maximize power output while preserving fuel efficiency.

KEYWORDS

Measurement Techniques, Performance Parameter, Power Output, testing methodologies, vehicle parameters.

INTRODUCTION

In order to assess and comprehend the operational effectiveness of internal combustion engines (IC engines), performance criteria and characteristics are essential. The power output, fuel efficiency, emissions, combustion efficiency, durability, and overall performance of IC engines are all greatly influenced by these characteristics. The analysis and optimization of these characteristics have become crucial due to technological breakthroughs and rising expectations for more economical and ecologically responsible transportation[1]–[3]. Another crucial performance factor that affects how much energy can be collected from a given amount of fuel is fuel efficiency. The most common way to represent it is in terms of fuel consumption per unit of travel distance, such as miles per gallon (MPG) or liters per kilometer (L/100 km). Enhancing fuel efficiency is crucial for both vehicle manufacturers and environmentally aware consumers since it lowers carbon dioxide (CO₂) emissions while simultaneously lowering operational expenses.

Due to their negative effects on the environment, emissions are a major concern with IC engines. The quantities of pollutants including carbon monoxide (CO), nitrogen oxides (NO_x), hydrocarbons (HC), and particulate matter (PM) are performance criteria connected to emissions. In order to lessen these dangerous particles and enhance air quality, more stringent emission rules have sparked the development of innovative technology including catalytic converters and exhaust gas recirculation (EGR). How efficiently fuel is burned inside of an engine is determined by its combustion efficiency. It is affected by things like the air-to-fuel ratio, the layout of the combustion chamber, and the timing of the ignition. Enhancing combustion efficiency results in increased power production, less fuel usage, and decreased

pollutants. The complicated combustion processes must be fully understood, and engine parameters must be calibrated with care, to get the best combustion efficiency.

The engine's capacity to tolerate challenging operating circumstances and retain performance over an extended period of time is reflected in its durability. Engine durability is heavily impacted by elements including material choice, lubrication systems, and cooling techniques. Engine longevity is increased and the need for regular maintenance or replacements is decreased by designing engines with durable parts and effective cooling systems.

Different measurement techniques and testing approaches are used to assess and improve these performance factors and attributes. These consist of data collecting systems, combustion analysis, pressure monitoring, and dynamometer testing. Engineers can collect precise data and spot potential improvement areas thanks to sophisticated diagnostic tools and sensors. In conclusion, the efficiency, emissions, and general performance of cars are significantly influenced by the performance parameters and traits of IC engines. Maximizing power output, boosting combustion efficiency, decreasing emissions, and assuring engine durability are the main goals of ongoing research and development. For the transportation industry to advance and to meet regulatory requirements while also increasing sustainability, these factors must be optimized.

The typical operating range of an internal combustion engine is speed. Through the use of a speed governor, some engines are designed to run at their rated speed. The power output varies and has a maximum usable value at each speed within the useful range. The load is defined as the ratio of developed power to the highest usable power at the same speed. The precise gasoline usage changes depending on the load and the speed. Within the useful range of speed and load, the interrelationship between power developed, speed, and the specific fuel consumption at each operating condition determines how well the engine performs [4]–[6].

When assessing an engine's performance, the following elements must be taken into account:

- (i) Within the practical speed range, the maximum power or torque attainable at each speed.
- (ii) The range of power production at constant speed needed for the engine to run steadily. In the useful speed range, the various speeds should be chosen at equal intervals.
- (iii) Fuel usage specific to braking at each operating condition within the practical operating range.
- (iv) Engine dependability and durability for the specified operating range.

Engine performance characteristics can be determined by the following two methods.

- (i) Using experimental findings from engine testing.
- (ii) Utilizing a theoretical data-based analytical calculation.

In reality, "engine performance" is a relative concept. Typical characteristic curves, which are functions of engine operating parameters, are used to represent it. The term "performance" typically refers to how effectively an engine produces useable energy in comparison to other comparable engines or how well it performs in relation to the input energy. Speed, input pressure and temperature, output, air-fuel ratio, and other critical characteristics are only a few. All of these parameters' useful ranges are constrained by a number of things, including mechanical strains, knocking, overheating, etc. As a result, the maximum power and efficiency that an engine is capable of producing have a practical limit. Engine power and engine efficiency are the two primary criteria used to evaluate an engine's performance. When discussing the theory, design, and operation of engines, numerous different efficiencies are

encountered in addition to the total efficiency. The following two sections go into greater information about these elements.

Engine Power

The energy flow through the engine is typically expressed in three different terms. They are called braking power (bp), friction power (f p), and indicated power (ip). Braking power may be calculated from the measurement of forces at the engine's crankshaft, and indicated power can be calculated from the measurement of forces in the cylinder. The engine can be run to estimate friction power, or alternative techniques are covered in Chapter 15. If the ip and bp are known, it can alternatively be calculated as the difference between the two,

$$ip = bp + f p$$

$$fp = ip - bp$$

The formulae for calculating power that are typically used are covered in the following sections.

DISCUSSION

Engine Performance Characteristics

Engine performance characteristics are an easy way to visualise an engine's performance. They are built using information gathered from the engine's real test runs, and they are especially helpful for comparing one engine's performance to another. Some of the significant SI engines' performance characteristics are addressed in this section. It should be remembered that the maximum charge will be introduced per cylinder every cycle at a specific speed within the range of a particular engine. Therefore, the piston can now be subjected to its maximum force. Practically speaking, this is also when the engine's torque, or ability to perform work, will be at its highest. As a result, there is a specific engine speed where the charge per cylinder each cycle is at its maximum, and at roughly the same speed, the engine's torque will also be at its maximum.

The amount of the specified charge will decrease as engine speed is raised above this speed. However, when speed increases, the engine's power output rises as more cycles are completed in a given amount of time. It should be noted that as engine speed is increased, air consumption will continue to rise until a certain point where the charge per the number of strokes per unit time is increasing, yet the cylinder per stroke is decreasing extremely quickly. The maximum air consumption point is not reached within the engine's operational speed because to the way engines are built. As air consumption rises, more fuel can be fed in a given amount of time, increasing power production. Actually, the amount of ip generated in the cylinder is almost precisely proportionate to the amount of air the engine consumes[7]–[9].

Fig. 1 shows the link between air consumption and ip as well as between air charge per cylinder per cycle and torque. You should take note that the maximum torque happens at a speed that is lower than the maximum ip.

Some other crucial performance parameters for a typical SI engine are shown in Figure 2. In this graph, torque, ip, bp, and f p are plotted against engine speed at full throttle and varied load across the engine's operational range. The f p is what separates the ip generated in the cylinder from the bp realised at the drive-shaft. The f p is relatively low and the bp is very close to the ip at low engine rpm. The f p grows more rapidly as engine speed rises. F p increases extremely quickly when the engine is running faster than its typical operating range.

Additionally, i_p will reach a maximum at these faster speeds before declining. When i_p and f_p are equal, b_p will eventually go to zero. Keep in mind that while the i_p has not achieved its maximum even at the engine's rated speed, the torque achieves its peak at about 60% of the engine's rated rpm.

Figure 3 displays the engine speed vs fuel consumption and bsfc for the same engine running in the same circumstances. With increased engine speed, more gasoline is utilised. On the other hand, the bsfc increases in the high-speed range, virtually levels out at middle speeds, and decreases as speed increases in the low speed region.

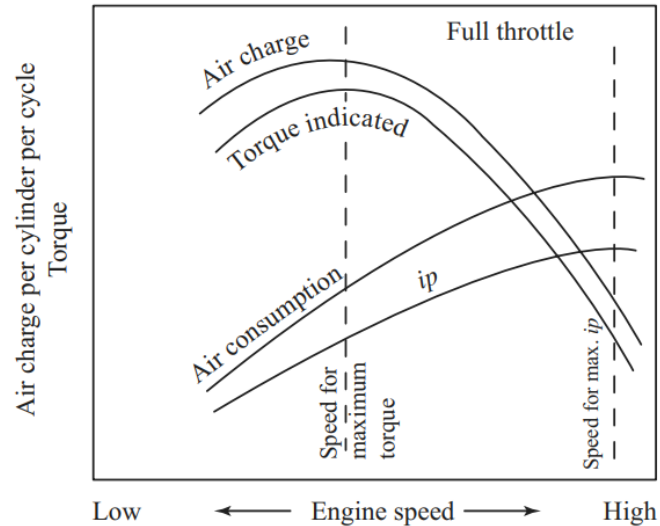


Figure 1: Typical performance plot with respect to speed (ftp.idu.ac.id)

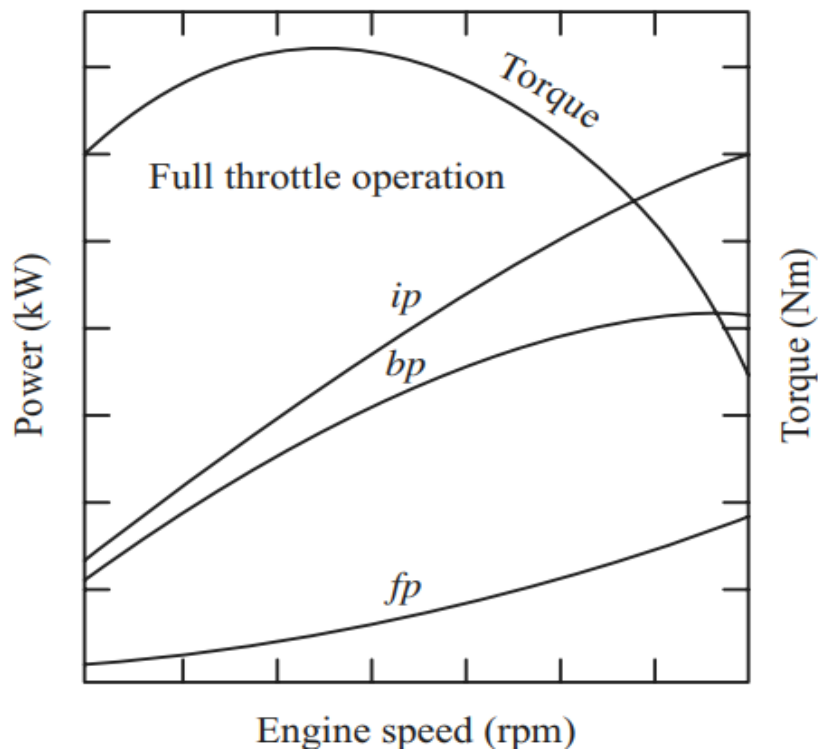


Figure 2: Typical SI engine performance curves (ftp.idu.ac.id)

Because of the proportionally increased heat loss to the combustion chamber walls and the lower combustion efficiency at low speeds, more fuel is needed to produce the same amount of power.

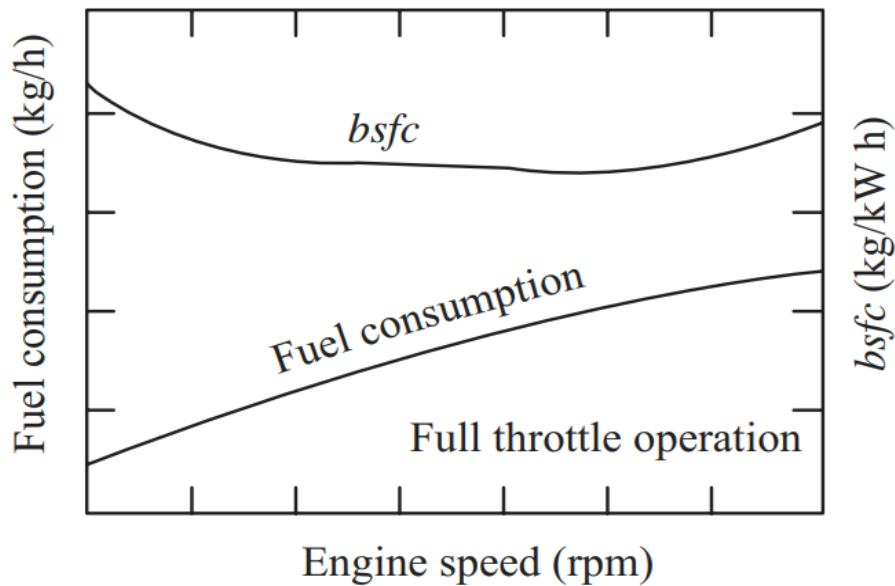


Figure 3: Typical fuel consumption curves for an SI engine (ftp.idu.ac.id)

The bsfc curve in Fig. 3 is for variable speed, full throttle operation. It reflects the bsfc that will be produced at any given speed when the engine is operating at maximum capacity. The same speed can be achieved by decreasing the load and opening of the throttle, but at loads that are lower than maximum. It is possible to obtain a series of curves that illustrate the impact of changing the load while maintaining a constant speed on the bsfc. The bsfc will rise steadily and quickly when the load (and throttle opening) is reduced under these circumstances of constant speed, variable load, and constant air-fuel ratio. The general contour of the curve for every particular rpm is shown in Figure 4. The fact that the f_p stays practically constant while the i_p is being reduced is what causes the quick increase in bsfc with the reduction in throttle opening. The b_p decreases more quickly than gasoline use, which causes the bsfc to increase.

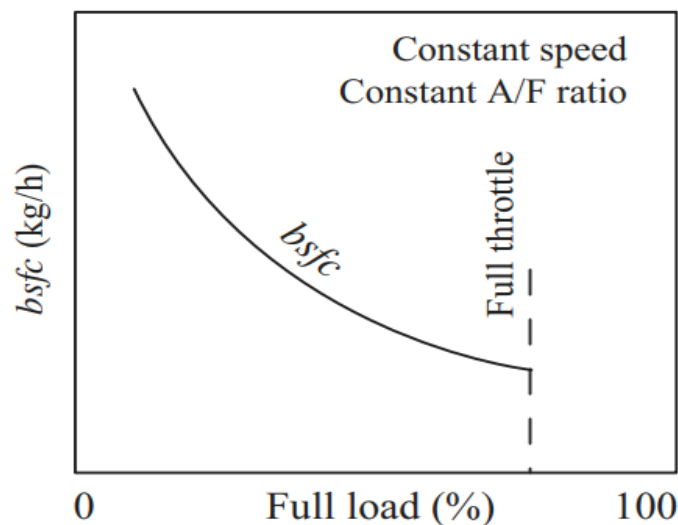


Figure 4: bsfc curve at constant speed and variable load (ftp.idu.ac.id)

Performance curves can be created for additional operating parameters including air consumption, bmep, imep, and more. The curves that are being shown, nevertheless, are normal and some of the most significant. The curves of torque, bp, and bsfc plotted against engine speed during full throttle operation are probably the most significant of these. The engine manufacturers' descriptive literature on their engine models most frequently includes these curves. A plot like this would resemble Fig. 5.

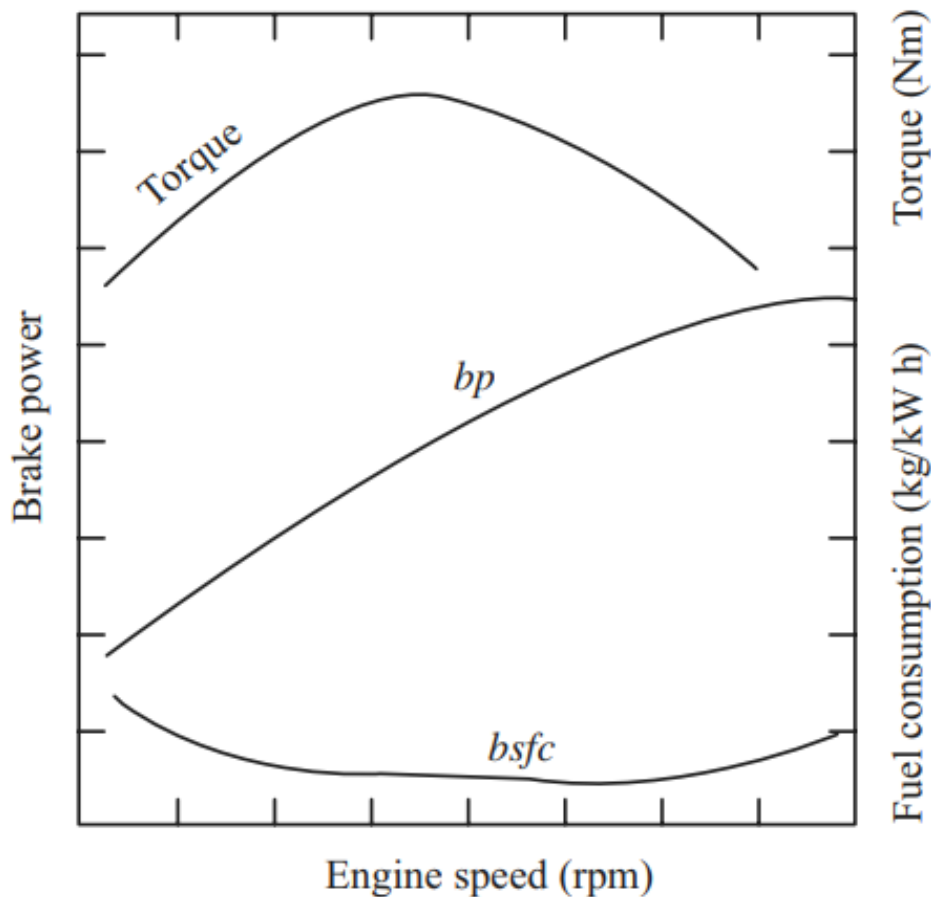


Figure 5: Variation of bsfc, torque and bp with respect to speed for an SI engine (ftp.idu.ac.id)

One can consult the curves offered by the major engine manufacturers to choose an engine with a specific capacity that will suit his demands. The best engine can then be chosen by looking at thorough performance curves. Indicated and braking power at full load as well as mean effective pressure for normally aspirated DI and IDI compression-ignition engines are shown in Figure 6. Since the diesel engine's intake system can have higher flow areas than the intake of SI engines due to their intake-system fuel transport requirements, braking torque and mep only slightly fluctuate with engine speed, even at high engine speeds. Similar to the full-load characteristics in Fig. 6, the part-load torque and bmep characteristics have a fixed amount of fuel injected per cycle.

Variables Affecting Performance Characteristics

Engine performance curves were covered in the section before. The control of numerous design and operational variables determines the shape of these curves or the engine performance. In this part, some of the crucial variables will be summarized and quickly addressed.

Combustion Rate and Spark Timing

To provide a smooth-running engine, the ignition timing and combustion rate should be adjusted such that the maximum pressure occurs as soon as feasible during the power stroke. In general, the piston reaches T DC on the compression stroke when the spark timing and combustion rate are adjusted so that roughly half of the total pressure rise due to combustion has occurred.

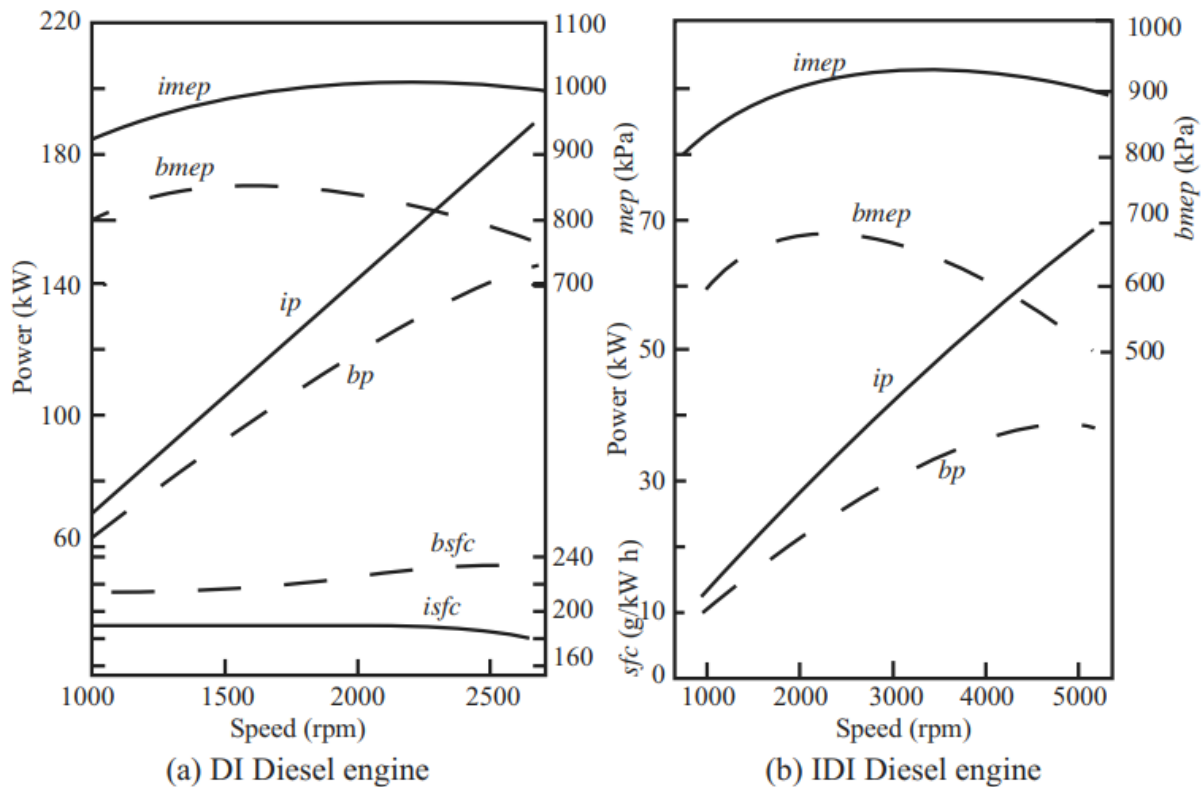


Figure 6: Performance graph of DI and IDI CI engine (ftp.idu.ac.id)

Air-Fuel Ratio

To provide a smooth-running engine, the ignition timing and combustion rate should be adjusted such that the maximum pressure occurs as soon as feasible during the power stroke. In general, the piston reaches T DC on the compression stroke when the spark timing and combustion rate are adjusted so that roughly half of the total pressure rise due to combustion has occurred.

Engine Speed

Low speeds allow for more time for heat to move to the cylinder walls, which results in a higher percentage of heat loss. Higher speeds result in greater air consumption and, hence, larger *ip*—up to a point. But as the speed is increased, the *f p* increases quickly and the inertia of the moving elements increases. The engine speed range must therefore be a compromise, even though the majority of modern designs seem to favor greater speeds.

Mass of Inducted Charge

The power generated increases with the mass of the charge introduced. Given that an engine's geometry is fixed, it is ideal to induct a charge at its highest density for optimal volumetric efficiency.

Heat Losses

It should be emphasized that a significant amount of the available energy is lost as heat, which cannot be used. Engine performance will often enhance with any technique that can be used to stop excessive heat loss and make this energy exit the engine in a useable form. Higher coolant temperatures, for example, offer a reduced temperature differential around the walls of the combustion chamber and a reduction in heat loss, but are constrained by the potential for engine component damage[8], [10], [11].

CONCLUSION

Future improvements to IC engines' performance metrics and features will be made possible by new technologies and methods that will increase engine efficiency, lower emissions, and boost overall performance. To increase fuel efficiency and reduce emissions, this includes the development of novel combustion processes including homogeneous charge compression ignition (HCCI) and lean-burn combustion. Additionally, improvements in materials, lubrication methods, and cooling techniques will help engines last longer. Additionally, the future of performance criteria in IC engines will be shaped by the combination of hybridization, electrification, and alternative fuels, encouraging sustainability and meeting growing regulatory requirements. The thermal efficiency rises with an increase in compression ratio, which is generally favorable. Most SI engines' compression ratios are constrained by knock and the usage of commercially viable antiknock grade fuels. There is a point at which further compression ratio increase would not be economical, even though this point looks to be very high. Increasing compression ratio also increases engine friction, notably between piston rings and the cylinder walls.

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CHAPTER 17

EXPLORING THE IC ENGINE COMPONENTS

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ABSTRACT

Engine electronics are the electrical devices and parts that modern internal combustion engines utilize to regulate numerous processes and enhance performance. These systems consist of communication networks, sensors, actuators, electronic control units (ECUs), and engine management systems. Fuel injection, ignition timing, airflow, exhaust emissions, and general engine operation are all monitored and controlled by engine electronics. This abstract gives a general introduction of engine electronics and discusses how important it is for increasing engine performance, cutting emissions, and improving engine economy. The engine management system (EMS) is one of the essential elements of engine electronics. Data from several sensors, including those measuring air intake, coolant temperature, throttle position, and exhaust pollutants, are strategically placed throughout the engine and collected by the EMS. This data is processed by the EMS, which then optimizes engine performance in real time. It regulates vital processes like air-fuel ratio, fuel injection timing, ignition timing, and exhaust gas recirculation (EGR), ensuring that the engine runs effectively under a variety of circumstances.

KEYWORDS

Electronics, Emission, Engine, Performance, System.

INTRODUCTION

By integrating cutting-edge electronic systems and components into internal combustion engines, engine electronics has completely changed the automotive industry. The control and optimization of numerous operations by these electronic systems is crucial for improving engine performance, fuel efficiency, emissions reduction, and overall vehicle performance. Engine electronics include a wide variety of parts and systems, including as electronic control units (ECUs), sensors, actuators, and communication networks. To monitor, control, and fine-tune important engine characteristics, these parts operate in unison.

As they give the information required for the EMS to make wise decisions, sensors are essential parts of engine electronics. Physical characteristics including temperature, pressure, airflow, and engine speed are detected by these sensors. These sensors' data collection is essential for maintaining precise engine management and raising performance. Another crucial component of engine electronics is the actuator, which converts impulses from the ECUs into mechanical actions. Fuel injectors, ignition coils, throttle valves, and exhaust valves are just a few of the engine components that are controlled by actuators. The engine may accomplish ideal fuel combustion, power delivery, and emissions management by carefully controlling these components.

The brain of the engine electronics system is an electronic control unit (ECU). Using complex algorithms, they take in data from the sensors, process it, and then send control signals to the actuators. These ECUs are extremely sophisticated, use strong microprocessors and intricate software algorithms to make exact modifications based on the engine's operating

circumstances. Additionally, they enable smooth integration and coordination of engine functions with other vehicle systems by facilitating communication networks between various electronic components within the vehicle [1]–[3].

Engine operation has changed as a result of the integration of engine electronics, which has made it possible to precisely manage and optimize a number of factors. Engine electronics help boost fuel efficiency, lower pollutants, and improve overall vehicle performance by continuously monitoring and modifying engine performance. Additionally, engine electronics provide the door for the adoption of cutting-edge technologies that further improve efficiency and sustainability, including hybridization, electrification, and intelligent engine management systems. As a result of better control, optimization, and performance improvement, engine electronics have evolved into a crucial component of contemporary internal combustion engines. Engine functions may be precisely monitored, adjusted, and coordinated thanks to the integration of engine management systems, sensors, actuators, ECUs, and communication networks. Engine electronics will be essential in advancing fuel economy, reducing pollutants, and improving overall vehicle performance as technology develops. The consumer demands higher performance levels with lower fuel consumption, which has made engine development challenging. Furthermore, using techniques that reduce emissions is necessary due to environmental issues. It is challenging to combine low emissions with good performance and maneuverability. Electronic engine management has grown in significance in this environment. The following are the main objectives of an engine developer:

- (i) With the least initial investment possible, high reliability and durability
- (ii) High power output and torque
- (iii) Low amounts of particle and gaseous emissions
- (iv) Low fuel consumption
- (v) Low noise levels and vibrations

Electronics has undergone a great deal of advancement over time. As a result, using sensors and actuators connected to an electronic control module, it is simple to monitor the regulation of the following parameters in petrol engines. The following are significant variables that can be controlled:

- (i) Air-fuel ratio
- (ii) distribution of mixture between cylinders
- (iii) Ignition timing
- (iv) Injection timing of the fuel
- (v) Idle speed

As is common knowledge, a diesel engine burns a mixture of gasoline and air. By adjusting the amount of fuel injected, the load is managed. For effective combustion to occur, the injected fuel must be atomized and blended with the air without leaving rich pockets. Very lean mixes can result from excessive mixing. Once more, incomplete fuel combustion will cause hydrocarbon emissions. Low mixing will result in large emissions of smoke, HC, CO, and fuel. Low combustion temperatures and slick engine running can result from proper ignition timing. Additionally, this will lower NO_x emissions.

Conventional mechanically operated fuel injection systems cannot meet the performance and pollution norms of today. To reduce fuel consumption and emissions, injection time, pressure, duration, and other factors can be simply regulated electronically. Once more, a system of sensors determines the engine operating conditions, and the electronic control module uses this information to send appropriate commands to actuators to regulate the

engine. Engine speed regulation can also be done with electronic controls. Under transients and significant load changes, these systems can provide very precise engine speed control.

The electronic control module in both SI and CI engines operates on specifically created software that makes use of sensor inputs and previously recorded information about the engine. This data is used to adjust various parameters. Actuators are used to achieve the control. A table containing the engine's data is kept and later examined to help in decision-making. The electronic control module typically needs to cooperate with mechanical systems. This chapter will introduce several instruments and sensors that can be applied in real-world settings.

DISCUSSION

Typical Engine Management Systems

Let's utilize the illustration of a typical petrol fuel injection system in Fig. 1 to give you an idea of the various sorts of sensors that are employed in an engine. As we can see, many sensors are employed to monitor various engine parameters, including engine speed, air flow rate, exhaust oxygen level, camshaft position, and EGR valve position, exhaust manifold pressure, knock, manifold air pressure, and manifold temperature[4]–[6].

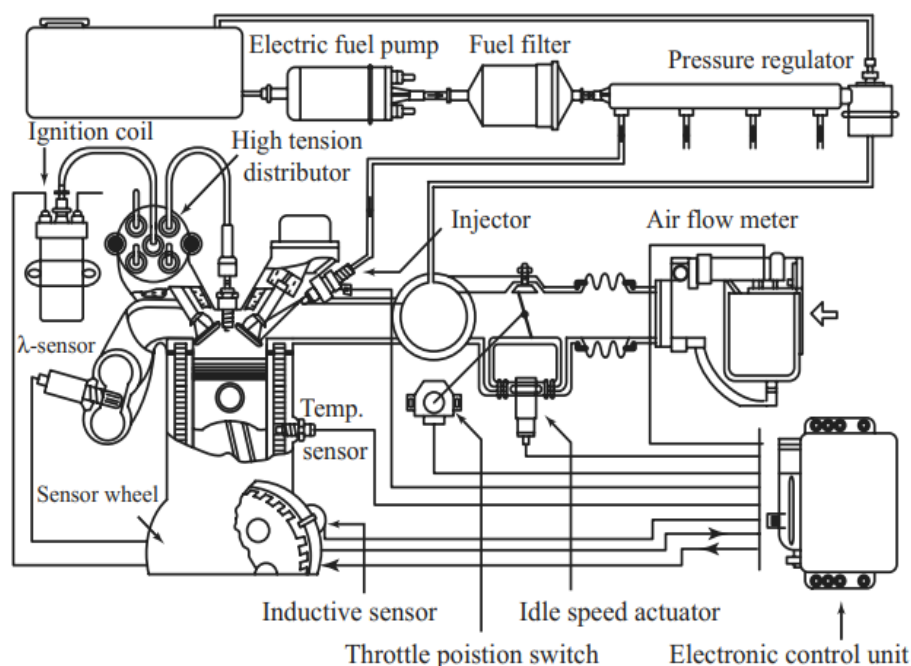


Figure 1: Bosch motronic system (ftp.idu.ac.id)

A typical common rail diesel fuel injection system is shown in Figure 2. We have sensors for the following in this situation: engine speed, air mass flow, crank speed, cam shaft position, turbo air boost pressure, fuel rail pressure, air temperature, coolant temperature, accelerator pedal position, etc. Thus, it is evident that information from various types of sensors is necessary for all electronically controlled fuel control systems to operate properly.

After processing and calculating these inputs from the sensors, the ECU (Electronic Control Unit) transmits outputs to change the timing of the injection of fuel, the amount of fuel used in the ignition, and other parameters. Additionally, it regulates systems including the coolant supply, particle trap regenerator (in a diesel engine), idle speed control unit, and fuel injection pump. A sensor, signal conditioner, analogue to digital converter, electronic control unit,

output signal, driver, and actuator are the basic components of any engine management system. At any given time, the engine will be under control by a combination of several inputs and multiple outputs. We'll examine various commonly utilized sensor types and associated measurement methodologies. The topic is divided into categories based on the parameter being measured for clarity, and relevant sensors are addressed under each category.

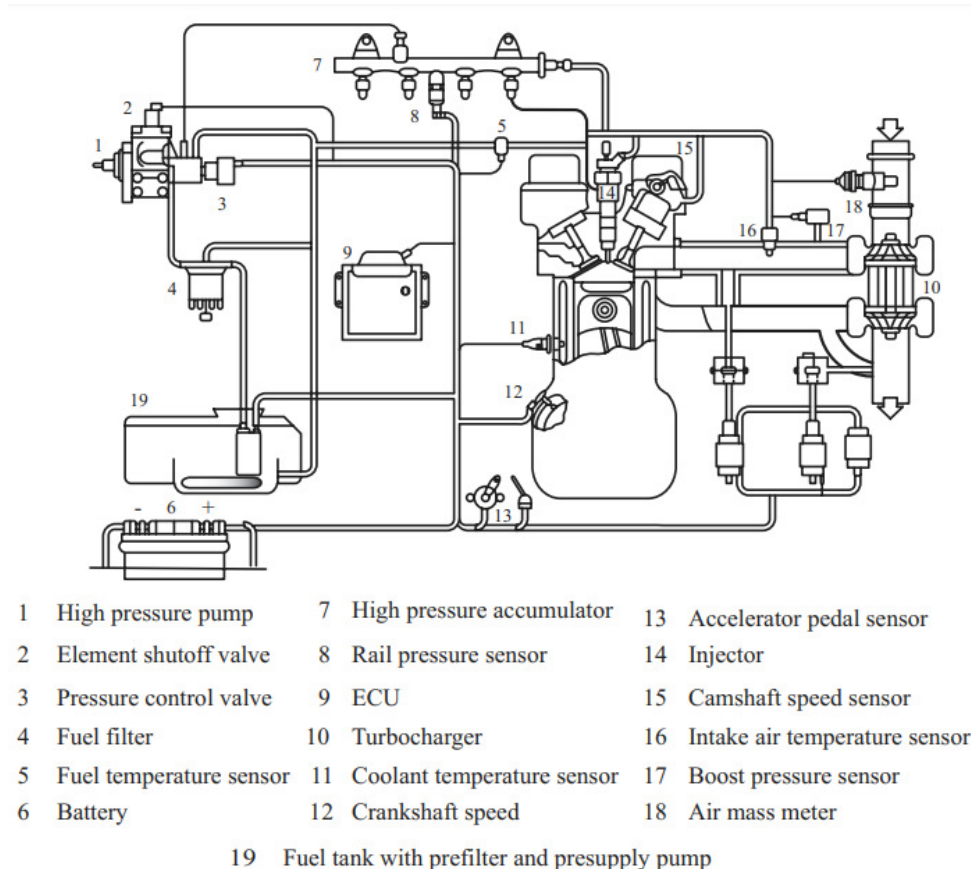


Figure 2: Bosch common rail diesel injection system (ftp.idu.ac.id)

Position Displacement and Speed Sensing

The engine management system places a lot of importance on position displacement and speed sensing. Inductive, hall effect, potentiometric, electro optical, differential transformer, and strain gauge sensors are frequently employed for this type of sensing, especially in cars and engine test labs. The next sections go through the specifics and operation of various sensors.

Inductive Transducers

Fig. 3 depicts a typical inductive transducer. There is a permanent magnet, as is evident. The magnetic flux is changed when the toothed piece moves because it alters the magnetic circuit's permeance. Thus, as the flux field changes as the toothed piece travels, a voltage is created. The plus and minus sides of the voltage output alternate about the mean. To further shape it into a square pulse, it can be transmitted to an electronic circuit. As shown in Fig. 1, the teeth on the engine's crank shaft can serve as the toothed piece, allowing the frequency of pulses to be used to determine shaft speed.

For this application, these transducers are frequently employed. As the flux rate fluctuates more quickly, their output gets stronger. They are therefore only employed in dynamic applications. If a tooth is removed from the cam shaft at a certain location on the shaft, as in

Fig. 1, they can also be used to calculate the position of the cam shaft. The signals in that situation will resemble those in Fig. 4. In essence, the signal is a sine wave, but there is a tooth missing, which will be detected as a missed pulse (Fig. 4). This information is useful to the ECU. The number of pulses before or after the missed pulse can provide the crank position, and the frequency of the pulses determines the speed. The ECU will start the injection and/or ignition process using the crank position. When the pickup comes close to each tooth's apex, the sine wave zeroes out.

A diesel engine's real timing of injection can also be determined using inductive pickups. To start the spark on the right cylinder, distributor rotors can also use inductive transducers. If the movement is quick or the sensor is close to the moving element, inductive pickups, which are straightforward and dependable, will produce a strong voltage [5]–[7].

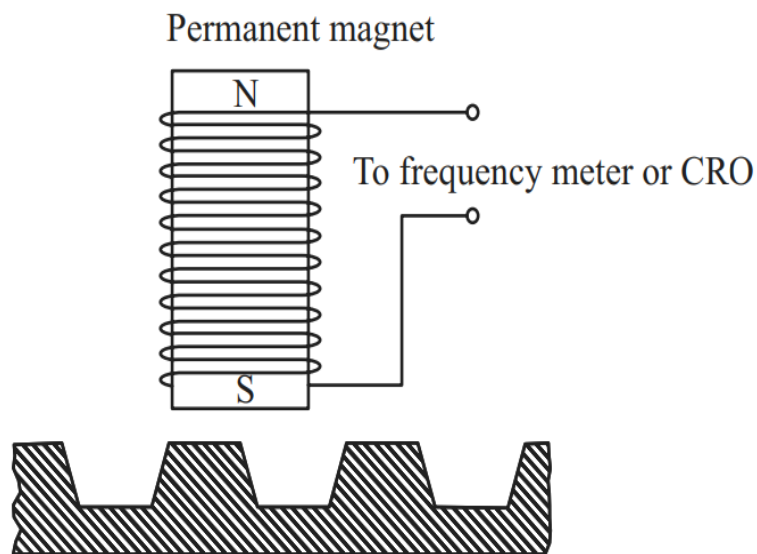


Figure 3: Variable reluctance pickup (ftp.idu.ac.id)

Hall Effect Pickup

A suitable semiconductor material is given a steady current in a hall effect pick-up. An emf develops at right angles to the supply current when this sensor is exposed to a magnetic field that is perpendicular to the direction of the current, as shown in Fig. 5. The moving object may be fastened with a permanent magnet.

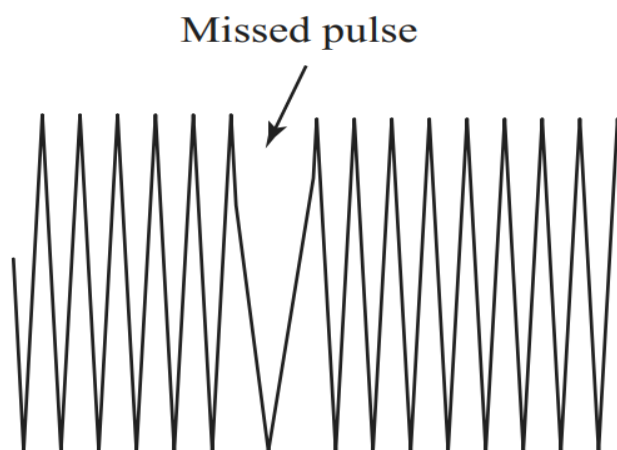


Figure 4: Output signal (ftp.idu.ac.id)

The velocity of the moving object is irrelevant when using this sensor to detect motion, in contrast to an inductive transducer. The magnetic field's strength affects the sensor's output. Typically, the output is changed to produce a square pulse that may be sent to the ECU. The position and speed of the shaft can be determined with this sensor. Additionally, it can be utilised to start the ignition system.

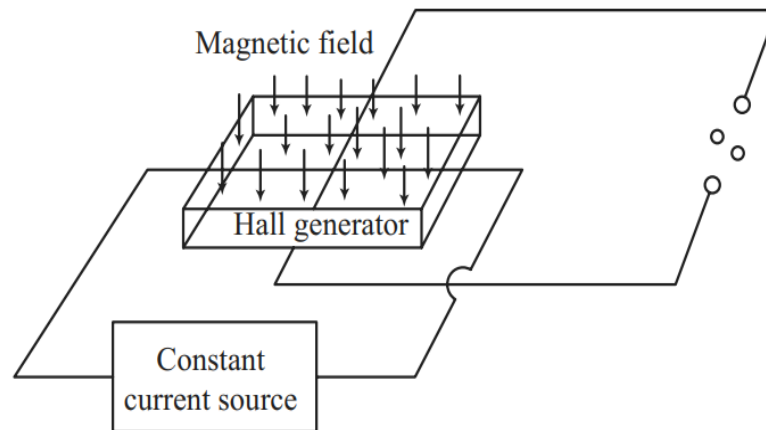


Figure 5: Hall Effect Sensor (ftp.idu.ac.id)

Potentiometers

A potentiometer is typically used to determine the throttle position. The potentiometer is a passive device; unless it is excited, it produces no voltage. In essence, the potentiometer is a changeable resistance. A constant voltage is used across the resistance. The output is the voltage measured between the resistance's one end and the moving leg that slides along it. Rotational or translational contact motion are also possible. In the case of multiple turn devices, it can also be helical. The resistance components may be wire wound, created from conductive ceramic, or created from conductive plastic. The type of resistance element affects resolution. Wound wires are utilised to achieve a high enough resistance. However, in this instance, the resistance changes in discrete steps. Typically, there are 500 to 1000 spins per inch. Applications like the detection of the throttle position and others use non-wire wound potentiometers. They can be resolved infinitely. In Fig. 6, a typical device is depicted. In this instance, a strip of resistance is put on a base insulator.

The key benefit in this situation is the ability to alter the deposition in order to generate any form of resistance variation depending on the location of the slider contact or wiper. This means that by properly designing the potentiometer, any non-linearity in the related hardware can be made up for. Due to roughness and minor changes in the resistance in the strip, which is made up of a mixture of conductive and non-conductive granules, the output varies by a factor of around a mean. As a result, the resolution is high but also constrained by the related roughness. Potentiometers only have a certain lifespan. For wire wound, ten million cycles are typical, while fifty million cycles are typical for conductive polymers. Position is often measured using a potentiometric instrument. It is inadequate for measuring speed.

Linear Variable Differential transformer (LVDT)

The LVDT has good linearity and resolution for measuring large displacements. They are somewhat hefty and can only be used for low frequency measurements. These transducers are excellent for measuring characteristics like valve lift, injector needle lift, fuel injection pump rack motion, etc. in a laboratory setting. They are not frequently utilized on moving vehicles.

The movement of the diaphragms of various pressure sensors has been detected using LVDTs. Fig. 7 depicts an LVDT's schematic. The gadget requires an input of alternating electricity. As opposed to the frequency at which the displacement will vary, this is typically of a much higher frequency. The carrier frequency, which is the frequency of the input signal, is normally around 5 kHz.

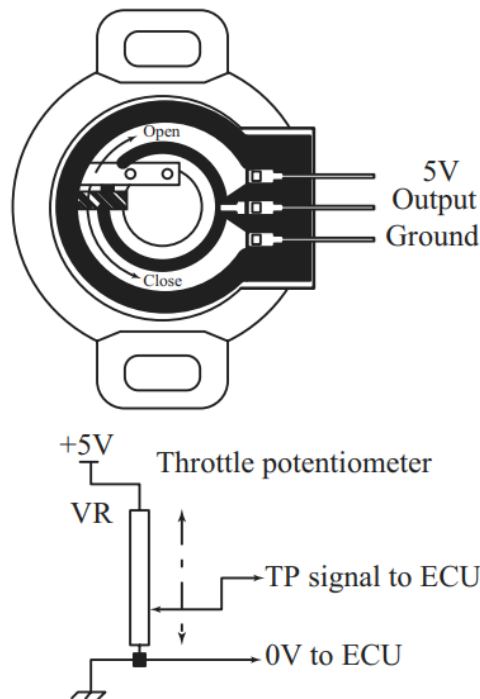


Figure 6: Throttle potentiometer (ftp.idu.ac.id)

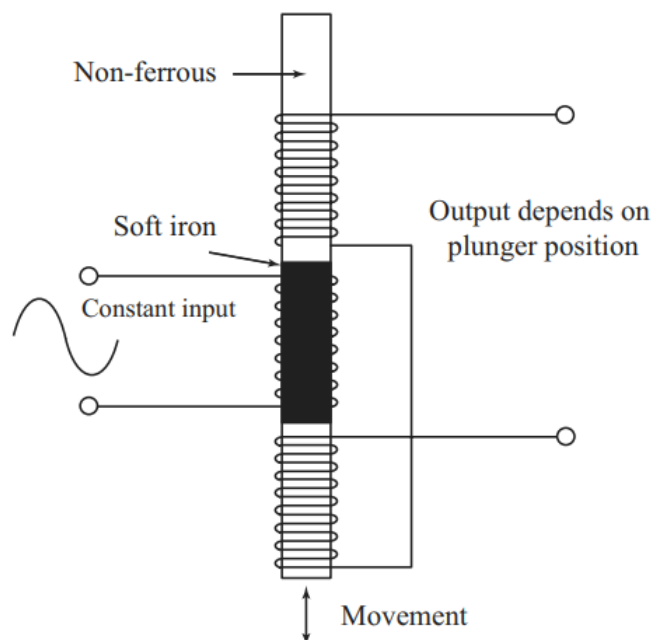


Figure 7: LVTD (ftp.idu.ac.id)

Electro Optical Sensors

It is possible to measure location and speed using electro-optical sensors. These gadgets may work at high frequencies, making it possible to detect moving things. Shaft encoders typically

contain two optical pickups, one of which provides a signal at predetermined crank angle intervals and the other of which provides one pulse per revolution to enable the detection of the absolute angle. To calculate the crank angle, incremental encoders gather pulses (Fig. 8). They cannot be utilised when the direction changes, though. Two crank angle-based pulses with a phase shift of $1/4$ th of a degree resolution can be employed in these circumstances. The direction may then be determined using the phase difference between the signals. Loss of power or noise signals can result in complete loss of position detection in all of these types of incremental encoders, at least temporarily. These issues are resolved by absolute encoders, which also have many outputs that are binary representations of the true location and tracks [8]–[10].

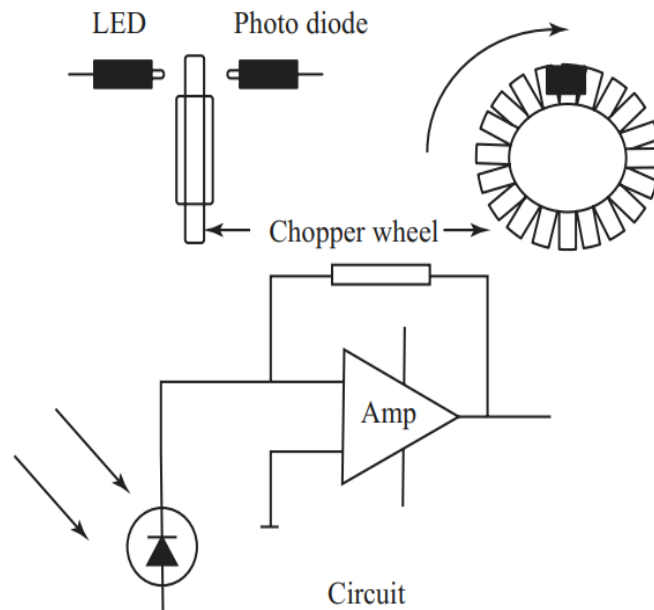


Figure 8: Photo detector (ftp.idu.ac.id)

CONCLUSION

Engine electronics is the integration of electronic devices and parts into internal combustion engines for the purpose of regulating and enhancing various operations. Engine management systems, sensors, actuators, ECUs, and communication networks are some of these systems. The development of technologies including cutting-edge sensor systems, artificial intelligence, machine learning, and networking options will shape the future of engine electronics. With the help of these advancements, engine performance can be controlled with greater accuracy, monitored in real-time, and adaptively optimised, further enhancing fuel economy, lowering emissions, and facilitating the incorporation of hybrid and electric powertrains. The two secondaries are linked in opposition and the primary is used for excitation. There won't be any output voltage in the core while it is at the mean or null position. The difference between the voltages generated in the two secondary windings is the output. A magnetic substance serves as the core. One of the windings' voltage changes as it is shifted, making it distinct from the other. According to the core's motion direction and in proportion to the deviation from zero, the output is an alternating voltage that is either in phase or out of phase with the input. To ascertain the size and direction of the object that is actually attached to the core, a phase sensitive detector is required. The output signal must be isolated because it is actually superimposed on the carrier signal. Demodulation is the name of this procedure.

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CHAPTER 18

EXPLORING THE ADVANTAGES OF SUPERCHARGING TECHNIQUE

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ABSTRACT

Internal combustion engines use supercharging as a technique to increase the volume of air entering the combustion chamber and consequently improve engine performance. An overview of supercharging and its importance in enhancing engine performance, torque, and overall engine efficiency is given in this abstract. It covers the many kinds of superchargers, including centrifugal and positive displacement superchargers, as well as their workings. High-performance engines require supercharging to enable increased power density, quicker throttle response, and superior performance at high altitudes. Compressing the incoming air before it enters the combustion chamber is the basic idea of supercharging. As a result, the air is denser, which increases the amount of fuel that can be burned and raises the amount of power generated. The restrictions imposed by air pressure are successfully removed by supercharging, allowing the engine to produce additional power.

KEYWORDS

Combustion, Engine, Performance, Supercharges Supercharging.

INTRODUCTION

Internal combustion engines frequently use supercharging to increase the amount of air fed to the combustion chamber, which improves performance. Supercharging raises the engine's power, torque, and overall efficiency by squeezing more air into it. Applications for this technology range from high-performance sports vehicles to heavy-duty trucks and even aviation engines. Positive displacement and centrifugal superchargers are the two main types of superchargers that are frequently employed. Positive displacement superchargers are renowned for producing immediate and significant low-end torque and deliver a fixed amount of air every revolution. They are frequently belt-driven and are most suited for situations where an immediate throttle response is necessary, like in drag racing or off-road vehicles[1]–[3].

On the other hand, centrifugal superchargers suck in and compress air using a compressor that is powered by a belt or gear drive. They are renowned for continuously producing more power as engine speed increases. High-performance street cars frequently use centrifugal superchargers because they provide a balance between power increases and efficiency. Internal combustion engines can benefit from supercharging in a variety of ways. A supercharger increases air density, which enables engines to burn more fuel and produce more power. Faster acceleration, increased towing capacity, and better overall vehicle performance are all impacted by the improved power-to-weight ratio. Supercharging also makes up for power loss at very high altitudes. The air is thinner as you go higher in height, which means there is less oxygen available for combustion. A supercharger guarantees that the engine receives a suitable amount of oxygen by compressing the air, ensuring engine performance and efficiency even at high altitudes. This is crucial for aeroplane engines because they fly at different heights.

Additionally, engine downsizing is made possible by supercharging, allowing smaller displacement engines to deliver power outputs that are comparable to bigger naturally aspirated engines. For automakers seeking to fulfil higher emission rules and increase fuel economy, this downsizing strategy offers increased fuel efficiency while retaining performance levels.

The potential of supercharging rests in the integration of electronic control systems as technology develops. This enhances performance and fuel economy by enabling accurate control and adjustment of boost pressure based on engine load and operating circumstances. The effectiveness and dependability of superchargers are further enhanced by material developments, such as lightweight and robust compressor components. In conclusion, supercharging is a useful technology for internal combustion engines since it increases overall engine efficiency and results in noticeable performance advantages. Supercharging is essential in many applications, from high-performance sports cars to heavy-duty trucks and aviation engines, due to its capacity to boost power output, torque, and make up for altitude-related power loss. The future of supercharging depends on the incorporation of electronic controls, which will further optimise performance and fuel efficiency in a variety of engine applications as technology develops. The primary goals of the engine designer will be to produce increased power output and minimal exhaust emissions. A naturally aspirated engine's power production is primarily influenced by the following five variables:

- (i) Amount of air inducted into the cylinder
- (ii) Extent of utilization of the inducted air.
- (iii) The speed of the engine.
- (iv) Quantity of fuel admitted and its combustion characteristics
- (v) Thermal efficiency of the engine.

Due to their interdependence and the potential need for significant adjustments to accomplish total combustion, the last two elements may not be the best approach. The engine's brake power is provided by

$$bp = (P_{bm} \times V_s \times n \times K) / 60000$$

In addition to the number of cylinders and engine cubic capacity, there are two more parameters in the expression that might boost power output. They consist of (i) n and p_{bm} . As a result, for an engine with a fixed cubic capacity, (V_s K), power output can be increased by speeding up (raising the number of power strokes). The speed will be constrained, nevertheless, as a result of rising friction. Increasing the mean effective pressure is the most recommended way to increase power production. By providing air or an air-fuel mixture at a pressure higher than atmospheric pressure, this can be accomplished. By increasing density, you can introduce more air or an air-fuel mixture for a given swept volume. The engine's power output will rise as a result. Supercharging is the process of providing air or a fuel-air combination through a boosting device at a pressure greater than the engine ordinarily aspirates. Supercharger is the name of the device that increases pressure.

Supercharging

Internal combustion engine supercharging has been used for a long time as a technique to increase engine power output. A new trend is emerging as the new millennium begins. In order to comply with pollution regulations regarding fuel economy and emission control, the trend points to tiny displacement engines. However, consumers continue to expect the same level of performance or perhaps higher.

Forced induction, also known as supercharging, is a suitable technique to address these needs. As was previously said, the goal of supercharging an engine is to increase the air charge's density before it enters the cylinders. As a result, more air will be injected into the system, where it will be compressed in each cylinder. Compared to the usual approach of spontaneously aspirating the new air charge into the cylinder, this increases the amount of oxygen available for burning. As a result, the engine will be pushed to burn more air and fuel per cycle, which can be burned efficiently to increase the engine's power output beyond what would otherwise be achievable [4]–[6].

The following are the important considerations while supercharging an engine:

- (i) Supercharging boosts an engine's power output. Per braking kW hour, the fuel consumption is not increased.
- (ii) When air is compressed, a certain amount of power is used. The engine itself must provide this power. There will be some power loss as a result. It is clear that the net power output will exceed that of an engine with the same capacity that isn't supercharged, though.
- (iii) The engine must be built to resist the increased forces brought on by supercharging.
- (iv) The fuel used needs to have higher antiknock properties since supercharging increases temperature and pressure, which could result in detonation.

Supercharging is a technique used in racing automobile engines. The following are the key areas where supercharging is crucial:

- (i) Engines in vehicles and ships must be light and compact.
- (ii) Engines operating at great heights. Supercharging can be used to make up for the power loss caused by elevation.

DISCUSSION

Types of Superchargers

The pressure-boosting device known as a supercharger delivers air (or a combination) at a higher pressure. It is typical to employ a centrifugal, axial flow, or displacement type compressor. Superchargers are referred to as mechanically driven if the engine crankshaft drives them. Some superchargers are powered by a gas turbine that draws energy from the exhaust gases of the engine. A turbocharger is one such supercharger. Three different types of superchargers exist [4]–[6].

- (i) Centrifugal type
- (ii) Root's type
- (iii) Vane type

Centrifugal Type Supercharger

In automobile engines, the centrifugal type supercharger is frequently employed (Fig. 1). The engine pulley's V-belt drives the supercharger. The air-fuel mixture first enters the core of the impeller. After that, it moves via the diffuser vanes and the impeller. In the end, air or a mixture enters the volute casing and exits from the casing into the engine. The mixture will be released under more pressure, and this state is referred to as a supercharged situation.

More air-fuel mixture is driven into the cylinder as a result of the higher pressure. It is possible to force about 30% more air-fuel combination into the combustion chamber. At

80,000 revolutions per minute, the impeller rotates quite quickly. The large strains generated by this speed should therefore be able to be handled by the impeller. Due of the tremendous stresses, impellers are typically composed of duralumin or alloy steels.

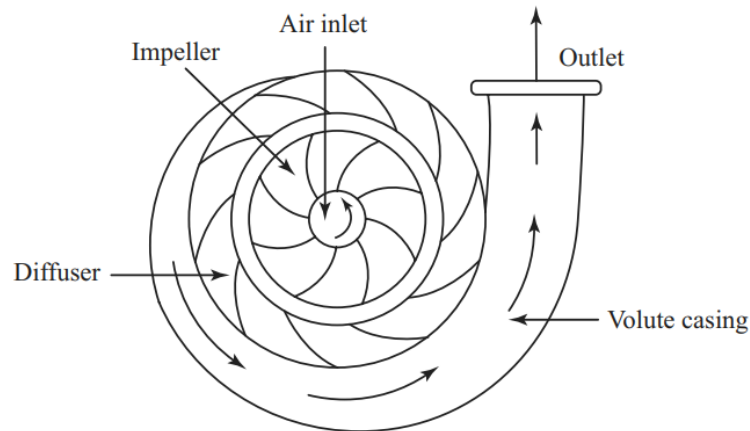


Figure 1: Centrifugal type of supercharger (ftp.idu.ac.id)

Root's Supercharger

Fig. 2 illustrates the Root supercharger in detail. The supercharger on the Root has two epicycloidal-shaped rotors that are each keyed to their respective shafts. Gears are used to connect one rotor to the other. Due to the identical size of the gears, both rotors spin at the same rate. The supercharger on The Root works like a gear pump. This supercharger's mixture will be under substantially greater pressure at the outlet than it was at the entrance.

Vane Type Supercharger

Figure. 3 depicts specifics of a common vane type supercharger. On the drum, which is located inside the body of the supercharger, are several vanes installed. Despite the spring's tension, the vanes might move in or out. The vanes are constantly in contact with the inner surface of the body as a result of this design. From the intake to the outer side, there is less gap between the inner surface of the body and the drum. In this manner, the volume of the mixture entering at the inlet drops, causing the pressure of the mixture to rise as it approaches the exit.

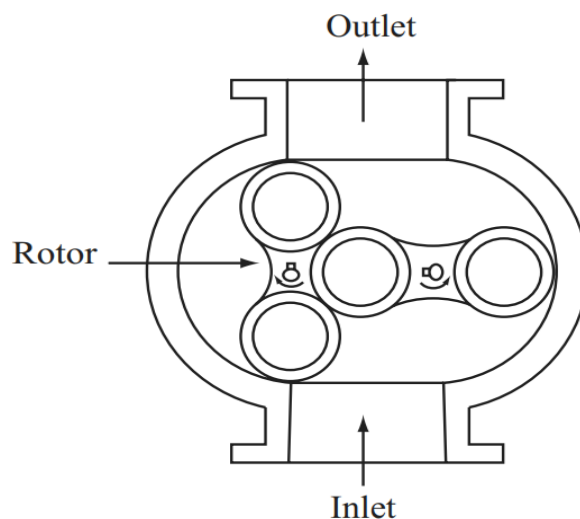


Figure 2: Root's supercharger (ftp.idu.ac.id)

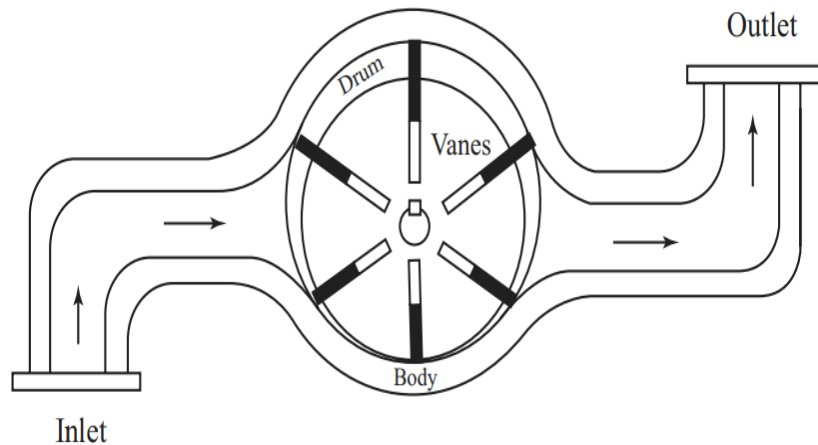


Figure 3: Vane type supercharger (ftp.idu.ac.id)

Comparison between the Three Superchargers

A centrifugal type supercharger has inadequate performance requirements and is only effective at low speeds. The supercharger on the root is easier to build, takes less maintenance, and lasts longer. A unique issue with the vane-type supercharger is the gradual deterioration of the vanes' tips. As a result, one must first consider the application before selecting the appropriate type of supercharger.

Methods of Supercharging

The engine can be fed with the necessary quantity of compressed air (or mixture) in the following methods.

- (i) Independently driven compressor or blower, usually driven by an electric motor.
- (ii) Ram effect.
- (iii) Under piston supercharging.
- (iv) Kadenacy system (applied to two stroke engines).
- (v) Engine driven compressor or blower

In the parts that follow, we'll briefly go through the specifics of the aforementioned five supercharging techniques. In the parts that follow, we'll go into more detail on the two other crucial superchargers, the gear-driven and exhaust-driven ones.

Electric Motor Driven Supercharging

In this design, an electric motor is typically used to operate the compressor independently. Because the supercharger's speed may be adjusted independently of engine speed, control is considerably simpler.

Ram Effect of Supercharging

Tuned inlet pipes make up the majority of the ram impact of supercharging system. Resonant harmonic air oscillations are produced by these pipes. These oscillations' kinetic energy produces a ramming effect. The engine speed needs to be consistent for this system to operate effectively.

Under Piston Supercharging

Large marine four-stroke engines of the crosshead type have been the sole exception to the under piston method of supercharging up to this point. The air is compressed using the

bottom side of the piston. The cylinder's bottom ends are sealed up and fitted with appropriate valves. Given that each suction stroke in the cycle is followed by two delivery strokes, this method provides an appropriate supply of compressed air.

Kadenacy System of Supercharging

The kadenacy system uses the exhaust system's energy to lower the pressure inside the cylinder. The scavenge air flows into the cylinder as a result of this depression. With this arrangement, a blower is an option but is not required.

The kadenacy system is based on the idea that when the exhaust ports or valves are opened quickly during the end of the expansion stroke, the gases have a need or impulse to depart the cylinder very quickly within the first few thousandths of a second. A pressure depression is left behind by the fleeing gases. By carefully timing the admission valve or ports, a new charge of air (or mixture) is now permitted to enter the cylinder following the exhaust gases. A proper timing and skilfully designed exhaust system are essential for the optimum outcome.

To achieve a decent-sized puff, the exhaust blow down needs to be quick. Exhaust ports outperform valves in this regard, and ports with square top edges perform better than those with rounded upper edges. 15 to 20 before the exhaust valve closes, the impulse must reach the concerned cylinder's exhaust duct at the proper time. This will depend on when the blow down occurs and how long the wave takes to travel from its origin. The exhaust pressure in this approach is low (even sub-atmospheric for the most of the scavenging duration). As a result, scavenging occurs against less resistance and low intake pressure is sufficient. Exhaust pulse supercharging is another name for this kind of supercharging.

Effects of Supercharging

One should be aware of the repercussions of supercharging an engine before doing so. The results of supercharging engines are as follows. CI engines are mentioned in a few of the points:

- (i) Increased power output
- (ii) More charge mass is induced.
- (iii) Improved fuel atomization
- (iv) Better fuel and air blending
- (v) Better product scavenging
- (vi) Improved torque characteristics across the whole speed spectrum
- (vii) Faster vehicle acceleration
- (viii) A more thorough and seamless combustion
- (ix) Use of fuel with inferior or inadequate ignition qualities
- (x) Improved efficiency and less inclination for diesel knock
- (xi) Increased likelihood of detonation in SI engines
- (xii) A better chilly start
- (xiii) Reduced emissions of smoke
- (xiv) Decreased specific fuel use during turbocharging
- (xv) Higher mechanical effectiveness
- (xvi) Additional thermal stresses
- (xvii) Greater heat losses as a result of more turbulent air
- (xviii) Higher gas loading
- (xix) Increased 60 to 160° crank angle valve overlap period
- (xx) Increased cooling needs for valves and pistons

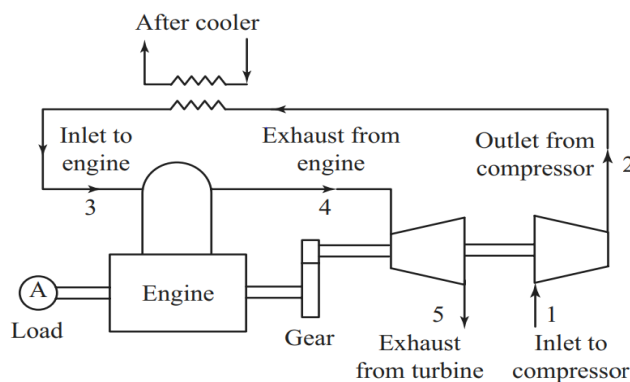
Supercharging Limitations

The heat load on the engine's numerous components increases as a result of supercharging. Some engines have a hollow chamber built into the piston crown. This area is cycled with oil or water, cooling the piston crown as a result. The piston crown, seat, and edges of the exhaust valves are typically constructed of better materials that can resist higher temperatures in some engines. The utilization of larger bearing surfaces and heavier engine parts is required due to the increased gas loading brought on by supercharging. If supercharging is to be implemented in an existing engine, one should first research the variables that restrict the amount of supercharging that can be tried. For the engine to be protected and to get the full benefits of supercharging, more modifications must be done. Below is a discussion of them:

The amount of supercharging that is permitted relies on the engine's capacity to resist the added thermal stress and gas loading. The key factors limiting an engine's level of supercharging are durability, dependability, and fuel efficiency. The piston crown, as well as the seat and edges of the exhaust valves, are more likely to burn as a result of the higher heat generation and heat transfer. In supercharged engines, the valve overlap is typically higher to address this issue. Depending on the crank travel, the valve overlap can range from roughly 80 to 160 degrees. More valve overlap extends the amount of time that cooled air can pass through the valves and the piston crown. The piston crown, the exhaust valves, and their seats are all cooled as a result[7]–[9].

Turbocharging

In turbocharging, a gas turbine that harnesses the energy in exhaust gases powers the supercharger. The supercharger and engine are not mechanically connected. Turbine wheel, turbine housing, turbo shaft, compressor wheel, compressor housing, and bearing housing are the key components of a turbocharger. Figures 5 and 6, respectively, illustrate the exhaust turbocharging principles of single-cylinder engines and Vee-type engines with charge cooling units. Hot exhaust gases from the engine exit via the exhaust valve opening and into the exhaust manifold while the engine is running. These gases enter the turbine housing via the exhaust manifold and connecting tubing. The turbine wheel's fins or blades are struck by the gases as they move through the turbine housing. The turbine wheel spins quickly when the engine load is high enough to allow for sufficient gas flow. The turbo shaft connects the compressor wheel to the turbine wheel. As a result, the turbine and compressor wheel both rotate, drawing air into the compressor housing. The air is thrown outward by centrifugal force. As a result, pressured air is forced into the engine cylinder by the turbocharger. Turbo lag is a phenomenon that occurs during turbocharging. It describes the brief interval before a surge in manifold pressure or both. This is because it takes the turbocharger assembly some time for the turbine and compressor wheel's turbine and compressor gas to accelerate.



(a)

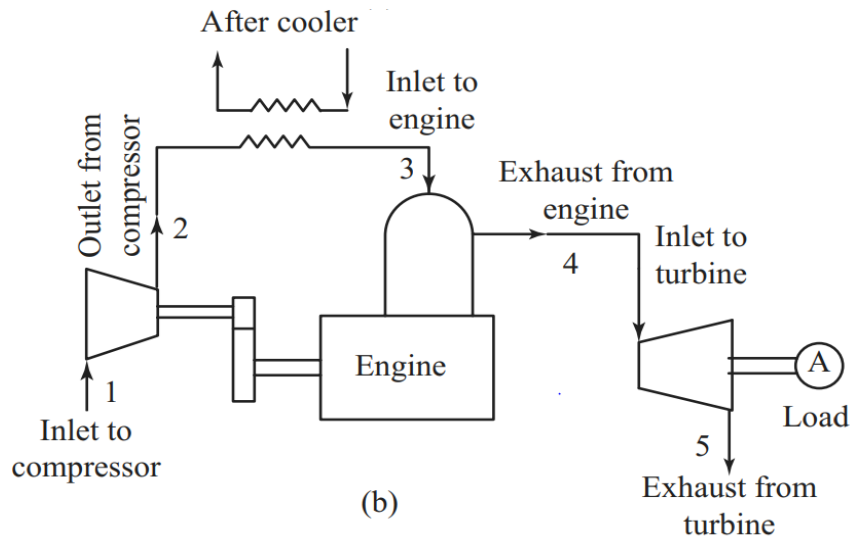


Figure 4: Methods of supercharging (ftp.idu.ac.id)

A waste gate control unit is a part of the turbocharger assembly. In order to avoid engine damage and SI engine detonation, this unit restricts the maximum boost pressure.

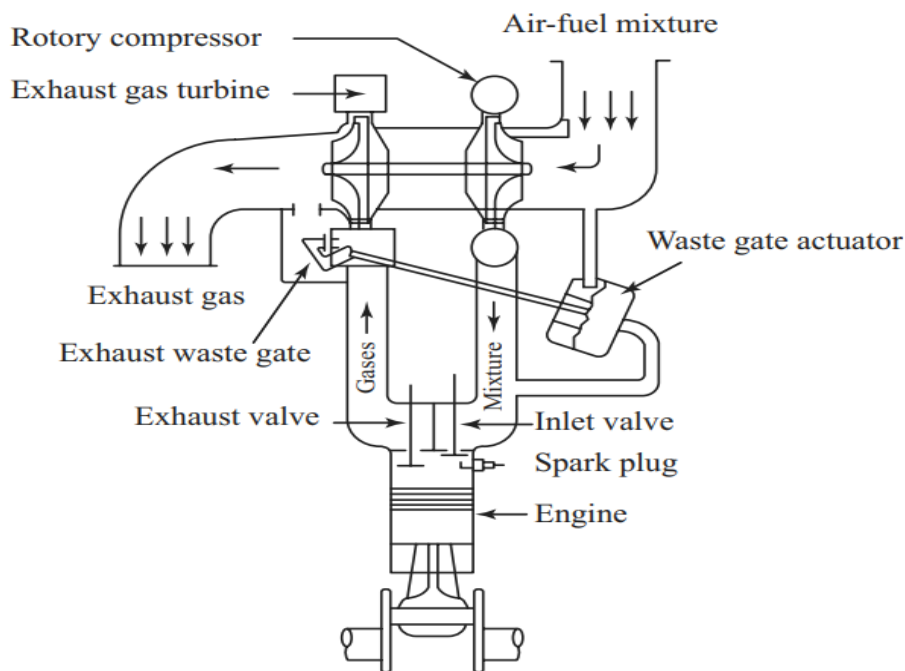


Figure 5: Principles of exhaust turbocharging of a single-cylinder engine (ftp.idu.ac.id)

When the manifold pressure is fairly high, this diaphragm-operated valve can bypass some of the gases near the turbine wheel. The waste gate solenoid, a microprocessor, and engine sensors are all components of the computer-controlled turbocharging system. The waste gate can open or close when the solenoid is powered on or off by the computer. The boost pressure can be precisely regulated in this manner.

Gear Driven and Exhaust Driven Supercharging Arrangements

In Figures 7 and 4, two primary categories of supercharging setups are depicted. The compressor is linked to the engine in Figure 7(a) using step-up gearing to accelerate the compressor's spin. In this instance, a portion of the engine's power is used to power the compressor. Subtracting this power from the engine's gross output yields the supercharging-

related increase in output. Additionally, an after cooler is displayed, via which cold air can optionally be delivered to the engine. The density of the intake air will rise much more as a result. An engine with a compressor operated by the free exhaust is shown in Figure 7(b). The engines with this equipment are allegedly turbo-supercharged. In this instance, a turbine connected to a compressor is driven by the engine's exhaust energy. The compressor or turbine are not mechanically connected to the engine. However, the turbine's inlet is joined to the engine's exhaust pipe. In this instance, the compressor is not driven by the engine's output.

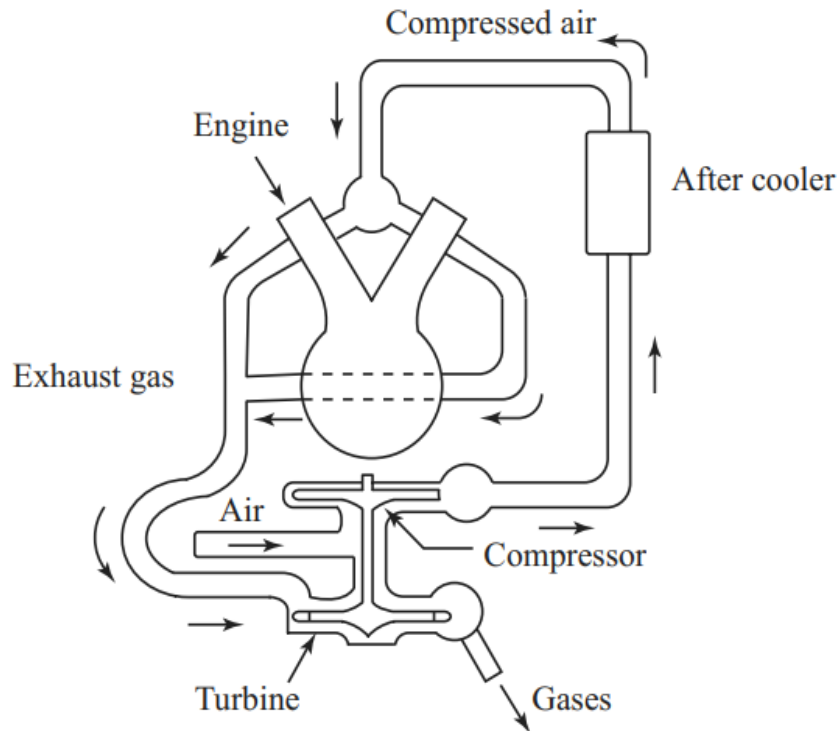
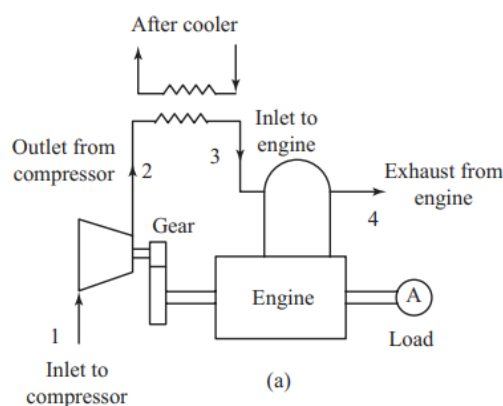
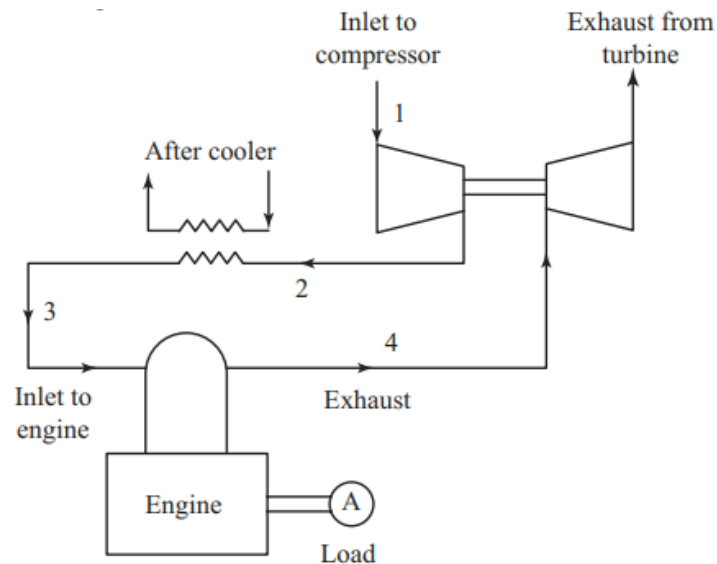


Figure 6: Exhaust turbocharging of a "V" type engine (ftp.idu.ac.id)

Compressor, engine, and turbine are all geared together in Fig. 4(a). An illustration is the Wright Turbo-compound jet engine. In this scenario, the engine power takes care of the remaining load on the compressor if the turbine output is insufficient to run it, especially at part loads. Additionally, the engine can receive the extra power generated by the turbine. The setup for a gas generator is depicted in Figure 4(b). In this scenario, the engine merely powers the compressor. The engine is fed air by the compressor, and the exhaust gases operate a power turbine. This idea underlies the majority of free-piston engines [6]. Heat is produced when air charge is compressed. The temperature of the air charge is substantially higher than the temperature of the surrounding air when it exits the compressor.





(b)

Figure 7: Supercharging arrangements (ftp.idu.ac.id)

CONCLUSION

Supercharging is the process of putting extra air into the combustion chamber to boost an internal combustion engine's power and performance. It often includes utilizing a turbine or a compressor to raise air pressure and density, which boosts engine performance and horsepower. Future developments in electric supercharging, hybrid systems, and more efficient compressor designs are anticipated to result in even greater power gains and lower emissions for both conventional and electric vehicles. Supercharging technology is also anticipated to continue advancing. The temperature of the air rises from 60 to 95 C during supercharging. As air warms up, it expands and loses density. The mass of air entering the cylinder decreases as a result. As a result, there is less oxygen available for combustion in the cylinder. Additionally, supplying hot air to the engine may raise the operating temperature of the engine. In order to solve these issues, charge is cooled using intercooling and/or after cooling. But this increases the system's complexity.

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CHAPTER 19

AN ANALYSIS OF TWO-STROKE ENGINES

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ABSTRACT

Two-stroke internal combustion engines, as opposed to four-stroke engines, may complete a power cycle with just two piston strokes. The advantages, disadvantages, and general operation of two-stroke engines are discussed in this study. The special scavenging method, port configurations, and lubrication system utilized in two-stroke engines are all covered. The worries about emissions and prospective gains in effectiveness and environmental performance due to developments in fuel injection, direct injection, and exhaust after-treatment technologies are also explored. The use of the piston as a sliding valve in conjunction with intake and exhaust ports drilled into the side of the cylinder substantially simplifies the mechanical design of many two-stroke engines. In contrast to the diesel or dual combustion cycles, where the charge is pure air, the Otto cycle engine's charge contains the proper ratio of fuel and air. Numerous obstacles must be overcome in order to apply the two-stroke principle to compact, high-speed CI engines, although they are by no means insurmountable.

KEYWORDS

Combustion, Engine, Four Stroke, Injection Two-Stroke.

INTRODUCTION

A two-stroke engine is one that completes its operational cycle in either two piston strokes or one crankshaft revolution. The incoming fresh charge, which is compressed either in the crankcase or through a separate blower when the engine piston approaches bottom dead center, handles the functions of the four-stroke engine's intake and exhaust processes in this engine. The new charge must only be compressed and the combustion products must be expanded by the engine piston. Theoretically, a two-stroke engine will produce twice as much power when running at the same mean effective pressure since it will have twice as many cycles per minute as a four-stroke engine working at the same speed and with the same number of cylinders. The amount of air available for combustion determines how much power this engine produces, just like it does with four-stroke engines, in kilograms per minute [1]–[3]. When it comes to SI engines, the two-stroke cycle principle has been used in a wide range of engines, from tiny single-cylinder model engines that produce just a few milliwatts to the largest aircraft engines that produce 2500 kW or more.

Two-stroke engines stand out as outstanding powerhouses in the large field of internal combustion engines and have established themselves in numerous sectors. These engines have come to be associated with effectiveness and performance due to their well-known lightweight design, high power-to-weight ratio, and straightforward assembly. We will examine the fundamentals of two-stroke engines, their historical relevance, their uses across numerous sectors, and their key advantages and limits in this succinct yet thorough introduction. Come along as we unlock the mysteries of these nimble and powerful mechanical wonders.

Historical Relevance

We must first acknowledge the historical context of two-stroke engines in order to comprehend their core. The idea of the two-stroke engine first appeared in the 19th century in response to the demand for more effective internal combustion engines. It was pioneers like Sir Dugald Clerk and Joseph Day who introduced the simplified operational cycle that would transform the engine industry and established the foundation for this technology. These engines have since been improved by ongoing research, making it possible for them to be used widely today.

Effectiveness and simplicity

A two-stroke engine's intrinsic simplicity is what makes it work. Two-stroke engines minimize the mechanical complexity by completing a combustion cycle in just two strokes of the piston, unlike its four-stroke counterparts. We'll quickly go through the core ideas behind these engines, focusing on how well-streamlined their intake, compression, combustion, and exhaust systems work. Two-stroke engines maximize power output while minimizing size and weight by optimizing these processes.

Applications in Different Industries

Due to its adaptability, two-stroke engines are commonplace in a wide range of industries. These engines have the benefit of being lightweight and simple to use, which makes them ideal for small-scale applications like handheld tools and gardening equipment. Two-stroke engines offer dependable and effective power for a variety of outdoor jobs, from chainsaws to leaf blowers and hedge trimmers. The maritime sector has also embraced two-stroke engines, using their high power-to-weight ratio to move boats and other watercraft, allowing for exhilarating speed and agility on the sea. The power and acceleration provided by these engines are also vital to the world of high-performance motorbikes, delivering an exhilarating riding experience.

Advantages and Limitations

Two-stroke engines provide a lot of benefits, but they also have some drawbacks. Their small size and few moving components result in cheaper production costs and simpler maintenance. They are also perfect for applications needing a lightweight but powerful engine due to their high power output. However, these engines often struggle with issues including pollution, fuel use, and oil-fuel mixes. These considerations demand careful thought when selecting the right settings for the use of two-stroke engines, as well as the constant search for improvements to lessen their restrictions.

In conclusion, two-stroke engines have solidified their position as an effective, potent, and adaptable subset of internal combustion engines. These engines continue to have a considerable impact on a number of sectors, both in their historical uses and in their contemporary ones. Their popularity in small-scale equipment, maritime boats, and high-performance motorbikes is due to their simplicity and outstanding power-to-weight ratio. We can use their potential while pursuing greater innovation if we are aware of their benefits and limits. Two-stroke engines continue to stand as a lasting example of the balance of power and simplicity as we traverse the always changing terrain of engine technology.

Classification According To the Scavenging Process

Crankcase compression, is the simplest technique for putting the charge into the cylinder. The term "crankcase scavenged engine" refers to this kind of engine. Another form of engine is

referred to as a separately scavenged engine since the charge is introduced via the intake port using a separate blower or pump.

Based on air flow, two-stroke cycle engines may also be categorized. Cross scavenging is the most typical configuration, as seen in Fig.1 (a). The exhaust gases are forced out via the oppositely situated exhaust ports by forcing the entering air upward and subsequently downward.

When using loop or reverse scavenging, the new air first passes over the top of the piston before moving upward, downward, and eventually out the exhaust. The exhaust and inlet ports are on the same side, with the exhaust being higher than the inlet, in the M.A.N. type of loop scavenging (Fig. 1 (b)). The ports are side by side in the Schnuerle type (Fig. 1 (c)). The Schnuerle kind of scavenging is identical to the Curtis style of scavenging, Fig. 1 (d), with the exception that upwards directed input apertures are also arranged opposite the exhaust ports.

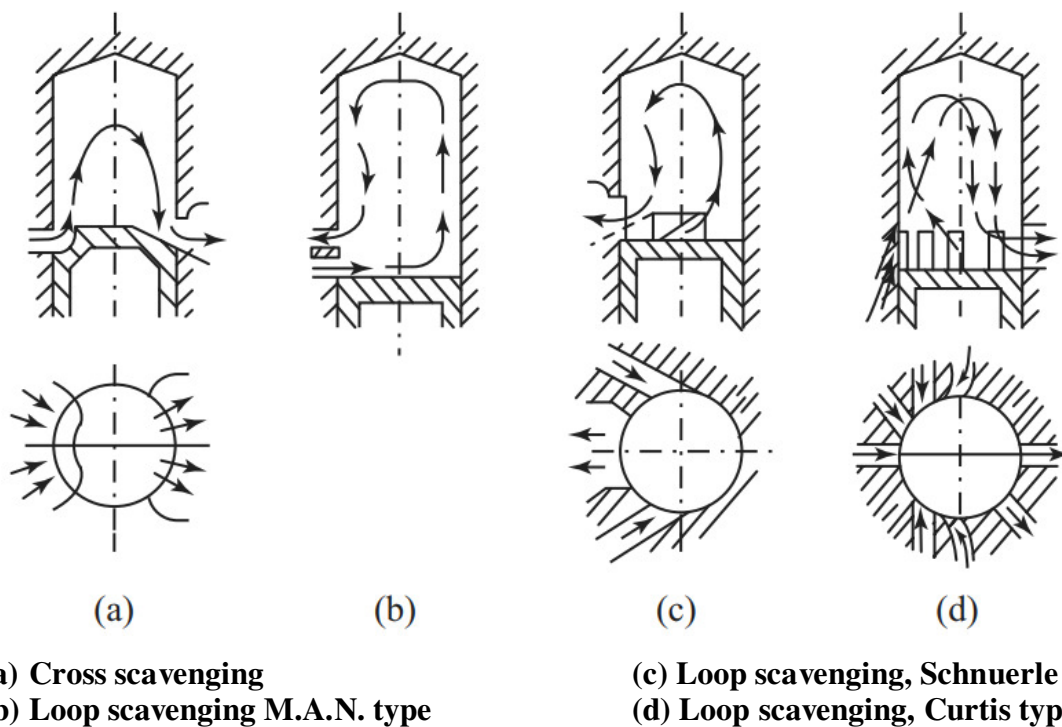


Figure 1: Methods of scavenging (ftp.idu.ac.id)

The uniflow approach, in which the fresh air charge is introduced at one end of the cylinder and the exhaust departs at the other, is the best technique for scavenging. There is minimal chance of short-circuit between the intake and exhaust apertures since the air flow is from end to end. Fig. 2 shows the three possible configurations for uniflow scavenging. A poppet valve is used in to allow input air or exhaust air, depending on the situation.

In b, distinct pistons that move in opposing directions each regulate the intake and exhaust ports. In the case of C, the combined motion of the piston and sleeve controls the intake and exhaust ports. An alternate configuration controls one set of ports with a piston and the other set with a sleeve or slide valve. All uniflow systems allow for asymmetric supercharging and scavenging. In Fig.3, reverse flow scavenging is seen.

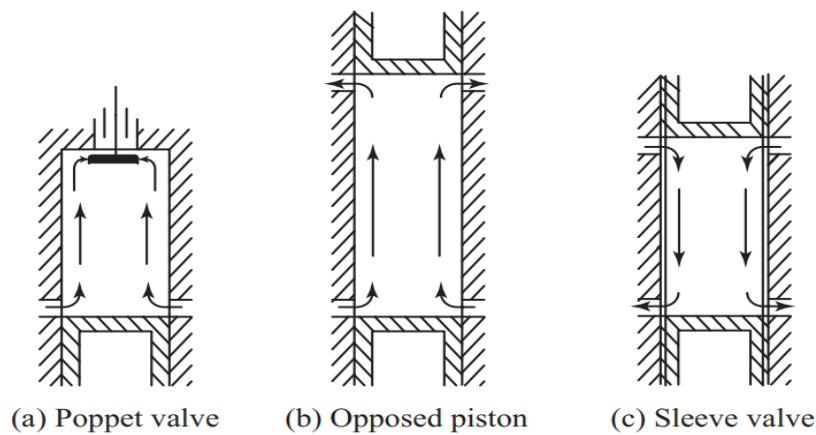


Figure 2: Uniflow scavenging (ftp.idu.ac.id)

In this design, inclined ports are employed, and scavenging air is pushed against the cylinder's opposing wall before being propelled back towards the outlet ports. The restriction on the port area is one clear drawback of this kind. This configuration has shown to be effective for long stroke engines running at slow piston speeds.

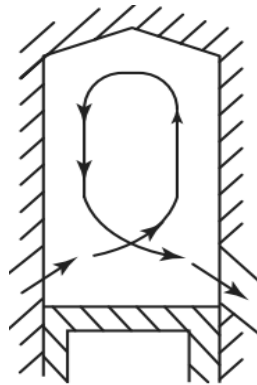


Figure 3: Reverse flow scavenging (ftp.idu.ac.id)

DISCUSSION

Types of Two-Stroke Engines

There are essentially two kinds of two-stroke engines, depending on the scavenging technique employed:

- (i) crankcase scavenged engine
- (ii) separately scavenged engine

The following sections provide a short discussion of the specifics of the aforementioned two kinds of two-stroke engines.

Crankcase Scavenged Engine

Fig.4 depicts one of the most basic two-stroke engine designs. In this engine, the underside of the piston compresses the charge (fuel-air mixture in SI engines and air in CI engines) in the crankcase during the expansion stroke. This engine has three ports.

- i. intake port at the crankcase
- ii. transfer port
- iii. exhaust port

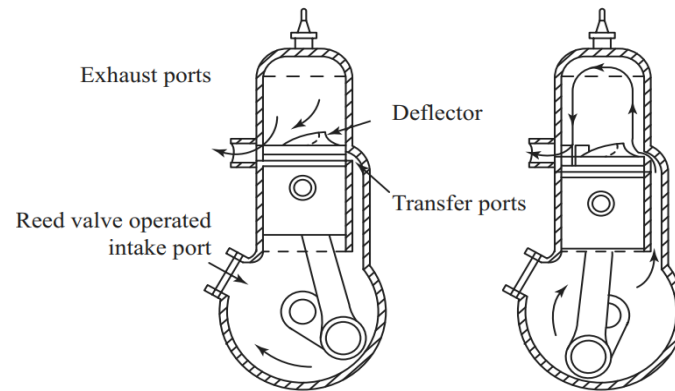


Figure 4: crankcases-scavenged two-stroke engine (ftp.idu.ac.id)

The compressed charge flushes the combustion products out of the engine cylinder as it enters via the transfer port. Scavenging is what this procedure is known as, and these engines are referred to as crankcase-scavenged engines. The exhaust ports are initially seen when the piston descends, and as the combustion products leave via these ports, the cylinder pressure decreases to ambient pressure. Additionally, when the piston descends, the transfer ports become visible, allowing the engine cylinder to be filled with either air or a slightly compressed mixture from the crankcase, depending on the kind of engine.

Typically, the ports and the top of the piston are designed such that the fresh air flows towards the top of the cylinder before reaching the exhaust ports. This serves to both minimise the flow of new fuel-air combination via the exhaust ports and scavenge the top portion of the cylinder of combustion by products. The deflector is a protrusion on the piston. The transfer ports and then the exhaust ports shut as the piston moves back towards bottom centre, starting compression of the charge. The fresh mixture or air is sucked into the crankcase via the intake reed valve as a result of the motion of the piston during compression, which decreases the pressure in the crankcase. The regular process of ignition and expansion occurs, and the cycle is repeated. Less than one cylinder's worth of displacement volume is charged into the engine because to flow restrictions in the transfer ports and the intake reed valve [4]–[6].

Separately Scavenged Engine

The term "externally" or "separately scavenged engine" refers to another kind of engine that scavenges combustion products using an external mechanism, such as a blower. Fig.5 depicts this sort of engine's specifics. These engines' intake manifolds receive air-fuel mixture at a little greater pressure. The exhaust ports are unveiled by the piston as it descends on the expansion stroke, around 60° before bottom centre. When the cylinder pressure has been significantly lowered, about 10 seconds later, the intake ports are exposed, and the scavenging process begins. Most of the air travels to the top of the cylinder on the intake side of the cylinder and moves down on the exhaust side, producing a loop (Fig. 5), before reaching the exhaust ports because of the geometry of the inlet ports (Fig. This guarantees that the top cylinder volume is scavenged. Because they are hefty and have a tendency to overheat at high power, piston deflectors are not employed. In comparison to the typical crankcase scavenged engine with deflector piston, the scavenging process is more effective in these sorts of engines.

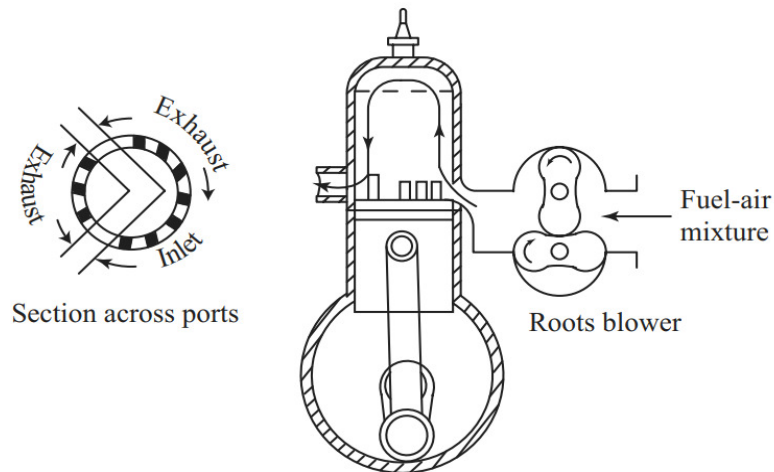


Figure 5: Loop-scavenged two-stroke engine (ftp.idu.ac.id)

Opposed Piston or End-to-End Scavenged Engine

The opposing piston kind of two-stroke engine, sometimes known as an end-to-end scavenged engine, is another form. Fig.6 depicts this sort of engine's specifics. The exhaust vents are opened first in this kind. The air or mixture enters tangentially via intake apertures that are designed to create a swirling motion (Fig. 6). During the scavenging phase, the swirl assists to avoid mixing of the new charge with combustion products. Exhaust ports shut first during the first phase of the compression stroke. The cylinder pressure rises as a result of the flow through the inlet port, and the inlet ports shut when the pressure is about equal to that of the inlet manifold. It should be observed that two pistons in this opposed piston engine are somewhat out of phase (Fig. 6). Because of the symmetrical port timing of the loop-scavenged engine, the exhaust port must shut after the intake port does. This sort of engine can't fill to full intake pressure because of the timing. End-to-end scavenged engines reduce the chance of fresh and waste gases mixing by eliminating counter flow inside the cylinder. Therefore, the scavenging ought to be more effective.

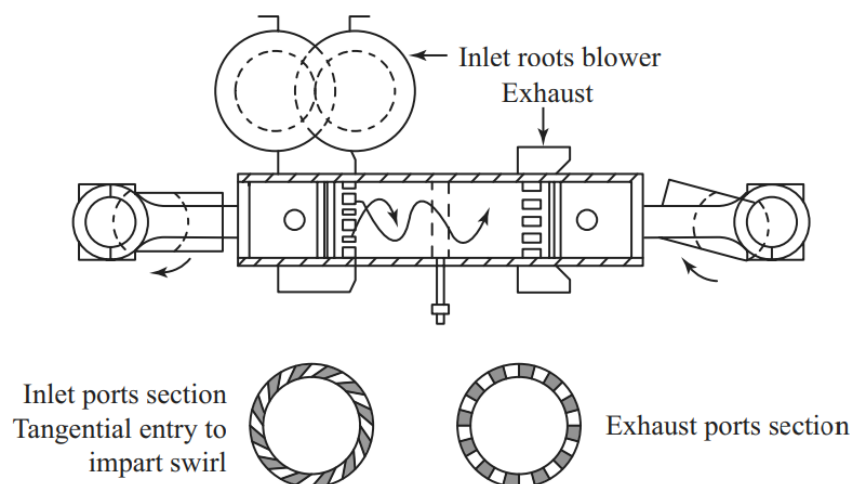


Figure 6: Port arrangement of a two-stroke end-to-end scavenged compression-ignition engine (ftp.idu.ac.id)

Actual Scavenging Process

The real scavenging procedure is neither an ideal scavenging nor an ideal mixing procedure. It presumably includes some perfect mixing, scavenging, and short circuiting. For three

distinct scavenging modes, namely perfect scavenging, perfect mixing, and intermediate scavenging, Fig. 7 illustrates the modification of delivery ratio and trapping efficiency with crank angle. Fig. 8 depicts the intermediate scavenging's scavenging parameters. This is an illustration of the scavenging procedure. This diagram demonstrates how certain combustion products are originally forced out of the cylinder without being mixed with new air.

The outflowing products are gradually diluted by more and more fresh air as a result of mixing and short circuiting, until ultimately the situation is the same as for perfect mixing, meaning that the initial stage of the scavenging process is a perfect scavenging process that gradually transforms into a complete mixing process.

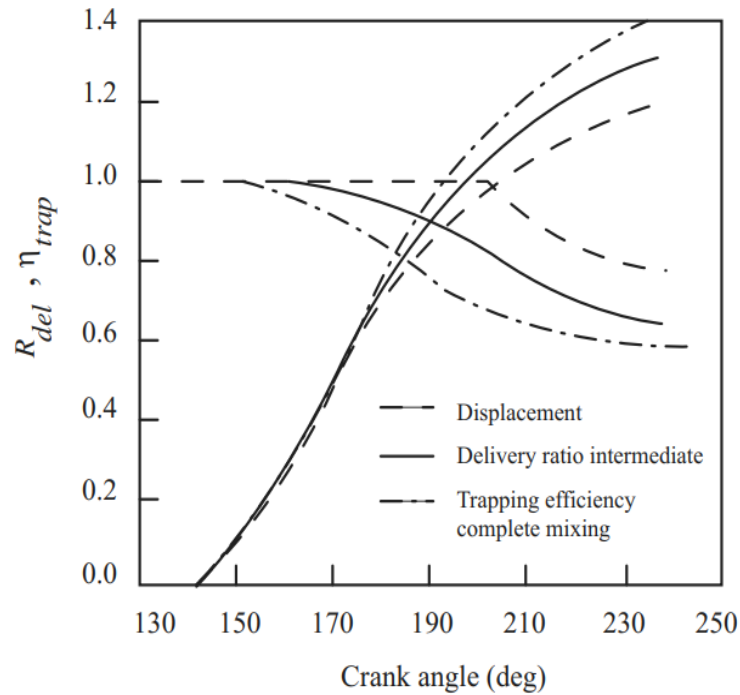


Figure 7: Delivery ratio and trapping efficiency variation for a crankcase scavenged engine (different scavenging modes) (ftp.idu.ac.id)

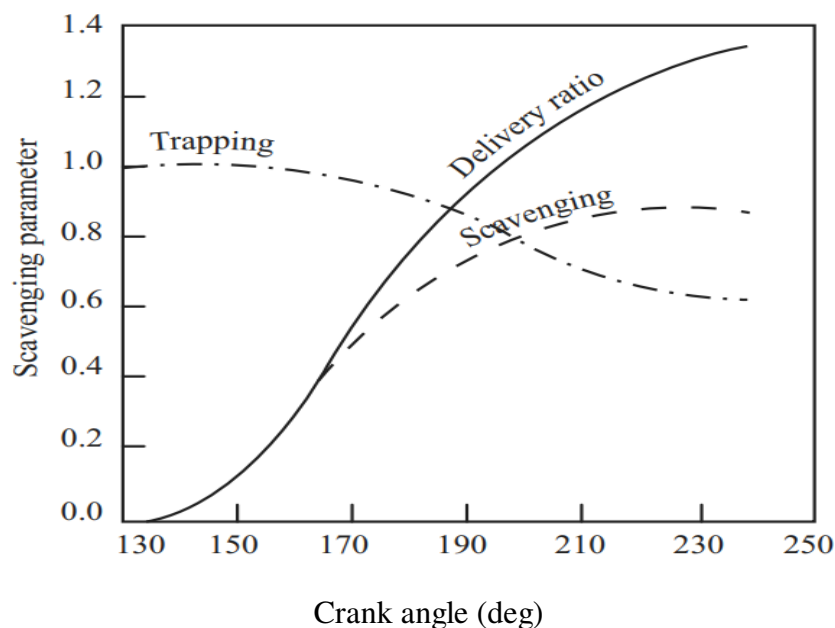


Figure 8: Scavenging parameters for the intermediate scavenging (ftp.idu.ac.id)

Comparison of Scavenging Methods

Fig. 9 presents an intriguing comparison of the advantages of several air scavenging techniques for two cycle engines. In actuality, the effectiveness of the scavenging system and its relationship to the brake mean effective pressure directly influence the engine's particular output. Scavenging efficiency varies depending on the delivery ratio and the method of scavenging, as illustrated in Fig. 9. Cross scavenging is the least effective method and results in the lowest brake mean effective pressure in this regard. This is mostly due to the fact that the scavenging air passes through the cylinder but ineffectively removes the exhaust residual gases. Cross scavenging is inferior to the loop scavenging approach. Scavenging efficiencies for cross scavenging, loop scavenging, and uniflow scavenging systems are around 53, 67, and 80 percent in all circumstances, with corresponding values of bmep as 3.5, 4.5, and 5.8 bar.

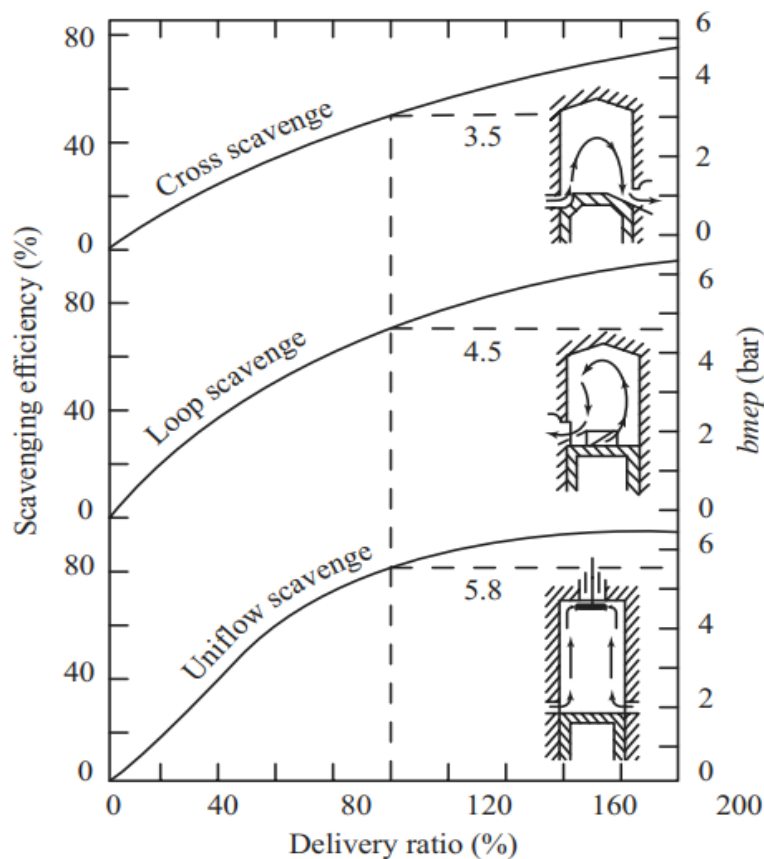


Figure 9: Scavenging efficiency (ftp.idu.ac.id)

Pros and cons of Two-Stroke Engines

Compared to four-stroke engines, two-stroke engines offer several benefits as well as downsides. Brief discussions of the key benefits and drawbacks are provided in the sections that follow.

(a) Pros of Two-stroke Engines

- (i) The power generated will be approximately double that of a four-stroke engine running at the same speed and with the same size since there is a working stroke for every revolution.

- (ii) The effort needed to reduce the exhaust and suction strokes' friction is reduced.
- (iii) Every revolution includes a working stroke, which results in a more uniform turning moment on the crankshaft and necessitates a lighter flywheel.
- (iv) For the same power output and speed, two-stroke engines are lighter than four-stroke engines.
- (v) The space used by two-stroke engines is smaller for the same power.
- (vi) Because ports are used in place of valves, two-stroke cycle engines are easy to build. As a result, fewer maintenance issues arise.
- (vii) Burnt gases do not accumulate in the clearing space as much in two-stroke engines as they do in four-stroke engines due of scavenging.

(b) Cons of Two-Stroke Engines

- (i) Due to the decreased volumetric efficiency, high speed two-stroke engines are less efficient.
- (ii) Otto cycle engines lose some of the new mixture during scavenging as it exits via the exhaust port. This increases fuel use and decreases thermal efficiency.
- (iii) Due to the ports' loss of a portion of the piston stroke, two-stroke engines' effective compression is lower.
- (iv) Two-stroke engines are likely to need more lubricating oil than four-stroke engines.
- (v) Two-stroke engines get overheated with large loads because of the extra heat generated. Due to the higher charge dilution, the engine's performance is also not particularly smooth at low loads.

A Comparison of Two-Stroke SI and CI Engines

The two-stroke SI engine has two significant drawbacks: fuel loss and troublesome idle. Since the two-stroke CI engine is not affected by these drawbacks, it is better suited for two-stroke operation. There won't be any fuel loss if gasoline is added to the cylinders after the exhaust ports are shut, and the two-stroke engine's suggested thermal efficiency will be on par with four-stroke engines. The scavenging is carried out using a fuel-air combination in a SI engine using a carburettor, and only the fuel combined with the retained air is utilised for combustion. Instead of using a carburetor shortly before the exhaust port closes, fuel injection may be utilised to prevent fuel loss [7]–[9].

CONCLUSION

Two strokes of the piston are all it takes for a power cycle to be completed in a two-stroke engine, which has a high power-to-weight ratio and a simple design. Future-oriented initiatives to enhance two-stroke engines are being driven by developments in technology and rising environmental concerns. Direct fuel injection, electronic controls, and enhanced pollution control systems are examples of innovations that are being used to increase efficiency, lower emissions, and broaden the uses of tiny engines in things like power tools, recreational vehicles, and transportation. The continuing development of two-stroke engines towards improved performance, fuel economy, and environmental friendliness is what will determine their future. When mean effective pressure is decreased to around 1.2 bar, the two-stroke SI engine begins to accelerate slowly and may even come to a complete halt at low speeds. This is as a result of a significant residual gas mixture (more so than in a four-stroke engine). Due to the slow combustion rate, there might be backfiring at low speeds. Fuel injection reduces backfiring and improves idling since there is no fuel in the intake system.

Because the charge in CI engines is merely air, there is no fuel loss, and there are no idling issues because the fresh charge (air) is not diminished.

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CHAPTER 20

NON-CONVENTIONAL ENGINES

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ABSTRACT

Nonconventional engines refer to alternative types of power plants that deviate from traditional internal combustion engines. These engines utilize innovative technologies and fuel sources to provide efficient and environmentally friendly solutions for various applications. This abstract provides a brief overview of nonconventional engines, highlighting their key characteristics, advantages, and potential future prospects. By exploring the advancements in this field, we aim to shed light on the promising future of nonconventional engines as viable alternatives in the realm of transportation and power generation. Through unconventional engines, the transportation industry is likewise going through a transformational wave. As more and more people choose environmentally friendly alternatives to internal combustion engines, electric vehicles (EVs) powered by cutting-edge battery technologies or fuel cells are becoming more and more common. Reduced reliance on fossil fuels, fewer emissions, and increased energy efficiency are all benefits of these unconventional power sources. Hybrid technology developments, which combine conventional and nonconventional engines, also provide a path to a cleaner and more fuel-efficient future for transportation.

KEYWORDS

Future Prospects, Innovative Technologies, Nonconventional Engines, Power Generation, Transportation.

INTRODUCTION

We've seen the specifics of the traditional engines up to this point, namely the Spark Ignition (SI) and Compression Ignition (CI) engines. Traditionally, CI engines utilize mechanical injection whereas SI engines use carburetion. Many studies have been conducted to enhance the performance of these engines during the last few decades. Numerous unconventional engines have been created as a result of ongoing work. This chapter will provide a thorough examination of a few nontraditional engines. In alphabetical order, the usual nonconventional engines are:

- (i) Common Rail Direct Injection (CRDI) engine
- (ii) Dual fuel and multi-fuel engine
- (iii) Free piston engine
- (iv) Gasoline Direct Injection (GDI) engine
- (v) Homogeneous Charge Compression Ignition (HCCI) engine
- (vi) Lean burn engine
- (vii) Stirling engine
- (viii) Stratified charge engine
- (ix) Variable Compression Ratio (VCR) engine
- (x) Wankel engine

Nonconventional engines have become a game-changing alternative to conventional internal combustion engines in the search for efficient and sustainable energy solutions. By using a

variety of technologies and fuel sources, these cutting-edge power plants vary from accepted standards in order to achieve better efficiency, a smaller environmental effect, and improved performance. We will examine the world of nonconventional engines, their relevance in power production and transportation, their fundamental characteristics, and the bright future possibilities they hold in this thorough but succinct introduction. Come along with us as we explore the world of alternative energy sources.

Rethinking Power Generation

A paradigm change in the field of power production has been brought about by unconventional engines. These engines transform numerous energy sources into electricity by using cutting-edge technology including fuel cells, gas turbines, and Sterling engines. Nonconventional engines provide cleaner and more environmentally friendly alternatives for producing electricity by using the potential of renewable energy, waste heat, or other fuels. Nonconventional engines are altering the energy environment, from stationary usage in residential and industrial settings to mobility applications in hybrid and electric cars.

Key Features and Advantages

A number of distinguishing qualities distinguish unconventional engines from their conventional equivalents. First off, since they employ cutting-edge technology and alternative fuels, they have a less environmental impact because they produce fewer greenhouse gases and other pollutants. Second, these engines often have superior energy efficiency, allowing for better resource utilization and less energy waste. Additionally, non-conventional engines provide the adaptability and resilience to shifting energy landscapes by enabling the flexibility to harness a variety of energy sources, including solar, wind, hydrogen, biofuels, and more.

Future Prospects

Non-conventional engines have a bright and hopeful future. The creation and use of unconventional engines are expected to increase as environmental concerns and the need for sustainable energy sources become more pressing. Technological developments are projected, including greater energy storage systems, higher efficiency, and increased affordability. In order to ensure a sustainable and resilient future, continuing research and development activities are concentrated on optimizing non-conventional engines for a variety of applications, including grid-scale power production, long-range electric cars, and decentralized energy systems.

When it comes to power production and transportation, non-conventional engines are a game-changer, providing creative solutions that put sustainability, efficiency, and minimal environmental effect first. These engines are accelerating the shift to a cleaner and greener future by challenging conventional conventions and embracing a variety of technology and fuel sources. Nonconventional engines have the ability to revolutionise how we produce electricity and transport people and products, paving the way for a more sustainable and energy-conscious society with their essential characteristics, benefits, and bright future possibilities [1], [2].

DISCUSSION

Common Rail Direct Injection Engine

Traditional SI and CI engines run on diesel and petrol, respectively. Diesel fuel has an issue since its particles are heavier and bigger than those of petrol. As a result, atomization is significantly more challenging. More unburned particles result from inefficient atomization,

which increases pollution, reduces fuel economy, and reduces power. High pressure fuel injection systems for both petrol and diesel engines have been developed to address these issues. CRDI engines are those that use this technology. CRDI is better suited for diesel engines, however, since common-rail technology aims to enhance atomization by the use of very high (up to 1800 bar) injection pressures. Diesel engines with conventional DI produce fuel pressure (about 300 bar) repeatedly with each injection. On the other hand, with CRDI engines, the pressure is continuously accessible in the fuel line and is built up irrespective of the injection sequence. The CRDI system provides each cylinder with real-time combustion data using the relevant sensors.

An accumulator is provided by the common rail, which is situated upstream of the cylinders. At a steady pressure of up to 1800 bar, it delivers the fuel to the injectors. It uses high-speed solenoid valves that an electronic engine management system controls. In accordance with the cylinder's real requirements, they independently regulate the time and volume of fuel to be injected. In other words, fuel injection and pressure generation operate independently of one another. Common-rail injection has this significant benefit over traditional injection.

Common rail direct injection enhances fuel atomization and the controllability of the separate injection processes. This decreases pollution while also conserving petrol. According to reports, fuel efficiency improvements over a traditional diesel engine of around 30% are achievable. A better synchronised time operation also results in a significant noise reduction. The CRDI concept may also be used in gasoline engines as Gasoline Direct Injection, which largely eliminates the disadvantages of the multi-point fuel injection system and traditional carburetors[3], [4].

The Working Principle

As was previously explained, the purpose of CRDI is to directly inject fuel into the cylinders of a diesel engine using a single, common rail that is linked to all of the fuel injectors. In Fig.20.1, a typical CRDI system is shown. As observed, gasoline from the fuel tank travels through a fuel filter and a low pressure pump before being increased to a respectable level of pressure. A temperature sensor keeps track of the fuel's temperature. Fuel is then transferred to a high pressure pump. The pressure is managed via a pressure regulator valve. The outlet pressure of a contemporary CRDI system may reach 1800 bar. The gasoline is delivered under high pressure to an accumulator known as common rail. The rail pressure sensor keeps an eye on the pressure in the common rail. A high pressure pump in the common rail system keeps a reservoir of fuel at very high pressure and delivers this high pressure fuel to the many injectors. Because the system simply has to physically or electrically maintain the prescribed pressure, this simplifies the high pressure pump's intended application. Typically, the fuel injectors are under ECU control. A hydraulic valve (made up of a nozzle and plunger) is mechanically or hydraulically opened when the fuel injectors are electrically triggered, allowing fuel to be shot into the cylinders at the proper pressure.

The Injector

Fig.20.2 (a) depicts a typical fuel injector used in CRDI. The injector is nothing more than an electrical valve. It may open and close several times per second and is fed with pressurised gasoline by the fuel pump. A plunger that opens a valve and causes the valve to open when the injector is powered up releases pressurised fuel via a small nozzle. Each injector has a nozzle, a hydraulic intensifier, and an electronic digital valve, making it complete and self-contained. A rapidacting solenoid valve at the end of each injector regulates the time and volume of fuel injected. The order of each valve's opening and shutting is managed by a microprocessor. The fuel is atomized by the nozzle, which creates a mist as tiny as possible

so that it may burn readily and dissipate quickly. The length of time the fuel injector is open affects how much fuel is delivered to the engine.

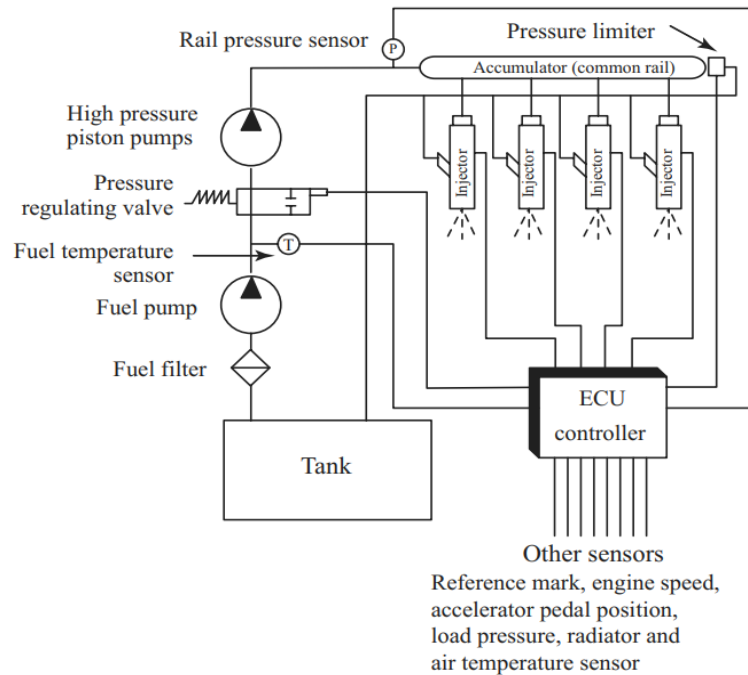


Figure 1: Schematic of common rail injection system

This is referred to as the pulse width, and the ECU regulates it. A separate chamber is required to sustain the high injection pressure of up to 1,800 bar in this division of work. Since the injectors are electronically controlled and the pressure energy of the fuel is remotely stored, the injection pressure at the beginning and conclusion of the injection is extremely near to the pressure in the accumulator (rail), resulting in a square injection rate.

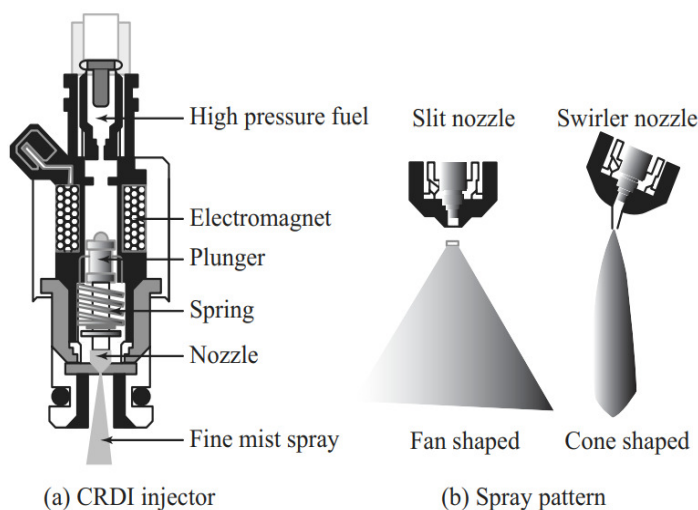


Figure 2: A typical CRDI injector

The injection pressure and rate will be the same for each of the many injection events if the accumulator, pump, and tubing are all sized appropriately. There are many nozzle options for the high-pressure injectors to accommodate various spray designs. A fan-shaped spray is produced by a slit nozzle, whereas a cone-shaped spray is produced by a swirler nozzle.

Sensor

The additional input sensors, in addition to the temperature and pressure sensors, are the throttle position sensor, crank position sensor, pressure sensor, lambda sensor, etc. The engine is controlled by sensors and a computer, which uses the gasoline most effectively. By operating the engine significantly better, this increases its performance, power, and fuel efficiency. Based on data gathered from different sensors on the cam and crankshafts, the electronic control unit (ECU) changes injection pressure precisely and as required. Additionally, an electronic display unit (EDU) is present and shows all pertinent data [5].

Electronic Control Unit (ECU)

Electronic fuel injection timing, volume, and pressure may all be precisely controlled by solenoids or piezoelectric valves. Better fuel atomization is provided by common rail technology, as was previously mentioned. The engine's electronic control unit has the ability to inject a tiny quantity of diesel ("pilot" injection) immediately before to the major injection event in order to reduce engine noise. This will lessen vibration and explosion. Additionally, the ECU can improve injection timing and volume taking into account variances in fuel quality, cold starting, and other factors.

A post-combustion procedure may be added using the common-rail injection method because to the exact timing control provided by the ECU. Little-scale combustion is created during this procedure by injecting a little quantity of fuel during the expansion stroke after the typical combustion has finished. By doing so, the unburned particles are removed and the temperature of the exhaust stream is raised. Additionally, the catalytic converters warm-up time is shortened. Post-combustion may, in essence, lower emissions.

Since the pump only delivers the engine with the amount of fuel it really needs, there is very little driving torque and pulsation within the high-pressure lines. Some cutting-edge common rail fuel systems can accurately regulate up to five injectors each stroke. CRDI is a sophisticated method of diesel engine control that makes use of contemporary electronics. However, with a CRDI system, this time interval and timing, among other things, are all controlled by a central computer or microprocessor-based electronic control unit.

With a normal, non-CRDI system, the interval between injections and the fuel amount are decided by mechanical components. The pipe can resist pressures of up to 1,600 bar in the first-generation design as it exists right now. The "Diesel Knock" feature, present in all diesel engines, whether or not they use direct injection, is lessened by precise timing. The CPU must be configured to properly handle input from various sensors in order for a CRDI system to function. The computer can determine the exact quantity of fuel and the moment at which it should be injected within the cylinder based on the data from these sensors. These calculations enable the CRDI control system to supply the appropriate quantity of diesel at the appropriate time, resulting in the best output with the least amount of pollutants and fuel waste [6], [7].

Microcomputer

A powerful microprocessor connected to other on-board control devices through the CAN (Controller Area Network) data bus controls direct-injection motors in a contemporary CRDI system. These gadgets have data-exchange software built in. The common rail system's key component is the electrical controls for the engine. In addition to controlling the solenoid valves installed in each cylinder, it controls injection pressure. It is necessary for the motor's variable control. Because only the speed and spontaneity of electronics can guarantee quick pressure injection adjustment and cylinder-specific control of the injector solenoid valves,

this electronic engine management network is a crucial component of the common rail system. Depending on the operating circumstances and the required output, the microprocessor controls how long the valves are open and, therefore, how much fuel is injected. Fuel injection halts instantly when the solenoid valves are closed. Modern common-rail direct fuel injection makes it possible to strike the perfect balance of efficiency, torque, riding comfort, and long life.

Status of CRDI Engines

The combustion chamber produces the appropriate swirl thanks to this technology. The better fuel-spray pattern and optimised piston head design of common-rail injectors, when combined, enable the air-fuel combination to perfectly create a vertical vortex. This leads to uniform combustion and much lower NO_x (nitrogen oxide) emissions than the present generation of diesel engines by around 50%.

In addition to other enhancements, the new common-rail engine reduces fuel consumption by 30%, doubles torque at low engine speeds, and boosts power by 25%. Additionally, it significantly lessens the noise and vibrations produced by ordinary diesel engines. Greenhouse gases (CO₂) are emitting at a 30% lower rate. Carbon monoxide (CO), unburned hydrocarbons (HC), and particle emissions are all decreased by 40%, 50%, and 60%, respectively, at a fixed level of NO_x.

Principle of CRDI in Gasoline Engines

Before the MPFI technology was developed, petrol engines supplied the air-fuel combination via carburetors. Carburetors are still used today because to their affordability and simplicity, nonetheless. Petrol engines may also use the CRDI idea. Today's petrol engines employ the latest technology known as petrol direct injection (GDI). Section 20.6 goes into further information about this.

Advantages of CRDI Systems

There are several benefits that may be attained by the use of CRDI, some of which include:

- i. Enhanced power, better fuel economy, and quieter operation.
- ii. Lowered emissions and exhaust particle levels.
- iii. Improved stability, extremely high injection pressure, and precise injection time.
- iv. Improved fuel pulverisation.
- v. improved combustion quality as a result of the pilot and post injection
- vi. Doubling the torque at lower speeds.

The biggest drawback of this technique is that it makes the engine more expensive to install and maintain.

Homogeneous Charge Compression Ignition Engine

In homogeneous charge compression ignition (HCCI), air and fuel are thoroughly mixed and compressed to the fuel's auto-ignition temperature. Exothermic reactions transfer chemical energy into a form that can be converted into work and heat in an engine. The two most common types of combustion employed in engines are reflected in HCCI:

- (i) homogeneous charge spark ignition
- (ii) stratified charge compression ignition

However, compression is used to increase the mixture's density and temperature rather than a spark to ignite it. The whole mixture reacts spontaneously when it reaches the temperature

required for self-ignition. The ignition takes place at several locations, which is the primary property of HCCI. The fuel-air combination burns almost simultaneously as a result. Combustion lacks a direct initiator. This makes it very difficult to regulate the combustion. The physical knowledge of the ignition mechanism and improvements in microprocessor technology, however, aid us.

If used correctly, the HCCI idea may provide emissions and efficiency similar to those of a petrol engine. In fact, it has been shown that HCCI engines can operate at very low nitrogen oxide (NO_x) emission levels without the need of a catalytic converter. Due to reduced peak temperatures, the unburned hydrocarbon and carbon monoxide emissions are nevertheless substantial, much as in petrol engines. As a result, in order to comply with vehicle pollution rules, exhaust gases must be treated [2]–[4]. Recent studies have shown that some of the challenges associated with managing HCCI ignition and burn rates may be overcome by combining two fuels with differing reactivities, such as petrol and diesel. Reactivity it has been shown that Controlled Compression Ignition (RCCI) offers very effective, low emission functioning throughout a broad load and speed range. The history of HCCI engines is lengthy. However, it should be emphasised that HCCI has not yet been used as extensively as diesel injection or spark ignition.

In essence, the HCCI engine uses an Otto cycle. In actuality, HCCI was well-liked prior to the advent of electronic spark ignition. The hot bulb engine is one example, which employed a hot vaporisation chamber to aid in the mixing of fuel and air. Compression and additional heat created the ideal environment for combustion to take place. When the temperature and reactant concentration are high enough, a fuel and air combination will ignite. By raising the compression ratio, pre-heating the induction gases, forcing induction, or retaining or reinducting exhaust gases, the concentration and/or temperature may be raised. After being lit, combustion moves extremely quickly. Combustion happens too quickly when auto-ignition starts too early or with too much chemical energy. This may result in very high in-cylinder pressures and perhaps wreck the engine. Because of this, HCCI is often used with low overall fuel mixes. The proper management of combustion is hence the difficult task in HCCI [8].

Control

The difficulty in controlling HCCI is a significant barrier to broader commercialization. A spark is utilised in a standard petrol engine to ignite the pre-mixed fuel and air. When diesel engine fuel is put into compressed air, combustion starts. The time of combustion is specifically managed in both situations. But with an HCCI engine, the homogenous mixture of fuel and air is compressed, and combustion starts as soon as the right circumstances are present. This implies that there is no clearly defined, directly controllable combustion initiator.

Engines may be made to have the ignition conditions happen at the ideal time. The control system must activate in an HCCI engine to bring about the circumstances that cause combustion in order to do this. As previously stated, the control must be implemented either by

- (i) Variable compression ratio, or
- (ii) Variable induction temperature, or
- (iii) Variable exhaust gas percentage, or
- (iv) Variable valve actuation, or
- (v) Variable fuel ignition quality.

Various control approaches mentioned above are briefly discussed in the following sections.

Variable Compression Ratio

The geometric and effective compression ratios may both be changed using a variety of techniques.

- i. With the use of a moveable plunger at the top of the cylinder head, the geometric compression ratio may be altered.
- ii. By shutting the intake valve very early or very late using a variable valve actuation system, the effective compression ratio may be varied from the geometric ratio.

Both of the aforementioned methods need some extra strength to provide quick results. Additionally, installation is costly. However, it is discovered that control of an HCCI engine utilising variable compression ratio schemes is successful. Numerous studies have been conducted on the impact of compression ratio on HCCI combustion.

Variable Induction Temperature

The auto-ignition event in HCCI engines is very temperature sensitive. Timing of combustion has been controlled by temperature using a variety of techniques. Resistance heaters are the easiest way to change the intake temperature, however this technique is sluggish. On a cycle-by-cycle basis, the temperature cannot be changed. Fast Thermal Management (FTM) approach is a different way. It is accomplished by quickly altering the intake charge temperature from cycle to cycle and quickly combining hot and cold air streams. It has a constrained scope of use and is similarly costly to implement.

Variable Exhaust Gas Percentage

If held, in-cylinder exhaust gas is very hot, or if it is re-inducted from the previous cycle via the intake as in traditional EGR systems, it is rather cold. The use of hot and cool EGR has both been tested. The exhaust affects HCCI combustion in two different ways. The new charge is diluted by cool EGR, which delays ignition and lowers chemical energy and engine effort. On the other hand, hot EGR will raise the temperature of the gases in the cylinder and hasten ignition. EGR-based combustion timing control in HCCI engines has also been the subject of in-depth research.

Variable Valve Actuation

The HCCI working zone has been shown to be extended by variable valve actuation (VVA), which provides finer control over the temperature-pressure-time history inside the combustion chamber. VVA may do this using two different techniques:

Controlling the effective compression ratio: The intake valve's point of closure may be adjusted using a variable valve actuation system. The compression ratio will vary if this is delayed beyond bottom dead centre, which will modify the in-cylinder pressure-time history before combustion. Controlling how much hot internal exhaust gas recirculation (EGR) is kept in the combustion chamber: A VVA system may be utilised to regulate how much hot EGR is kept in the combustion chamber. Both adjustments in valve overlap and valve reopening may be used to accomplish this. It may be able to regulate the in-cylinder temperature by balancing the ratio of cold external gas recirculation to hot internal gas recirculation produced by a VVA system. While camless VVA and electro-hydraulic systems may be employed to provide a considerable lot of control over the valve event, the fabrication of the components for such systems is presently difficult and costly. However, mechanical systems with variable lift and duration are far less expensive and complex. It is quite easy to build such systems to provide the required control over the valve lift curve if the desired VVA characteristic is known.

Variable Fuel Ignition Quality

Controlling the start of ignition and the pace at which heat is released is another way to increase the working range. This is done by adjusting the fuel's ignition quality. This is often accomplished by combining several fuels. Examples include the mixing of commercial petrol and diesel, as well as ethanol and natural gas. The following methods may be used to accomplish this:

Blending fuels upstream of the engine

Two fuels are combined in the liquid phase, one of which has a lower ignition resistance (like diesel fuel) and the other of which has a higher resistance (like petrol). By altering the ratio of these fuels, the time of ignition may be controlled. Then, either port or direct injection are used to supply the fuel.

Having two fuel circuits

The dual fuel approach is adaptable. Fuel A may be injected directly into the cylinder, whereas Fuel B can be pumped directly into the intake duct (port injection). Controlling ignition, the rate of heat release, and exhaust emissions may be done by adjusting the amount of these fuels.

Power

More fuel may be added to the combustion chamber to boost power in both compression ignition and spark ignition engines. These engines' sluggish rate of heat release allows them to sustain the greater power. However, the whole mixture burns almost simultaneously in HCCI engines. Peak pressures and heat release rates will rise even more with a larger fuel-to-air ratio. Furthermore, a lot of effective HCCI control methods need on thermal preheating of the charge. Because of the decreased mass and density of the fuel-air charge in the combustion chamber, power will be reduced. These elements make it difficult to increase the power output of HCCI engines.

Utilising fuels with various autoignition qualities is one technique to boost power. As a result, the heat release rate and peak pressures will be reduced, allowing the equivalency ratio to be raised. Thermal stratification of the charge is an additional method. This will cause various places in the compressed charge to burn at different times and at different temperatures, slowing the pace at which heat is released.

It is feasible to enhance power using this strategy. A third option is to only operate the engine in the HCCI mode during partial loads, and to operate it as a diesel or spark ignition engine under full or almost full loads. The final strategy is being pursued more intensively since more study is necessary to properly execute thermal stratification in the compressed charge [9].

Emissions

Peak temperatures for HCCI engines are lower than for spark ignition and compression ignition engines because they run on lean mixes. The generation of NO_x is decreased by the lower peak temperatures. Compared to traditional engines, this results in reduced NO_x emissions. However, the low peak temperatures also result in partial fuel combustion, particularly close to the combustion chamber's walls. High hydrocarbon and carbon monoxide emissions result from this. Because the exhaust is still oxygen-rich, an oxidising catalyst is needed to remove the regulated species.

Difference in Engine Knock

When part of the unburned gases ahead of the flame spontaneously auto-ignite, it causes knocking or pinging in traditional spark-ignition engines. As a result, an expansion wave travels towards the end gas area and a shock wave travels out of it. Standing waves with a large amplitude are created when the two waves interact and reflect off the combustion chamber's walls. Piston compression causes ignition in HCCI engines. The whole reactant mixture ignites (almost) simultaneously in this engine. There is no banging because there are no pressure variations or very little pressure changes across the various areas of the gas. However, even with HCCI, knocking is a distant possibility at high loads (i.e., high fuel-air ratios).

Advantages and Disadvantages of HCCI Engine

Advantage

- (a) Despite maintaining current pollution standards, a 30% fuel savings.
- (b) They run at compression ratios similar to diesel engines and are fuel-lean.
- (c) Compared to traditional SI engines, it is capable of better efficiency.
- (d) Lower emissions and better combustion are the results of homogeneous mixing.
- (e) Compared to ordinary SI engines, peak temperatures are lower.
- (f) Emissions of NO_x and soot are almost nonexistent.
- (g) It is compatible with practically all alternative fuels, as well as petrol and diesel.
- (h) Low throttling losses contribute to the higher than average HCCI efficiency.

Disadvantage

- (a) In-cylinder peak pressures that are too high might harm the engine.
- (b) Engine wear is influenced by rapid rates of heat emission and pressure increase.
- (c) Contrary to the SI and CI engines, the auto-ignition is difficult to manage.
- (d) The power range of HCCI engines is restricted.
- (e) Lean flammability restrictions cause limitations at low loads.
- (f) Due to constraints on in-cylinder pressure, there is a problem with high loads.
- (g) Compared to traditional engines, CO emissions are greater.
- (h) Higher hydrocarbon emissions before the catalyst.

CONCLUSION

Alternative power sources that differ from conventional internal combustion engines are referred to as non-conventional engines. These engines provide effective and sustainable solutions for transportation and power production via the use of cutting-edge technology and fuel sources.

As technology advances, their performance, efficiency, and environmental friendliness continue to improve, their future use has enormous promise. The rising need for sustainable energy solutions and the need to cut greenhouse gas emissions are what are driving the continuous development of non-conventional engines, such as electric cars, fuel cells, and alternative fuels. Their hopes for the future include expanding uses in a number of industries, which will help create a future that is greener and more sustainable.

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